

# 2008 Title 24 Refrigerated Warehouse CASE Initiative

Public Workshop February 22, 2006

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## **Background Research**

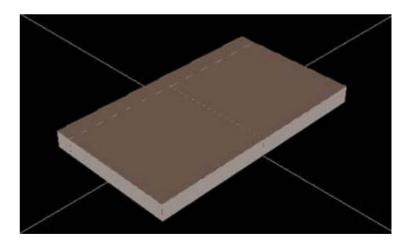
#### Literature review

- Utility efficiency programs
- ASHRAE "Design Essentials for RWHs"
- Purdue study for IARW
- Pacific Northwest VFD Evaporator Fan Initiative
- DEER measure cost study
- Manufacturers' data
- Contractor and designer telephone surveys
  - DOE-2 simulations



### **Prototype Refrigerated Warehouse Simulations**

- DOE-2.2R simulation program used
- Ammonia based system with screw compressors and evaporative condenser
- 92,000 SF
- Combination of freezer, cooler and shipping dock space
- See report for details





GSE Electric Company.

#### **Areas Addressed**

- Evaporators
- Condensers
- Compressors
- Lighting
- Insulation levels
- Under floor heat
- Defrost



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#### **Evaporator Fan VSD**





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# **Evaporator Fan VSD Analysis**

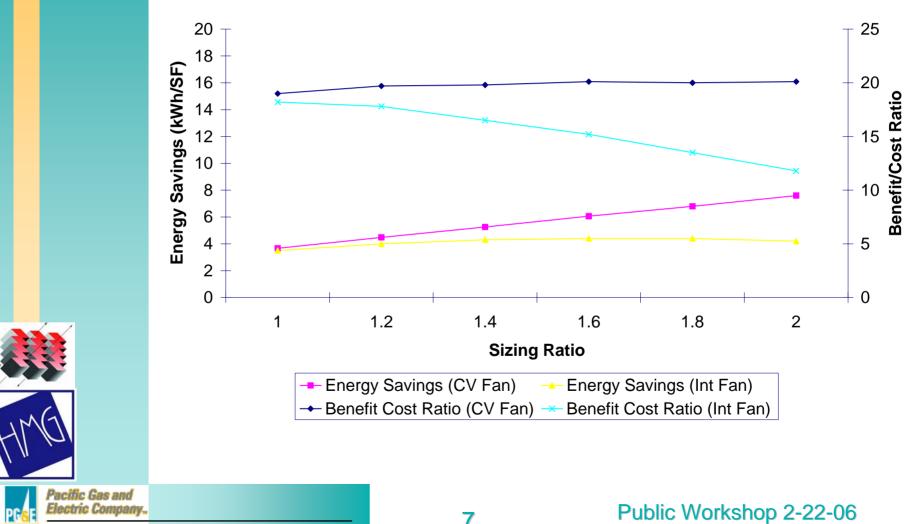
- Used hourly "Time Dependent Valuation" costs,
  - 15 year measure life,
  - 3% real (net of inflation) discount rate
- Impact of over-sizing investigated
- Evaporator fan VSD costs of \$577 per hp from the Northwest Energy Efficiency Alliance
  - Demonstrated energy savings
  - Reduced fruit mass loss in controlled atmosphere rooms



- Greater over-sizing, greater savings
- B/C ratio of 20 to 1 fairly constant for CV fans, above 10 for intermittent fans



### **Evaporator Fan VSD Energy Savings and Cost Effectiveness**









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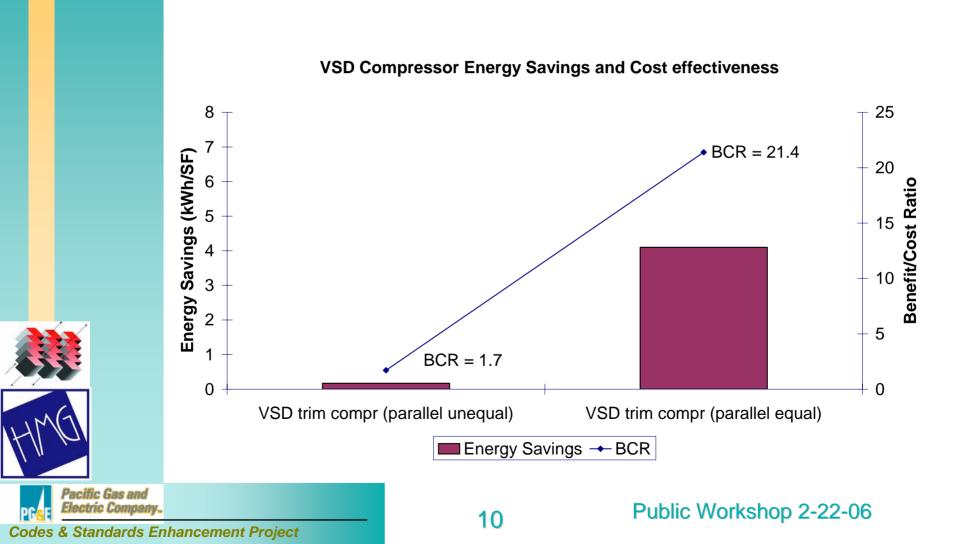
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# **VSD Compressor Analysis**

- Used 15 year TDV to value energy cost savings
- Looked at VSD on 3 compressor parallel equal line
- Looked at VSD also on smallest compressor in 3 compressor parallel unequal line
- VSD costs from 2005 DEER Measure Cost Study
  @ \$171/ton



### VSD Trim Compressor Energy Savings and Cost Effectiveness



#### **Evaporative Condensers**





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## Condenser Oversizing and Floating Head Pressure

- Used 15 year TDV to value energy cost savings
- Varied design approach temperature from 25°F to 13°F
- Condenser fan and pump power improved from 330 Btu/hr-watt to 400 Btu/hr-watt
- Minimum condensing temperature dropped from 85°F to 70°F



## **Oversized Evaporative Condenser and Floating Head Pressure**

#### Scenarios presented:

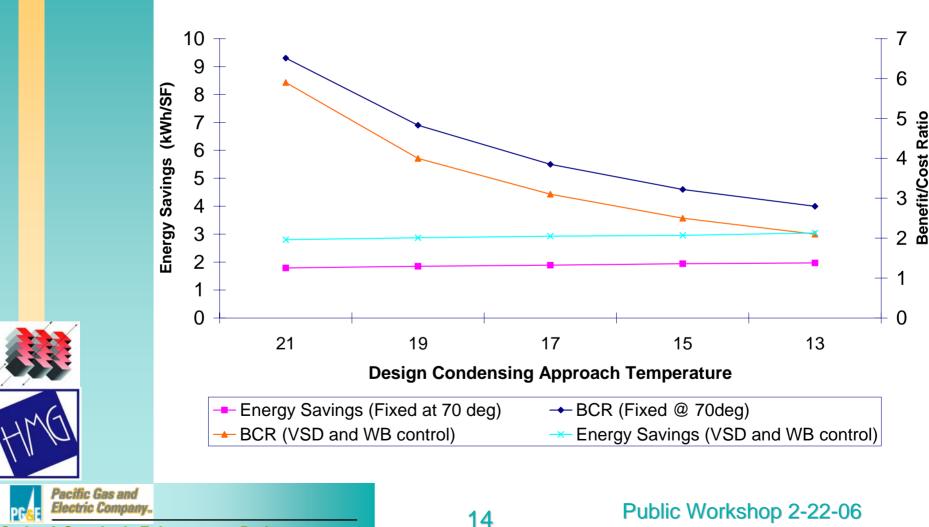
- Fixed condensing temperature
- 9°F wetbulb offset with VSD

#### 2005 DEER Measure Cost Study used

- \$88 per ton @ 5°F reduction
- \$203 per ton @ 5°F reduction with VSD and wetbulb control
- Scaled based on reduction modeled



### **Oversized Condenser Energy Savings and Cost Effectiveness**



### **Shell Insulation**



#### **Freezer Floor Insulation**



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#### **Insulated Panels**



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# Insulation Energy And Economic Analysis

- Used 30 year "Time Dependent Valuation" costs
- Insulation incremental cost (R.S. Means)
  - Roof insulation
    - Polyisocyanurate @ R-7.1 per inch
    - \$0.25 per SF in.
  - Wall insulation
    - \$0.63 per SF in.
    - Polyurethane @ R-5 per inch
  - Floor insulation
    - Extruded polystyrene @ R-5 per inch
    - \$0.32 per SF in.



# **Insulation Analysis**

|      | Shell       | Common       | ASHRAE         | CASE           | Benefit/Cost      |                              |
|------|-------------|--------------|----------------|----------------|-------------------|------------------------------|
|      | omponent    | Practice     | Recommendation | Recommendation | Coastal<br>(CZ 3) | Central<br>Valley<br>(CZ 13) |
| Free | zer Ceiling | R-46         | R-45 to R-50   | R-49           | 3.7               | 4.3                          |
| Fre  | ezer Wall   | R-32         | R-35 to R-40   | R-40           | 1.9               | 2.2                          |
| Fre  | ezer Floor  | R-30         | R-27 to R-32   | R-30           | 4.8               | 5.0                          |
| Coo  | ler Ceiling | R-24 to R-40 | R-30 to R-35   | R-35           | 1.1               | 2.0                          |
| Co   | oler Walls  | R-25         | R-25           | R-25           | 0.6               | 1.1                          |



Note: BCR varies with refrigeration plant efficiency – efficient plant used

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### **Under Floor Heat**

#### Energy consumption for under floor heat

- 2.0 Btu/hr-SF (-20°F freezer, R-30 insulation)
- 0.6 W/SF (5.2 kWh/SF-yr)
- 15 yr NPV TDV electricity costs for electric heat = \$15.43 per SF
- Costs taken from R.S. Means
  - \$2.26 per SF for glycol system
  - \$2.04 per SF for electric system
- Benefit / Cost Ratio = 70+



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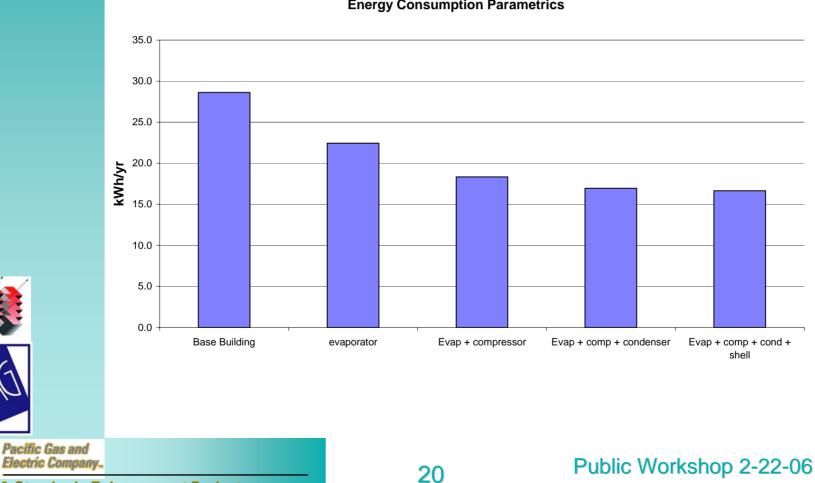
## **Temperature Terminate Defrost**

- Defrost needs vary by location within the facility, time of day and season
- Time on, time terminate defrost controls may be set for "worse case" conditions
- Time on, temperature terminate controls are standard practice
- Require additional temperature sensor at evaporator
- Case studies cited quick paybacks (< 1 year)</li>



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## **Energy Consumption Parametrics**



#### **Energy Consumption Parametrics**

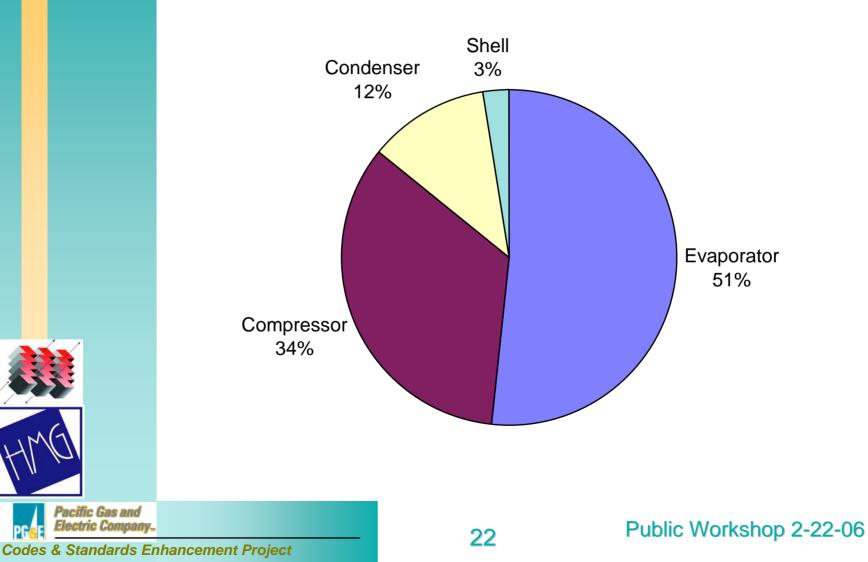
# **Overall Energy Savings Analysis**

- Energy savings of about 12 kWh/SF and non-coincident peak demand savings of 1.4 W/SF
- Represents about 40% of annual consumption of prototype model



Cost effective

# **Energy Savings Breakdown**



# **Code Change Proposal**

#### Mandatory Measures

- Lack of ACM for refrigerated warehouses precludes prescriptive approach
- New section in Standards
  - Section 120 Mandatory Requirements for Refrigerated Warehouses

Applies to freezers and coolers > 3000 SF



## **Proposed Code Provisions -Evaporators**

- Require VSDs on evaporator fan motors
- Limits on evaporator fan motor power of 0.15 W/cfm
- Limit electric defrost
  - Exception based on system size
- Require temperature termination on defrost controls



### **Proposed Code Provisions -Compressors**

- Require compressors and accessories supplied by manufacturers to operate at 70°F condensing
- Require VSD on at least one compressor per suction group



# **Proposed Code Provisions -Lighting**

- Use same lighting provisions as for nonrefrigerated warehouses
  - Max lighting power of 0.6 W/SF
  - Require bi-level lighting controls in storage spaces



## **Proposed Code Provisions -Condensers**

- Require evaporative condensers on all ammonia systems
- Limits on condenser wetbulb approach temperature
  - 20°F at design conditions
- Limits on condenser fan and pump power
  - 400 Btu/hr-watt
- Require floating head pressure control to 70°F
- VSD on evaporative condenser fans controlled by wetbulb or load



## **Proposed Code Provisions – Insulation**

#### Minimum R-values for freezers

- R-40 Wall
- R-49 Ceiling
- R-30 Floor
- Minimum R-values for coolers
  - R-25 Wall
  - R-35 Ceiling



- Limit on electric resistance under floor heating
  - Exception based on facility size
  - Resistance heat must be controlled off during summer on-peak periods





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# **Information and Contacts:**

#### Full CASE report available for download:

www.energy.ca.gov/title24/2008standards/documents/ 2006-02-22+23\_workshop/2006-02-15\_DRAFT\_REP\_PG&E.PDF

#### Please send written comments to:

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#### Thank you very much!!

