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DRAFT
NONRESIDENTIAL
ALTERNATIVE
CALCULATION METHOD
REFERENCE MANUAL

FOR THE ~~2016-2019~~ BUILDING
ENERGY EFFICIENCY
STANDARDS

2019

~~August 2017~~ June 2018 ~~May~~ ~~January~~

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Acknowledgments

The *Building Energy Efficiency Standards* were adopted and put into effect in 1978 and have been updated periodically in the intervening years. The standards are a unique California asset and have benefitted from the conscientious involvement and enduring commitment to the public good of many persons and organizations along the way. The ~~2016-2019~~ *Standards* development and adoption continued that long-standing practice of maintaining the standards with technical rigor, challenging but achievable design and construction practices, public engagement, and full consideration of stakeholders' views.

The ~~2016-2019~~ *Standards* and the supporting documents were conceptualized, evaluated, and justified through the excellent work of California Energy Commission staff and consultants working under contract to the Energy Commission, Pacific Gas & Electric Company, Southern California Edison Company, San Diego Gas & Electric Company, and Southern California Gas Company. At the Energy Commission, Maziar Shirakh, P.E. and Larry Froess, P.E. served as the project managers and senior engineers. Bill Pennington, Senior Technical and Program Advisor, provided overall guidance. Christopher Meyer is the Office Manager for the Building Standards Office. Kristen Driskell, ~~while serving as~~ Appliances and Outreach and Education Office Manager, provided legal counsel to the staff. Other technical staff contributors included Todd Ferris; ~~Thao Chau~~; RJ Wichert; Michael Shewmaker; Jeff Miller, P.E.; Payam Bozorgchami, P.E.; Danny Tam; ~~and~~ Simon Lee; ~~and Mark Alatorre, P.E.~~ Additional staff input and assistance came from Energy Hotline staff and the Energy Commission's web team. Critical support for the staff in conceptualizing, evaluating, and reviewing this document came from NORESO, L'Monte Information Services, Gard Analytics, National Renewable Energy Laboratory, Hitchcock Consulting, 360 Analytics, Wrightsoft Corporation, Taylor Engineering, McHugh Energy, Gabel Associates, Energy Solutions, E3, PECL, and the Heschong Mahone Group.

The authors are grateful to the many people and organizations that contributed to the development and production of the standards and supporting documents. The documents reflect, to a large extent, comments made by the many people who took time to carefully review earlier versions. Reviewers who significantly contributed to the content include members of California Association of Building Energy Consultants and California Building Officials.

ABSTRACT

The California Energy Commission's *2016-2019 Building Energy Efficiency Standards for Nonresidential Buildings* allow compliance by either a prescriptive or performance method. The performance compliance approach uses computer modeling software to trade off efficiency measures. Performance compliance is typically the most popular compliance method because of the flexibility it provides in building design.

Compliance software must be certified by the Energy Commission, following rules established for modeling software. This document establishes the rules for the process of creating a building model, describing how the proposed design (energy use) is defined, how the standard design (energy budget) is established, and ends with what is reported on the Performance Compliance Certificate (PRF-01). This *Nonresidential Alternative Calculation Method Reference Manual* explains how the proposed and standard designs are determined. This document also establishes the procedure for performance calculation, necessary rule sets, reference method for testing compliance software accuracy, and the minimum reporting requirements.

The *2016-2019* compliance software is the simulation and compliance rule implementation software specified by the Energy Commission. The compliance manager, called California Building Energy Code Compliance (CBECC), models all features that affect the energy performance of the building. This document establishes the process of creating a building model. Each section describes how a given component, such as a wall or fenestration, is modeled for the proposed design, standard design, and ends with what is reported on the Performance Compliance Certificate (PRF-01) for verification by the building enforcement agency.

Keywords: ACM, Alternative Calculation Method, Building Energy Efficiency Standards, California Energy Commission, California Building Energy Code Compliance, CBECC, Performance Compliance Certificate (PRF-01), compliance manager, compliance software, computer compliance, energy budget, Time Dependent Valuation (TDV), energy standards, energy use, prescriptive compliance, performance compliance, design, proposed design, standard design, VRF

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~~Summary of Changes~~

~~2016 Nonresidential Alternative~~ ~~Calculation Method (ACM) Reference~~ ~~Manual~~

~~Version 31.0 Released August~~December, 2017~~8~~

This version of the Nonresidential ACM Reference Manual incorporates changes made for the 2016.3.0 release summarized below. In addition to the updates describing the 3.0 release the ACM underwent a large scale cleanup to revise formatting and remove language not applicable to the current software.

GBECC Com 2016.3.0 Updates:

- ~~Added capability to model Chilled Beams – Active/Passive~~
- ~~Added capability to model Heat Pump Water Heater~~
- ~~Added capability to model Core and Shells projects (partial compliance) – auto generate proposed systems~~
- ~~Added capability to model unknown HVAC system for existing projects~~
- ~~Added capability to model cooling system in proposed case when no cooling system is defined~~
- ~~Added capability to model Thermal Energy Storage (Chilled Water)~~
- ~~Added capability for scaling of plant for Existing+Addition+Alteration projects~~
- ~~Added capability to model ventilation standards other than Title 24 (where standard will be modeled same as proposed)~~
- ~~Updates to modeling and reporting of Parking Garage systems~~
- ~~Added capability to model a new zone system of type Furnace (heating only)~~
- ~~Added new air system(s) type to support modeling of DOAS systems~~
- ~~Updates and enhancements to Domestic Hot Water Heating~~
- ~~Added capability to include doors and glass doors for modeling interlocks (Section 140.4(n))~~
- ~~Updates to material library adding more insulation options and including capability to select metal wall configuration~~
- ~~Updates and fixes to building envelope rules and mandatory checks~~
- ~~Updates to rules and reporting for cooling coil efficiency input precision~~
- ~~Implement fix for EnergyPlus U-factor limitation~~
- ~~Implemented a non-compliance option to bypass mandatory envelope checks~~
- ~~Added new zone system for reporting of Mini-Splits~~
- ~~Various updates and enhancements to user Interface including, but not limited to, Compliance Option Checkboxes~~
- ~~Nonresidential ACM updates as needed for new capability/enhancements and/or bug fixes~~
- ~~Reporting updates as needed for new capability/enhancements and/or bug fixes~~

- ~~Enhanced security updates for the new release under open source code~~
- ~~Various improvements and fixes to source code and rule enhancements as needed for various bug fixes~~

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1. Overview

1.1 Purpose

The *Nonresidential Alternative Calculation Method (ACM) Reference Manual* explains the requirements for approval of nonresidential compliance software in California. Approved software is used to demonstrate minimum compliance with the Energy Standards, reach codes, and to achieve beyond-code energy performance needed to qualify for reach or CALGreen standards. The procedures and processes described in this manual are designed to provide consistency and accuracy while preserving the integrity of the process of compliance. This manual addresses software for nonresidential buildings, hotels and motels, and high-rise residential buildings as outlined in Title 24, Part 6, Subchapter 5, §-140.1. A separate ACM reference manual applies to low-rise residential buildings. The approval process for nonresidential software programs is specified in Title 24, Part 1, §-10-101 through §-10-110 of the California Code of Regulations.

1.2 Modeling Assumptions

When calculating annual energy use, it is necessary to make assumptions about how the proposed building is operated. Operating assumptions include thermostat settings, number of occupants, receptacle loads, process loads, hot water loads, and operation schedules for HVAC systems, lighting systems, and other systems. Sometimes these data are known with some certainty, and other times (for instance, for speculative buildings), it is necessary to make estimates. ~~These inputs are prescribed. (They are fixed for both the proposed design and for the baseline building and can't be changed.)~~ Some of these inputs are prescribed (they are fixed for both the proposed and standard design ~~and for the baseline buildings~~ and can't be changed), while others are defaults.

1.3 Scope

This manual is intended to be used as both a reference for the modeling methods of the CBECC-Com software and as a guide to software programs seeking certification as Title 24 compliance software for nonresidential buildings.

The *ACM Reference Manual* can be modified during a code cycle without a formal rulemaking. Therefore, the goal of the software development team is to provide periodic updates to improve the accuracy and usability of compliance software.

1.4 Organization

This document is organized in five chapters and several appendices, as follows:

Chapter	Description
1. Overview	The purpose, organization, and content of the manual (this chapter).
2. General Modeling Procedures	An overview of the modeling process, outlining the modeling rules and assumptions that are implemented in the same way for both the standard design and the proposed design, and procedures for determining system types and equipment sizes.
3. Software Requirements	Requirements for the simulation engines and software shells used to make calculations, and special reporting requirements for non-standard building features.
4. Content and Format of Standard Reports	The content and organization of the standard reports produced by qualifying software.
5. Building Descriptors Reference	The acceptable range of inputs for the proposed design and a specification for the standard design.

In addition, there are several appendices that contain reference material supporting definition of the proposed design and standard design. The numbering for these appendices generally aligns with the section numbers in the main manual that reference the appendices.

1.5 Reference Method

The reference procedures and method described in this manual establish the basis of comparison for all software. The approval process ensures that a minimum level of energy efficiency is achieved regardless of the software used. This is accomplished by:

- Specifying a series of reference method comparison tests that candidate software passed.
- Specifying input that may vary for credit, and input that are fixed or restricted.
- Defining standard report output requirements.
- Certifying the software vendor requirements in this manual.

The *Nonresidential ACM Reference Manual* is an approved document, separate from the formally adopted ACM regulations. This approval gives the Energy Commission flexibility to incorporate new modeling procedures or features, or fix errata, within the code cycle. The document is said to be in continuous maintenance. Software may be certified with the capability of modeling specific building systems or features.

The Energy Commission's purpose in approving additional capabilities is to accommodate new technologies which have only begun to penetrate the market and new modeling algorithms. Newly added capabilities that evaluate measures already in relatively common use shall have their standard design for the measure based on the common construction practice (or the typical base situation) for that measure since common practice is the inherent basis of the standards for all measures not explicitly regulated. For example, the Commission has no interest in an

optional capability that evaluates the energy effects of dirt on windows unless a new technology produces substantial changes in this aspect of a building relative to buildings without this technology. The burden of proof that an additional capability should be approved lies with the applicant.

Companion documents that are helpful in preparing software for certification include the latest editions of the following Energy Commission publications:

- Energy Efficiency Standards
- Appliance Efficiency Regulations
- Nonresidential Compliance Manual
- Nonresidential ACM Approval Manual
- Reference Nonresidential Appendices
- Reference Joint Appendices

In this manual the term "standards" means the *Building Energy Efficiency Standards*, Title 24, Part 6 of the California Code of Regulations. The term "compliance" means that a building design in an application for a building permit complies with the "standards" and meets the requirements described for building designs therein.

There are a few special terms that are used in this manual. The Energy Commission approves software for use in demonstrating compliance. Energy Commission approval means that the Commission accepts the applicant's certification that software meets the requirements of this manual. The proponent of candidate software is referred to as a *vendor*. The vendor shall follow the procedure described in this document to publicly certify to the Energy Commission that the software meets the criteria in this document for:

- Accuracy and reliability when compared to the reference method.
- Suitability in terms of the accurate calculation of the correct energy budget, the generation of output for transmission to standardized forms, and documentation on how the program demonstrates compliance.

In addition to explicit technical criteria, Energy Commission approval will also depend upon the Commission's evaluation of:

- Enforceability in terms of reasonably simple, reliable, and rapid methods of verifying compliance and application of energy efficiency features modeled by the software and the inputs used to characterize those features by the software users.
- Dependability of the installation and energy savings of features modeled by the software. The Energy Commission will evaluate the probability of the measure actually being installed and remaining functional. The Commission shall also determine that the energy impacts of the features that the software is capable of modeling will be reasonably and accurately reflected in real building applications of those features. In particular, it is important that the software does not encourage the replacement of actual energy savings with theoretical energy savings due to tradeoffs allowed by the software.

For the vendor, the process of receiving approval of software includes preparing an application, working with the Energy Commission staff to answer questions from either Commission staff or the public, and providing any necessary additional information regarding the application. The application includes the four basic elements outlined below. Commission staff evaluates the

software based on the completeness of the application and overall responsiveness to staff and public comment.

The four basic requirements for approval include:

1. Required capabilities.
 - a) Software shall have all the required input capabilities explained in Chapter 2.
 - b) Software shall meet software requirements and documentation requirements for applicable features supported by the software, as described in Chapter 3.
2. Accuracy of simulation.
 - a) The software shall demonstrate acceptable levels of accuracy by performing and passing the required certification tests discussed in Appendix 3B.
 - b) The software vendor conducts the specified certification tests in Appendix 3B, evaluates the results, and certifies in writing that the software passes the tests. The Energy Commission will perform spot checks and may require additional tests to verify that the proposed software is appropriate for compliance.
 - c) When energy analysis techniques are compared, two potential sources of discrepancies are the differences in user interpretation when entering the building specifications, and the differences in the software algorithms (mathematical models) for estimating energy use. The approval tests minimize differences in interpretation by providing explicit detailed descriptions of the test buildings that must be analyzed. For differences in the software algorithms, the Energy Commission allows algorithms that yield equivalent results.
3. User's Manual.

The vendor shall include a user's manual and/or help system that provides appropriate guidance for specifying inputs and running a simulation for compliance.
4. Program support.

The vendor shall provide ongoing user and enforcement agency support as described in the *Nonresidential ACM Approval Manual*.

The Energy Commission may hold one or more public workshops with vendor participation to allow for public review of the vendor's application. Such workshops may identify problems or discrepancies that may necessitate revisions to the application.

Commission approval of software programs is intended to provide flexibility in complying with the standards. In achieving this flexibility, however, the software shall not degrade the standards or evade the intent of the standards to achieve a particular level of energy efficiency. The vendor has the burden of proof to demonstrate the accuracy and reliability of the software relative to the reference method and to demonstrate the conformance of the software to the requirements of this manual.

1.6 Compliance

1.6.1 Type of Project Submittal

Software shall require the user to identify the type of compliance for the project. The software shall require the user to choose one of the following options:

- New Building or Addition Alone. Software may do this by treating an addition alone as a new building, but an addition modeled in this way shall be reported on all output forms as an addition (modeled alone).
- Addition Plus Alteration of Existing Building (if software is approved for this optional capability).
- Alteration of Existing Building (if software is approved for this optional capability).

1.6.2 Scope of Compliance Calculations

For each building or separately permitted space, software shall also require the user to identify the scope of the compliance submittal from a combination of the following list:

- Envelope
- Lighting or Partial Lighting
- Mechanical or Partial Mechanical (may include or exclude Domestic Hot Water)

Each combination requires specific assumptions, input procedures, and reporting requirements. Modeling assumptions are documented in Chapter 5. Reporting requirements are documented in Chapter 4. Software shall produce only compliance reports specific to the scope of the submittal determined for the run. For example, if the scope is envelope only, only the PRF-01 forms with envelope only components are produced.

Lighting compliance for a partial compliance scenario may be for the entire building, or may be specified for only portions of the building. When the building applies for partial lighting compliance, the space(s) where lighting for the space is unknown or undefined shall be marked as “undefined,” and the compliance software shall use the standard design lighting power for the user-defined space type for both the proposed design and standard design. Under this compliance scope, the entire building shall be modeled, and the compliance forms shall indicate the spaces for which lighting compliance is not performed.

The combination of the above scopes will determine the standard design that the proposed design is compared to. When a scope is excluded from the performance calculation, the standard design will match the proposed for all features covered by that scope. Specific rules for each building model descriptor can be found in chapter 5 of this manual.

1.6.3 Climate Zones

The program shall account for variations in energy use due to the effects of the California climate zones and local weather data. Climate information for compliance simulations shall use the applicable data set in Reference Appendix JA2.

1.6.4 Time Dependent Valuation

The candidate software shall calculate the hourly energy use for both the standard design and the proposed design by applying a time dependent valuation (TDV) factor for each hour of the reference year. TDV factors have been established by the Energy Commission for residential and nonresidential occupancies, for each of the climate zones, and for each fuel (electricity, natural gas, and propane). The procedures for TDV energy are documented in *Reference Appendix JA3*.

1.6.5 Reporting Requirements for Unsupported Features

The compliance software shall meet required capabilities and pass applicable certification tests as defined in Appendix 3A, Appendix 3B, and Appendix 3C. While the vendor's software does not need to implement every modeling rule in the *ACM Reference Manual*, all software features, systems, components, and controls that are modeled must follow the modeling guidelines in the *ACM Reference Manual*. Vendors seeking certification for software programs to be used for Title 24 compliance should clearly state the extent of the capabilities of their software with respect to compliance. Support of a modeling feature includes correctly processing user input, specifying the standard design correctly, applying that information to simulation models, and processing the results.

Any building features or systems that cannot be modeled in a compliance software program shall show compliance using prescriptive forms.

1.7 Approval Process

1.7.1 Application Checklist

The following items shall be included in an application package submitted to the Energy Commission for software approval:

- Compliance Software Vendor Certification Statement. A copy of the statement contained in Appendix A, signed by the software vendor, certifying that the software meets all Energy Commission requirements, including accuracy and reliability when used to demonstrate compliance with the energy standards.
- Computer Runs. Copies of the computer runs specified in Chapter 3 of this manual on machine-readable form as specified in Chapter 3 to enable verification of the runs.
- Help System and/or User's Manual. The vendor shall submit a complete copy of the help system and/or software user's manual, including material on the use of the software for compliance.
- Copy of the Compliance Software and Weather Data. A machine-readable copy of the software for random verification of compliance analyses. The vendor shall provide weather data for all 16 climate zones.
- TDV Factor Documentation. The software shall be able to apply the TDV multipliers described in Reference Appendix JA3.
- Application Fee. The vendor shall provide an application fee of \$1,000.00 as authorized by § 25402.1(b) of the Public Resources Code, payable to the "State of California" to cover costs of evaluating the application and to defray reproduction costs.

A cover letter acknowledging the shipment of the completed application package should be sent to:

Executive Director
California Energy Commission
1516 Ninth Street, MS-39
Sacramento, CA 95814-5512

Two copies of the full application package should be sent to:

Compliance Software Nonresidential Certification
California Energy Commission
1516 Ninth Street, MS-26
Sacramento, CA 95814-5512

Following submittal of the application package, the Energy Commission may request additional information under Title 24, Part 1, § 10-110. This additional information is often necessary due to complexity of software. Failure to provide such information in a timely manner may be considered cause for rejection or disapproval of the application. A resubmittal of a rejected or disapproved application will be considered a new application, and must include a new application fee.

1.7.2 Types of Approval

This manual addresses two types of software approval: full program approval (including amendments to programs that require approval) and approval of new program features and updates.

If software vendors make a change to their programs as described below, the change must be approved by the Energy Commission. Furthermore, any software change that affects the energy use calculations for compliance, the modeling capabilities for compliance, the format and/or content of compliance forms, or any other change that would affect compliance ~~or reach~~ requires approval.

Changes that do not affect compliance ~~or reach~~, such as changes to the user interface, may follow a simplified or streamlined procedure for approval. To comply with this simpler process, the software vendor shall certify to the Energy Commission that the new program features do not affect the results of any calculations performed by the program; shall notify the Commission of all changes; and shall provide the Commission with one updated copy of the program and help system/user's manual. Examples of such changes include fixing logical errors in computer program code that do not affect the numerical results (bug fixes) and new interfaces.

1.7.2.1 Full Approval and Reapproval of Compliance Software

The Energy Commission requires program approval when candidate software has never been approved by the Commission, when the software vendor changes the program algorithms, or when any other change occurs that in any way affects the compliance results. The Commission may also require that all approved software be approved again whenever substantial revisions are made to the standards or to the Commission's approval process.

The Energy Commission may change the approval process and require that all software be approved again for several reasons, including:

- If the standards undergo a major revision that alters the basic compliance process, then software would have to be updated and reapproved for the new process.
- If new analytic capabilities come into widespread use, then the Energy Commission may declare them to be required software capabilities, and may require all software vendors to update their programs and submit them for reapproval.

When reapproval is necessary, the Energy Commission will notify all software vendors of the timetable for renewal, a new version of this manual will be published, and the Commission will provide instructions for reapproval.

Reapproval shall be accompanied by a cover letter explaining the type of amendment(s) requested and copies of other documents, as necessary. The timetable for reapproval of amendments is the same as for full program approval.

1.7.2.2 Approval of New Features and Updates

Certain types of changes may be made to previously approved nonresidential software through a streamlined procedure, including implementing a computer program on a new machine and changing executable program code that does not affect the results.

Modifications to previously approved software, including new features and program updates, are subject to the following procedure:

- The software vendor shall prepare an addendum to the compliance supplement or software user's manual when new features or updates affect the outcome or energy efficiency measure choices, describing the change to the software. If the change is a new modeling capability, the addendum shall include instructions for using the new modeling capability for compliance.
- The software vendor shall notify the Energy Commission by letter of the change that has been made to the software. The letter shall describe in detail the nature of the change and the reason it is being made. The notification letter shall be included in the revised compliance supplement or software user's manual.
- The software vendor shall provide the Energy Commission with an updated copy of the software and include any new forms created by the software (or modifications in the standard reports).

The Energy Commission will respond within 45 days. The Energy Commission may approve the change, request additional information, refuse to approve the change, or require that the software vendor make specific changes to either the compliance supplement addendum or the software program itself.

With Energy Commission approval, the vendor may issue new copies of the software with the compliance supplement addendum, and notify software users and building officials.

1.7.3 Challenges

Building officials, program users, program vendors, Energy Commission staff, or other interested parties may challenge any nonresidential software approval. If any interested party believes that a compliance program, an algorithm or method of calculation used in a compliance program, a particular capability, or other aspect of a program provides inaccurate results or results that do not conform to the criteria described in his manual, the party may challenge the program.

1.7.4 Decertification of Compliance Software Programs

The Energy Commission may decertify (rescind approval of) an alternative calculation method through the following means:

- All software programs are decertified when the standards undergo substantial changes, which occur about every three years.
- Any software can be decertified by a letter from the software vendor requesting that a particular version (or versions) of the software be decertified. The decertification request

shall briefly describe the nature of the program errors or "bugs" that justify the need for decertification.

- Any "initiating party" may begin decertifying any software according to the steps outlined below. The intent is to include a means whereby unfavorable software tests, serious program errors, flawed numeric results, improper forms, and/or incorrect program documentation not discovered in the certification process can be verified and use of the particular software version discontinued. In this process, there is ample opportunity for the Energy Commission, the software vendor, and all interested parties to evaluate any alleged problems with the software program.

NOTE 1: The primary rationale for a challenge is unfavorable software tests, which means that for some particular building design with a set of energy efficiency measures, the software fails to meet the criteria used for testing software programs described in Chapter 3.

NOTE 2: Another challenge rationale is flawed numeric results, where the software meets the test criteria in Chapter 3, in particular, when software fails to properly create the standard design ~~baseline~~ building.

Following is a description of the process for challenging software or initiating a decertification procedure:

1. Any party may initiate a review of software approval by sending a written communication to the Executive Director. (The Energy Commission may be the initiating party for this type of review by noticing the availability of the same information listed here.)

The initiating party shall:

- State the name of the software and the program version number(s) that contain the alleged errors.
 - Identify concisely the nature of the alleged errors in the software that require review.
 - Explain why the alleged errors are serious enough in the effect on analyzing buildings for compliance to justify a decertification procedure.
 - Include appropriate data on any media compatible with Windows 7 or newer and/or information sufficient to evaluate the alleged errors.
2. The Executive Director shall make a copy or copies of the initial written communication available to the software vendor and interested parties within 30 days.
 3. Within 75 days of receipt of the written communication, the Executive Director may request any additional information needed to evaluate the alleged software errors from the party who initiated the decertification review. If the additional information is incomplete, this procedure will be delayed until the initiating party submits complete information.
 4. Within 75 days of receipt of the initial written communication, the Executive Director may convene a workshop to gather additional information from the initiating party, the software vendor, and interested parties. All parties will have 15 days after the workshop to submit additional information regarding the alleged program errors.
 5. Within 90 days after the Executive Director receives the application or within 30 days after receipt of complete additional information requested of the initiating party, whichever is later, the Executive Director shall either:

- Determine that the software need not be decertified.
 - Submit to the Energy Commission a written recommendation that the software be decertified.
6. The initial written communication, all relevant written materials, and the Executive Director's recommendation shall be placed on the consent calendar and considered at the next business meeting after submission of the recommendation. The matter may be removed from the consent calendar at the request of one of the Commissioners.
 7. If the Commission approves the software decertification, it shall take effect 60 days later. During the first 30 days of the 60-day period, the Executive Director shall send out a notice to building officials and interested parties announcing the decertification.

All initiating parties have the burden of proof to establish that the review of alleged software errors should be granted. The decertification process may be terminated at any time by mutual written consent of the initiating party and the Executive Director.

As a practical matter, the software vendor may use the 180- to 210-day period outlined here to update the software program, get it reapproved by the Commission, and release a revised version that does not have the problems initially brought to the attention of the Energy Commission. The software vendor may wish to be the initiating party to ensure that a faulty program version is taken off the market.

1.8 Vendor Requirements

Each vendor shall meet all of the following requirements as part of the software approval and as part of an ongoing commitment to users of the particular program.

1.8.1 Availability to Energy Commission

All software vendors are required to submit at least one fully working program version of the software to the Energy Commission. An updated copy or access to the approved version of the software shall be kept by the Commission to maintain approval for compliance use of the software.

The Energy Commission agrees not to duplicate the software except for analyzing it, for verifying building compliance with the compliance software, or for verifying that only approved versions of the software are used for compliance.

1.8.2 Enforcement Agency Support

Software vendors shall provide a copy of the software user's manual/help system to all enforcement agencies who request one in writing.

1.8.3 User Support

Software vendors shall offer support to their users with regard to the use of the software for compliance or reach purposes. Vendors may charge a fee for user support.

1.8.4 Compliance Software Vendor Demonstration

The Energy Commission may request that software vendors offer a live demonstration of the capabilities of their software. One or more demonstrations may be requested before approval is granted.

2. General Modeling Procedures

2.1 General Requirements for User-Entered Data

2.1.1 General

This document lists the building descriptors that are used in the compliance simulation. Users must provide valid data for all descriptors that do not have defaults specified and that apply to parts of the building that must be modeled.

2.1.2 Building Envelope Descriptions

The user shall provide accurate descriptions for all building envelope assemblies including exterior walls, windows, doors, roofs, exterior floors, slab-on-grade floors, below-grade walls, and below-grade floors. The user shall provide data for all of the required descriptors listed in Section 5.5 that correspond with these assemblies. However, the following exception applies:

- Exterior surfaces with an azimuth orientation and tilt differing by no more than 45° that is otherwise the same, may be described as a single surface or described as using multipliers. This specification would permit a circular form to be described as an octagon.

2.1.3 Space Use Classification

The user must designate space use classifications that best match the uses for which the building or spaces are designed. Space use classifications determine the default occupant density, occupant activity level, receptacle power, service water heating, lighting load, area-based minimum outdoor ventilation air, daylighting setpoints, and operating schedules used in the analysis. Process loads and refrigeration loads are also provided for applicable space types.

The user must specify the space use classifications using the area category method. The area category method uses the area categories in the standard design, which were developed for lighting requirements. The area category method requires area category entry of floor area and space use designations. More than one building area category may be used if the building is a mixed-use facility.

The user may override the default assumptions for some building descriptors dependent on the space use classification with supporting documentation. Details are provided in Section 5.4 of this manual.

2.1.4 Treatment of Descriptors Not Fully Addressed by This Document

This document provides input and rating rules covering a full range of energy-related features encountered in commercial buildings. However, this goal is unlikely to ever be achieved due to the large number of features that must be covered, and the continuous evolution of building materials and technologies. Building features or systems not covered in this manual must apply for approval via the exceptional calculation method to the Energy Commission. This manual may be amended with provisions to model additional features or HVAC systems during the code cycle. When this occurs, it is the responsibility of the software vendor to pass the necessary acceptance tests and apply for approval for the new building feature(s).

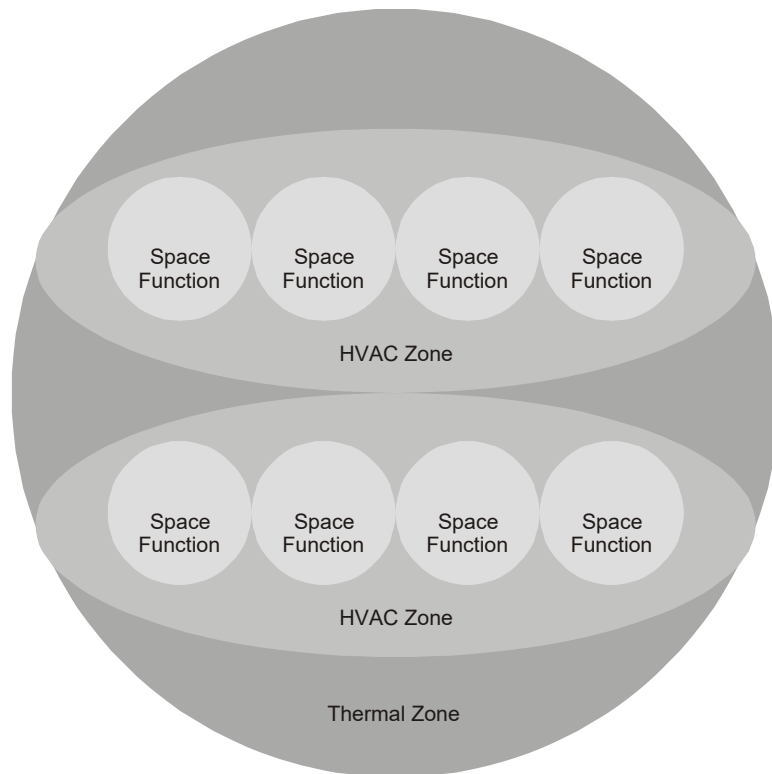
2.2 Thermal Zones, HVAC Zones, and Space Functions

2.2.1 Definitions

An *HVAC zone* is a physical space within the building that has a thermostat and zonal system for maintaining thermal comfort. HVAC zones are identified on the HVAC plans. HVAC zones should not be split between thermal zones; however, a thermal zone may include more than one HVAC zone.

A *space function* is a subcomponent of a thermal zone that has specific standard design lighting requirements and for which there are associated defaults for outside air ventilation, occupancy, receptacle loads, and hot water consumption. An HVAC zone may contain more than one space function. Appendix 5.4A lists the space functions that may be used with the software. Daylighted areas should be assigned to specific spaces, even if they have the same classification from Appendix 5.4A, so that lighting reductions due to daylighting can be determined at the appropriate resolution.

Figure 1 shows the hierarchy of space functions, HVAC zones, and thermal zones.

Figure 1: Hierarchy of Space Functions, HVAC Zones, and Thermal Zones

2.3 Software Modeling Requirements for Zones

2.3.1 Required Zone Modeling Capabilities

For California compliance, software shall accept input for and be capable of modeling a minimum of 50 thermal zones, each with a control. Compliance software may use zone multipliers for identical zones.

2.3.2 Modeling Requirements for Unconditioned Spaces

Unconditioned space is enclosed space that is neither directly nor indirectly conditioned. Examples include stairways, warehouses, unoccupied adjacent tenant spaces, attached sunspaces, attics, and crawl spaces.

Unconditioned spaces shall be modeled if they are part of the permitted space. All applicable envelope information shall be specified in a similar manner to conditioned space.

If unconditioned space is not a part of the permitted space, the space may be either explicitly modeled or the impact thereof on the permitted space may be approximated by modeling the space as outdoor space. For unconditioned spaces that are explicitly modeled, all internal gains and operational loads (occupants, water heating, receptacle, lighting, and process loads) shall be modeled as specified in Appendix 5.4A.

Return air plenums are considered indirectly conditioned spaces and shall be modeled as part of the adjacent conditioned space with equipment, lighting, and occupant loads at zero.

Indirectly conditioned spaces can either be occupied or unoccupied. For spaces that are unoccupied, such as plenums, attics, or crawlspaces, lighting, receptacle, and occupant loads shall be zero. For spaces that can be occupied, such as stairwells or storage rooms, modeling assumptions shall be taken from Appendix 5.4A.

Unconditioned spaces may not be located in the same zone as conditioned spaces. Conditioned spaces and indirectly conditioned spaces may be located in the same zone; when this occurs, the indirectly conditioned spaces will assume the space temperature schedule of the conditioned space.

2.3.3 Space Use Classification Considerations

Thermal zones shall be combined only if the spaces have similar space conditioning requirements and operating schedules. Space function inputs, as how they translate to thermal zone and HVAC system analysis assumptions, are defined by the following rules:

Schedule Group: There are many different schedule groups defined in Appendix 5.4B for California compliance. Each schedule group defines building specific hourly profiles for thermostat setpoints, HVAC system availability, occupancy, lighting, and so forth.

Space Functions: Each building space is assigned one space function. Design internal loads and other space function input assumptions, including the assigned schedule group described above, are defined in Appendix 5.4A. The schedule group and the schedule values for each space function are prescribed for compliance analysis.

These space functions are common to many different building types and, therefore, the user can assign any of the available schedule groups defined in Appendix 5.4B. This addresses the issue of conflicting schedule profiles if these common functions are combined into a single thermal zone or served by the same HVAC system as surrounding zones. In the event the user does not assign a schedule group to these common space types, default assumptions are defined in the Appendix 5.4B.

Thermal Zones: Spaces can be combined into thermal zones. In this situation, peak internal loads and other design inputs for the thermal zone are determined by weight-averaging the space function design inputs by floor area. When spaces are combined into thermal zones, the thermal zone schedules (occupancy, HVAC schedule, lighting schedule, space setpoint schedule), are based on the schedule group of the predominant space function on the building floor (by floor area) included in the thermal zone.

HVAC Systems: In many cases, more than one conditioned thermal zone is served by an HVAC system, which has scheduled availability (ON or OFF) to address the occupancy and internal load patterns of the thermal zones it serves. For systems that serve more than one thermal zone, the HVAC schedule group and availability schedule are determined by the most predominant schedule group (by floor area) represented in the thermal zones served.

The schedule group in the standard design is defined for each building story according to the predominant space function type and the schedule group assignment in Appendix 5.4A. Residential spaces and covered process spaces shall be served by dedicated systems, separate from nonresidential spaces.

2.4 Unmet Load Hours

This manual uses the term “unmet load hours” (UMLH) as a criterion for sizing equipment and for other purposes. The concept of unmet load hours applies to thermal zones but is summed

for hours whenever any thermal zone in the building has unmet loads. For a thermal zone, it represents the number of hours during a year when the HVAC system serving the thermal zone is unable to maintain the setpoint temperatures for heating and/or cooling. During periods of unmet loads, the space temperature drifts above the cooling setpoint or below the heating setpoint. A thermal zone is considered to have ~~unmet load hours~~ (UMLH) if the space is outside the throttling range for heating or cooling. The throttling range is defined in Chapter 5 as the space temperature difference between no cooling and full cooling, or between no heating and full heating. It is assumed that the cooling and heating setpoints are “centered” on the throttling range so that a cooling setpoint of 75°F results in an acceptable temperature band of 74°F to 76°F. The throttling range is fixed at 2°F for simulating both the standard design and proposed design.

An UMLH can occur only during periods when the HVAC system is scheduled to operate. UMLH are accounted for in each zone of the building. No zone in the building should exceed the maximum allowed UMLH.

UMLH can occur because fans, air flows, coils, furnaces, air conditioners, or other equipment are undersized. UMLH can also occur due to user errors, including mismatches between the thermostat setpoint schedules and HVAC operating schedules or from other input errors, for instance, high internal gains or occupant loads. The term, as used in this manual, addresses only equipment that is undersized. It is the responsibility of the user to address other causes of UMLH in the proposed design.

UMLH apply to thermal zones that contain any space type that is normally occupied. Thermal zones that contain only the space types listed below will not have UMLH applied to them:

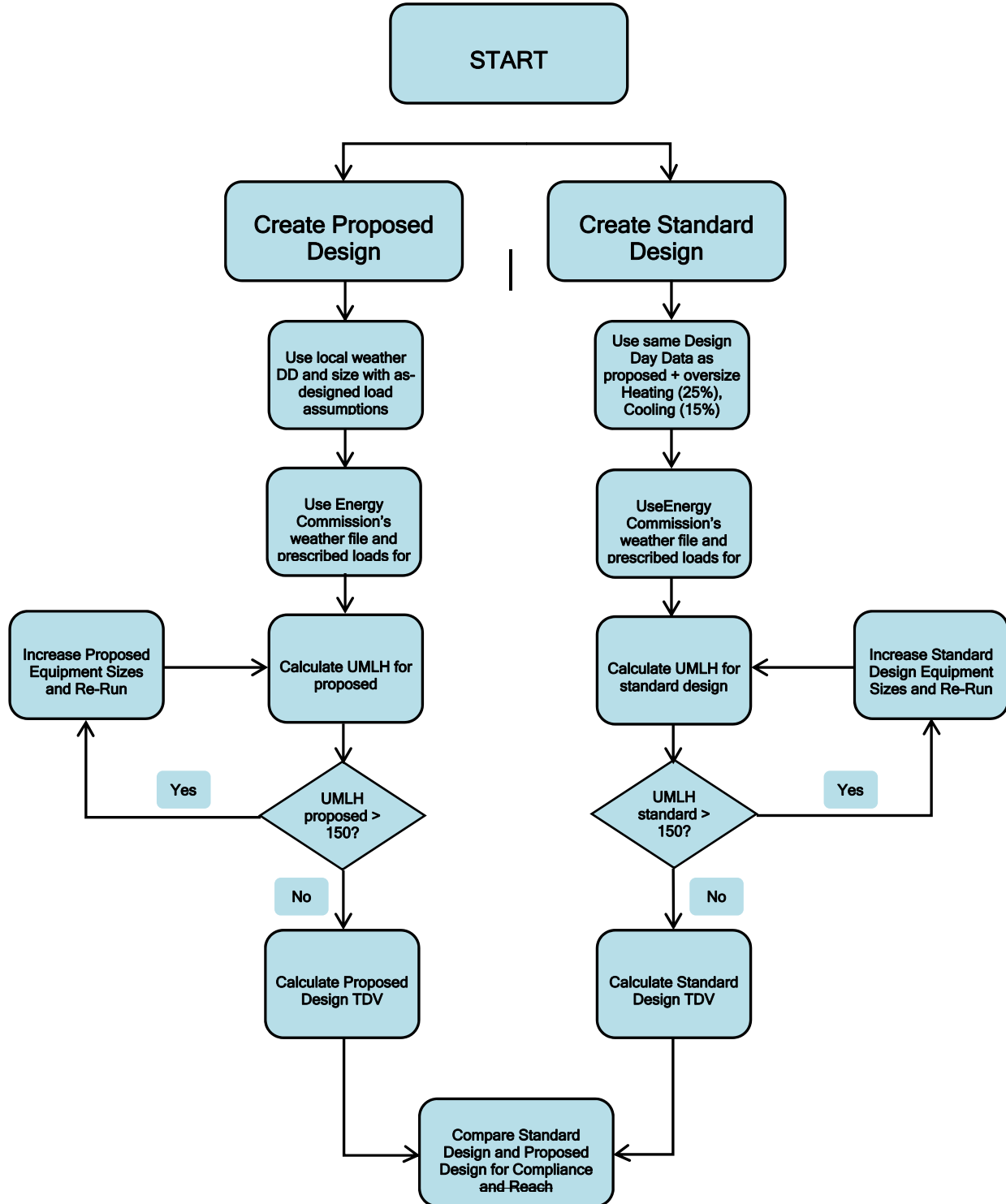
- Commercial and industrial storage areas
- Corridors, restrooms, stairs, and support areas
- Electrical, mechanical, telephone rooms
- Laundry rooms
- Locker/dressing rooms
- Parking garage areas
- Unoccupied gross floor areas

Spaces that are not subject to any UMLH checks or restrictions are listed in Appendix 5.4A.

Calculation Procedures

The general calculation procedure is illustrated below in Figure 2. The proposed design TDV energy use is compared to the standard design.

Figure 2: Calculation Process for Title 24 Compliance



1. The process begins with a detailed description of the proposed design. Information is provided in enough detail to enable an estimate of annual energy use for a typical weather year. This information includes the building envelope, lighting systems, HVAC systems, water heating systems, and other important energy-using systems. This collection of information is referred to in this manual as building descriptors. Details on the building descriptors are provided in Chapter 5.
2. Before the calculations are performed, some of the building descriptors are modified for the proposed design to incorporate prescribed modeling assumptions. Prescribed modeling assumptions include occupant density, equipment power density, ventilation rates, and water heating loads.
3. The next step is to make a simulation of the proposed design to determine how well the heating and cooling loads are being satisfied. The indicator is UMLH, the number of occupied hours during the year when the space temperature in one or more thermal zones is outside the throttling range. A large number of hours indicate that the equipment is undersized.
4. Test the number of UMLH and proceed only if the hours for each zone in the building are fewer than or equal to 150 for the year of the proposed design simulation.
5. If the UMLH are greater than 150 for the year, then user adjusts the proposed building simulation model to reduce the UMLH to fewer than or equal to 150; the user has the option to continue with the simulation but the thermal zones that exceed 150 hours will be reported on the compliance form as an exceptional condition. No zone, other than the irregularly occupied space types listed above, may exceed 150 UMLH. If the problem is heating, then the size of the boiler or furnace may need to be increased. If the problem is cooling, then the size of the coils or chillers may need to be increased. It is up to the designer to adjust equipment sizes as necessary. In some cases, adjusting the zone airflows may solve the unmet load issue.
6. If the UMLH are fewer than or equal to 150, then the final simulation is performed. If no changes are made in the model, this may be the same simulation in step 3. These calculations produce the results that are compared to the standard design, which is calculated in steps 7 through 16.
7. Create the standard design following the rules in this manual. The standard design has the same floor area, number of floors, and spatial configuration as the proposed design; however, systems and components are modified to be in minimum compliance with the standard design. The HVAC systems for the standard design are established according to rules in this manual and depend on the primary building activity (residential or nonresidential), the floor area, and the number of stories. See Section 5.1.
8. Sizing calculations are performed for the standard design and heating equipment is oversized by 25 percent and cooling equipment by 15 percent.
9. The standard design is simulated to determine the number of UMLH. This process is the same as performed for the proposed design in step 3.
10. The number of unmet UMLH for the standard design is then tested to see if they are greater than 150 for any zone(s). This is not likely since the heating and cooling equipment is oversized by 15 percent for cooling, and 25 percent for heating in step 8.

11. If the UMLH in the standard design are greater than 150, then equipment capacity in the standard design is increased so that the unmet hours are fewer than or equal to 150. See the discussion below on how equipment sizes are increased.
12. Once the tests on UMLH are satisfied, then the energy consumption of the standard design is calculated. If the tests on unmet hours are satisfied the first time through, this step is the same as step 9.
13. Finally, the proposed design TDV energy use and standard design TDV energy use are compared for compliance or reach.

2.5 HVAC Capacity Requirements and Sizing

To ensure that the simulated space-conditioning loads are adequately met, adequate capacity must be available in each of the components of the HVAC system; for example, supply-air flow rates, cooling coils, chillers, and cooling towers. If any component of the system is incapable of adequate performance, the simulation may understate the required energy inputs for space conditioning and report unmet load hours. Adequate capacities are required in the simulations of both the proposed design and standard design. The subsections below describe the procedures that shall be followed to ensure that both versions of the design are simulated with adequate space-conditioning capacities.

The UMLH requirement may be updated to prevent HVAC systems that show as undersized according to compliance software modeling constraints from receiving a compliance credit. This does not mandate that a specific cooling or heating capacity be specified; that is the role of the engineer of record for the building. With this change, if the proposed design appears as undersized, the user will be prompted to adjust plant, system, and/or zonal capacities as needed to meet the UMLH criteria.

The special case of a building designed with no cooling system (typically, in a temperate coastal climate) is accommodated by the compliance software automatically adding a minimally compliant packaged constant volume single zone system.

2.5.1 Specifying HVAC Capacities for the Proposed Design

As shown in ~~Figure 2~~~~Figure 2~~Figure 2, the proposed design shall have no more than 150 UMLH for any thermal zone. If this limit is exceeded, the software allows the user to make changes to the proposed design building description to bring the UMLH equal to or below 150. This process is not automated by the software. There are two tests that must be met to avoid excess UMLH:

- Space loads must be satisfied. Space temperatures in all thermal zones must be maintained within one-half of the throttling range (1°F with a 2°F throttling range) of the scheduled heating or cooling thermostat setpoints. This criterion may be exceeded for no more than 150 hours for a typical year.
- System loads must be satisfied. Plant equipment must have adequate capacity to satisfy the HVAC system loads. This criterion may be exceeded for no more than 150 hours for a typical year.

If either the space or system loads do not meet the above criteria, the equipment in the proposed design shall be resized by the user to meet the criteria. If the space conditioning criteria are not met because the HVAC equipment in the proposed design lacks the capability to provide either heating or cooling, equipment capable of providing the needed space conditioning must be specified by the user.

Equipment sizes for the proposed design shall be entered into the model by the energy analyst and shall agree with the equipment sizes specified in the construction documents. When the simulations of these actual systems indicate that specified space conditions are not being adequately maintained in one or more thermal zone(s), the user shall be prompted to make changes to equipment sizes or zones as necessary. This occurs when the UMLH exceed 150 for the year. The use of equipment sizes that do not match the actual equipment sizes as indicated on construction documents triggers an exceptional condition that is noted on the compliance forms.

2.5.2 Sizing Equipment in the Standard Design

For sizing heating and cooling equipment capacities, the compliance software shall use design day schedules as specified in Section 5.3. For cooling capacity sizing, compliance software shall use the OnDay schedule from Appendix 5.4B for occupant, lighting, and equipment schedules, respectively. For heating capacity sizing, compliance software shall use the OffDay schedule from Appendix 5.4B for occupant, lighting, and equipment schedules, respectively.

Equipment in the standard design is automatically oversized by the program (25 percent for heating and 15 percent for cooling). If the automatic oversizing percentage is not sufficient to meet demands, then UMLH are evaluated at the building level by looking at the UMLH for each of the thermal zones being modeled. The zone with the greatest number of UMLH shall not exceed 150.

If the total number of UMLH for cooling and/or heating exceeds 150, then equipment capacities of cooling and/or heating equipment must be increased by the software incrementally.

1. The first step is to determine whether heating or cooling UMLH are the bigger problem. If heating UMLH are the bigger problem, upsize the heating equipment capacity. If cooling UMLH is the problem, upsize the cooling equipment capacity.
2. If the cooling is undersized, the equipment is resized by first increasing the design airflow of all zones with significant UMLH (greater than 150 for a zone) by 10 percent, then the equipment capacity for the system(s) serving the affected zones is increased to handle the increased zone loads. For a central plant, the chiller(s) and towers are resized proportionally to handle the increased system loads.

If heating is undersized, the same procedure is followed with zone terminal units resized first, heating equipment second, and finally boilers as necessary.

The capacity of the boiler or furnace shall be increased in 5 percent increments and the simulation rerun until the loads are met. For heat pumps, the capacity of the coil is increased so that the additional load is not met by auxiliary heat.

2.5.3 Handling Proposed Design ~~With~~with No HVAC Equipment

~~This section is under review, to ensure that designs for buildings with no cooling system specified in temperate climates can be accommodated in the performance compliance approach.~~

If a compliance model does not contain an HVAC system, and if the number of UMLH exceeds 150 hours for any zone, then the compliance software shall prompt the user to enter a cooling capacity for an HVAC cooling system. The system type and efficiency characteristics shall match that of the standard design system. The compliance software shall make an appropriate note on compliance documentation indicating that the modeled HVAC system does not match

design requirements. If the compliance software provides a means for the user to identify that the building has no cooling system, this information is reported on the compliance reports.

2.6 Ventilation Requirements

Design decisions regarding outside air ventilation shall be based on Section 120.1 of the Energy Standards. If local codes do not apply, minimum values from Appendix 5.4A shall be used. Chapter 5.6 of the ACM has additional information on the ventilation requirements used in the building descriptors for the proposed and standard design. While no compliance credit can be claimed for reducing ventilation rates in the proposed design below the required levels, the user can specify higher ventilation rates in the proposed design.

3. Compliance Software Test Requirements

This chapter contains the procedures used to test and certify vendor's compliance software as acceptable for compliance with Title 24, Part 6. Compliance software must also follow all modeling rules specified in Chapter 5. The tests used to verify software functionality and accuracy of simulation results are referred to as the reference method. The tests fall into the following categories:

- Tests to verify that the software is evaluating thermal loads and the response of the HVAC systems to these loads in an acceptable manner. These tests reference *ASHRAE Standard 140-2007, Standard Method of Test for Evaluation of Building Energy Analysis Computer Programs*.
- Tests that verify that compliance software is capable of modeling envelope, lighting, HVAC, and water heating efficiency features and provides precise estimates of energy tradeoffs and reasonably accurate predictions of building energy consumption.
- Tests to verify that the standard design (~~baseline-building~~) building is created correctly. For example, the ~~baseline-standard design~~ HVAC system is properly specified, other components of the ~~baseline standard design~~ are correctly defined, and rules that fix and restrict inputs (such as schedules and plug loads) are properly applied. These tests do not verify simulation outputs but may require simulations to be run to specify inputs that depend on system sizing.
- The reference method is designed to cover representative software functionality for building envelope, space uses, lighting, daylighting, HVAC, and water heating, both for simulation performance and for proper implementation of ACM rules specified in Chapter 5. The Energy Commission reserves the right to add ruleset implementation tests or software sensitivity tests to verify existing or future compliance software requirements. Moreover, the Energy Commission reserves the right to adjust the passing criteria for the software sensitivity tests to reflect the capabilities of commonly available energy simulation programs.

3.1 General Requirements

3.1.1 Scope

The compliance software must satisfy the requirements contained in this section.

The compliance software shall be capable of modeling at least 50 thermal zones.

The compliance software shall be capable of modeling at least 15 HVAC systems.

3.1.2 Calculation Methods

The compliance software shall calculate the annual consumption of all end uses in buildings, including fuel and electricity for:

- HVAC (heating, cooling, fans, and ventilation).
- Lighting (both interior and exterior).
- Receptacles and miscellaneous electric.
- Service water heating.

- Process energy uses.
- All other energy end uses that typically pass through the building meter.

The compliance software shall perform a simulation on an hourly time interval (at a minimum) over a one-year period (8,760 hours) with the ability to model changes in weather parameters, schedules, and other parameters for each hour of the year. This is achieved by specifying a 24-hour schedule for each day of the week plus holidays.

3.1.2.1 Calculating Design Loads

The software shall be capable of performing design load calculations for determining required HVAC equipment capacities and air and water flow rates using accepted industry calculation methods for the standard design.

3.1.2.2 Checking Simulation Output for Unmet Loads

The software shall be capable of checking the output of the energy analysis module for the proposed design to ensure that space conditions are maintained within the tolerances specified (maximum of 150 thermal zone UMLH per year).

3.1.2.3 Adjusting Capacities

For the standard design ~~baseline~~ building, the software shall be capable of modifying capacities, temperatures, or flow rates for standard design ~~baseline~~ building HVAC system components causing excessive UMLH according to the procedures in Chapter 2.

3.1.2.4 Error Handling

The software shall send a warning to the user when unmet loads exceed 150 hours and provide information to the user describing the error that has occurred.

3.1.3 Climate Data

The compliance software shall perform simulations using the official Energy Commission weather files and design conditions documented in Joint Appendix 2.

The compliance software shall calculate solar radiation on exterior surfaces on an hourly basis from the values of direct normal irradiance and diffuse horizontal irradiance contained in the climate data, taking ground reflectance into account.

3.1.5 Time Dependent Valued (TDV) Energy

The compliance software shall be capable of applying the Energy Commission TDV multipliers for each hour of the simulation. See Energy Commission Joint Appendix 3.

3.1.6 Thermal Mass

The calculation procedures used in the compliance software shall account for the effect of thermal mass on loads due to occupants, lights, solar radiation, and transmission through building envelope on the amount of heating and cooling required to maintain the specified space temperature schedules; and on variation in space temperature.

3.1.7 Modeling Space Temperature

The compliance software shall incorporate a dynamic simulation of space temperature that accounts for:

- Dynamics in change in heating and cooling setpoint temperatures.

- Deadband between heating and cooling thermostat settings.
- Temperature drift in transition to setback or setup thermostat schedules.
- Temperature drift in periods when heating or cooling capability are scheduled off.
- Temperature drift when heating or cooling capability of the system is limited by heating or cooling capacity, air flow rate, or scheduled supply air temperature.
- Indirectly conditioned thermal *zones*, where the temperature is determined by internal loads, heat transfer through building envelope, and heat transfer between thermal zones.

3.1.8 Heat Transfer Between Thermal Zones

The compliance software shall be capable of modeling heat transfer between a thermal zone and adjacent thermal zones.

The compliance software shall account for the effect of this heat transfer on the space temperature, space conditioning loads, and resulting energy use in the thermal zone and in adjacent thermal zones.

3.1.9 Control and Operating Schedules

The compliance software shall be capable of modeling control and operating schedules that can vary by:

- The hour of the day.
- The day of the week.
- Holidays, which are treated as a special day of the week.

The compliance software shall be capable of explicitly modeling all of the schedules specified in Appendix 5.4B of this manual.

3.1.10 Loads Calculation

The load calculations described in this section relate to the simulation engine, and not to the procedure used by the design engineer to size and select equipment.

3.1.10.1 Internal Loads

The compliance software shall be capable of calculating the hourly cooling loads due to occupants, lights, receptacles, and process loads.

The calculation of internal loads shall account for the dynamic effects of thermal mass.

The compliance software shall be capable of simulating schedules for internal loads in the form given in Appendix 5.4B.

The simulation of cooling load due to lights shall account for:

- The effect of the proportion of radiant and convective heat, which depends on the type of light and on the dynamic response characteristic.
- A portion of heat from lights going directly to return air; the amount depends on the type and location of fixture.

3.1.10.2 Building Envelope Loads

The compliance software shall calculate heat transfer through walls, roofs, and floors for each thermal zone, accounting for the dynamic response due to thermal characteristics of the particular construction as defined in the Building Descriptors in Chapter 5.

The calculation of heat transfer through walls and roofs shall account for the effect of solar radiation absorbed on the exterior surface, which depends on orientation and absorptance of the surface.

The compliance software shall calculate heat transfer through windows and skylights, accounting for both temperature difference and transmission of solar radiation through the glazing.

Calculation of cooling load due to transmission of solar radiation through windows and skylights shall account for:

- The variation of thermal properties of the fenestration system with ambient temperature.
- Orientation (azimuth and tilt of surface).
- The effect of shading from overhangs or side fins.

3.1.10.3 Infiltration

The compliance software shall be capable of simulating infiltration that varies by the time of day and day of the week. Schedules are provided in Appendix 5.4B.

3.1.11 Systems Simulation

3.1.11.1 General

The compliance software shall be capable of modeling:

- The standard design-baseline building systems defined in Chapter 5.
- The lighting, water heating, HVAC, and miscellaneous equipment detailed in Chapter 5.
- All compulsory and required features, as detailed in Chapter 5.

The capability to model multiple zone systems shall allow at least 15 thermal zones to be served by one multiple-zone system.

The compliance software shall be capable of modeling plenum air return.

3.1.11.2 HVAC Zone Level Systems

The compliance software shall be capable of simulating the effect on space temperature and energy use of:

- Limited capacity of terminal heating devices.
- Limited capacity of terminal cooling devices.
- Limited rate of air flow to thermal zones.

3.1.11.3 HVAC Secondary Systems and Equipment

The compliance software shall be capable of simulating the effect on energy use and space temperature in thermal zones served by the HVAC system of:

- Limited heating capacity.
- Limited cooling capacity.

The simulation of HVAC systems shall account for:

- Temperature rise of supply air due to heat from supply fan, depending on the location of the fan.
- Temperature rise of return air due to heat from return fan.
- Temperature rise of return air due to heat from lights to return air stream.
- Fan power as a function of supply air flow in variable air volume systems.

3.1.11.4 HVAC Primary Systems and Equipment

The compliance software shall be capable of simulating the effect on energy use of limited heating or cooling capacity of the central plant system.

If the compliance software is not capable of simulating the effect of limited heating or cooling capacity of the central plant system on space temperature in affected thermal zones, then it shall issue a warning message when loads on the central plant system are not met.

3.1.11.5 Equipment Performance Curves

The compliance software shall be capable of modeling the part-load efficiency and variation in capacity of equipment as follows:

- Furnaces as a function of part load
- Boilers as a function of part load, supply hot water temperature, and return hot water temperature
- Water-cooled compressors including heat pumps and chillers as a function of part load, evaporator fluid, or air temperature and condensing fluid temperature
- Air-cooled compressors including heat pumps, direct expansion cooling and chillers as a function of part load, ambient dry-bulb temperature, and wet-bulb temperature returning to the cooling coil
- Evaporative cooling systems as a function of ambient wet-bulb temperature
- Cooling towers as a function of range and ambient wet-bulb temperature

3.1.11.6 Economizer Control

The compliance software shall be capable of modeling integrated air- and water-side economizers.

3.2 Special Documentation and Reporting Requirements

3.2.1 Building Envelope

3.2.1.1 Roof Radiative Properties

The user shall enter three-year aged roof reflectance and emittance for roofs that have been certified by the Cool Roof Rating Council. The software shall report the product identification number(s) of any roofing products used on the building, so that aged reflectance and emittance can be verified by the code official.

3.2.2 Interior Lighting

3.2.2.1 Regulated Interior Lighting Power

Complete lighting plans and space plans are required for the tailored method. Prescriptive compliance forms for the tailored method shall be developed and the plans examiner shall verify them.

Whenever any of the special allowance exceptions (footnotes in Standards Table 140.6-C) are claimed, the compliance software shall indicate on the compliance forms that verification is required.

3.2.2.2 Indoor Lighting Power (see 5.4.4)

Compliance software shall print all applicable lighting forms and report the lighting energy use and the lighting level (watts/ft²) for the entire project. Compliance software shall report “no lighting installed” for nonresidential spaces with no installed lighting. Compliance software shall report “default residential lighting” for housing units of high-rise residential buildings and hotel/motel guest rooms.

If the modeled lighting power density (LPD) is different than the actual LPD calculated from the fixture schedule for the building, the compliance software shall model the larger of the two values for sizing the mechanical systems and for the compliance run. Compliance software shall report the larger value on PRF-01. Lighting levels schedules shall be adjusted by any lighting control credit watts, if input by the user.

Lighting power is not modeled in unconditioned spaces that are modeled but lighting in those spaces is required to meet the prescriptive requirements for regulated unconditioned spaces, such as commercial and industrial storage spaces, and parking garages. When these types of spaces are entered, the compliance software must report in the Special Features section that these spaces must comply with the prescriptive requirements for such spaces.

3.2.2.3 Design Illumination Setpoint

Spaces that have low design illuminance levels, below the ranges specified in Appendix 5.4A, shall provide documentation showing the design illuminance to be used as the daylight illumination setpoint.

3.2.3 HVAC Exceptional Conditions

3.2.3.1 Equipment Sizing

When any proposed equipment size for secondary equipment or central plant equipment does not match the equipment size listed on construction documents, an exceptional condition shall be reported on compliance forms.

3.2.3.2 Process and Filtration Pressure Drop Allowance

Any nonzero value entered for supply fan process and filtration pressure drop are flagged as an exceptional condition in the compliance documentation. ~~An allowance of up to 1-inch water column (in. wc) is provided in the static pressure for process and filtration requirements (such as clean rooms). Any amount may be claimed for the proposed building, but the standard design (baseline) is adjusted by a maximum of 1 in.wc.~~

3.2.3.3 Natural Ventilation Specified

Anytime natural ventilation is specified by the user for the proposed design, ~~either for residential or nonresidential~~ and Hotel/Motel spaces, the software shall report an exceptional condition that the conditions in Section 120.1(~~b~~c) of the energy standards must be met:

~~120.1(b)1A. Naturally ventilated spaces shall be permanently open to and within 20 feet of operable wall or roof openings to the outdoors, the operable area of which is not less than 5 percent of the conditioned floor area of the naturally ventilated space. Where openings are covered with louvers or otherwise obstructed, operable area shall be based on the free unobstructed area through the opening.~~

~~EXCEPTION to Section 120.1(b)1A: Naturally ventilated spaces in high-rise residential dwelling units and hotel/motel guest rooms shall be open to, and within 25 feet of, operable wall or roof openings to the outdoors.~~

~~120.1(b)1B. The means to open required operable openings shall be readily accessible to building occupants whenever the space is occupied.~~

3.3 ASHRAE Standard 140-2007 Tests

This method of testing is provided for analyzing and diagnosing building energy simulation software using software-to-software and software-to-quasi-analytical-solution comparisons. The method allows different building energy simulation programs, representing different degrees of modeling complexity, to be tested by comparing the predictions from other building energy programs to the simulation results provided by the compliance software in question.

Compliance software must pass ASHRAE 140-~~2007~~2014 tests, but these tests are not part of the reference method.

3.4 Rule Set Implementation Tests

3.4.1 Introduction

The tests in this section are intended to verify that the software correctly constructs the standard design model, and applies rules of the Nonresidential ACM appropriately to the proposed and standard design models. The ruleset implementation tests cover representative portions of the rules for building envelope, lighting, daylighting, space use data, and HVAC. For each test, a set of three models is defined:

- User Model – The user model contains the user inputs for the as-designed building. In most cases, the values for the proposed design will be taken from user inputs with no modification. However, there are some cases where the building input is prescribed for the proposed design, or constrained by mandatory minimums or other rules.
- Proposed Design Model – The proposed model is defined by the rules in the *ACM Reference Manual*, created by the vendor software, and is the building modeled for compliance. This model takes user inputs for building geometry, building envelope, lighting, and HVAC, and is used in the compliance simulation.
- Standard Design Model – This is the baseline model defined by the *ACM Reference Manual* modeling rules and is the basis for comparison that determines whether a building passes compliance using the performance method.

These tests do not require that simulation outputs be verified, but they do require that simulation input files for the proposed design and standard design are properly constructed according to the rules in the *ACM Reference Manual*. Some tests require that sizing runs be performed for HVAC inputs with values that are dependent on autosized standard design systems.

3.4.2 Overview

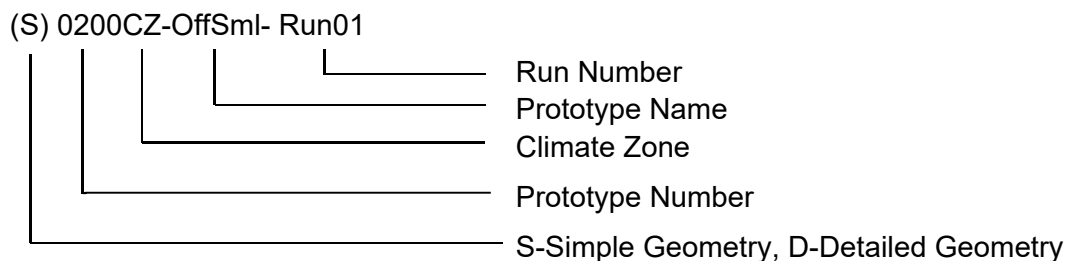
The test runs described in this section represent the Title 24 Nonresidential ACM code compliance calculation and use the following prototype models: small office building, medium office building, large office building, warehouse building, and small hotel. For further details on the prototype models, refer to Appendix 3C. Each standard design test case shall be created by modifying the prototype model as described in Section IV of this document. The modified prototype model shall form the proposed case for each test run. The standard design model shall be generated by compliance software as per the rules in the *Nonresidential ACM Manual*. The standard design and proposed model files for each test case shall then be evaluated to verify that:

- The standard design building envelope constructions are correctly substituted for exterior opaque surfaces and fenestrations.
- The fenestration area in the standard design building is reduced in accordance with the *Nonresidential ACM Manual*, when the proposed design fenestration area is greater than 40 percent of the exterior wall.
- The skylight area in the standard design building is adjusted in accordance with the *Nonresidential ACM Manual*, when applicable.
- Default schedules of operation are applied for both the standard design building and the proposed design building.
- The proposed and standard design cases use the same defaults, or tailored inputs, for internal loads as required by the *Nonresidential ACM Manual*.
- The standard design building lighting system is correctly specified, and that exterior lighting is modeled.
- Receptacle loads and process loads are modeled according to the rules in the *Nonresidential ACM Manual*.
- The standard design building uses the correct system types as prescribed in Table 5 of the *Nonresidential ACM Manual*.
- An economizer (of the right type) is included in the standard design building, if required.
- The primary and secondary standard design building systems are properly specified and sized.
- Fan brake horsepower is correctly specified for the standard design building.

- Prescribed modeling assumptions are applied for both the standard design building and the proposed design building.
- Overhangs are modeled in the proposed design for test case 7 but not the standard design building.
- Conditioned, indirectly conditioned, and Unconditioned spaces are modeled.
- Other standard design building specifications and/or modeling assumptions are correctly applied.

As the software developer verifies the various test conditions, the input and output files should be annotated with comments or other methods to demonstrate that the modeling rules specified in the *Nonresidential ACM Manual* are correctly applied. Software developers should use the spreadsheets included in Appendix 3C to report the results of these tests. These annotated files shall then be submitted to the Energy Commission for further evaluation. Any errors discovered shall be corrected by making modifications to the software, the runs shall be repeated, and the new results shall be annotated for submittal to the Commission.

The standard design tests are labeled using the format:



3.4.3 Ruleset Implementation Tests

The following tests shall be performed to verify that the compliance software correctly creates the standard design model and applies modeling rules as per the requirements of the *Nonresidential ACM Reference Manual*. The user model shall be created by modifying the appropriate prototype model type. The standard design model for each test case shall be generated automatically by the compliance software.

The characteristics of the user model and inputs to be verified in the proposed and standard design models are described in Appendix 3.

3.4.3.1 Results Comparison

The applicant shall perform all tests specified in Section IV and report the outputs in the forms provided in Appendix 3C. The standard design for some inputs, such as cooling efficiency and pump power, depend upon the autosizing of the HVAC equipment. The ruleset implementation tests do not check that the autosized capacity matches the reference method but that the standard design input is properly defined in relation to the autosized capacity.

3.5 Software Sensitivity Tests

This section details the eligibility requirements for an applicant simulation program for use as compliance software. A series of quantitative tests called software sensitivity tests shall be performed to measure the change in energy consumption when changing specified input parameters. Applicant software results will be compared against predetermined reference results to demonstrate that the applicant software is acceptable for use in code compliance. All the test cases described here shall be performed and results summarized in the forms contained in Appendix 3B.

3.5.1.1 Overview

The applicant software shall perform a suite of software sensitivity tests to demonstrate that the performance is acceptable for code compliance. The applicant software test results shall be compared against a base case called the reference test case. The reference test case is the corresponding match of a particular test case simulated already on EnergyPlus engine. The reference test case results, as determined by the Energy Commission, are tabulated in Appendix 3B.

Test cases specific for simplified geometry are only for software with 2D inputs for building geometry. Software with a 2D geometry approach shall seek certification by submitting the simplified geometry test cases. In addition, they are also required to produce results for HVAC tests that will be compared against the HVAC reference test results that are common for both simplified and detailed geometry.

The test cases will assess the sensitivity of the applicant software to various inputs ranging from envelope thermal conductance to HVAC system performance. Each case tests the effect of the input component on building end-use energy and annual TDV. The following six building components will be tested through a series of tests:

- Opaque envelope
- Glazing
- Lighting
- Daylighting
- Receptacle loads
- HVAC system parameters

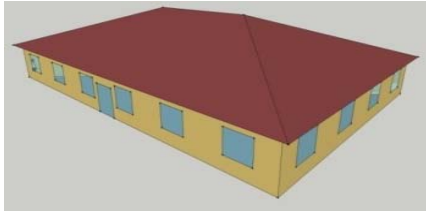
3.5.1.2 Prototype Models

The software sensitivity tests are performed on four prototype models that are a subset of the U.S. Department of Energy (DOE) prototype building models developed by PNNL for analysis of ASHRAE Standard 90.1. The prototype models are EnergyPlus model input files of the DOE prototype building models, modified to comply with the requirements of Title 24. The prototype models will be the reference baseline runs for the test cases. The applicant software shall replicate the building models below using the same inputs as the prototype models. The models so replicated will be the applicant baseline models for the test cases.

A summary of the prototype models is provided in Appendix 3A. Detailed input files of the reference baseline models are available from the Energy Commission's Building Energy Efficiency Software Consortium web page at <http://bees.archenergy.com/>.

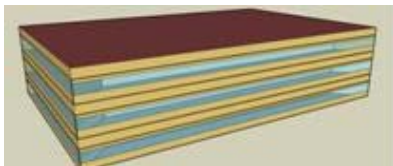
Prototype models used for software sensitivity test cases are:

- Small Office (02000CZ-OffSml):



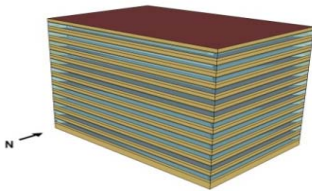
The small office building model is a single-story rectangular building of 5,500 square feet. It has punched windows and a hipped roof with an attic. There are five zones, each served by packaged single-zone air conditioner units. For more details refer Appendix 3A. This prototype is used for simple geometry test cases only.

- Medium Office Building (0300CZ-OffMed):



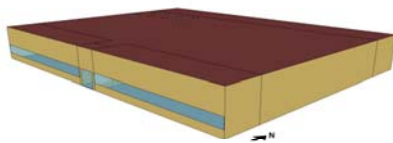
The medium office building model is a three-story rectangular building with an overall area of 53,600 square feet. It has a window-to-wall ratio of 33 percent with fenestration distributed evenly across all four façades. The zones are served by DX cooling and gas furnace heating with hot water reheat. For more details refer Appendix 3A. This prototype is used for both detailed geometry and simple geometry test cases.

- Large Office Building (0400CZ-OffLrg):



The large office building has 12 floors and a basement floor with glass windows with a window-to-wall ratio of 40 percent on the above-grade walls. The total area of the building is 498,600 square feet. The HVAC system type used is a variable air volume (VAV) system. For more details refer Appendix 3A.

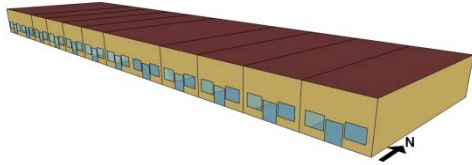
- Stand-Alone Retail (0500CZ-RetlMed):



The stand-alone retail building is a single-story rectangular building measuring 178 feet by 139 feet. The total area is 24,695 square feet. Windows are located only on the street-facing façade and occupy 25.4 percent of that façade. The building is divided into five thermal zones that are

served by packaged single-zone systems as described in Appendix 3A. This prototype is used for both detailed geometry and simple geometry test cases.

- Strip Mall Building Strip Mall-PSZ System (1000CZ-RetlStrp):



The strip mall building area is 22,500 square feet. It has 10 zones each with rooftop units. The building has windows in the street-facing façade and has an overall window-to-wall ratio of 10.5 percent. For more details refer to Appendix 3A.

3.5.1.3 Climate Zones

The software sensitivity test cases use building models for 5 of the 16 California climate zones. Most tests are performed with two or three climate zones to capture the sensitivity of the input characteristics to extremes in weather conditions. The test cases are performed on Climate Zones 6, 15, and 16, which represent mild, hot, and cold climates respectively. Daylighting tests are performed on Climate Zone 7, which has a high annual daylighting potential, and Climate Zone 1, representative of the climate with the least annual daylighting potential.

Table 1: Climate Zones Tested

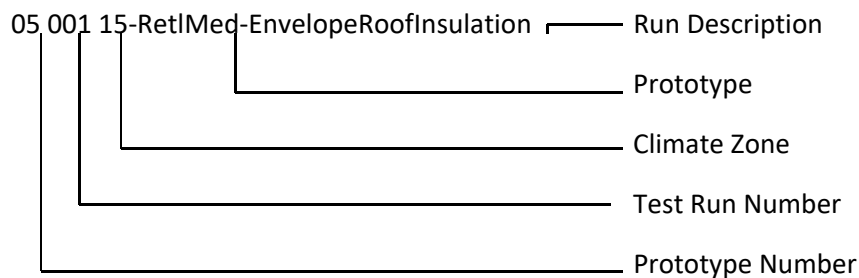
Climate Zone	Example City/Weather File
1	Arcata/ARCATA_725945
6	Torrance/TORRANCE_722955
7	San Diego Lindbergh/ SAN-DIEGO-LINDBERGH_722900
15	Palm Springs/PALM-SPRINGS-INTL_722868
16	Blue Canyon/BLUE-CANYON_725845

3.5.1.4 Labeling Test Runs

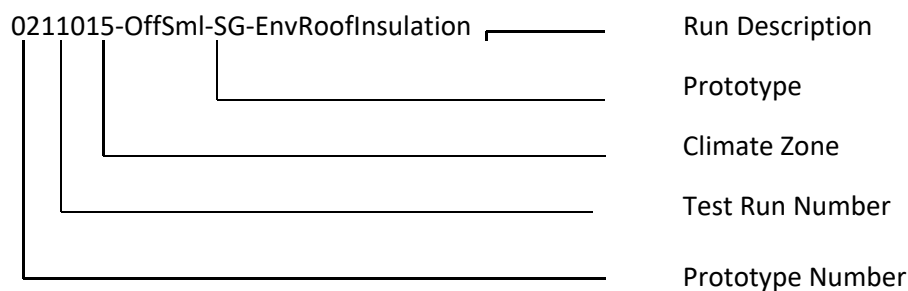
Each test case in the software sensitivity test is labeled uniquely to make it easier to keep track of the runs and to facilitate analysis. The following scheme is used:

XXYYYYZZ-Prototype-RunDescription
 Where XX denotes the Prototype Number
 YY denotes Test Run Number
 ZZ denote Climate zone

Example:



Example for Simple Geometry Test case



3.5.1.5 Test Criteria

Applicant software vendors shall perform a series of computer runs. Each of these runs shall be a systematic variation of the applicant base case model as described in Section 3.5.1.7. The applicant test case results will be compared to the reference results to verify that applicant software meets the requirements of the ACM. Simulation results for each test case will be compiled in forms provided in Appendix 3B. Compiled results will include annual site energy consumption for each end use, overall site energy consumption, total unmet load hours and annual TDV and percent variation of annual TDV and total end use site energy.

The annual TDV percent variation shall be calculated using the formula:

$$TDV\% = (TDV_b - TDV_n) / TDV_b$$

Where, TDV% is the TDV percent variation,

TDV_n is the annual TDV for test case number n and

TDV_b is the annual TDV for the base case run.

To be accepted, the applicant software should fulfill the passing criteria as determined by the Energy Commission:

For each test case the change in energy for the test case must be in the same direction as the Reference Method test case result, and must be equal to the Reference Method test case percentage change in TDV energy, plus or minus 0.5 percent of baseline TDV energy.

If any of the tests required for the Title 24 compliance feature set fails to meet these criteria, the applicant software will not be accepted for compliance use.

3.5.1.6 Reporting Test Results

For each test case, the TDV energy use of the modeled building is reported (kBtu/ft²), along with the TDV energy use attributed to the major fuel types (electricity, gas), site energy use, and energy end-use intensity for the regulated end uses (cooling, heating, lighting, and so forth). The following energy totals are reported:

1. Annual TDV EUI (kBtu/ft²)
2. Annual Site EUI – Electricity (kWh/ft²)
3. Annual SiteEUI – Natural Gas (therm/ft²)
4. Annual Total End Use Site Energy EUI – kBtu/ft²
Site Energy End Uses
5. Site Energy: Heating (kBtu/ft²)
6. Site Energy: Cooling (kBtu/ft²)
7. Site Energy: Interior Lighting (kBtu/ft²)
8. Site Energy: Interior Equipment (kBtu/ft²)
9. Site Energy: Fans (kBtu/ft²) (Airside Fans, does not include tower fans)
10. Site Energy: Pumps (kBtu/ft²)
11. Site Energy: Towers (kBtu/ft²)Water heating (kBtu/ft²)
12. TDV percent Variation – this field is used for the compliance test
13. Total End Use Site Energy percent - percentage change in site energy use
14. Pass/Fail – test fails if it does not meet passing criteria
15. Unmet load hours (UMLH) – defined as the zone with the most UMLH
 - a. Reference Model Occupied UMLH
 - b. Applicant Model Occupied UMLH
 - c. Reference Model Number of Zones with excess UMLH (>150)
 - d. Applicant Model Number of Zones with excess UMLH (>150)

Figure 3: Results Spreadsheet Excerpt From Appendix 3B

Test Case	11		12		13		Pass/Fail
	Water Heating (kBtu/sqft)		TDV % variation		Total End Use Site Energy % Variation		
	Applicant Model	Reference Model	Applicant Model	Reference Model	Applicant Model	Reference Model	
CZ15StandAloneRetailBaseline		7.58					
01CZ15StandAloneRetail Envelope RoofInsulation		7.57		2.52%		2.37%	Fail
02CZ15StandAloneRetail Envelope WallInsulation		7.58		0.74%		0.64%	Fail
03CZ15StandAloneRetail Envelope Heavy		7.57		-5.16%		-3.69%	Fail
CZ16StandAloneRetailBaseline		9.73					
04CZ16StandAloneRetail Envelope RoofInsulation		9.73		2.01%		2.04%	Fail
05CZ16StandAloneRetail Envelope WallInsulation		9.73		0.55%		0.79%	Fail
06CZ16StandAloneRetail Envelope Heavy		9.72		-2.84%		-0.32%	Fail
CZ06StandAloneRetailBaseline		8.65					
07CZ06StandAloneRetail Envelope RoofInsulation		8.65		0.81%		0.57%	Fail
08CZ06StandAloneRetail Envelope WallInsulation		8.65		0.49%		0.48%	Fail
09CZ06StandAloneRetail Envelope Heavy		8.64		-1.25%		0.72%	Fail
CZ15MediumOfficeBaseline		2.03					
10CZ15MediumOffice Envelope FloorslabInsulation		2.03		-0.77%		-0.53%	Fail
11CZ15MediumOffice Envelope Infiltration		2.03		-0.26%		-0.29%	Fail
CZ15MediumOfficeBaseline		2.03					

The results spreadsheet provides the results of the reference method for each test and provides a column (in orange) for the vendor to report the results from the candidate compliance software.

The variation from baseline section of the spreadsheet shows the percentage change in TDV energy use (kBtu/ft²) from the base case for testing. The percentage must be within the passing criteria for the candidate software to pass this test.

Also reported is the number of UMLH during occupied hours for the building. An UMLH for a specific zone in Title 24 compliance is defined as any hour when the zone has an unmet cooling or heating load. This is typically reported by the software for each zone in the building. For the test case results, two unmet load hour metrics must be reported: the number of UMLH for the zone with the greatest number of UMLH, and the number of zones that fail the *ACM Reference Manual* criteria for acceptable UMLH. (Any zones with greater than 150 hours fail the criteria.)

The spreadsheet where the results are documented indicates whether the candidate software passes or fails a test. The result in column AL of the spreadsheet indicates whether the candidate software passes the test.

3.5.1.7 Software Sensitivity Test Cases

Test cases assess the energy impact of one or more of the building or system input characteristics on the baseline model. Each test suite consists of a series of unique test cases aimed to test the effect of a specific characteristic on building energy performance. Simulations are grouped according to test criteria and subgrouped based on the reference model type to allow for direct comparison of results. For each test case, the applicant software will modify the applicant baseline model with specific inputs as described in the test case description section.

The test cases are simulated on multiple California weather files to evaluate the sensitivity of the building or system input to extremes in climate. Results of the test case runs and the TDV percentage variation over the baseline run shall be compiled and compared against the reference results.

Detailed descriptions of the standard design models are provided in [Appendix 3A](#). CBECC input files for all baseline and test case models are available from the Energy Commission's, Building Energy Efficiency Software Consortium web page <http://bees.archenergy.com>. Details on each test description can be found in Appendix 3A under the test criteria tab.

3.5.1.8 Results Documentation

The applicant shall perform simulations for all tests specified above. A detailed description of each test case is provided in Appendix 3A and report results in the forms is provided in Appendix 3B. Some of the prototype models have variants of the baseline model. These include 1) stand-alone duct loss baseline, a variant of the stand-alone retail model; 2) StripMall-PTAC model, a variant of StripMall-PSZ model; and 3) StripMall-Fan Coil model, a variant of StripMall-PSZ model. For details refer to Appendix 3A.

Three test cases are presented here as an example: one for building envelope, one for lighting and daylighting, and one for HVAC. The development of the other required test cases follows the same process.

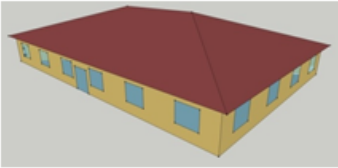
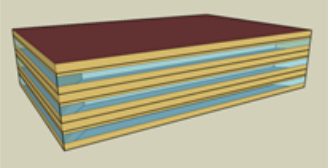
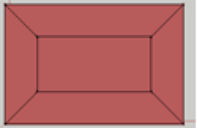
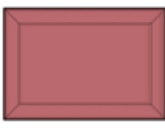
Example Test Case: 0301315-OffMed-GlazingWindowSHGC

For this test case, the U-factor and solar heat gain coefficient (SHGC) of all vertical fenestration is decreased by 20 percent. The prototype used for this test case is a medium office building.

Before the test cases are run, the first step is to generate the prototype models for the four reference buildings, which are required for all of the tests. The four prototype models are defined in the prototype model spreadsheet of Appendix 3A. (Note: While many of the prototype model inputs are based on Title 24 prescriptive requirements, the prototype models do not exactly conform to minimum Title 24 requirements but are intended to test the sensitivity of the candidate software simulation results to common variations in building inputs.)

Step 1. Generate prototype models. The first step is to generate the prototype building for the medium office building. The detailed specification of the medium office building is listed in Appendix 3A. A portion of the inputs are shown in Figure 4. The prototypes are defined for the reference models on the prototype model tab of Appendix 3A.

Figure 4: Prototype Model Definition From Appendix 3A

Prototype Description	Small Office Building	Medium Office Building
Vintage	New Construction	New Construction
Location	CZ-6/15/16	CZ-3/6/15/16
Fuel Type	gas, electricity	gas, electricity
Total Floor Area (sq feet)	5500 (90.8 ft x 60.5ft)	53600 (163.8 ft x 109.2 ft)
Building shape		
Aspect Ratio	1.5	1.5
Number of Floors	1	3
Window Fraction (Window-to-Wall Ratio)	24.4% for South and 19.8% for the other three orientations (Window Dimensions: 6.0 ft x 5.0 ft punch windows for all façades)	33% (Window Dimensions: 163.8 ft x 4.29 ft on the long side of facade 109.2 ft x 4.29 ft on the short side of the façade)
Window Locations	evenly distributed along four façades	evenly distributed along four façades
Shading Geometry	none	none
Azimuth	non-directional	non-directional
Thermal Zoning	Perimeter zone depth: 16.4 ft. Four perimeter zones, one core zone and an attic zone. Percentages of floor area: Perimeter 70%, Core 30%	Perimeter zone depth: 15 ft. Each floor has four perimeter zones and one core zone. Percentages of floor area: Perimeter 40%, Core 60%
		
Floor to floor height (feet)	10	13

The prototype model definition in the spreadsheet contains links to other input definitions:

Rows 19, 26, 45: Links to layer-by-layer exterior construction assembly definitions in the *Construction Assembly* tab

Row 52: links to layer-by-layer interior construction assembly definitions in the *Construction Assembly* tab

Step 2. Define base case and variation for test run.

The base case is defined as the starting point for each test. In many tests, the base case will be one of the prototype models. However, in some cases a variation of the prototype may serve as the base case for the test.

For this test, the base case is found by looking at the test criteria tab of Appendix 3A.

Figure 5: Base Case Definition From Appendix 3A

Y4 fx Decrease U value & SHGC of windows by 20% compared to baseline case					
	A	U	V	W	X
2	Test Run Name	20CZ06MediumOffice Envelope FloorslabInsulation	21CZ06MediumOffice Envelope Infiltration	22CZ06MediumOffice Glazing WindowU	23CZ06MediumOffice Glazing WindowSHGC
3	Baseline	CZ06MediumOffice	CZ06MediumOffice	CZ06MediumOffice	CZ06MediumOffice
4	Test Description	Change Floor slab F factor to 0.45	Increase Exterior Wall Infiltration by 10% compared to baseline case	Decrease U value of windows by 20% compared to baseline case	Decrease SHGC of windows by 20% compared to baseline case
5	Location	CZ06	CZ06	CZ06	CZ06

For this test, the baseline field in row 3 of the *Test Criteria* tab shows that the baseline is *CZ06MediumOffice*, the medium office prototype in Climate Zone 6.

This same *Test Criteria* tab shows the input(s) to be verified, which are highlighted in purple. For this test, the SHGC of all vertical fenestration is reduced by 20 percent, from 0.25 to 0.20.

Figure 6: Input Parameter Variation for Medium Office From Appendix 3A

A	U	V	W	X
Test Run Name	20CZ06MediumOffice Envelope FloorslabInsulation	21CZ06MediumOffice Envelope Infiltration	22CZ06MediumOffice Glazing WindowU	23CZ06MediumOffice Glazing WindowSHGC
Baseline	CZ06MediumOffice	CZ06MediumOffice	CZ06MediumOffice	CZ06MediumOffice
Test Description	Change Floor slab F factor to 0.45	Increase Exterior Wall Infiltration by 10% compared to baseline case	Decrease U value of windows by 20% compared to baseline case	Decrease SHGC of windows by 20% compared to baseline case
Location	CZ06	CZ06	CZ06	CZ06
Dimensions				
Tilts and orientations		Refer MediumOffice		
Window				
Dimensions				
Glass-Type and frame				
U-factor (Btu / h * ft ² * °F)			0.29	
SHGC				0.2
Visible transmittance				
Operable area				

Step 3. Run the base case model and generate test results.

Once the base case model is developed, the simulation is run and the results are recorded onto the spreadsheet of test cases. See Appendix 3B.

The candidate software shall report electricity use by end use, gas use by end use, TDV energy, and UMLH. For compliance, UMLH are defined at the zone level, and the zone with the greatest number of UMLH must pass the criteria specified in the sizing procedure.

For the reference tests, the capacities and flow rates of the HVAC system can be found in Appendix 3A under the *Sizing Values* tab.

Step 4. Run the test case model (with the reduced SHGC) and report the results.

The model is rerun and the energy results and outputs are reported. The percentage change in energy use is reported.

Step 5. Report the change in regulated TDV energy use from the base case as a percentage change.

The reported percentage change in energy use from the candidate compliance software must fall within the passing criteria for the reference method.

4. Content and Format of Standard Reports

Consult the *Nonresidential Compliance Manual* for the reports required to be manually generated for any project. For nonresidential compliance, the PRF-01 report is generated by the software.

5. Building Descriptors Reference

5.1 Overview

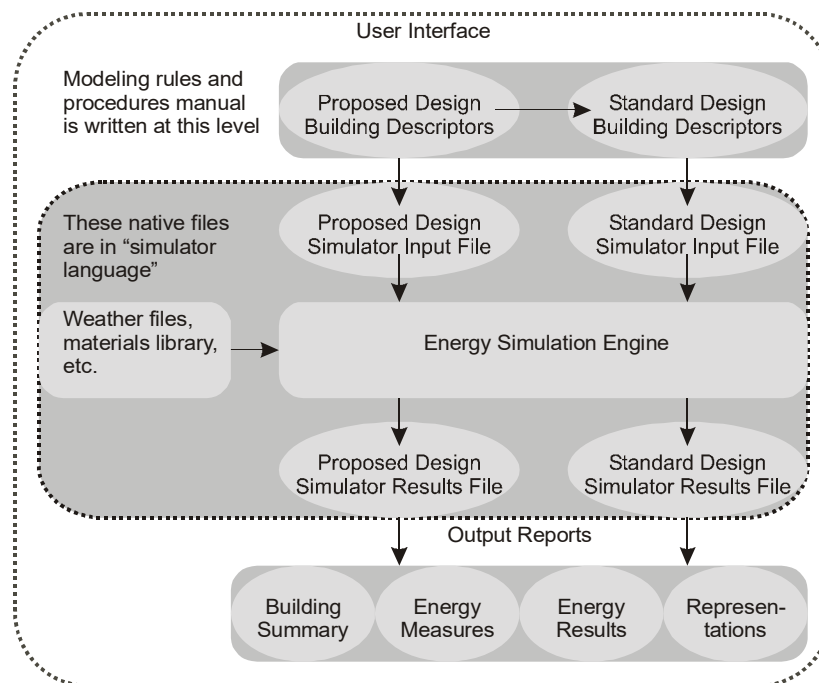
This chapter specifies, for each building descriptor, the rules that apply to the proposed design and to the standard design.

5.1.1 Definition of Building Descriptors

Building descriptors provide information about the proposed design and the standard design. In this chapter, the building descriptors are discussed in the generic terms of engineering drawings and specifications. By using generic building descriptors, this manual avoids bias toward one particular energy simulation engine. The building descriptors in this chapter are compatible with commonly used simulation software.

Each energy simulation program has a unique way of accepting building information. EnergyPlus uses a comma delimited data file called an IDF file. DOE-2 uses BDL (building design language) to accept information. It is the responsibility of the compliance software to translate the generic terms used in this chapter into the “native language” of the simulation program. [Figure 7](#) illustrates the flow of information.

Figure 7: Information Flow



5.1.2 HVAC System Map

The HVAC system in the standard design depends on the primary building activity, the size of the building, and the number of floors. Details about these systems are provided in subsequent sections.

Many of the building descriptors have a one-to-one relationship between the proposed design and the standard design; for example, every wall in the proposed design has a corresponding wall in the standard design. For HVAC systems, however, this one-to-one relationship generally does not hold. The HVAC system serving the proposed design and the standard design may be completely different, each with different components.

The HVAC system in the standard design shall be selected from Table 2: HVAC System Map, and be based on building type, number of floors, conditioned floor area, and heating source. Moreover, the selected system shall conform to the descriptions in **Table 5: System Descriptions**.

For systems 1, 2, 3, 7, 10, and 11, each thermal zone shall be modeled with a respective HVAC system. For systems 5, 6, and 9, each floor shall be modeled with a separate HVAC system. Floors with identical thermal zones and occupancies can be grouped for modeling. The standard design heating source is natural gas.

Table 2: HVAC System Map

Building Type	Standard Design
Residential or hotel/motel guestrooms in a building with three <u>seven</u> or fewer floors <u>above grade</u>	System 4-31 - <u>PTACSZAC</u>
Residential or hotel/motel guestrooms in a building with four <u>eight</u> or more floors <u>above grade</u>	System 2 - FPFC
<u>Retail buildings 2 floors or fewer</u>	<u>System 7 — SZVAV*</u>
Warehouse and light manufacturing space types (per the Appendix 5.4A Schedule column) that do not include cooling in the proposed design	System 9 - HEATVENT
Covered process	See <u>Table 4: System Map for Covered Processes</u>
<u>Healthcare Facilities</u>	<u>Same as the Proposed Design</u>
All other space types	See <u>Table 3: Nonresidential Spaces (Not Including Covered Processes)</u>

Table 3: Nonresidential Spaces (Not Including Covered Processes)

Building Area	Floors	Standard Design	Description
\leq <u>4025,000</u> ft ²	\leq <u>3</u> floors	<u>System 7 -</u>	<u>Packaged Single</u>

		<u>PSZSZVAV*</u>	<u>ZoneSingle Zone VAV</u>
	<u>>1 floor4 or 5 floors</u>	<u>System 5 - PVAV</u>	<u>Packaged VAV Unit</u>
	<u>> 5 floors</u>	<u>System 6 — VAVS</u>	<u>Built-up VAV Unit</u>
<u>1025,000 ft²–150,000 ft²</u>	<u>≤Any≤ 5 floors</u>	<u>System 5 - PVAV</u>	<u>Packaged VAV Unit</u>
	<u>> 5 floors</u>	<u>System 6 — VAVS</u>	<u>Built-up VAV Unit</u>
<u>>150,000 ft²</u>	<u>1 floor</u>	<u>SZVAV</u>	<u>Single-zone VAV Unit</u>
	<u>>1 floorAny</u>	<u>System 6 - VAVS</u>	<u>Built-up VAV Unit</u>

* Single zone VAV system (SZVAV) shall have a minimum fan speed ratio of 0.5 if the standard design total cooling capacity ~~≥exceeds~~ 65 kBtu/h, and a minimum fan speed ratio of 1 (constant volume) for a standard design total cooling capacity ~~efless than 65 kBtu/h or lower~~; system ~~t~~shall have an integrated economizer if the standard design cooling capacity exceeds 54 kBtu/h.

Retail building Standard Design rules shall apply to zones on floors (building stories) whose predominant space type is retail.

Table 4: System Map for Covered Processes

Building Type or Space Type	Floors	Baseline-Standard Design System
Total computer room design cooling load is over 3,000,000 Btu/h Note: if the user chooses computer room for the space type and enters a receptacle load less than 20 W/ft ² , then the proposed and <u>standard design-baseline</u> shall use a receptacle load of 20 W/ft ² .	Any	System 10 – CRAH Unit
Computer rooms that do not meet the conditions for System 10, CRAH	Any	System 11 – CRAC Unit
Laboratory Space	Any	System 12 – LAB
Restaurant Kitchen	Any	System 13 – KITCH

Table 5: System Descriptions

System Type	Description	Detail
System 1 – <u>PTAGSZAC</u>	Packaged Terminal Air Conditioner <u>Residential Air Conditioner</u>	Ductless single zone DX unit with hot water natural gas boiler <u>Single zone system with constant volume fan, no economizer, DX cooling and furnace.</u> <u>Applicable to HRR and hotel motel guestrooms</u>
System 2 – FPFC	Four-Pipe Fan Coil	Central plant with terminal units with hot water and chilled water coils, with separate ventilation source
System 3 – PSZ	Packaged Single Zone	Single-zone constant volume DX unit with gas heating
System 4 – RESERVED		
System 5 – PVAV	Packaged VAV Unit	VAV reheat system; packaged variable volume DX unit with gas heating and with hot water reheat terminal units
System 6 – VAVS	Built-up VAV Unit	Variable volume system with chilled water and hot water coils, water-cooled chiller, tower and central boiler
System 7 – SZVAV	Packaged Single-Zone VAV Unit	Single-zone variable volume DX unit with variable-speed drive and gas heating
System 8 – RESERVED		
System 9 – HEATVENT	Heating and Ventilation Only	Gas heating and ventilation
System 10 – CRAH	Computer Room Air Handler	Built-up variable volume unit with chilled water, no heating
System 11 – CRAC	Computer Room Air Conditioner	Packaged variable volume DX unit with no heating
System 12 – LAB	Laboratory HVAC System	<u>For buildings with a total lab floor area less than 10,000 sf: a single zone VAV system (SZVAV), with variable-speed drive for supply fan, DX cooling and a gas furnace (CEC review)</u> For buildings with lab floor area of 10,000 sf or greater: For building floor area < 150,000 ft ² : packaged variable volume system with 100% OA and minimum

		<p>ventilation rate of 6 ACH</p> <p>For building Floor Area >= 150,000 ft², built-up VAV (VAVS) with water-cooled chiller and central boiler</p> <p>Gas furnace heating if building floor area is 10,000 ft² or less; hot water coils/boiler if greater than 10,000 ft²</p>
System 13 – KITCH	Kitchen HVAC System	<p>Dedicated makeup air unit (MAU) – CHW if building is VAVS, DX otherwise. Dedicated exhaust fan</p> <p>Gas furnace heating if building floor area is 10,000 ft² or less; hot water coils/boiler if greater than 10,000 ft²</p>

Separate Residential (24-hour) occupancies in mixed-use buildings are served by separate standard design systems than nonresidential space types. Also, spaces containing covered processes are served by dedicated standard design systems separate from systems serving other nonresidential space types. Examples include residential spaces located over retail and other similar conditions. For example, a 100,000 ft² building that has retail and restaurant on floor 1, offices on floors 2, 3, and 4, a 20-ton computer room on each office floor, and residential on floors 5, 6, and 7 would have the following systems in the standard design:

- A KITCH serving the restaurant
- One or more single zone VAV systems serving the retail spaces Retail spaces follow the system map, since the building has more than 2 stories
- A VAVS serving all retail and office spaces
- Separate CRAC systems serving each computer room
- Separate FPFC systems serving each residential space

The standard design baseline building shall have only one central chilled or hot water plant, so if there are multiple systems that incorporate a plant (for example, CRAH and VAVS), then a single plant shall serve all plant loads.

5.1.2.1 Additions and Alterations System Modification

For additions and alterations projects, the standard design building shall follow the same rules as the HVAC system map above, except that the building that will follow the logic of the system map rules may be the modeled building (the addition or alteration alone, or the addition or alteration and a portion of the existing building), or the entire building (the entire existing building, plus an addition, if present).

The decision on the existing building basis for applying the system map rules is:

General System Modification

The following rules apply to any building that has both heating and cooling systems.

1. Plant: If the change in plant cooling capacity exceeds 50 percent of the existing total cooling capacity of all cooling systems, the system map is based on the entire building characteristics. (See Section 5.2.2.)
2. Airside System: If the change in cooling capacity of the airside system (for example, air handling units, DX packaged units) of all cooling sources other than chilled water exceeds 50 percent of the existing rated cooling capacity for the building, then the HVAC system map is based on the entire building characteristics. Also, if the combined net cooling capacity of all

altered airside systems exceeds 90 percent of the building cooling capacity, then the HVAC system map is based on the entire building characteristics.

3. Zone Level: If the change in the cooling capacity of the zonal systems (for example, ~~PTAC~~ SZAC units, FPFC units) exceeds 50 percent of the rated total cooling capacity of all zonal systems in the existing building, then the HVAC system map is based on the entire existing building characteristics. Also, if the combined net cooling capacity of all altered zonal systems exceeds 90 percent of the building cooling capacity, then the HVAC system map is based on the entire building characteristics.

4. If none of these three conditions apply, then the HVAC system map is based on the building characteristics of the modeled building for additions and alterations compliance, which may be just a portion of the entire building.

Since some additions and alterations projects will trigger the HVAC system map for the standard design, the user must enter a minimum set of building characteristics for the entire building (existing plus any addition); existing building floor area and number of stories must be entered.

Heating-Only System Modification

The following rules apply to any building that has only heating-only systems.

1. Plant: If the change in plant heating capacity exceeds 50 percent of the existing total space heating capacity of all heating systems, the system map is based on the entire building characteristics.

2. Airside System: If the change in heating capacity of the airside system (unitary DX equipment, heat pumps, for example) of all heating sources other than heating hot water exceeds 50 percent of the existing rated cooling capacity for the building, then the HVAC system map is based on the entire building characteristics. Also, if the combined net heating capacity of all altered airside systems exceeds 90 percent of the building heating capacity, then the HVAC system map is based on the entire building characteristics.

3. Zone Level: If the change in the heating capacity of the zonal systems (~~PTAC~~ SZAC units, for example), exceeds 50 percent of the rated total heating capacity of all zonal systems in the existing building, then the HVAC system map is based on the entire existing building characteristics. Also, if the combined net cooling capacity of all altered zonal systems exceeds 90 percent of the building cooling capacity, then the HVAC system map is based on the entire building characteristics.

4. If none of these three conditions above apply, then the HVAC system map is based on the building characteristics of the modeled building for additions and alterations compliance, which may be just a portion of the entire building.

Since some additions and alterations projects will trigger the HVAC system map for the standard design, the user must enter a minimum set of building characteristics for the entire building (existing plus any addition): existing building floor area and number of stories must be entered.

5.1.3 Organization of Information

Building descriptors are grouped under objects or building components. A wall or exterior surface (an object) would have multiple building descriptors dealing with the geometry, thermal performance, and so forth. Each building descriptor contains the following pieces of information:

Building Descriptor Title	
<i>Applicability</i>	Information on when the building descriptor applies to the proposed design
<i>Definition</i>	A definition for the building descriptor
<i>Units</i>	The units that are used to prescribe the building descriptor; A “List” indicates that a fixed set of choices applies and the user shall be allowed to enter only one of the values in the list
<i>Input Restrictions</i>	Any restrictions on information that may be entered for the proposed design
<i>Standard Design</i>	This defines the value for the “standard design” or baseline building is applied for this building descriptor. A value of “same as proposed” indicates that the building descriptor is neutral, i.e., the value is set to match the proposed design value. In many cases, the value may be fixed or may be determined from a table lookup. In some cases, the input may not be applicable. For example, heat recovery effectiveness is not applicable because the standard design (baseline building) does not have heat recovery.

5.1.4 Special Requirements for Additions and Alterations Projects

Compliance projects containing additions and/or alterations require that the user designate each building component (envelope construction assemblies and fenestration, lighting, HVAC, and water heating) as either new, altered, or existing. Many of the building descriptors in Chapter 5 of this manual do not have explicit definitions for the standard design when the project is an addition and/or alterations project. For these terms, the standard design rules for existing, altered components follow the same rule as the standard design rule for new construction.

For example, the receptacle loads are prescribed for both the proposed design building and standard design building for a new construction compliance project. For additions or alterations to an existing building, since the rules are not explicitly defined in the building descriptor in Section 5.3.3, the same rules apply to the proposed design and standard design for the additions or alterations compliance project.

Building descriptors that are prescribed for the proposed and standard design models for new construction projects are also prescribed for the proposed and standard design models for additions and alterations projects.

For additions and alterations projects, there are three modeling approaches that can be taken when modeling the existing building:

1. Model the addition or altered portion alone. For this option, the addition or alteration is modeled as a stand-alone building, and the boundary or interface between the addition and/or alteration and the preexisting building is modeled as an adiabatic partition (an adiabatic wall, ceiling, roof or floor).

2. Model the entire existing building and any additions and alterations. For this option, the existing, unaltered components of the building would be modeled “as designed” (as specified by the user), with the standard design components modeled the same as the proposed design.
3. Model part of the existing building and any additions and alterations. For this option, all components of the existing, unaltered building (HVAC, lighting, envelope, spaces) would have to be distinguished from the components that are added and altered. The existing building components would be modeled “as designed” (as specified by the user), with the standard design components modeled the same as the proposed design. Added or altered building components would follow the rules for additions and alterations.

When either Option 1 or Option 3 is used, the adiabatic partitions shall not be considered as part of gross exterior wall area or gross exterior roof area for the window/wall ratio (WWR) and skylight/roof ratio (SRR) calculations.

5.2 Project Data

5.2.1 General Information

Project Name	
<i>Applicability</i>	All projects
<i>Definition</i>	Name used for the project, if one is applicable
<i>Units</i>	Up to 50 alphanumeric characters
<i>Input Restrictions</i>	Input is optional for the proposed design
<i>Standard Design</i>	Not applicable

Building Location	
<i>Applicability</i>	All projects
<i>Definition</i>	Street address, city, state, and zip code
<i>Units</i>	Up to 50 alphanumeric characters on each of two lines
<i>Input Restrictions</i>	Input is optional for the proposed design
<i>Standard Design</i>	Not applicable

Project Owner	
<i>Applicability</i>	All projects
<i>Definition</i>	Owner(s) of the project or individual or organization for whom the building permit is sought; should include name, title, organization, email, and phone number
<i>Units</i>	Up to 50 alphanumeric characters
<i>Input Restrictions</i>	Input is optional for the proposed design
<i>Standard Design</i>	Not applicable

Envelope Designer	
<i>Applicability</i>	All projects
<i>Definition</i>	Person responsible for the building design; information should include name, title, organization, email, and phone number
<i>Units</i>	Up to 50 alphanumeric characters
<i>Input Restrictions</i>	Input is optional for the proposed design
<i>Standard Design</i>	Not applicable

Mechanical Designer	
<i>Applicability</i>	All projects
<i>Definition</i>	Person responsible for the mechanical design; information should include name, title, organization, email, and phone number
<i>Units</i>	Up to 50 alphanumeric characters
<i>Input Restrictions</i>	Input is optional for the proposed design
<i>Standard Design</i>	Not applicable

Lighting Designer	
<i>Applicability</i>	All projects
<i>Definition</i>	Person responsible for the lighting design; information should include name, title, organization, email, and phone number
<i>Units</i>	Up to 50 alphanumeric characters
<i>Input Restrictions</i>	Input is optional for the proposed design
<i>Standard Design</i>	Not applicable

Documentation Author	
<i>Applicability</i>	All projects
<i>Definition</i>	Person responsible for inputting building information and performing the compliance analysis; information should include name, title, organization, email, and phone number
<i>Units</i>	Up to 50 alphanumeric characters
<i>Input Restrictions</i>	Input is optional for the proposed design
<i>Standard Design</i>	Not applicable

Date	
<i>Applicability</i>	All projects
<i>Definition</i>	Date of completion of the compliance analysis or the date of its most-recent revision
<i>Units</i>	Date format
<i>Input Restrictions</i>	Input is optional for the proposed design
<i>Standard Design</i>	Not applicable

Compliance Type	
<i>Applicability</i>	All projects
<i>Definition</i>	Type of compliance project (new construction, partial compliance or additions and alterations)
<i>Units</i>	<p>List:</p> <p>NewComplete: new construction project</p> <p>NewEnvelope: new construction, partial compliance with envelope</p> <p>NewEnvelopeAndLighting: new construction, partial compliance with envelope and lighting</p> <p>NewEnvelopeAndPartialLighting: new construction, partial compliance with envelope and lighting compliance for some spaces</p> <p>NewEnvelopeAndMechanical: new construction, partial compliance with mechanical and envelope only</p> <p>NewMechanical: new construction, partial compliance with mechanical. This is the complement of a partial compliance with envelope and lighting, which should have already been performed.</p> <p>NewMechanicalAndLighting: new construction, partial compliance with mechanical and lighting only. The building should have already passed an Envelope Only partial compliance.</p> <p>NewMechanicalAndPartialLighting: new construction, partial compliance with mechanical and lighting compliance for some spaces. The building should have already passed an Envelope Only partial compliance.</p> <p>ExistingAlteration: alteration project</p> <p>ExistingAdditionAndAlteration: project with both additions and alterations</p> <p>AdditionComplete: an addition modeled alone</p> <p>AdditionEnvelope: an existing building with partial envelope compliance for a new addition</p> <p>AdditionEnvelopeAndLighting: an existing building with partial envelope and lighting compliance for a new addition</p> <p>AdditionEnvelopeAndPartialLighting: an addition to an existing building that includes the building envelope and lighting for some of the spaces. For the spaces with lighting defined in the proposed design, the space function type must be defined as well. For undefined spaces, the lighting status is “future” and both the proposed design and standard design are set to match the prescriptive lighting power limits.</p> <p>AdditionEnvelopeAndMechanical: an addition modeled with as-designed envelope and mechanical components, but the interior lighting design has not yet been defined. For this option, interior lighting must comply prescriptively.</p>

	<p>AdditionMechanical: an addition modeled with the as-designed mechanical system, including any plant, system or zone level equipment, as well as ventilation. For this option, all envelope and lighting components are modeled as defined by the user in both the proposed design and <u>standardbaseline</u> designs.</p> <p>AdditionMechanicalAndLighting: an addition modeled with as designed mechanical and lighting equipment. In this design, all building envelope components in the standard design are set to match those in the proposed design, since the envelope is assumed to have complied via a separate permit.</p> <p>AdditionMechanicalAndPartialLighting: an addition modeled with as designed mechanical equipment and with lighting systems defined for part of the building. This model is the complement to the <u>AdditionEnvelopeAndParitalLighting</u> compliance option, since <u>Envelope and lighting compliance for the spaces not included in this compliance permit are assumed to have already been modeled (and permitted).</u></p>
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Same as proposed

5.2.2 Existing Building Classification

Existing Building Number of Stories	
<i>Applicability</i>	Additions and alterations
<i>Definition</i>	Total number of stories of the building (For information and reporting purposes only)
<i>Units</i>	Integer
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable
<i>Existing Buildings</i>	Not applicable

Existing Building Floor Area	
<i>Applicability</i>	Additions and alterations
<i>Definition</i>	Total floor area of an existing building, including any additions, if present (For information and reporting purposes only)
<i>Units</i>	ft ²
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable
<i>Existing Buildings</i>	Not applicable

5.2.3 Partial Compliance Model Input Classification

Earlier sections of this reference manual have described the available partial compliance scenarios. The compliance software that supports these scenarios must define the inputs for both the proposed design and the standard design for unpermitted portions of the building.

- 1) **Envelope Only:** The user specifies the building envelope and all spaces, space types, and thermal zones in the building. The standard design rules are applied to the envelope components. For all lighting and HVAC inputs, the proposed design values are prescribed and follow the rules for the standard design, including modeling the same HVAC systems determined using the new construction HVAC system map in Section 5.1.2.
- 2) **Envelope and Lighting Only:** The user specifies the building envelope, spaces, space types, thermal zones and all lighting, and any daylighting, where present. For all HVAC inputs, the proposed design values are prescribed and follow the rules for the standard design, including modeling the same HVAC systems determined using the new construction HVAC system map in Section 5.1.2.
- 3) **Envelope and Mechanical Only:** This compliance option assumes that the building will use separately permitted prescriptive lighting compliance. The user specifies the building envelope, spaces, space types, thermal zones, and mechanical systems in the building. For all lighting inputs, the proposed design values are prescribed and follow the rules for the standard design.
- 4) **Envelope and Partial Lighting Only:** This compliance option is used for projects where the building envelope is defined, and where the lighting in some of the spaces is defined. The user specifies the building envelope, all spaces, space types, thermal zones, and lighting for spaces with lighting systems defined, and any daylighting, where present. For all HVAC inputs, the proposed design values are prescribed and follow the rules for the standard design, including modeling the same HVAC systems determined using the new construction HVAC system map in Section 5.1.2.
- 5) **Mechanical Only:** This compliance option assumes that the building has already been permitted for envelope and lighting. The envelope and lighting systems for both the proposed design and the standard design are modeled as designed. (For example, if the building vertical fenestration area exceeds prescriptive WWR limits, the limits are NOT applied to the standard design. Instead, the actual vertical fenestration area is used.) The mechanical systems of the proposed model are described as-designed, and the new construction rules and system map are applied to the HVAC system of the standard design.
- 6) **Mechanical and Lighting Only:** This compliance option assumes that the building has already been permitted for envelope compliance. All spaces and space types must be defined by the user, and all envelope components for the proposed design are “as designed” (must be defined by the user). The standard design lighting and HVAC components are set to match the standard design.
- 7) **Mechanical and Partial Lighting Compliance:** This compliance option assumes that the building has already been permitted for Envelope and Partial Lighting compliance (option 3 above). The envelope components, spaces and space types, and permitted lighting spaces are entered as designed for the proposed design and, for these components, the standard design is set to be the same as the proposed. For the other components as part of the permit application, the mechanical systems and new lighting systems are entered by the user for the proposed design as designed, and the standard design

components for the mechanical (HVAC) system and new lighting systems are defined by the new construction standard design rules.

- 8) Envelope and Partial Mechanical: For projects where mechanical systems are not defined for all thermal zones. When the “HVAC is Unknown” checkbox is enabled at the thermal zone, the proposed mechanical system for that zone will be defaulted to match the standard design. Possible uses for this compliance option include “core and shell” projects and existing/addition/alteration projects with unknown existing mechanical equipment.

Building descriptors with inputs for both the proposed design and standard design that are restricted to prescribed values (for example, equipment performance curves) follow the same rules for prescribed values for any of the partial compliance projects listed above.

5.2.4 Building Model Classification

~~The structure of this manual supports multiple purposes and multiple baselines. For example, a baseline used for Title 24 compliance might be separate from a baseline used for green building ratings or beyond code programs. A building descriptor “purpose” could be used to identify different baselines but is not within the scope of this manual.~~

Space Classification Type	
<i>Applicability</i>	All projects
<i>Definition</i>	One of two available classification methods for identifying the function of the building or the functions of spaces within the building, which in turn determine energy-related requirements for the standard design. Appendix 5.4A lists the building classifications that are available under the area category method. The Area Category method uses a separate space classification for each space in the building according to its function.
<i>Units</i>	List (See Appendix 5.4)
<i>Input Restrictions</i>	For multiuse buildings, the building may be divided and a different building classification may be assigned to each part.
<i>Standard Design</i>	Same as proposed

5.2.5 Geographic and Climate Data

The following data needs to be specified or derived in some manner. Software developers may use any acceptable method to determine the data. For California, city, state, and county are required to determine climate data from the available data in Reference Appendix JA2.

Zip Code	
<i>Applicability</i>	All projects
<i>Definition</i>	California postal designation

<i>Units</i>	List (see Appendix 5.4)
<i>Input Restrictions</i>	None
<i>Standard Design</i>	Not applicable

Latitude	
<i>Applicability</i>	All projects
<i>Definition</i>	The latitude of the project site
<i>Units</i>	Degrees (°)
<i>Input Restrictions</i>	None Not a User Input
<i>Standard Design</i>	Latitude of representative city from Reference Appendix JA2

Longitude	
<i>Applicability</i>	All projects
<i>Definition</i>	The longitude of the project site
<i>Units</i>	Degrees (°)
<i>Input Restrictions</i>	None Not a User Input
<i>Standard Design</i>	Longitude of representative city from Reference Appendix JA2

Elevation	
<i>Applicability</i>	All projects
<i>Definition</i>	The height of the building site above sea level
<i>Units</i>	Feet (ft)
<i>Input Restrictions</i>	None
<i>Standard Design</i>	Elevation of representative city from Reference Appendix JA2

California Climate Zone	
<i>Applicability</i>	All projects
<i>Definition</i>	One of the 16 California climate zones
<i>Units</i>	List (see Reference Appendices JA2)
<i>Input Restrictions</i>	None
<i>Standard Design</i>	Same as proposed

City	
<i>Applicability</i>	All projects
<i>Definition</i>	The city where the project is located
<i>Units</i>	Alphanumeric string
<i>Input Restrictions</i>	None
<i>Standard Design</i>	Representative city from Reference Appendix JA2

Design Day Data	
<i>Applicability</i>	All projects
<i>Definition</i>	A data structure indicating design day information used for the sizing of the proposed system. Note: this information may not necessarily match the information used in the annual compliance simulation.
<i>Units</i>	Data structure: contains the following: Design DB (0.5%), mean coincident wet-bulb, early <u>daily</u> range, day of year
<i>Input Restrictions</i>	The design day information is taken from one of the 86 pre-defined California weather files, for the location within the same climate zone that is closest to the proposed building's location. (This is not input by the user.)
<i>Standard Design</i>	Not applicable

Weather File	
<i>Applicability</i>	All projects
<i>Definition</i>	The hourly (i.e., 8,760 hour per year) weather data to be used in performing the building energy simulations. Weather data must include outside dry-bulb temperature, outside wet-bulb temperature, atmospheric pressure, wind speed, wind direction, cloud amount, cloud type (or total horizontal solar and total direct normal solar), clearness number, ground temperature, humidity ratio, density of air, and specific enthalpy.
<i>Units</i>	Data file
<i>Input Restrictions</i>	The weather file selected shall be in the same climate zone as the proposed design. If multiple weather files exist for one climate zone then the weather file closest in distance to the proposed design and in the

	same climate zone shall be used.
<i>Standard Design</i>	Weather data shall be the same for both the proposed design and standard design.

Ground Reflectance	
<i>Applicability</i>	All Projects
<i>Definition</i>	Ground reflectance affects daylighting calculations and solar gain. The reflectance can be specified as a constant for the entire period of the energy simulation or it may be scheduled, which can account for snow cover in the winter.
<i>Units</i>	Data structure: schedule, fraction
<i>Input Restrictions</i>	Prescribed. The weather file determines the ground reflectance. The ground reflectance shall be set to 0.2 when the snow depth is 0 or undefined, and set to 0.6 when the snow depth is greater than 0.
<i>Standard Design</i>	Same as proposed

Local Terrain																			
<i>Applicability</i>	All projects																		
<i>Definition</i>	An indication of how the local terrain shields the building from the prevailing wind. Estimates of this effect are provided in the ASHRAE Handbook of Fundamentals.																		
<i>Units</i>	<p>List: the list shall contain only the following choices:</p> <table border="1"> <thead> <tr> <th>Description</th> <th>Exponent (α)</th> <th>Boundary layer thickness, δ (m)</th> </tr> </thead> <tbody> <tr> <td>Flat, open country</td> <td>0.14</td> <td>270</td> </tr> <tr> <td>Rough, wooded country, Suburbs</td> <td>0.22</td> <td>370</td> </tr> <tr> <td>Towns and cities</td> <td>0.33</td> <td>460</td> </tr> <tr> <td>Ocean</td> <td>0.10</td> <td>210</td> </tr> <tr> <td>Urban, industrial, forest</td> <td>0.22</td> <td>370</td> </tr> </tbody> </table>	Description	Exponent (α)	Boundary layer thickness, δ (m)	Flat, open country	0.14	270	Rough, wooded country, Suburbs	0.22	370	Towns and cities	0.33	460	Ocean	0.10	210	Urban, industrial, forest	0.22	370
Description	Exponent (α)	Boundary layer thickness, δ (m)																	
Flat, open country	0.14	270																	
Rough, wooded country, Suburbs	0.22	370																	
Towns and cities	0.33	460																	
Ocean	0.10	210																	
Urban, industrial, forest	0.22	370																	

	<p>The exponent and boundary layer are used in the following equation to adjust the local wind speed:</p> $V_z = V_{met} \left(\frac{\delta_{met}}{Z_{met}} \right)^{\alpha_{met}} \left(\frac{Z}{\delta} \right)^{\alpha}$ <p>Where:</p> <p>Z = altitude, height above ground (m)</p> <p>V_z = wind speed at altitude Z (m/s)</p> <p>α = wind speed profile exponent at the site</p> <p>δ = wind speed profile boundary layer thickness at the site (m)</p> <p>Z_{met} = height above ground of the wind speed sensor at the meteorological station (m)</p> <p>V_{met} = wind speed measured at the meteorological station (m/s)</p> <p>α_{met} = wind speed profile exponent at the meteorological station</p> <p>δ_{met} = wind speed profile boundary layer thickness at the meteorological station. (m)</p> <p>The wind speed profile coefficients α, δ, α_{met}, and δ_{met}, are variables that depend on the roughness characteristics of the surrounding terrain. Typical values for α and δ are shown in the table above.</p>
<i>Input Restrictions</i>	Weather data should be representative of the long term conditions at the site
<i>Standard Design</i>	The standard design terrain should be equal to the proposed design

5.2.6 Site Characteristics

Shading of Building Site	
<i>Applicability</i>	All projects
<i>Definition</i>	Shading of building fenestration, roofs, or walls by surrounding terrain, vegetation, and the building itself
<i>Units</i>	Data structure
<i>Input Restrictions</i>	The default and fixed value is for the site to be unshaded. External shading from other buildings or other objects is not modeled for Title 24 compliance in the ACM. Building self-shading is accounted for using the detailed geometry method.
<i>Standard Design</i>	The proposed design and standard design are modeled with identical assumptions regarding shading of the building site.

Site Fuel Source

<i>Applicability</i>	All projects
<i>Definition</i>	The fuel source that is available at the site for water heating, space heating or other fuel purposes. For most buildings connected to a utility service, this will be natural gas.
<i>Units</i>	List
<i>Input Restrictions</i>	The following choices are available: Natural Gas Propane
<i>Standard Design</i>	Natural gas

5.2.7 Calendar

Year for Analysis	
<i>Applicability</i>	All projects
<i>Definition</i>	The calendar year to be used for the annual energy simulations. This input determines the correspondence between days of the week, and the days on which weather events on the weather tape occur and has no other impact.
<i>Units</i>	List: choose a year (other than a leap year)
<i>Input Restrictions</i>	Use year 1991
<i>Standard Design</i>	Same calendar year as the proposed design

Schedule of Holidays									
<i>Applicability</i>	All projects								
<i>Definition</i>	A list of dates on which holidays are observed and on which holiday schedules are used in the simulations								
<i>Units</i>	Data structure								
<i>Input Restrictions</i>	<p>The following ten holidays represent the prescribed set. When a holiday falls on a Saturday, the holiday is observed on the Friday preceding the Saturday. If the holiday falls on a Sunday, the holiday is observed on the following Monday.</p> <table> <tr> <td>New Year's Day</td> <td>January 1</td> </tr> <tr> <td>Martin Luther King Day</td> <td>Third Monday in January</td> </tr> <tr> <td>Presidents Day</td> <td>Third Monday in February</td> </tr> <tr> <td>Memorial Day</td> <td>Last Monday in May</td> </tr> </table>	New Year's Day	January 1	Martin Luther King Day	Third Monday in January	Presidents Day	Third Monday in February	Memorial Day	Last Monday in May
New Year's Day	January 1								
Martin Luther King Day	Third Monday in January								
Presidents Day	Third Monday in February								
Memorial Day	Last Monday in May								

	Independence Day Labor Day Columbus Day Veterans Day Thanksgiving Day Christmas Day	July 4 First Monday in September Second Monday in October November 11 Fourth Thursday in November December 25
<i>Standard Design</i>	The standard design shall observe the same holidays specified for the proposed design.	

5.3 Thermal Zones

A thermal zone is a space or collection of spaces having similar space-conditioning requirements, the same heating and cooling setpoint, and is the basic thermal unit (or zone) used in modeling the building. A thermal zone will include one or more spaces. Thermal zones may be grouped together, but systems serving combined zones shall be subject to efficiency and control requirements of the combined zones. High-rise residential and nonresidential buildings with identical floors served by like systems may be modeled with floor multipliers.

5.3.1 General Information

Thermal Zone Name	
<i>Applicability</i>	All projects
<i>Definition</i>	A unique identifier for the thermal zone made up of 50 or fewer alphanumeric characters.
<i>Units</i>	Alphanumeric string
<i>Input Restrictions</i>	None
<i>Standard Design</i>	Not applicable

Thermal Zone Description	
<i>Applicability</i>	All projects
<i>Definition</i>	A brief description of the thermal zone that identifies the spaces which make up the thermal zone or other descriptive information. The description should tie the thermal zone to the building plans.
<i>Units</i>	Alphanumeric string
<i>Input Restrictions</i>	None

<i>Standard Design</i>	Not applicable
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Thermal Zone Type	
<i>Applicability</i>	All projects
<i>Definition</i>	Designation of the thermal zone as <u>a directly conditioned space</u> , unconditioned indirectly conditioned space (i.e., conditioned only by passive heating or cooling from an adjacent thermal zone) , or plenum (i.e., unoccupied but partially conditioned as a consequence of its role as a path for returning air) <u>space</u> .
<i>Units</i>	List: directly conditioned, unconditioned or plenum
<i>Input Restrictions</i>	The default thermal zone type is “directly conditioned.”
<i>Standard Design</i>	The descriptor is identical for the proposed design and standard design.

System Name	
<i>Applicability</i>	All projects
<i>Definition</i>	The name of the HVAC system that serves this thermal zone. The purpose of this building descriptor is to link the thermal zone to a system (child points to parent). Software can make this link in other ways.
<i>Units</i>	Text, unique
<i>Input Restrictions</i>	None
<i>Standard Design</i>	The standard design may have a different system mapping if the standard design has a different HVAC type than the proposed design.

Floor Area	
<i>Applicability</i>	All projects
<i>Definition</i>	The gross floor area of a thermal zone; including walls and minor spaces for mechanical or electrical services such as chases that are not assigned to other thermal zones.
<i>Units</i>	Square feet (ft ²)
<i>Input Restrictions</i>	The floor area of the thermal zone is derived from the floor area of the individual spaces that make up the thermal zone.
<i>Standard Design</i>	Same as proposed design

5.3.2 Interior Lighting

Inputs for interior lighting are specified at the space level (see specification below). In those instances, when thermal zones contain just one space, the inputs here will be identical to the inputs for the single space that is contained within the thermal zone.

For those instances when a thermal zone contains more than one space, the software shall either model the lighting separate for each space and sum energy consumption and heat gain for each time step of the analysis, or it must incorporate some procedure to sum inputs or calculate weighted averages such that the lighting power used at the thermal zone level is equal to the combination of lighting power for each of the spaces contained in the thermal zone.

In some cases, combining lighting power at the space level into lighting power for the thermal zone may be challenging and would have to be done at the level of each time step in the simulation. These cases include:

- A thermal zone that contains some spaces that have daylighting and others that do not.
- A thermal zone that contains spaces with different schedules of operation.
- A thermal zone that contains some spaces that have a schedule adjusted in some way for lighting controls and other spaces that do not.
- Combinations of the above.

5.3.3 Receptacle Loads

Inputs for receptacle and process loads are specified at the space level (see specification below). In those instances, when thermal zones contain just one space, the inputs here will be identical to the inputs for the single space that is contained within the thermal zone.

For those instances when a thermal zone contains more than one space, the software shall either model the receptacle and process loads separate for each space and sum energy consumption and heat gain for each time step of the analysis or it must incorporate some procedure to sum inputs or calculate weighted averages such that the receptacle and process loads used at the thermal zone level are equal to the combination of receptacle and process loads for each of the spaces contained in the thermal zone.

When the spaces contained in a thermal zone have different schedules, combining receptacle and process loads from the space level may be challenging and would have to be done at the level of each time step in the simulation. See discussion above on lighting.

5.3.4 Occupants

Inputs for occupant loads are specified at the space level (see specification below). In those instances, when thermal zones contain just one space, the inputs here will be identical to the inputs for the single space that is contained within the thermal zone.

For those instances when a thermal zone contains more than one space, the software shall either model the occupant loads separate for each space and the heat gain for each time step of the analysis or it must incorporate some procedure to sum inputs or calculate weighted averages such that the occupant loads used at the thermal zone level are equal to the combination of occupant loads for each of the spaces contained in the thermal zone.

When the spaces contained in a thermal zone have different occupant schedules, rolling up occupant loads from the space level may be challenging and would have to be done at the level

of each time step in the simulation. Spaces with differences in full-load equivalent operating hours of more than 40 hours per week shall not be combined in a single zone. See discussion above on lighting.

5.4 Space Uses

Each thermal zone discussed above may be subdivided into spaces. This section presents the building descriptors that relate to the space uses. Space uses and the defaults associated with them are listed in Appendix 5.4A. Every thermal zone shall have at least one space, as defined in this section. Daylit spaces should generally be separately defined by space type and/or orientation.

5.4.1 General Information

Space <u>Function</u> Type	
<i>Applicability</i>	All projects
<i>Definition</i>	<p>The space function type that defines occupancy, internal load, and other characteristics, as indicated in Appendix 5.4A.</p> <p>If lighting compliance is not performed, use either approach but actual LPDs cannot be entered for the spaces; the LPDs of the building match the standard design.</p> <p>The allowed space function types in area category are available from Appendix 5.4A. The building or space type determines the following baseline inputs:</p> <ul style="list-style-type: none"> Number of occupants (occupant density) Equipment power density Lighting power density Hot water load Ventilation rate Schedules (from Appendix 5.4B)
<i>Units</i>	List
<i>Input Restrictions</i>	<p>Only selections shown in Appendix 5.4A may be used.</p> <p>For unconditioned spaces, the user must enter “unconditioned” as the occupancy and ventilation; internal loads and uses are set to zero. Compliance software shall require the user to identify if lighting compliance is performed (lighting plans are included or have already been submitted).</p>
<i>Standard Design</i>	Same as proposed
<i>Existing Buildings</i>	Same as proposed

Floor Area	
<i>Applicability</i>	All projects
<i>Definition</i>	The floor area of the space The area of the spaces that make up a thermal zone shall sum to the floor area of the thermal zone.
<i>Units</i>	Square feet (ft ²)
<i>Input Restrictions</i>	Area shall be measured to the outside of exterior walls and to the center line of partitions
<i>Standard Design</i>	Area shall be identical to the proposed design
<i>Existing Buildings</i>	Same as proposed

5.4.2 Infiltration

Infiltration Method	
<i>Applicability</i>	All projects
<i>Definition</i>	Energy simulation programs have a variety of methods for modeling uncontrolled air leakage or infiltration. Some procedures use the effective leakage area which is generally applicable for small residential scale buildings. The component leakage method requires the user to specify the average leakage through the building envelope per unit area (ft ²). Other methods require the specification of a maximum rate, which is modified by a schedule.
<i>Units</i>	List effective leakage area, component leakage, or air changes per hour
<i>Input Restrictions</i>	For the purpose of California compliance and reach, the The component leakage area is prescribed; a fixed infiltration rate shall be specified and calculated as a leakage per area of exterior envelope, including the gross area of exterior walls and fenestration but excluding roofs and exposed floors.
<i>Standard Design</i>	The infiltration method used for the standard design shall be the same as the proposed design.

Infiltration Data	
<i>Applicability</i>	All projects
<i>Definition</i>	Information needed to characterize the infiltration rate in buildings. The required information will depend on the infiltration method selected above. For the effective leakage area method, typical inputs are leakage per exterior wall area in ft ² or other suitable units and information to indicate the height of the building and how shielded the site is from wind pressures. Only zones with exterior wall area are assumed to be subject

	to infiltration.
Units	<p>A data structure is required to define the effective leakage area model.</p> <p>For the purpose of California compliance and reach, infiltration <u>Infiltration</u> shall be calculated each hour using the following equation:</p> $Infiltration = I_{design} \cdot F_{schedule} \cdot (A + B \cdot t_{zone} - t_{odb} + C \cdot ws + D \cdot ws^2)$ <p>Where:</p> <ul style="list-style-type: none"> $Infiltration$ = zone infiltration airflow (m³/s-m²) I_{design} = design zone infiltration airflow (m³/s-m²) $F_{schedule}$ = fractional adjustment from a prescribed schedule, based on HVAC availability schedules in Appendix 5.4B(unitless) t_{zone} = zone air temperature (°C) t_{odb} = outdoor dry bulb temperature (°C) ws = the windspeed (m/s) A = overall coefficient (unitless) B = temperature coefficient (1/°C) C = windspeed coefficient (s/m) D = windspeed squared coefficient (s²/m²)
Input Restrictions	<p>For the proposed design, I_{design} shall have a fixed value of 0.0448 cfm/ft² (0.000228 m³/s-m²) times the gross wall area exposed to ambient outdoor air. A, B and D shall be fixed at zero. C shall be fixed at 0.10016 hr/mile (0.224 s/m).</p> <p>For nonresidential spaces with operable windows that do not have <u>mechanical system</u> interlocks, the CBECC software shall automatically increase infiltration to the space by 0.15 cfm/ft² whenever the outside air temperature is between 50°F and 90°F and when the HVAC system is operating. <u>High-rise dwelling units are exempt from mechanical system interlocks.</u></p> <p>For high-rise residential spaces with operable windows that do not have interlocks, the CBECC software shall automatically increase infiltration to the space by 0.02 cfm/ft² whenever the outside air temperature is between 50°F and 90°F and when the HVAC system is operating.</p>
Standard Design	<p>The standard design shall use the equation listed above, with coefficients A, B, and D set to 0. C shall be set to 0.10016 hr/mile (0.224 s/m). I_{design} shall be 0.0448 cfm/ft².</p>

Infiltration Schedule	
Applicability	When an infiltration method is used that requires the specification of a schedule
Definition	With the ACH method and other methods (see above), it may be necessary to specify a schedule that modifies the infiltration rate for each hour or time step of the simulation. Typically the schedule is either on or

	off but can also be fractional.
<i>Units</i>	Data structure: schedule, fractional
<i>Input Restrictions</i>	<p>The infiltration schedule shall be prescribed based on the HVAC system operating schedules from Appendix 5.4B. The infiltration schedule shall be set equal to 1 when the HVAC system is scheduled off and 0.25 when the HVAC system is scheduled on. This is based on the assumption that when the HVAC system is on it brings the pressure of the interior space above the pressure of the exterior, decreasing the infiltration of outside air. When the HVAC system is off, interior pressure drops below exterior pressure and infiltration increases.</p> <p><u>A possible incorporation</u>The implementation of the prescriptive requirement for interlocks for operable windows will model mixed mode ventilation as an increased infiltration rate when outside air conditions allow <u>for nonresidential buildings only, excluding healthcare and highrise residential buildings and spaces.</u></p>
<i>Standard Design</i>	The infiltration schedule for the standard design shall be set equal to 1 when the HVAC system is scheduled off and 0.25 when the HVAC system is scheduled on.

5.4.3 Occupants

For space level information on occupancy, lighting, and plug load schedules, as well as occupant density, allowed lighting power density. Appendix 5.4A provides a table of allowed space types.

Fixed Seating in Space	
<i>Applicability</i>	All projects that have a space with fixed seating (such as a theater or auditorium)
<i>Definition</i>	This is a flag that indicates that the space has fixed seating. If checked, this flag allows the user to override the default occupancy with values that comply with the California Building Code.
<i>Units</i>	Boolean
<i>Input Restrictions</i>	<p>As designed</p> <p>May not be used with high-rise residential, hotel/motel, unoccupied, and unleased tenant area spaces. The default is false.</p>
<i>Standard Design</i>	Same as proposed
<i>Existing Buildings</i>	The number of occupants must be identical for both the proposed and standard baseline design cases.

Dwelling Units per Space	
<i>Applicability</i>	High-rise residential projects

<i>Definition</i>	The number of residential living units within a single compliance model space
<i>Units</i>	Boolean
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	1
<i>Existing Buildings</i>	1

Number of Bedrooms	
<i>Applicability</i>	High-rise residential projects
<i>Definition</i>	The number of bedrooms per dwelling unit
<i>Units</i>	Integer
<i>Input Restrictions</i>	As designed but constrained to a minimum of 0 (studio) and a maximum of 5
<i>Standard Design</i>	Same as proposed
<i>Existing Buildings</i>	Same as proposed

Number of Occupants	
<i>Applicability</i>	High-rise residential projects
<i>Definition</i>	The number of people in a space. The number of people is modified by an hourly schedule (see below), which approaches but does not exceed 1.0. Therefore, the number of people specified by the building descriptor is similar to design conditions as opposed to average occupancy.
<i>Units</i>	The number of people may be specified in an absolute number, ft ² /person, or people/1000 ft ² .
<i>Input Restrictions</i>	The number of occupants is prescribed, and the values are given by Space Type in Appendix 5.4A, For high-rise residential spaces, the number of occupants is defined as: Max (number of bedrooms +1, 2).
<i>Standard Design</i>	The number of occupants must be identical for both the proposed and <u>standard baseline</u> design cases.
<i>Standard Design:</i> <i>Existing Buildings</i>	The number of occupants must be identical for both the proposed and <u>standard baseline</u> design cases.

Occupant Heat Rate	
<i>Applicability</i>	All projects

<i>Definition</i>	The sensible and latent heat produced by each occupant in an hour. This depends on the activity level of the occupants and other factors. Heat produced by occupants must be removed by the air conditioning system as well as the outside air ventilation rate and can have a significant impact on energy consumption.
<i>Units</i>	Btu/h specified separately for sensible and latent gains
<i>Input Restrictions</i>	The occupant heat rate is prescribed for California compliance
<i>Standard Design</i>	The occupant heat rate for the standard design <u>baseline building</u> shall be the same as the proposed design.
<i>Standard Design: Existing Buildings</i>	Same as proposed

Occupancy Schedule	
<i>Applicability</i>	All projects
<i>Definition</i>	The occupancy schedule modifies the number of occupants to account for expected operational patterns in the building. The schedule adjusts the heat contribution from occupants to the space on an hourly basis to reflect time-dependent usage patterns. The occupancy schedule can also affect other factors such as outside air ventilation, depending on the control mechanisms specified.
<i>Units</i>	Data structure: schedule, fractional
<i>Input Restrictions</i>	The occupant schedule is prescribed for California compliance. For California compliance, an appropriate schedule from Appendix 5.4B shall be used.
<i>Standard Design</i>	Occupancy schedules are identical for proposed and standard <u>baseline design</u> buildings designs .
<i>Standard Design: Existing Buildings</i>	Same as proposed

5.4.4 Interior Lighting

The building descriptors in this section are provided for each lighting system. Typically a space will have only one lighting system but, in some cases, it could have two or more. Examples include a general and task lighting system in offices, or hotel multi-purpose rooms that have lighting systems for different functions. It may also be desirable to define different lighting systems for areas that are daylit and those that are not.

Lighting Classification Method	
<i>Applicability</i>	Each space in the building
<i>Definition</i>	Indoor lighting power can be specified using the area category method or

	<p>the tailored method.</p> <p>Area category method can be used for all areas of the building with space types listed in Appendix 5.4A. This method can be used by itself or with the tailored lighting method.</p> <p>Tailored lighting method can be used for spaces with primary function listed in Table 140.6-D of the standards. The tailored lighting method is intended to accommodate special lighting applications. The tailored lighting method can be used by itself for all areas of the building or with the area category method. For a given area only one classification type can be used.</p>
<i>Units</i>	List
<i>Input Restrictions</i>	Only area category or tailored lighting are allowed
<i>Standard Design</i>	Same as proposed
<i>Standard Design: Existing Buildings</i>	Same as proposed

Table 6: Lighting Specification

Options: Lighting Classification Method	Area category method	Tailored lighting Method
Allowed combinations with other lighting classification methods	May be combined with tailored method <u>in same building, but not in same space.</u>	May be combined with area category method, <u>in same building, but not in same space.</u>
Allowed Regulated lighting power types	General lighting power <u>Additional Custom lighting power</u>	General lighting power <u>Wall display Custom lighting power</u> <u>Floor display and task lighting power</u> <u>Ornamental/special effect lighting power</u> <u>Very valuable display case lighting power</u>
Allowed Trade-offs	General lighting between conditioned spaces using area category method General lighting between conditioned spaces using area category and tailored method	General lighting between conditioned spaces using tailored method General lighting between conditioned spaces using tailored and area category method
Exception: With the area category method, custom lighting power can be used only if the tailored lighting method is not used in any area of the building.		

Regulated Interior Lighting Power Density	
<i>Applicability</i>	All projects when lighting compliance is performed
<i>Definition</i>	<p>Total connected lighting power density for all regulated interior lighting power</p> <p>This includes the loads for lamps and ballasts. The total regulated interior lighting power density is the sum of general lighting power and applicable custom lighting power per floor area in a space. Calculation of lighting power for conditioned spaces is done separately from unconditioned spaces.</p> <p>Lighting in unconditioned spaces can be modeled, but total lighting power in unconditioned spaces is not enforced in the compliance software. Lighting in unconditioned spaces must follow prescriptive compliance, and must be documented on appropriate compliance forms. No tradeoffs are allowed between lighting in conditioned spaces and lighting in unconditioned spaces.</p>
<i>Units</i>	W/ft ²
<i>Input Restrictions</i>	<p>Proposed value is:</p> <p>a) For the area category method: the sum of the proposed general lighting power and the proposed general lighting exceptional power within a conditioned space or a user input value if no interior lighting systems are modeled.</p> <p>b) For the tailored lighting method: the sum of the proposed general lighting power and the proposed custom lighting power within a conditioned space or a user input value if no interior lighting systems are modeled.</p> <p>When lighting compliance is not performed, the lighting power may not be entered and is set equal to the lighting level of the <u>standard-baseline design-building</u>, which is set to the levels for the selected occupancy from Appendix 5.4A.</p>
<i>Standard Design</i>	<p>For spaces without special task lighting, wall display lighting or similar requirements, this input will be the same as the general lighting power density. See the general lighting power building descriptor for details.</p> <p>With the area category and tailored method regulated interior lighting power for each space will be the sum of general lighting power and allowed custom lighting power.</p> <p>For alterations where less than 40 luminaires have been modified the standard design is the existing lighting condition before the alteration. If 40 or more luminaires have been modified, the prescriptive requirements for new construction apply.</p>

General Lighting Power

<i>Applicability</i>	All spaces or projects
<i>Definition</i>	General lighting power is the power used by installed electric lighting that provides a uniform level of illumination throughout an area, exclusive of any provision for special visual tasks or decorative effect, and also known as ambient lighting.
<i>Units</i>	Watts
<i>Input Restrictions</i>	<p>As designed</p> <p>For spaces without special task lighting, wall display lighting or similar requirements, this input will be the same as the regulated lighting power.</p> <p>Trade-offs in general lighting power are allowed between spaces all using the area category method, between spaces all using the tailored lighting method and between spaces that use area category and tailored methods. See <u>Table 6: Lighting Specification</u> for details.</p>
<i>Standard Design</i>	<p>With the area category method, general lighting power is the product of the lighting power densities for the space type from Appendix 5.4A and the floor areas for the corresponding conditioned spaces.</p> <p>With the tailored lighting method, general lighting power is the product of the lighting power density for the primary function type in Table 140.6-D of the standards and the floor area of the space. The lighting power density is given as a function of room cavity ratio (RCR) and interior illumination level in Table 140.6-G. No interpolation is allowed for this table.</p> <p>The general lighting power in the tailored method is calculated by the following steps:</p> <p>Step 1. Determine illumination level from Table 140.6-D by matching the primary function area in Table 140.6-D with the space type in Appendix 5.4A.</p> <p>Step 2. Calculate the room cavity ratio (RCR) by using the applicable equation in Table 140.6-F.</p> <p style="padding-left: 40px;">Rectangular Rooms: $RCR = 5 \times H \times (L+W) / (L \times W)$</p> <p style="padding-left: 40px;">Irregular Rooms: $RCR = 2.5 \times H \times P / A$</p> <p>Where: L = length of room; W = width of room; H = vertical distance from the work plane to the centerline of the lighting fixture; P = perimeter of room, and A = area of room</p> <p>Step 3. Determine the general lighting in the space(s) using the tailored method by a look-up in Table 140.6-G, where the general lighting LPD is a function of illuminance level and RCR. No interpolation is allowed for this table. A space between two illuminance levels (for example, 150 lux) uses the applicable LPD from the next lower illuminance level (100 lux).</p> <p>The standard design uses the irregular room RCR equation for both simplified and detailed geometry models.</p>

	The standard design lighting power is modified by a factor of 1/1.20 (0.833) if the simplified geometry approach is used and if the visible transmittance of any fenestration in the space does not meet the prescriptive requirements established in Section 140.3 of the standards.
<i>Standard Design: Existing Buildings</i>	<p>When the lighting status is “existing” (and unaltered) for the space, the standard design is the same as the existing, proposed design.</p> <p>When the lighting status is “altered” for the space, and at least 10 percent of existing luminaires have been altered:</p> <ol style="list-style-type: none"> If the lighting status is “existing”, then the standard design LPD is the same as the proposed design. If the lighting status is “new”, then the standard design LPD is same as new construction. If the lighting status is “altered”, then the standard design LPD is the same as new construction.

General Lighting Exceptional Power					
<i>Applicability</i>	Spaces that use the area category method; note that some exceptional allowances are only applicable to certain space types. See Table 140.6-C of the standards.				
<i>Definition</i>	The standards provide an additional lighting power allowance for special cases. Each of these lighting system cases is treated separately as “use-it-or-lose-it” lighting--the user receives no credit (standard design matches proposed) but there is a maximum power allowance for each item). There are eight lighting power allowances, as defined in the standards Table 140.6-C footnotes:				
<i>Units</i>	Data structure. This input has eight data elements: <ol style="list-style-type: none"> Specialized task work, laboratory (W/ft²) Specialized task work, other approved areas (W/ft²) Ornamental lighting (W/ft²) Precision commercial and industrial work (W/ft²) White board or chalk board lighting (W/linear foot) Accent, display and feature lighting (W/ft²) Decorative Lighting (W/ft²) Videoconferencing studio lighting (W/ft²) 				
<i>Input Restrictions</i>	As designed				
<i>Standard Design</i>	<p>The standard design general lighting exceptional power (GLEP) is given by the following equation:</p> $GLEP_{std} = \sum_{i=1}^8 \min(GLEP_{prop,i} \times GLETA_i, GLEA_i \times GLETA_i)$ <p>Where:</p> <table border="1"> <tr> <td>$GLEP_{std}$</td> <td>The GLEP of the standard design</td> </tr> <tr> <td>$GLEP_{prop,i}$</td> <td>The proposed GLEP of the footnote allowance</td> </tr> </table>	$GLEP_{std}$	The GLEP of the standard design	$GLEP_{prop,i}$	The proposed GLEP of the footnote allowance
$GLEP_{std}$	The GLEP of the standard design				
$GLEP_{prop,i}$	The proposed GLEP of the footnote allowance				

		i in the data structure above, or in the footnotes to Table 140.6-C of the standards
	$GLEA_i$	The general lighting exceptional allowance (GLEA) , which is the maximum allowed added lighting power in the rightmost column in Table 140.6-C of the standards; these allowances are, for GLEA1 through GLEA8, 0.2 W/ft ² , 0.5 W/ft ² , 0.5 W/ft ² , 1.0 W/ft ² , 5.5 W/linear foot, 0.3 W/ft ² , 0.2 W/ft ² and 1.5 W/ft ² , respectively
	$GLETA_i$	The general lighting exceptional task area (GLETA) for the i th exception, where the exception number corresponds to the area category exception number in the footnotes to Table 140.6-C of the standards

<i>Standard Design:</i> <i>Existing Buildings</i>	
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General Lighting Exceptional Task Area	
<i>Applicability</i>	Spaces that use area category method
<i>Definition</i>	The area associated with each of the exceptional lighting allowances in the GLEP building descriptor
<i>Units</i>	ft ²
<i>Input Restrictions</i>	As designed but cannot exceed the floor area of the space
<i>Standard Design</i>	Same as proposed
<i>Standard Design:</i> <i>Existing Buildings</i>	Same as proposed

White Board Length	
<i>Applicability</i>	Spaces that use area category method and take GLEP allowance #5
<i>Definition</i>	The linear length of the white board or chalk board in feet
<i>Units</i>	Ft
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Same as proposed
<i>Standard Design:</i> <i>Existing Buildings</i>	Same as proposed

Custom Lighting Power	
<i>Applicability</i>	All spaces or projects that use the tailored lighting method
<i>Definition</i>	<p>Custom lighting power covers lighting sources that are not included as general lighting, including task lighting, display lighting, and other specialized lighting designated in the footnotes to Table 140.6-C and lighting systems in Table 140.6-D of the standards. This lighting must be entered separately from the general lighting because it is not subject to tradeoffs.</p> <p>Software shall allow the user to input a custom lighting input for the allowed lighting system. If area category method is used, custom lighting power cannot be used if the tailored method is used for any area of the building. See <u>Table 6: Lighting Specification</u> for details.</p>
<i>Units</i>	Watts
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<p>Same as proposed but subject to the maximum limits specified in the footnotes to Table 140.6-C and Table 140.6-D of the standards. For spaces using the tailored method, the maximum allowed custom power is defined by the following procedure:</p> <p>The standard design custom lighting power is calculated by the sum of the following four terms:</p> <ol style="list-style-type: none"> 1) The product of the standard design wall display power and the standard design wall display length; 2) The product of the standard design floor and task lighting power and the standard design floor and task lighting area; 3) The product of the standard design ornamental and special effect lighting power, and the standard design ornamental and special effect lighting area; and 4) The product of the standard design very valuable display case power and the standard design very valuable display case area.
<i>Standard Design: Existing Buildings</i>	For alterations where less than 10 percent of existing luminaires have been modified, the <u>standard design baseline</u> is the existing lighting condition before the alteration. If 10 percent or more luminaires have been altered, the custom lighting power for the <u>standard design baseline</u> is the same as proposed, but subject to the limits specified in the footnotes to Table 140.6-C of the standards.

Wall Display Power	
<i>Applicability</i>	All spaces that use the tailored method
<i>Definition</i>	The lighting power allowed for wall display, as specified in standards

	Table 140.6-D, column 3
<i>Units</i>	W/ft
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	The standard design lighting power is the lesser of the proposed design wall display power or the limit specified in Table 140.6-D for the applicable space type.
<i>Standard Design: Existing Buildings</i>	Same as proposed

Wall Display Length	
<i>Applicability</i>	All spaces that use the tailored method
<i>Definition</i>	The horizontal length of the wall display lighting area using the tailored method for the space
<i>Units</i>	ft
<i>Input Restrictions</i>	As designed but this value cannot exceed the floor area of the space
<i>Standard Design</i>	Same as proposed
<i>Standard Design: Existing Buildings</i>	Same as proposed

Floor and Task Lighting Power	
<i>Applicability</i>	All spaces that use the tailored method
<i>Definition</i>	The lighting power allowed for floor display and task lighting, as specified in Table 140.6-D, column 4, of the standards
<i>Units</i>	W/ft ²
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	The standard design floor and task lighting power is the lesser of the proposed design floor and task lighting power or the limit specified in Table 140.6-D, column 4, for the applicable space type.
<i>Standard Design: Existing Buildings</i>	Same as proposed

Floor and Task Lighting Area	
<i>Applicability</i>	All spaces that use the tailored method
<i>Definition</i>	The lighting area that is served by the floor and task lighting defined

	using the tailored method for the space
<i>Units</i>	ft ²
<i>Input Restrictions</i>	As designed but this value cannot exceed the floor area of the space
<i>Standard Design</i>	Same as proposed
<i>Standard Design: Existing Buildings</i>	Same as proposed

Ornamental and Special Effect Lighting Power	
<i>Applicability</i>	All spaces that use the tailored method
<i>Definition</i>	The lighting power allowed for ornamental and special effect lighting, as specified in Table 140.6-D, column 5, of the standards
<i>Units</i>	W/ft ²
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	The standard design ornamental and special effect lighting power is the lesser of the proposed design ornamental and special effect lighting power or the limit specified in Table 140.6-D, column 5, for the applicable space type.
<i>Standard Design: Existing Buildings</i>	Same as proposed

Ornamental and Special Effect Lighting Area	
<i>Applicability</i>	All spaces that use the tailored method
<i>Definition</i>	The lighting area that is served by the ornamental and special effect lighting defined using the tailored method for the space
<i>Units</i>	ft ²
<i>Input Restrictions</i>	As designed but this value cannot exceed the floor area of the space
<i>Standard Design</i>	Same as proposed
<i>Standard Design: Existing Buildings</i>	Same as proposed

Very Valuable Display Case Lighting Power	
<i>Applicability</i>	All spaces that use the tailored method
<i>Definition</i>	The lighting power allowed for very valuable display case lighting, as

	specified in standards section 140.6(c)3L
<i>Units</i>	W/ft ²
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	The standard design very valuable display case lighting power is the lesser of: <ul style="list-style-type: none"> a) The product of the area of the primary function and 0.8 W/ft²; b) The product of the area of the display case and 12 W/ft²; or c) The proposed very valuable display lighting power.
<i>Standard Design: Existing Buildings</i>	Same as proposed

Very Valuable Display Case Lighting Area	
<i>Applicability</i>	All spaces that use the tailored method
<i>Definition</i>	The area of the very valuable display case(s) in plan view
<i>Units</i>	ft ²
<i>Input Restrictions</i>	As designed but this value cannot exceed the floor area of the space
<i>Standard Design</i>	Same as proposed
<i>Standard Design: Existing Buildings</i>	Same as proposed

Non-Regulated Interior Lighting Power	
<i>Applicability</i>	All projects
<i>Definition</i>	For California, §140.6(a)3 of the energy efficiency standards identifies non-regulated (exempted) lighting.
<i>Units</i>	W/ft ² or Watts
<i>Input Restrictions</i>	As designed The non-regulated lighting power should be cross-referenced to the type of exception and to the construction documents. The default for non-regulated lighting power is zero.
<i>Standard Design</i>	The non-regulated interior lighting in the <u>standard design</u> - baseline building shall be the same as the proposed design.
<i>Standard Design: Existing Buildings</i>	Same as proposed

Lighting Schedules	
<i>Applicability</i>	All projects
<i>Definition</i>	Schedule of operation for interior lighting power used to adjust the energy use of lighting systems on an hourly basis to reflect time-dependent patterns of lighting usage
<i>Units</i>	Data structure: schedule, fractional
<i>Input Restrictions</i>	The lighting schedule is prescribed for California compliance. An appropriate schedule from Appendix 5.4B shall be used.
<i>Standard Design</i>	The non-regulated interior lighting in the <u>standard design-baseline</u> building shall be the same as the proposed design.
<i>Standard Design: Existing Buildings</i>	Same as proposed

Tailored Lighting General Illumination Height	
<i>Applicability</i>	Spaces that have special tailored lighting power allowances
<i>Definition</i>	The illumination height is the vertical distance from the work plane to the centerline of the luminaire. This distance is used in the room cavity ratio (RCR) calculation which determines the allowed general lighting power density for a tailored lighting space.
<i>Units</i>	Ft
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Same as proposed The illumination height, H, is used to calculate the RCR and therefore the <u>standard design-baseline</u> general lighting power. See general lighting power for details.
<i>Standard Design: Existing Buildings</i>	Same as proposed

Floor/Wall Display Mounting Height Above Floor	
<i>Applicability</i>	Spaces that have wall display or floor display lighting and tailored lighting power allowances
<i>Definition</i>	The mounting height of wall display or floor display lighting above the floor
<i>Units</i>	List one of three <u>four</u> choices: 1) <u><10'-7"</u> 12 ft 2) <u>10'-7" – 14'-0"</u> 12-16 ft 3) <u>>14'-0" – 18'-0"</u>

	3)4) _____ > 168'-0"-ft
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	As designed The entered value maps to Table 140.6-E of the standards, that provides an adjustment multiplier for the tailored lighting wall power allowance in Table 140.6-D. The multiplier is 1.15 if the mounting height is 42'-12 ft to 46'-16 ft, and 1.30 if greater than 46'-16 ft. The compliance software must perform input processing to perform the necessary requirements.
<i>Standard Design: Existing Buildings</i>	Same as proposed

Fixture Type	
<i>Applicability</i>	All interior light fixtures
<i>Definition</i>	The type of lighting fixture, which is used to determine light heat gain distribution
<i>Units</i>	List: one of three choices: 1) Recessed with lens 2) Recessed/downlight 3) Not in ceiling
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Recessed/downlight
<i>Standard Design: Existing Buildings</i>	Recessed/downlight

Luminaire Type	
<i>Applicability</i>	All interior light fixtures
<i>Definition</i>	The type of lighting luminaire used to determine the light heat gain distribution The dominant luminaire type determines the daylight dimming characteristics, when there is more than one type of luminaire in the space.
<i>Units</i>	List one of three choices: a) Linear fluorescent b) Compact fluorescent lamp c) Incandescent d) Light emitting diode e) Metal halide f) Mercury vapor

	g) High pressure sodium
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Linear fluorescent
<i>Standard Design: Existing Buildings</i>	Linear fluorescent

Light Heat Gain Distribution	
<i>Applicability</i>	All projects
<i>Definition</i>	<p>The distribution of the heat generated by the lighting system that is directed to the space, the plenum, the HVAC return air, or to other locations</p> <p>This input is a function of the luminaire type and location. Luminaires recessed into a return air plenum contribute more of their heat to the plenum or the return air stream if the plenum is used for return air; while pendant mounted fixtures hanging in the space contribute more of their heat to the space. Common luminaire type/space configurations are listed in Table 3, Chapter 18, 2009 ASHRAE Handbook of Fundamentals, summarized in Table 7Table 7Table 7. Typically the data will be linked to list of common luminaire configurations similar to Table 7Table 7Table 7 so that the user chooses a luminaire type category and heat gain is automatically distributed to the appropriate locations.</p>
<i>Units</i>	List (of luminaire types) or data structure consisting of a series of decimal fractions that assign heat gain to various locations
<i>Input Restrictions</i>	<p>Heat gain distribution is fixed to Table 7Table 7Table 7 values based on the luminaire, fixture, and distribution type.</p> <p>Where lighting fixtures having different heat venting characteristics are used within a single space, the wattage weighted average heat-to-return-air fraction shall be used.</p>
<i>Standard Design</i>	The standard design baseline building shall use the values in Table 7 Table 7 Table 7 for recessed fluorescent luminaires without lens.
<i>Standard Design: Existing Buildings</i>	Same as new construction

Table 7: Light Heat Gain Parameters for Typical Operating Conditions

Based on Table 3, Chapter 18, 2009 ASHRAE Handbook – Fundamentals

Fixture Type	Luminaire Type	Return Type	Space Fraction	Radiative Fraction
Recessed with Lens	Linear Fluorescent	Ducted/Direct	1.00	0.67
		Plenum	0.45	0.67

Recessed/Downlight	Linear Fluorescent	Ducted/Direct	1.00	0.58	
		Plenum	0.69	0.58	
	CFL	Ducted/Direct	1.00	0.97	
		Plenum	0.20	0.97	
	Incandescent	Ducted/Direct	1.00	0.97	
		Plenum	0.75	0.97	
	LED	Ducted/Direct	1.00	0.97	
		Plenum	0.20	0.97	
	Metal Halide	Ducted/Direct	1.00	0.97	
		Plenum	0.75	0.97	
	Non In Ceiling	Linear Fluorescent	Ducted/Direct	1.00	0.54
			Plenum	1.00	0.54
CFL		Ducted/Direct	1.00	0.54	
		Plenum	1.00	0.54	
Incandescent		Ducted/Direct	1.00	0.54	
		Plenum	1.00	0.54	
LED		Ducted/Direct	1.00	0.54	
		Plenum	1.00	0.54	
Metal Halide		Ducted/Direct	1.00	0.54	
		Plenum	1.00	0.54	
Mercury Vapor		Ducted/Direct	1.00	0.54	
		Plenum	1.00	0.54	
High Pressure Sodium		Ducted/Direct	1.00	0.54	
		Plenum	1.00	0.54	

In this table, the Space Fraction is the fraction of the lighting heat gain that goes to the space; the radiative fraction is the fraction of the heat gain to the space that is due to radiation, with the remaining heat gain to the space due to convection.

Lighting Power Adjustment Factors (PAF)	
<i>Applicability</i>	All projects
<i>Definition</i>	Automatic controls that are not already required by the baseline standard <u>Title 24 Energy Standards</u> and which reduce lighting power more or less uniformly over the day can be modeled as power adjustment factors. Power adjustment factors represent the percent reduction in lighting power that will approximate the effect of the control. Models

	<p>account for such controls by multiplying the controlled watts_by (1-PAF). Eligible California power adjustment factors are defined in Table 140.6-A. Reduction in lighting power using the PAF method can be used only for nonresidential controlled general lights. Only one PAF can be used for a qualifying lighting system unless multiple adjustment factors are allowed in Table 140.6.A of the standards. Controls for which PAFs are eligible are listed in Table 140.6-A of the standards and include:</p> <ul style="list-style-type: none"> a) Occupancy Sensing Controls for qualifying enclosed spaces and open offices. b) Demand Response Controls – Demand responsive lighting control that reduces lighting power consumption in response to a demand response signal for qualifying building types. c) Institutional tuning – lighting tuned to not use more than 85 percent of rated power, per Section 140.6 of the standards. d) <u>Daylight dimming plus off controls – daylight dimming controls that automatically shut off luminaires when natural lighting provides an illuminance level of at least 150 percent of the space requirement, as specified by the standards.</u> e) <u>Horizontal slats – interior or exterior horizontal slats on fenestration adjacent to daylit areas</u> f) <u>-Light shelves – interior or exterior light shelves adjacent to daylit areas</u> e) <u>Clerestories are modeled as Power Adjustment Factors, and are not modeled directly by compliance software. Compliance software shall have a means of disregarding daylight through clerestory windows when using the PAF.</u>
<i>Units</i>	List: eligible control types (see above) linked to PAFs
<i>Input Restrictions</i>	PAF shall be fixed for a given control and area type
<i>Standard Design</i>	PAF is zero
<i>Standard Design: Existing Buildings</i>	PAF is zero

5.4.5 Daylighting Control

This group of building descriptors is applicable for spaces that have daylighting controls or daylighting control requirements.

California prescribes a modified version of the split flux daylighting methods to be used for compliance. This is an *internal daylighting method* because the calculations are automatically performed by the simulation engine. For top-lighted or sidelit daylighted areas, California compliance prescribes an internal daylighting model consistent with the split flux algorithms used in many simulation programs. With this method the simulation model has the capability to model the daylighting contribution for each hour of the simulation and make an adjustment to the lighting power for each hour, taking into account factors such as daylighting availability, geometry of the space, daylighting aperture, control type, and the lighting system. The assumption is that the geometry of the space, the reflectance of surfaces, the size and

configuration of the daylight apertures, and the light transmission of the glazing are taken from other building descriptors.

For daylight control using a simplified geometry approach, daylight control for both the primary daylight zone (mandatory) and secondary daylight zone (prescriptive) must be indicated on the compliance forms. If the simplified geometry approach is used and the visible transmittance of fenestration does not meet prescriptive requirements, the standard design lighting power is reduced by 20 percent as a penalty. See Interior Lighting.

Daylight Control Requirements	
<i>Applicability</i>	All spaces with exterior fenestration
<i>Definition</i>	The extent of daylighting controls in skylit and sidelit areas of the space
<i>Units</i>	List
<i>Input Restrictions</i>	When the installed general lighting power in the primary daylight zone exceeds 120W, daylighting controls are required, per the Title 24 mandatory requirements.
<i>Standard Design</i>	<p>For nonresidential spaces, when the installed general lighting power in the skylit or primary sidelit daylight zone exceeds 120W, daylighting controls are required in the primary daylight zone, per the Title 24 mandatory requirements.</p> <p>For parking garages, when the installed general lighting power in the primary sidelit or secondary sidelit daylight zone exceeds 120W, daylighting controls are required, per the Title 24 mandatory requirements. Luminaires located in daylight transition zones or dedicated ramps are exempt from this requirement.</p> <p>For nonresidential spaces, daylighting controls are specified when the installed general lighting power in the skylit, primary sidelit, or secondary sidelit daylight zone(s) exceeds 120W.</p> <p>For parking garages, when the installed general lighting power in the primary sidelit or secondary sidelit daylight zone exceeds 120W, daylighting controls are required. Luminaires located in daylight transition zones or dedicated ramps are exempt from this requirement.</p>
<i>Standard Design: Existing Buildings</i>	<p>When lighting systems in an existing altered building are not modified as part of the alteration, daylighting controls are the same as the proposed design.</p> <p>When an alteration increases the area of a lighted space, increases lighting power in a space, or when luminaires are modified in a space where proposed design lighting power density is greater than 85 percent of the standard design LPD, daylighting control requirements are the same as for new construction.</p>

Skylit, Primary, and Secondary Daylighted Area

<i>Applicability</i>	All daylighted spaces
<i>Definition</i>	<p>The floor area that is daylighted.</p> <p>The skylit area is the portion of the floor area that gets daylighting from a skylight. Two types of sidelit daylighted areas are recognized. The primary daylighted area is the portion that is closest to the daylighting source and receives the most illumination. The secondary daylighted area is an area farther from the daylighting source, which still receives useful daylight.</p> <p>The primary daylight area for sidelighting is a band near the window with a depth equal to the distance from the floor to the top of the window and width equal to window width plus 0.5 times window head height wide on each side of the window opening. The secondary daylight area for sidelighting is a band beyond the primary daylighted area that extends a distance double the distance from the floor to the top of the window and width equal to window width plus 0.5 times window head height wide on each side of the window opening. Area beyond a permanent obstruction taller than 6 feet should not be included in the primary and secondary daylight area calculation.</p> <p>The skylit area is a band around the skylight well that has a depth equal to the 70 percent of the ceiling height from the edge of the skylight well. The geometry of the skylit daylit area will be the same as the geometry of the skylight. Area beyond a permanent obstruction taller than 50 percent of the height of the skylight from the floor should not be included in the skylit area calculation.</p> <p>Double counting due to overlaps is not permitted. If there is an overlap between secondary and primary or skylit areas, the effective daylit area used for determining reference position shall be the area minus the overlap.</p>
<i>Units</i>	ft ²
<i>Input Restrictions</i>	The daylit areas in a space are derived using other modeling inputs like dimensions of the fenestration and ceiling height of the space.
<i>Standard Design</i>	The daylit areas in the <u>standard design baseline building</u> are derived from other modeling inputs, including the dimensions of the fenestration and ceiling height of the space. Daylit area calculation in the standard design is done after window to wall ratio and skylight to roof ratio rules in Section 5.5.7 of this manual are applied.
<i>Standard Design: Existing Buildings</i>	Same as new construction when skylights are added/replaced and general lighting altered

Installed General Lighting Power in the Primary and Skylit Daylit Zone	
<i>Applicability</i>	All spaces
<i>Definition</i>	The installed lighting power of general lighting in the primary and skylit

	<p>daylit zone.</p> <p>The primary and skylit daylit zone shall be defined on the plans, and be consistent with the definition of the primary and skylit daylit zone in the standards. Note that a separate building descriptor, fraction of controlled lighting, defines the fraction of the lighting power in the space that is controlled by daylighting.</p>
<i>Units</i>	Watts
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	The installed lighting power for the standard design is the product of the primary daylit area and the LPD for general lighting in the space.
<i>Standard Design: Existing Buildings</i>	Same as new construction when skylights are added/replaced and general lights are altered

Installed General Lighting Power in the Secondary Daylit Zone	
<i>Applicability</i>	All spaces
<i>Definition</i>	<p>The installed lighting power of general lighting in the secondary daylit zone.</p> <p>The secondary daylit zone shall be defined on the plans and be consistent with the definition of the secondary daylit zone in the standards. Note that a separate building descriptor, fraction of controlled lighting, defines the fraction of the lighting power in the space that is controlled by daylighting.</p>
<i>Units</i>	W
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	The installed lighting power for the standard design is the product of the secondary daylit area and the LPD for general lighting in the space.
<i>Standard Design: Existing Buildings</i>	Same as new construction when skylights are added/replaced and general lights are altered

Reference Position for Illuminance Calculations	
<i>Applicability</i>	All spaces or thermal zones, depending on which object is the primary container for daylighting controls
<i>Definition</i>	<p>The position of the two daylight reference points within the daylit space.</p> <p>Lighting controls are simulated so that the illuminance at the reference position is always maintained at or above the illuminance setpoint. For step switching controls, the combined daylight illuminance plus uncontrolled electric light illuminance at the reference position must be greater than the setpoint illuminance before the controlled lighting can be</p>

dimmed or tuned off for stepped controls. Similarly, dimming controls will be dimmed so that the combination of the daylight illuminance plus the controlled lighting illuminance is equal to the setpoint illuminance.

Preliminary reference points for primary and secondary daylit areas are located at the farthest end of the daylit area aligned with the center of the each window. For skylit area, the preliminary reference point is located at the center of the edge of the skylit area closest to the centroid of the space. In each case, the Z – coordinate of the reference position (elevation) shall be located 2.5 feet above the floor.

Up to two final reference positions can be selected from among the preliminary reference positions identified in for each space.

Units Data structure

Input Restrictions The user does not specify the reference position locations; reference positions are automatically calculated by the compliance software based on the procedure outlined below. Preliminary reference positions are each assigned a relative daylight potential (RDP) which estimates the available illuminance at each position, and the final reference position selection is made based on the RDP.

RDP: An estimate of daylight potential at a specific reference position. This is NOT used directly in the energy simulation, but it used to determine precedence for selecting the final reference points. The relative daylight potential is calculated as a function of effective aperture, azimuth, illuminance setpoint and the type (skylit, primary sidelit, or secondary sidelit) of the associated daylit zone. RDP is defined as:

$$RDP = C_1 \times EA_{dz} + C_2 \times SO + C_3$$

Where: $C_1, C_2,$ and C_3 are selected from the following table.

Illuminance Setpoint	Skylit Daylit Zones			Primary Sidelit Daylit Zones			Secondary Sidelit Daylit Zones		
	C_1	C_2	C_3	C_1	C_2	C_3	C_1	C_2	C_3
≤ 200 lux	3927	0	3051	1805	-0.40	3506	7044	-3.32	1167
≤ 1000 lux	12046	0	-421	6897	-7.22	475	1512	-2.88	-22
> 1000 lux	5900	0	-516	884	-5.85	823	212	-0.93	57

Illuminance Setpoint: This is defined by the user, and is entered by the user, subject to the limits specified in Appendix 5.4A, determined from the space type.

Source Orientation (SO): The angle of the outward facing normal of the daylight source’s parent surface projected onto a horizontal plane, expressed as degrees from south. This is not a user input but is calculated from the geometry of the parent surface. For skylights, the source orientation is not applicable. For vertical fenestration, it is defined:

$$SO = |(180 - Azimuth)|$$

Where: Azimuth is defined as the azimuth of the parent object containing the fenestration associated with the preliminary reference point.

	<p>Effective Aperture (EA): For this calculation, effective aperture represents the effectiveness of all sources which illuminate a specific reference position in contributing to the daylight available to the associated daylight zone. In cases where daylight zones from multiple fenestration objects intersect, the effective aperture of an individual daylight zone is adjusted to account for those intersections according to the following rules:</p> <ul style="list-style-type: none"> • For skylit and primary sidelit daylight zones, intersections with other skylit or primary sidelit daylight zones are considered. • For secondary sidelit daylight zones, intersections with any toplit or sidelit (primary or secondary) daylight zones are considered. <p>Effective aperture is defined as follows:</p> $EA_{dz} = (VT_{fdz} \times A_{fdz} + \sum F_i \times VT_i \times A_i) / A_{dz}$ <p>Where:</p> <p>EA_{dz} is the combined effective aperture of all daylight sources illuminating a specific daylight zone</p> <p>VT_{fdz} is the user specified visible transmittance of the fenestration object directly associated with the daylight zone</p> <p>A_{fdz} is the area of the fenestration object directly associated with the daylight zone</p> <p>VT_i is the user specified visible transmittance of the fenestration object associated with each intersecting daylight zone</p> <p>A_i is the area of the fenestration object directly associated with each intersecting daylight zone</p> <p>F_i is the fraction of intersecting area between the daylight zone in question and each intersecting daylight zone:</p> $F_i = A_{intersection} / A_{dzi}$ <p>A_{dzi} is the area of each intersecting daylight zone (including area that might fall outside a space or exterior boundary)</p> <p>A_{dz} is the area of the daylight zone (including area that might fall outside a space or exterior boundary).</p> <p>First Reference Position: Select the preliminary reference point with the highest relative daylight potential (RDP) from among all preliminary reference points located within either top or primary sidelit daylight zones. If multiple reference points have identical RDPs, select the reference point geometrically closest to the centroid of the space.</p> <p>Second Reference Position: Select the preliminary reference point with the highest RDP from amongst all remaining preliminary reference points located within either top or primary sidelit daylight zones. If multiple reference points have identical RDPs, select the reference point geometrically closest to the centroid of the space.</p>
<p><i>Standard Design</i></p>	<p>Reference positions for the standard design shall be selected using the</p>

	same procedure as those selected for the proposed design.
<i>Standard Design: Existing Buildings</i>	Additions or alternations of lighting in spaces trigger the daylighting control requirements whenever the total installed lighting in the daylit zone is 120 W or greater, and the reference positions shall be determined in the same manner as with new construction. This only applies when alterations or additions to the lighting in an existing building trigger daylighting control requirements.

Illumination Adjustment Factor	
<i>Applicability</i>	All Daylighted Spaces
<i>Definition</i>	Recent studies have shown that the split flux interreflection component model used in many simulation programs overestimates the energy savings due to daylighting, particularly deep in the space. A set of two adjustment factors is provided, one for the primary daylit zone and one for the secondary daylit zone. For simulation purposes, the input daylight illuminance setpoint will be modified by the illuminance adjustment factor as follows: $LightSetpoint_{adj} = LightSetpoint \times Adjustment\ Factor$
<i>Units</i>	Unitless
<i>Input Restrictions</i>	Prescribed values for space type in Appendix 5.4A
<i>Standard Design</i>	The <u>standard design</u> baseline building illumination adjustment factors shall match the proposed
<i>Standard Design: Existing Buildings</i>	Same as new construction when skylights are added/replaced and general light is altered.

Fraction of Controlled Lighting	
<i>Applicability</i>	Daylighted Spaces
<i>Definition</i>	The fraction of the general lighting power in the (daylighted) primary and skylit daylit zone, or secondary sidelit daylit zone that is controlled by daylighting controls.
<i>Units</i>	Numeric: fraction for primary and skylit daylit zone, and fraction for secondary zone
<i>Input Restrictions</i>	As designed for secondary daylit areas. If the proposed design has no daylight controls in the secondary daylit area the value is set to 0 for the general lights in the secondary daylit area. Primary and skylit daylit area fraction of controlled general lighting shall be as designed when the daylight control requirements building descriptor indicates that they are not required, and shall be 1 when controls are required.
<i>Standard Design</i>	When daylight controls are required according to the daylight control

	requirements building descriptor in either the primary daylit and skylit zone, or the secondary daylit zone, or both, the fraction of controlled lighting shall be 1.
<i>Standard Design: Existing Buildings</i>	Same as for new construction when skylights are added/replaced, and general light is altered.

Daylighting Control Type

<i>Applicability</i>	Daylighted Spaces
<i>Definition</i>	<p>The type of control that is used to control the electric lighting in response to daylight available at the reference point.</p> <p>Options:</p> <ul style="list-style-type: none"> Stepped switching controls vary the electric input power and lighting output power in discrete equally spaced steps. At each step, the fraction of light output is equal to the fraction of rated power. Continuous dimming controls have a fraction to rated power to fraction of rated output that is a linear interpolation of the minimum power fraction at the minimum dimming light fraction to rated power (power fraction = 1.0) at full light output. See Figure 8: Example Continuous Dimming Control <p>Continuous dimming + off controls are the same as continuous dimming controls except that these controls can turn all the way off when none of the controlled light output is needed. See the example control chart below.</p>
	Figure 88: Example Continuous Dimming Control
<i>Units</i>	List (see above)

<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Standard design Baseline does not have <u>uses continuous</u> daylighting control (continuous) .
<i>Standard Design: Existing Buildings</i>	Same as for new construction when skylights are added/replaced, and general light is altered.

Minimum Dimming Power Fraction	
<i>Applicability</i>	Daylit spaces
<i>Definition</i>	The minimum power fraction when controlled lighting is fully dimmed. Minimum power fraction = minimum power / full rated power.
<i>Units</i>	Numeric: fraction
<i>Input Restrictions</i>	As designed, specified from luminaire type (not a user input)
<i>Standard Design</i>	Standard design Baseline building uses continuous dimming control with a minimum dimming power fraction from <u>Table 8: Standard Design Power/Light Output Fraction</u> Table 8: Baseline Power/Light Output Fraction . Where the controlled luminaire type, input by the user, determines the minimum dimming power fraction.
<i>Standard Design: Existing Buildings</i>	Same as for new construction when skylights are added/replaced, and general light is altered.

Minimum Dimming Light Fraction	
<i>Applicability</i>	Daylighting and dimming controls
<i>Definition</i>	The minimum light output when controlled lighting is fully dimmed. Minimum light fraction = minimum light output / rated light output.
<i>Units</i>	Numeric: fraction
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Standard design Baseline building uses continuous dimming control with a minimum dimming light fraction from <u>Table 8: Standard Design Power/Light Output Fraction</u> Table 8: Baseline Power/Light Output Fraction . Where the controlled luminaire type, input by the user, determines the minimum dimming power fraction.
<i>Standard Design: Existing Buildings</i>	Same as for new construction when skylights are added/replaced, and general light is altered.

Table 8: Standard Design~~Baseline~~ Power/Light Output Fraction

Light Source	Power Fraction	Light Output Fraction
--------------	----------------	-----------------------

LED	0.1	0.1
Linear Fluorescent	0.2	0.2
Mercury Vapor	0.3	0.2
Metal Halide	0.45	0.2
High Pressure Sodium	0.4	0.2
CFL	0.4	0.2
Incandescent	0.5	0.2

5.4.6 Receptacle Loads

Receptacle loads contribute to heat gains in spaces and directly use energy.

Receptacle Power													
<i>Applicability</i>	All building projects												
<i>Definition</i>	Receptacle power is power for typical general service loads in the building. Receptacle power includes equipment loads normally served through electrical receptacles, such as office equipment and printers, but does not include either task lighting or equipment used for HVAC purposes. Receptacle power values are slightly higher than the largest hourly receptacle load that is actually modeled because the receptacle power values are modified by the receptacle schedule, which approaches but does not exceed 1.0.												
<i>Units</i>	<p>Total power (W) or the space power density (W/ft²)</p> <p>Compliance software shall also use the following prescribed values to specify the latent heat gain fraction and the radiative/convective heat gain split.</p> <p>For software that specifies the fraction of the heat gain that is lost from the space, this fraction shall be prescribed at 0.</p> <p>Heat Gain Fractions:</p> <table border="1"> <thead> <tr> <th></th> <th>Radiative</th> <th>Latent</th> <th>Convective</th> </tr> </thead> <tbody> <tr> <td>Receptacle Power</td> <td>0.20</td> <td>0.00</td> <td>0.80</td> </tr> <tr> <td>Gas Equipment Power</td> <td>0.15</td> <td>0.00</td> <td>0.00</td> </tr> </tbody> </table>		Radiative	Latent	Convective	Receptacle Power	0.20	0.00	0.80	Gas Equipment Power	0.15	0.00	0.00
	Radiative	Latent	Convective										
Receptacle Power	0.20	0.00	0.80										
Gas Equipment Power	0.15	0.00	0.00										
<i>Input Restrictions</i>	Prescribed to values from Appendix 5.4A												
<i>Standard Design</i>	Same as proposed												
<i>Standard Design: Existing Buildings</i>	Same as for new construction												

Receptacle Schedule	
<i>Applicability</i>	All projects
<i>Definition</i>	Schedule for receptacle power loads used to adjust the intensity on an hourly basis to reflect time-dependent patterns of usage.
<i>Units</i>	Data structure: schedule, fraction
<i>Input Restrictions</i>	Prescribed to schedule in Appendix 5.4A
<i>Standard Design</i>	Same as proposed
<i>Standard Design: Existing Buildings</i>	Same as for new construction

5.4.7 Commercial Refrigeration Equipment

Commercial refrigeration equipment includes the following:

- Walk-in refrigerators
- Walk-in freezers
- Refrigerated casework

~~Walk-in refrigerators and freezers typically have remote condensers. Some refrigerated casework has remote condensers, while some have a self-contained condenser built into the unit. Refrigerated casework with built-in condensers rejects heat directly to the space while remote condensers reject heat in the remote location, typically on the roof or behind the building. Refrigeration equipment are modeled as neutral plug loads, with standard design power matching the proposed design.~~

~~Refrigerated casework can be further classified by the purpose, the type of doors and, when there are no doors, the configuration: horizontal, vertical or semi-vertical. DOE has developed standards for refrigerated casework.~~

~~Table 9 shows these classifications along with the standard level of performance, expressed in kWh/d, which depends on the class of equipment, the total display area, and the volume of the casework.~~

Table 9: DOE Requirements for Refrigerated Casework (kWh/d)

TABLE I-1—STANDARD LEVELS FOR COMMERCIAL REFRIGERATION EQUIPMENT

Equipment class ²	Standard level ^{***} (kWh/day)	Equipment class	Standard level ^{***} (kWh/day)
VOP.RC.M	0.82 × TDA + 4.07	VCT.RC.I	0.66 × TDA + 3.05
SVO.RC.M	0.83 × TDA + 3.18	HCT.RC.M	0.16 × TDA + 0.13
HZO.RC.M	0.35 × TDA + 2.88	HCT.RC.L	0.34 × TDA + 0.26
VOP.RC.L	2.27 × TDA + 6.85	HCT.RC.I	0.4 × TDA + 0.31
HZO.RC.L	0.57 × TDA + 6.88	VCS.RC.M	0.11 × V + 0.26
VCT.RC.M	0.22 × TDA + 1.95	VCS.RC.L	0.23 × V + 0.54
VCT.RC.L	0.56 × TDA + 2.61	VCS.RC.I	0.27 × V + 0.63
SOC.RC.M	0.51 × TDA + 0.11	HCS.RC.M	0.11 × V + 0.26
VOP.SC.M	1.74 × TDA + 4.71	HCS.RC.L	0.23 × V + 0.54
SVO.SC.M	1.73 × TDA + 4.59	HCS.RC.I	0.27 × V + 0.63
HZO.SC.M	0.77 × TDA + 5.55	SOC.RC.L	1.08 × TDA + 0.22
HZO.SC.L	1.92 × TDA + 7.08	SOC.RC.I	1.26 × TDA + 0.26
VCT.SC.I	0.67 × TDA + 3.29	VOP.SC.L	4.37 × TDA + 11.82
VCS.SC.I	0.38 × V + 0.88	VOP.SC.I	5.55 × TDA + 15.02
HCT.SC.I	0.56 × TDA + 0.43	SVO.SC.L	4.34 × TDA + 11.51
SVO.RC.L	2.27 × TDA + 6.85	SVO.SC.I	5.52 × TDA + 14.63
VOP.RC.I	2.89 × TDA + 8.7	HZO.SC.I	2.44 × TDA + 9.
SVO.RC.I	2.89 × TDA + 8.7	SOC.SC.I	1.76 × TDA + 0.36
HZO.RC.I	0.72 × TDA + 8.74	HCS.SC.I	0.38 × V + 0.88

^{*} TDA is the total display area of the case, as measured in the Air-Conditioning and Refrigeration Institute (ARI) Standard 1200–2006, Appendix D.

^{**} V is the volume of the case, as measured in ARI Standard 1200–2006, Appendix C.

^{***} Kilowatt hours per day.

² For this rulemaking, equipment class designations consist of a combination (in sequential order separated by periods) of: (1) An equipment family code (VOP=vertical open, SVO=semivertical open, HZO=horizontal open, VCT=vertical transparent doors, VCS=vertical solid doors, HCT=horizontal transparent doors, HCS=horizontal solid doors, or SOC=service over counter); (2) an operating mode code (RC=remote condensing or SC=self contained); and (3) a rating temperature code (M=medium temperature (38 °F), L=low temperature (0 °F), or I=ice-cream temperature (-15 °F)). For example, "VOP.RC.M" refers to the "vertical open, remote condensing, medium temperature" equipment class. See discussion in section V.A.2 and chapter 3 of the TSD, market and technology assessment, for a more detailed explanation of the equipment class terminology. See Table IV-2 for a list of the equipment classes by category.

~~Walk in refrigerators and freezers are not covered by the DOE standards and test procedures. Title24 default values for these are given in Table 10: Default Power for Walk-In Refrigerators and Freezers (W/ft²). These values are expressed in W/ft² of refrigerator or freezer area. This power is assumed to occur continuously. Some walk-ins have glass display doors on one side so that products can be loaded from the back. Glass display doors increase the power requirements of walk-ins. Additional power is added when glass display doors are present. The total power for walk-in refrigerators and freezers is given in Equation 1.~~

Equation 1:

$$P_{Walk-in} = (A_{Ref} PD_{Ref} + N_{Ref} D_{Ref}) + (A_{Frz} PD_{Frz} + N_{Frz} D_{Frz})$$

Where:

$P_{Walk-in}$ is the estimated power density for the walk-in refrigerator or freezer in (W)

A_{xxx} the area of the walk in refrigerator or freezer (ft²)

N_{xxx} the number of glass display doors (unitless)

PD_{xxx} the power density of the walk-in refrigerator or freezer taken from ~~Table 10: Default Power for Walk-In Refrigerators and Freezers (W/ft²)~~ **Error! Reference source not found.** (W/ft²)

D_{xxx} the power associated with a glass display door for a walk-in refrigerator or freezer (W/door)

xxx subscript indicating a walk-in freezer or refrigerator (Ref or Frz)

Table 10: Default Power for Walk-In Refrigerators and Freezers (W/ft²)

Floor Area	Refrigerator	Freezer
100 ft ² or less	8.0	16.0
101 ft ² to 250 ft ²	6.0	12.0
251 ft ² to 450 ft ²	5.0	9.5
451 ft ² to 650 ft ²	4.5	8.0
651 ft ² to 800 ft ²	4.0	7.0
801 ft ² to 1,000 ft ²	3.5	6.5
More than 1,000 ft ²	3.0	6.0
Additional Power for each Glass Display Door	105	325

Source: These values are determined using the procedures of the Heatcraft Engineering Manual, Commercial Refrigeration Cooling and Freezing Load Calculations and Reference Guide, August 2006. The EER is assumed to be 12.39 for refrigerators and 6.33 for freezers. The specific efficiency is assumed to be 70 for refrigerators and 50 for freezers. Operating temperature is assumed to be 35 F for refrigerators and -10 F for freezers.

Refrigeration Modeling Method	
<i>Applicability</i>	All buildings that have commercial refrigeration for cold storage or display
<i>Definition</i>	<p>The method used to estimate refrigeration energy and to model the thermal interaction with the space where casework is located. Two methods are included in this manual:</p> <ul style="list-style-type: none"> Title 24 defaults. With this method, the power density values provided in Appendix 5.4A¹ are used; schedules are assumed to be continuous operation. DOE performance ratings. With this method, the energy modeler takes inventory of the refrigerated casework in the rated building and sums the rated energy use (typically in kWh/day). Walk-in refrigerators and freezers shall use the defaults from Equation 1 and the values from Table 9. All refrigeration equipment is then assumed to operate continuously. <p>The remaining building descriptors in this section apply to buildings that use either the Title 24 defaults or the DOE performance ratings.</p>
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	None. For California compliance, the The Title 24 defaults shall be used. Otherwise, there are no input restrictions.
<i>Standard Design</i>	Same as proposed Title 24 defaults

¹ See Table C-43, p. 146 of NREL/TP-550-41956, Methodology for Modeling Building Energy Performance across the Commercial Sector, Technical Report, Appendix C, March 2008. The values in this report were taken from Table 8-3 of the California Commercial End-Use Survey, Consultants Report, March 2006, CEC-400-2006-005

<i>Standard Design: Existing Buildings</i>	Same as for new construction
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Refrigeration Power	
<i>Applicability</i>	All buildings that have commercial refrigeration for cold storage or display
<i>Definition</i>	Commercial refrigeration power is the average power for all commercial refrigeration equipment, assuming constant year-round operation. Equipment includes walk-in refrigerators and freezers, open refrigerated casework, and closed refrigerated casework. It does not include residential type refrigerators used in kitchenettes or refrigerated vending machines. These are covered under receptacle power.
<i>Units</i>	W/ft ²
<i>Input Restrictions</i>	With the Title 24 defaults method, the values in Appendix 5.4A are prescribed. These values are multiplied times the floor area of the rated building to estimate the refrigeration power. With the DOE performance ratings method, refrigeration power is estimated by summing the kWh/day for all the refrigeration equipment in the space and dividing by 24 hours. The refrigeration power for walk-in refrigerators and freezers is added to this value.
<i>Standard Design</i>	Refrigeration power is the same as the proposed design when the Title 24 defaults are used. When the DOE performance ratings method is used, refrigeration power for casework shall be determined from Table 9 ; the power for walk-in refrigerators and freezers shall be the same as the proposed design.
<i>Standard Design: Existing Buildings</i>	Same as for new construction

Remote Condenser Fraction	
<i>Applicability</i>	All buildings that have commercial refrigeration for cold storage or display and use the Title 24 defaults or DOE performance ratings methods
<i>Definition</i>	<p>The fraction of condenser heat that is rejected to the outdoors. For self-contained refrigeration casework, this value will be zero. For remote condenser systems, this value is 1.0. For combination systems, the value should be weighted according refrigeration capacity.</p> <p>For refrigeration with self-contained condensers and compressors, the heat that is removed from the space is equal to the heat that is rejected to the space, since the evaporator and condenser are both located in the same space. There may be some latent cooling associated with operation of the equipment, but this may be ignored with the Title 24 defaults or DOE performance ratings methods. The operation of self-contained refrigeration units may be approximated by adding a</p>

	<p>continuously operating electric load to the space that is equal to the energy consumption of the refrigeration units. Self-contained refrigeration units add heat to the space that must be removed by the HVAC system.</p> <p>When the condenser is remotely located, heat is removed from the space but rejected outdoors. In this case, the refrigeration equipment functions similar to a continuously running split system air conditioner. Some heat is added to the space for the evaporator fan, the anti-fog heaters and other auxiliary energy uses, but refrigeration systems with remote condensers remove more heat from the space where they are located than they add. The HVAC system must compensate for this imbalance.</p> <p>For remotely located condensers using the Title 24 defaults or DOE performance ratings methods, the heat that is removed from the space is determined as follows:</p> $Q = [(1 - F) \times kW - (F \times kW \times COP)] \times 3.413$ <p>Where:</p> <ul style="list-style-type: none"> Q The rate of heat removal from the space due to the continuous operation of the refrigeration system (kBtu/h). A negative number means that heat is being removed from the space; a positive number means that heat is being added. kW The power of the refrigeration system determined by using the Title 24 defaults or the DOE performance ratings method (kW) F The remote condenser fraction (see building descriptor below) (unitless) COP The coefficient of performance of the refrigeration system (unitless) <p>The simple approach outlined above assumes that there is no latent cooling associated with the refrigeration system. The heat addition or removal resulting from the above equation can be modeled in a number of ways to accommodate the variety of calculation engines available. It can be scheduled if the engine can accommodate a heat removal schedule. It can be modeled as a separate, constantly running air conditioner if the engine can accommodate two cooling systems serving the same thermal zone. Other modeling techniques are acceptable as long as they are thermodynamically equivalent.</p>
<i>Units</i>	Fraction
<i>Input Restrictions</i>	None
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	Same as for new construction

Refrigeration COP

<i>Applicability</i>	All buildings that have commercial refrigeration for cold storage or display and use the Title 24 defaults or DOE performance ratings methods
<i>Definition</i>	The coefficient of performance of the refrigeration system. This is used only to determine the heat removed or added to the space, not to determine the refrigeration power or energy.
<i>Units</i>	Fraction
<i>Input Restrictions</i>	This value is prescribed to be 3.6 for refrigerators and 1.8 for freezers ²
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	Same as for new construction

Refrigeration Schedule	
<i>Applicability</i>	All buildings that have commercial refrigeration for cold storage or display
<i>Definition</i>	The schedule of operation for commercial refrigeration equipment used to convert refrigeration power to energy use.
<i>Units</i>	Data structure: schedule, fractional
<i>Input Restrictions</i>	Continuous operation is prescribed.
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	Same as for new construction

5.4.8 Elevators, Escalators and Moving Walkways

Elevators, escalators and moving walkways account for 3 percent to 5 percent of electric energy use in buildings.³ Buildings up to about five to seven stories typically use hydraulic elevators because of their lower initial cost. Mid-rise buildings commonly use traction elevators with geared motors, while high-rise buildings typically use gearless systems where the motor directly drives the sheave. The energy-using components include the motors and controls as well as the lighting and ventilation systems for the cabs.

Elevators, escalators, and moving walkways are modeled as a plug loads, with the standard design power matching the proposed design.

~~Elevators are custom designed for each building. In this respect they are less like products than they are engineered systems, e.g. they are more akin to chilled water plants where the engineer~~

² ~~These values are consistent with the assumptions for the default values for walk-ins, which assume an EER of 12.39 for refrigerators and 6.33 for freezers.~~

³ Sachs, Harvey M., Opportunities for Elevator Energy Efficiency Improvements, American Council for an Energy Efficiency Economy, April 2005.

chooses a chiller, a tower, pumping and other components which are field engineered into a system. The main design criteria are safety and service. Some manufacturers have focused on energy efficiency of late and introduced technologies such as advanced controls that optimize the position of cars for minimum travel and regeneration motors that become generators when a loaded car descends or an empty car rises. These technologies can result in 35 percent to 40 percent savings.⁴

The motors and energy using equipment is typically located within the building envelope so it produces heat that must be removed by ventilation or by air conditioning systems. In energy models, a dedicated thermal zone (elevator shaft) will typically be created and this space can be indirectly cooled (from adjacent spaces) or positively cooled.

Little information is known on how to model elevators. As engineered systems, the model would need information on the number of starts per day, the number of floors, motor and drive characteristics, and other factors. Some work has been done to develop and categorize energy models for elevators;⁵ however a simple procedure is recommended based on a count of the number of elevators, escalators, and moving walkways in the building. This data is shown in Table 11.⁶

Table 11: Unit Energy Consumption Data for Elevators, Escalators, and Moving Walkways

Mode	Elevators		Escalators and Moving Walkways	
	Power (W)	Annual Hours	Power (W)	Annual Hours
Active	10,000	300	4,671	4,380
Ready	500	7,365	n.a.	0
Standby	250	1,095	n.a.	0
Off	0	0	0	4,380
Typical Annual Energy Use	7,000 kWh/y		20,500 kWh/y	

Elevator/Escalator Power	
<i>Applicability</i>	All buildings that have commercial elevators, escalators, or moving walkways
<i>Definition</i>	The power for elevators, escalators and moving walkways are modeled for different modes of operation. Elevators typically operate in three modes: active (when the car is moving passengers), ready (when the lighting and ventilation systems are active but the car is not moving), and standby (when the lights and ventilation systems are off). Escalators and moving walkways are either active or turned off as a plug loads.

⁴ Ibid.

⁵ Al-Sharif, Lutfi, Richard Peters and Rory Smith, Elevator Energy Simulation Model, Elevator World, November 2005, Volume LII, No14

⁶ TIAx, Commercial and Residential Sector Miscellaneous Electricity consumption: Y20005 and Projections to 2030, Final Report to the U.S. Department of Energy's Energy Information Administration (EIA) and Decision Analysis Corporation (DAC), September 22, 2006, Reference Number D0366.

<i>Units</i>	W/unit
<i>Input Restrictions</i>	The power values from Table 11 for different modes of operation are prescribed for the proposed design.
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	Not applicable

Elevator/Escalator Schedule	
<i>Applicability</i>	All buildings that have commercial elevators, escalators, or moving walkways
<i>Definition</i>	The schedule of operation for elevators, escalators, and moving walkways. This is used to convert elevator/escalator power to energy use.
<i>Units</i>	Data structure: schedule, state
<i>Input Restrictions</i>	The operating schedule is prescribed <u>and indicated in Appendix 5.4B</u> . For California compliance, an appropriate schedule from Appendix 5.4B shall be used. If values other than those shown in Appendix 5.4B are used, this will be reported as a condition requiring an exceptional condition review by a third party reviewer.
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	Not applicable

5.4.9 Process, Gas

Commercial gas equipment includes the following:

- Ovens
- Fryers
- Grills
- Other equipment

The majority of gas equipment is located in the space and may contribute both sensible and latent heat. Gas equipment is ~~typically~~ modeled by specifying the rate of peak gas consumption and modifying this with a fractional schedule. ~~Energy consumption data for gas equipment is only beginning to emerge.~~

~~Because of these limits, the procedure for commercial gas is limited.~~ The procedure consists of prescribed power and energy values for use with both the proposed and standard design ~~and the baseline~~ buildings. No credit for commercial gas energy efficiency features is offered.

The prescribed values are provided in Appendix 5.4A. Schedules are defaulted to be continuous operation.

Gas Equipment Power	
<i>Applicability</i>	All buildings that have commercial gas equipment
<i>Definition</i>	Commercial gas power is the average power for all commercial gas equipment, assuming constant year-round operation.
<i>Units</i>	<p>Btu/h-ft²</p> <p>Compliance software shall also use the following prescribed values to specify the latent heat gain fraction and the radiative/convective heat gain split.</p> <p>For software that specifies the fraction of the heat gain that is lost from the space, this fraction shall be prescribed at 0.</p> <p>Gas Equipment Power Heat Gain Fractions: Radiative = 0.15, Latent = 0, Convective = 0</p>
<i>Input Restrictions</i>	The values in Appendix 5.4A are prescribed.
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	Not applicable

Gas Equipment Schedule	
<i>Applicability</i>	All buildings that have commercial gas equipment
<i>Definition</i>	The schedule of operation for commercial gas equipment. This is used to convert gas power to energy use.
<i>Units</i>	Data structure: schedule, fractional
<i>Input Restrictions</i>	Continuous operation is prescribed.
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	Not applicable

Gas Equipment Location	
<i>Applicability</i>	All buildings that have commercial gas equipment
<i>Definition</i>	The assumed location of the gas equipment for modeling purposes. Choices are in the space or external.
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed.

<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	Not applicable

Radiation Factor	
<i>Applicability</i>	Gas appliances located in the space
<i>Definition</i>	The fraction of heat gain to appliance energy use
<i>Units</i>	Fraction
<i>Input Restrictions</i>	Default value is 0.15. Other values can be used when a detailed inventory of equipment is known. The override value shall be based on data in Table 5C, Chapter 18, ASHRAE HOF, 2009, or similar tested information from the manufacturer.
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	Not applicable

Usage Factor	
<i>Applicability</i>	Gas appliances located in the space
<i>Definition</i>	<p>A duty cycle or usage factor to appliance energy use.</p> <p>The radiation factor and usage factor are used together to determine the sensible heat gain to the space:</p> $Q_{sens} = Q_{input} \times F_U \times F_R$ <p>Where Q_{input} is the heat input of the equipment in Btu/h or Btu/h-ft², F_U is the usage factor and F_R is the radiation factor</p>
<i>Units</i>	Fraction
<i>Input Restrictions</i>	Default value is 0.70. Other values can be used when a detailed inventory of equipment is known. The override value shall be based on data in Table 5C, Chapter 18, ASHRAE HOF, 2009, or similar tested information from the manufacturer.
<i>Standard Design</i>	Same as the proposed design

<i>Standard Design:</i> <i>Existing Buildings</i>	Not applicable
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Gas Process Loads	
<i>Applicability</i>	Spaces with process loads
<i>Definition</i>	<p>Process load is the gas energy consumption in the conditioned space of a building resulting from an activity or treatment not related to the space conditioning, lighting, service water heating, or ventilating of a building as it relates to human occupancy. Process load may include sensible and/or latent components.</p> <p>Compliance software shall model and simulate process loads only if the amount of the process energy and the location and type of process equipment are specified in the construction documents. This information shall correspond to specific special equipment shown on the building plans and detailed in the specifications. The compliance software shall inform the user that the software will output process loads including the types of process equipment and locations on the compliance forms.</p>
<i>Units</i>	Data structure: sensible (Btu/h), latent (Btu/h)
<i>Input Restrictions</i>	Compliance software shall receive input for sensible and/or latent process load for each zone in the proposed design. The process load input shall include the amount of the process load (Btu/h-ft ²) and the thermal zone where the process equipment is located. The modeled information shall be consistent with the plans and specifications of the building.
<i>Standard Design</i>	The standard design shall use the same gas process loads and sensible and latent contribution and radiative/convective split for each zone as the proposed design.
<i>Standard Design:</i> <i>Existing Buildings</i>	Same as new construction

Electric Process Loads	
<i>Applicability</i>	Spaces with electric process loads
<i>Definition</i>	<p>Process load is the electrical energy consumption in the conditioned space of a building resulting from an activity or treatment not related to the space conditioning, lighting, service water heating, or ventilating of a building as it relates to human occupancy.</p> <p>Data center loads including transformers, uninterruptible power supplies, power delivery units, server fans and power supplies are considered receptacle loads, not process loads, and the equipment schedules are</p>

	<p>given in Appendix 5.4B.</p> <p>Compliance software shall model and simulate process loads only if the amount of the process energy and the location and type of process equipment are specified in the construction documents. This information shall correspond to specific special equipment shown on the building plans and detailed in the specifications. The compliance software shall inform the user that the software will output process loads including the types of process equipment and locations on the compliance forms.</p>
<i>Units</i>	<p>Data structure: load (kW)</p> <p>For electric process loads, the radiative fraction shall be defaulted to 0.2 and the convective fraction shall be defaulted to 0.8 by the compliance software. The user may enter other values for the radiative/convective split, but the compliance software shall verify that the values add to 1.</p>
<i>Input Restrictions</i>	<p>Compliance software shall receive input for sensible and/or latent process load for each zone in the proposed design. The process load input shall include the amount of the process load (Btu/h-ft²) and the thermal zone where the process equipment is located. The modeled information shall be consistent with the plans and specifications of the building.</p>
<i>Standard Design</i>	<p>The standard design shall use the same process loads and radiative/convective split for each zone as the proposed design.</p>
<i>Standard Design: Existing Buildings</i>	<p>Same as new construction</p>

Gas Process Load Schedule	
<i>Applicability</i>	All buildings that have commercial gas equipment
<i>Definition</i>	The schedule of process load operation. Used to convert gas power to energy use.
<i>Units</i>	Data structure: schedule, fractional
<i>Input Restrictions</i>	As designed.
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	Not applicable

Electric Process Load Schedule	
<i>Applicability</i>	All buildings that have commercial gas equipment
<i>Definition</i>	The schedule of electric process load operation.
<i>Units</i>	Data structure: schedule, fractional

<i>Input Restrictions</i>	As designed.
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	Not applicable

5.4.10 Water Heating Use

This section defines the water heating load (use rate) and system requirements on a space level.

<u>Space Water Heating Use Rate</u>	
<i>Applicability</i>	<u>All spaces</u>
<i>Definition</i>	<u>The water heating use rate for a space in a building</u>
<i>Units</i>	<u>Gal/h</u>
<i>Input Restrictions</i>	<u>The values in Appendix 5.4A are prescribed.</u>
<i>Standard Design</i>	<u>Same as the proposed design</u>
<i>Standard Design: Existing Buildings</i>	<u>Not applicable</u>

<u>Space Water Heating Fuel Type</u>	
<i>Applicability</i>	<u>All spaces</u>
<i>Definition</i>	<u>A mapping that defines the standard design water heating fuel type for a space</u>
<i>Units</i>	<u>List; gas or electric</u>
<i>Input Restrictions</i>	<u>As designed</u>
<i>Standard Design</i>	<u>Prescribed from the table in Appendix 5.4A</u>
<i>Standard Design: Existing Buildings</i>	<u>Not applicable</u>

5.5 Building Envelope Data

5.5.1 Materials

Energy simulation programs commonly define construction assemblies by listing a sequence of material layers that make up the construction assembly. Appendix 5.5A has a list of standard

materials that may be referenced by construction assemblies. Additional materials not listed in Appendix 5.5A may be defined as described below. Alternate methods may be used to define construction assemblies such as specifying the U-factor and optionally, a metric describing thermal mass such as heat capacity (HC). These alternate methods may not require identification of materials. When a material is defined, all of the properties listed below must be defined. Some materials listed in Appendix 5.5A are non-homogeneous, for instance, framing members with insulation in the cavity. Typical construction assemblies and their respective material layers are defined in Reference Appendix JA4. Additionally, the properties of the each material layer can be found in ACM Appendix 5.5A.

Material Name	
<i>Applicability</i>	Opaque constructions
<i>Definition</i>	The name of a construction material used.
<i>Units</i>	Text: unique
<i>Input Restrictions</i>	Material name is a required input for materials not available from the standard list in ACM Appendix 5.5A. The user may not modify entries for predefined materials.
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	Not applicable

Density	
<i>Applicability</i>	Opaque constructions
<i>Definition</i>	The density, mass per unit volume, of the construction material as documented in Appendix 5.5A.
<i>Units</i>	lb/ft ³
<i>Input Restrictions</i>	Prescribed from Appendix 5.5A
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	Not applicable

Specific Heat	
<i>Applicability</i>	Opaque constructions
<i>Definition</i>	The specific heat capacity of a material is numerically equal to the quantity of heat that must be supplied to a unit mass of the material to increase its temperature by 1°F

<i>Units</i>	Btu/lb·°F
<i>Input Restrictions</i>	Prescribed from Appendix 5.5A
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	Not applicable

Thermal Conductivity	
<i>Applicability</i>	All non-standard materials
<i>Definition</i>	The thermal conductivity of a material of unit thickness is numerically equal to the quantity of heat that will flow through a unit area of the material when the temperature difference through the material is 1°F.
<i>Units</i>	Btu/lb·°F
<i>Input Restrictions</i>	Prescribed from Appendix 5.5A Not applicable
<i>Standard Design</i>	Not applicable Prescribed from Appendix 5.5A
<i>Standard Design: Existing Buildings</i>	Not applicable

Thickness	
<i>Applicability</i>	All non-standard materials
<i>Definition</i>	The thickness of a material
<i>Units</i>	Inches
<i>Input Restrictions</i>	Prescribed from Appendix 5.5A
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	Not applicable

5.5.2 Construction Assemblies

For California compliance, construction assemblies for the proposed design shall be created by selecting from a library of building construction layers in ACM Appendix 5.5A. The compliance software shall all specify composite layers that consist of both framing and insulation. It shall use established methods defined in the ASHRAE Handbook of Fundamentals for calculating effective R-values of composite layers.

Geometry

The geometry of roofs, walls, floors, doors and fenestration should match the construction documents or as-built drawings as accurately as possible. Unusual curved surfaces such as a dome or semi-circular wall may be approximated by a series of constructions.

Mass Walls

For mass walls, the user first chooses the mass layer from Appendix 5.5A. After that, the user may select an insulating layer from Appendix 5.5A for outside and/or inside the mass wall.

Ballasted Roofs, Vegetated Roofs, Concrete Pavers, and Other Mass Roofs

An additional layer may be added to the roof construction assembly when thermal mass is used above the roof membrane. This exception is intended to allow ballasted roofs, concrete pavers and other massive elements to be explicitly modeled. To qualify, the weight of the stone ballast, the concrete pavers or other elements must exceed ~~15~~25 lb/ft². The thickness, heat capacity, conductance and density of the additional mass layer shall be based on the measured physical properties of the material. If the surface properties of the additional mass material have been verified through the Cool Roof Rating Council (CRRC), the CRRC reported properties may be used for the proposed design. Otherwise, the mass layer shall be modeled with an aged reflectance of 0.10 and an emittance of ~~0.85~~0.75.

Assembly Name	
<i>Applicability</i>	All projects
<i>Definition</i>	The name of a construction assembly that describes a roof, wall, or floor assembly. The name generally needs to be unique so it can be referenced precisely by surfaces.
<i>Units</i>	Text
<i>Input Restrictions</i>	Required input and name must be unique.
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	Not applicable

Specification Method	
<i>Applicability</i>	All projects
<i>Definition</i>	The method of describing a construction assembly. The more <u>more</u> simple <u>simpler</u> method is to describe the U-factor of the construction assembly which can account for thermal bridging and other factors. However with this method, the time delay of heat transfer through the construction assembly is not accounted for. Generally, with the U-factor method, heat transfer is assumed to occur instantly. The more complex method is to describe the construction assembly as a series of layers, each layer representing a material. With this method, heat transfer is delayed in accord with the thermal mass and other properties of the assembly. For below-grade constructions, a C-factor can be specified; for slab-on-grade constructions, an F-factor is specified.

<i>Units</i>	List: layers, U-factor, C-factor, F-factor
<i>Input Restrictions</i>	The layers method shall be used for all above-grade constructions
<i>Standard Design</i>	For each construction, the proposed design specification method shall be used.
<i>Standard Design: Existing Buildings</i>	Same as new construction

Layers	
<i>Applicability</i>	All construction assemblies that use the layers method of specification
<i>Definition</i>	A structured list of material names that describe a construction assembly, beginning with the exterior finish and progressing through the assembly to the interior finish. Material names must be from the standard list (Appendix E) or defined (see above) ACM Appendix 5.5A.
<i>Units</i>	List: layers of construction assembly
<i>Input Restrictions</i>	The user is required to describe all layers in the actual assembly and model the proposed design based the layer descriptions.
<i>Standard Design</i>	See building descriptors for roofs, exterior walls, exterior floors, doors, fenestration and below-grade walls.
<i>Standard Design: Existing Buildings</i>	Same as new construction

5.5.3 Roofs

Roof Name	
<i>Applicability</i>	All roof surfaces
<i>Definition</i>	A unique name or code that identifies the roof and ties it to the construction documents submitted for energy code review. It is not mandatory to name roofs.
<i>Units</i>	Text
<i>Input Restrictions</i>	Name must be unique
<i>Standard Design</i>	N/A
<i>Standard Design: Existing Buildings</i>	N/A

Roof Type

<i>Applicability</i>	All roof surfaces
<i>Definition</i>	A roof classification defined in the standards. This descriptor can be derived from other building descriptors and it may not be necessary for the software user to specify it directly.
<i>Units</i>	List: attic and other roofs; metal building roofs; and roofs with insulation entirely above deck metal building, wood framed or other
<i>Input Restrictions</i>	Not applicable for new construction; as designed for existing buildings
<i>Standard Design</i>	All roofs in the <u>standard design</u> baseline building are modeled as wood-framed and other.
<i>Standard Design: Existing Buildings</i>	Same as proposed

Roof Geometry	
<i>Applicability</i>	All roofs
<i>Definition</i>	Roof geometry defines the position, orientation, azimuth, tilt, and dimensions of the roof surface. The details of how the coordinate system is implemented may vary between software programs. The data structure for surfaces is described in the reference section of this chapter.
<i>Units</i>	Data structure: surface
<i>Input Restrictions</i>	There are no restrictions other than that the surfaces defined must agree with the building being modeled, as represented on the construction drawings or as-built drawings.
<i>Standard Design</i>	Roof geometry will be identical in the proposed and standard design building designs. For alterations, roof geometry will be fixed, based on one of the building prototypes (office, retail, etc.)
<i>Standard Design: Existing Buildings</i>	Same as new construction

Roof Solar Reflectance	
<i>Applicability</i>	All opaque exterior roof surfaces exposed to ambient conditions
<i>Definition</i>	The solar reflectance of a material. For roofing materials, the three-year aged reflectance value from CRRC testing should be used if available.
<i>Units</i>	
<i>Input Restrictions</i>	For roofs that are part of new construction, if asphalt shingles or composition shingles are not rated by the CRRC, the default aged solar reflectance shall be equal to 0.08 for asphalt roofs and 0.10 for all other

	<p>roof types. The default value may be overridden when roof materials are used that the CRRC has tested and are called for in the construction documents. In cases where the default value is overridden, the user is required to submit documentation identifying the test procedure that was used to establish the non-default values. If the aged CRRC reflectance is not known, the aged CRRC reflectance may be calculated from the initial CRRC reflectance using the following equation:</p> $p_{\text{aged}} = 0.2 + \beta \cdot (p_{\text{init}} - 0.2)$ <p>Where,</p> <p>p_{aged} = the calculated aged reflectance</p> <p>β = 0.65 for field-applied coatings, 0.7 for all other roof surfaces</p> <p>p_{init} = the initial CRRC reflectance</p> <p>As a compliance option, low-sloped roofs that use aggregate may specify an aged reflectance of 0.50 if the product meets the following criteria:</p> <ol style="list-style-type: none"> 1. Conforms to material standard ASTM D1863. 2. Conforms to ASTM D448, size number equal between No.6 and No.7. 3. Has a CRRC-tested initial solar reflectance that meets or exceeds 0.55 using the ASTM E1918 test procedure with aggregate passing a No. 4 sieve and is retained by a No. 8 sieve that conforms to ASTM D448, conducted by a CRRC-accredited independent laboratory meeting the requirement of Section 10-113(d)4 of the Building Energy Efficiency Standards. 4. Has a label on bags or containers of aggregate stating that the materials conform to ASTM D1863 and ASTM D448.
<p><i>Standard Design</i></p>	<p>For new construction, the standard design reflectance is defined in Table 140.3-B for nonresidential buildings, Table 140.3-C for high-rise residential buildings and hotel-motel buildings containing guestrooms, and Table 140.3-D for relocatable classroom buildings.</p> <p>For alterations to more than 50 percent of the roof area or roof areas above 2,000 ft², the standard design shall be modeled as the more efficient of either the existing conditions or the values required for cool roofs under Section 141.0 of the standards.</p>
<p><i>Standard Design: Existing Buildings</i></p>	<p>Same as new construction</p>

<p>Roof Thermal Emittance</p>	
<p><i>Applicability</i></p>	<p>All opaque exterior roof surfaces exposed to ambient conditions</p>
<p><i>Definition</i></p>	<p>The thermal emittance of a material. For roofing materials, the three-year aged emittance value from CRRC testing should be used if available.</p>

<i>Units</i>	
<i>Input Restrictions</i>	<p>For roofs, new construction: as designed, from CRRC values. If CRRC rating information is not available, the default thermal emittance shall be 0.75<u>0.85</u>. Aggregate that meets the following criteria may specify a thermal emittance of 0.85<u>0.85</u>:</p> <ol style="list-style-type: none"> 1. Conforms to material standard ASTM D1863. 2. Conforms to ASTM D448, size number equal between No.6 and No.7. 3. Has a CRRC-tested initial solar reflectance that meets or exceeds 0.55 using the ASTM E1918 test procedure with aggregate passing a No. 4 sieve and is retained by a No. 8 sieve that conforms to ASTM D448, conducted by a CRRC accredited independent laboratory meeting the requirement of Section 10-113(d) 4 of the Building Energy Efficiency Standards. 4. Has a label on bags or containers of aggregate stating that the materials conform to ASTM D1863 and ASTM D448.
<i>Standard Design</i>	<p>For roofs, new construction, the standard design thermal emittance shall be 0.85<u>0.85</u>.</p> <p>For alterations to more than 50 percent of the roof area or roof areas above 2,000 ft², the standard design shall be modeled as the more efficient of either the existing conditions or a thermal emittance of 0.85.</p>
<i>Standard Design: Existing Buildings</i>	<u>If the existing roof is unaltered, same as proposed. For alterations, the standard design is 0.78<u>0.85</u>.</u>

Roof Construction			
<i>Applicability</i>	All roofs		
<i>Definition</i>	A specification containing a series of layers that result in a construction assembly for the proposed design. The first layer in the series represents the outside (or exterior) layer and the last layer represents the inside (or interior) layer. See the building descriptors above for roof construction type.		
<i>Units</i>	List: layers		
<i>Input Restrictions</i>	<p>The area-weighted average of the roof construction assembly U-factors, defined by a series of layers, must be equal to or more efficient than the mandatory U-factor requirements of Section 120.7 of the standards for new construction, and Section 141.0 of the standards for alterations. Note that these U-Factor requirements assume an exterior air film of R-0.17 and an interior air film of R-0.61. Each layer specified must be listed in the materials database in the ACM Appendix 5.5A.</p> <p>New Construction</p> <table border="1"> <tr> <td>Metal Building</td> <td>U – 0.098</td> </tr> </table>	Metal Building	U – 0.098
Metal Building	U – 0.098		

	<table border="1" style="width: 100%;"> <tr> <td style="width: 50%;">Wood Framed and Others</td> <td style="width: 50%;">U – 0.075</td> </tr> </table> <p>Additions and Alterations</p> <table border="1" style="width: 100%;"> <tr> <td style="width: 50%;">Roof / Ceiling Insulation</td> <td style="width: 50%;">See 141.0(b)2Biii of the standards</td> </tr> </table> <p>Appropriate R-values for insulation can be calculated using the formula below.</p> $R_{insulation} = (1/U_{Factor}) - R_{Layer(1)} - R_{Layer(2)} - R_{Layer(3)} - R_{Layer(n)}$ $R_{insulation} = R_{ins_{continuous}} + R_{ins_{framing}}$ <p>Ceilings that form the boundary between the modeled building of an additions and alterations project and the existing, un-modeled portion of the building may be modeled as adiabatic roofs (no heat transfer).</p>	Wood Framed and Others	U – 0.075	Roof / Ceiling Insulation	See 141.0(b)2Biii of the standards													
Wood Framed and Others	U – 0.075																	
Roof / Ceiling Insulation	See 141.0(b)2Biii of the standards																	
<p><i>Standard Design</i></p>	<p>Roofs in the standard design baseline building are of the type “insulation entirely above deck.” The insulation requirement is determined by climate zone and baseline standard. The standard design baseline building roof construction shall be modeled as layers as defined. See Appendix 5.5B for details.</p> <p>For new construction, the standard design roof type is wood framed and other, and the roof is a standing seam metal roof, with the R-value of continuous insulation adjusted to match the prescriptive standards for wood-framed and other roofs. The U-factor required for roof construction is defined in Table 140.3-B, C or D of the standards. Programs that model a U-factor shall include an exterior and interior air film resistance. The standard design construction is based on JA4-10 Table 4.2.7 and assumes an exterior air film of R-0.17 and an interior air film of R-0.61.</p> <p>The standard design construction shall include the following layers:</p> <table border="1" style="width: 100%;"> <tr> <td style="width: 15%;">Layer 1</td> <td style="width: 55%;">Metal Standing Seam 1/16 in.</td> <td style="width: 30%;">R - 0.00</td> </tr> <tr> <td>Layer 2</td> <td>Continuous Insulation</td> <td>R - Based on Climate Zone</td> </tr> <tr> <td>Layer 3</td> <td>Open Framing + No Insulation</td> <td>R — 0.00</td> </tr> </table> <p>The value of the continuous insulation layer entirely above framing shall be set to achieve the following R-values:</p> <p>Nonresidential Buildings: Continuous Insulation</p> <table border="1" style="width: 100%;"> <tr> <td style="width: 55%;">Climate Zones 2, 3, 4, 9-16</td> <td style="width: 45%;">R — 28.63 (U-0.034)</td> </tr> <tr> <td>Climate Zones 1, 5</td> <td>R - 28.63 (U-0.034)</td> </tr> <tr> <td>Climate Zones 7, 8</td> <td>R — 19.62 (U-0.049)</td> </tr> <tr> <td>Climate Zones 6</td> <td>R — 19.62 (U-0.049)</td> </tr> </table> <p>High-Rise Residential Buildings and Hotel/Motel Guestrooms: Continuous Insulation</p>	Layer 1	Metal Standing Seam 1/16 in.	R - 0.00	Layer 2	Continuous Insulation	R - Based on Climate Zone	Layer 3	Open Framing + No Insulation	R — 0.00	Climate Zones 2, 3, 4, 9-16	R — 28.63 (U-0.034)	Climate Zones 1, 5	R - 28.63 (U-0.034)	Climate Zones 7, 8	R — 19.62 (U-0.049)	Climate Zones 6	R — 19.62 (U-0.049)
Layer 1	Metal Standing Seam 1/16 in.	R - 0.00																
Layer 2	Continuous Insulation	R - Based on Climate Zone																
Layer 3	Open Framing + No Insulation	R — 0.00																
Climate Zones 2, 3, 4, 9-16	R — 28.63 (U-0.034)																	
Climate Zones 1, 5	R - 28.63 (U-0.034)																	
Climate Zones 7, 8	R — 19.62 (U-0.049)																	
Climate Zones 6	R — 19.62 (U-0.049)																	

	Climate Zones 1, 2,4,8-16	R — 34.93 (U-0.028)
	Climate Zone 7	R — 24.86 (U-0.039)
	Climate Zones 3, 5, 6	R — 28.63 (U-0.034)
	<p>For mixed-use buildings, the roof standard design requirements shall be determined by which space type (nonresidential or residential) is the majority of the floor area of the adjoining conditioned spaces.</p> <p>For re-locatable classroom buildings, the standard design shall use the construction assembly corresponding to the most stringent of requirements in any climate zone, or R-28.63 continuous insulation.</p> <p>For alterations, any approved roof type may be used. The U-factor in the standard design shall be modeled as the more efficient of either the existing conditions or the values stated in Section 141.0 of the standards. Where applicable, selection shall be based on building type, assembly, and climate zone. A construction of layers shall be defined to yield an equivalent U-factor.</p>	
<i>Standard Design: Existing Buildings</i>	<p>For existing buildings, if the roof component is not altered, the standard design roof construction shall match the proposed design roof construction of the existing building. If the roof is altered, the roof component shall meet the prescriptive requirements for new construction for the roof type of the existing building.</p> <p>The roof type of the existing building is either a metal building roof or a wood-framed or other roof. The standard design roof assemblies for altered roofs are shown below for the appropriate climate zones.</p> <p>The Reference Appendix JA4 reference and U-factor are provided for reference only. The U-factor does not need to exactly match the JA4 value, but the layer shall match the layer described below.</p> <p>Alterations Roof Standard Design:</p> <p>For alterations, any approved roof type may be used. The U-factor in the standard design shall be modeled as the more efficient of either the existing conditions or the values stated in Section 141.0 of the standards. Where applicable, selection shall be based on building type, assembly, and climate zone. A construction of layers shall be defined to yield an equivalent U-factor.</p>	

5.5.4 Exterior Walls

Wall Name	
<i>Applicability</i>	All walls
<i>Definition</i>	A unique name or code that relates the exterior wall to the design documents. This is an optional input since there are other acceptable ways to key surfaces to the construction documents.
<i>Units</i>	Text

<i>Input Restrictions</i>	Must be unique
<i>Standard Design</i>	None
<i>Standard Design: Existing Buildings</i>	None

Wall Type	
<i>Applicability</i>	All walls
<i>Definition</i>	<p>One of four categories of above-grade wall assemblies used to determine minimum insulation requirements for walls. The five wall type categories are as follows:</p> <ol style="list-style-type: none"> 1. Mass Light 2. Mass Heavy 3. Metal building 4. Metal framing 5. Wood framing and other walls <p>A mass light wall is defined as a wall with total heat capacity greater than 7 but less than 15 Btu/ft²-. A mass heavy wall is defined as a wall with a total heat capacity of 15 Btu/ft²-°F or greater. (Heat capacity is defined as the product of the specific heat in Btu/lb-°F, the thickness in ft, and the density in lb/ft³.)</p>
<i>Units</i>	List: mass light, mass heavy, metal building walls, metal framing walls, and wood framing and other walls
<i>Input Restrictions</i>	This input is required for existing buildings when any wall is altered. This input is not required for new construction.
<i>Standard Design</i>	All walls in the standard design building are modeled as “metal framed.”
<i>Standard Design: Existing Buildings</i>	Same as proposed

Wall Geometry	
<i>Applicability</i>	All walls
<i>Definition</i>	Wall geometry defines the position, orientation, azimuth, and tilt of the wall surface. The data structure for surfaces is described in the reference section of this chapter.
<i>Units</i>	Data structure: surface
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	All walls in the standard design building are modeled as “metal framed.” Same as proposed

<i>Standard Design: Existing Buildings</i>	Wall geometry in the standard design building is identical to the proposed design. <u>Same as proposed</u>
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Wall Solar Reflectance	
<i>Applicability</i>	All opaque exterior walls exposed to ambient conditions
<i>Definition</i>	The solar reflectance of a material.
<i>Units</i>	Unitless ratio
<i>Input Restrictions</i>	For walls and other non-roof surfaces, the value is prescribed to be 0.3.
<i>Standard Design</i>	For walls and other non-roof surfaces, the value is prescribed to be 0.3. For new construction, the standard design reflectance shall meet the requirements stated in Section 140.3 of the standards.
<i>Standard Design: Existing Buildings</i>	<u>0.3</u>

Wall Thermal Emittance	
<i>Applicability</i>	All opaque exterior walls exposed to ambient conditions
<i>Definition</i>	The thermal emittance of a material.
<i>Units</i>	<u>Unitless ratio</u>
<i>Input Restrictions</i>	For walls and other non-roof surfaces, the value is prescribed to be 0.9
<i>Standard Design</i>	For walls and other non-roof surfaces, the thermal emittance is 0.9
<i>Standard Design: Existing Buildings</i>	<u>For walls and other non-roof surfaces, the thermal emittance is 0.9</u>

Wall Construction	
<i>Applicability</i>	All walls that use the layers method
<i>Definition</i>	A specification containing a series of layers that result in a construction assembly for the proposed design. The first layer in the series represents the outside (or exterior) layer and the last layer represents the inside (or interior) layer. See the building descriptors above for wall construction type.
<i>Units</i>	List: Layers
<i>Input Restrictions</i>	The area weighted-average of the construction assembly U-factors, defined by a series of layers, must be equal to or more efficient than the

	<p>mandatory U-factor requirements of Section 120.7 of the standards for new construction. Note that these U-Factor requirements assume an exterior air film of R-0.17 and an interior air film of R-0.68. Each layer specified, with the exception of composite layers, must be listed in the materials database in the ACM Appendix 5.5A.</p> <p>Newly Constructed</p> <table border="1"> <tr> <td>Metal Building</td> <td>U – 0.113</td> </tr> <tr> <td>Metal Framed</td> <td>U – 0.151 (R-13 cavity + R-2 continuous insulation, or equivalent)</td> </tr> <tr> <td>Light Mass Walls</td> <td>U – 0.440</td> </tr> <tr> <td>Heavy Mass Walls</td> <td>U – 0.690</td> </tr> <tr> <td>Wood Framed and Others</td> <td>U – 0.110</td> </tr> <tr> <td>Spandrel Panels / Glass Curtain Walls</td> <td>U – 0.280</td> </tr> </table> <p>Additions and Alterations</p> <table border="1"> <tr> <td>Metal Building</td> <td>U – 0.113</td> </tr> <tr> <td>Metal Framed</td> <td>U – 0.217</td> </tr> <tr> <td>Wood Framed and Others</td> <td>U – 0.110</td> </tr> <tr> <td>Spandrel Panels / Glass Curtain Walls</td> <td>U – 0.280</td> </tr> </table> <p>Appropriate R-values for insulation can be calculated using the formula below.</p> $R_{insulation} = (1/U_{Factor}) - R_{Layer(1)} - R_{Layer(2)} - R_{Layer(3)} - R_{Layer(n)}$ $R_{insulation} = R_{ins_{continuous}} + R_{ins_{framing}}$ <p>Walls that form the boundary between the modeled building of an additions and alterations project and the existing, un-modeled portion of the building may be modeled as adiabatic walls (no heat transfer).</p>	Metal Building	U – 0.113	Metal Framed	U – 0.151 (R-13 cavity + R-2 continuous insulation, or equivalent)	Light Mass Walls	U – 0.440	Heavy Mass Walls	U – 0.690	Wood Framed and Others	U – 0.110	Spandrel Panels / Glass Curtain Walls	U – 0.280	Metal Building	U – 0.113	Metal Framed	U – 0.217	Wood Framed and Others	U – 0.110	Spandrel Panels / Glass Curtain Walls	U – 0.280
Metal Building	U – 0.113																				
Metal Framed	U – 0.151 (R-13 cavity + R-2 continuous insulation, or equivalent)																				
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Heavy Mass Walls	U – 0.690																				
Wood Framed and Others	U – 0.110																				
Spandrel Panels / Glass Curtain Walls	U – 0.280																				
Metal Building	U – 0.113																				
Metal Framed	U – 0.217																				
Wood Framed and Others	U – 0.110																				
Spandrel Panels / Glass Curtain Walls	U – 0.280																				
<p><i>Standard Design</i></p>	<p>The U-factor required for wall construction of the standard design building is defined in Table 140.3-B, C or D of the standards. Programs that model a U-factor shall use an exterior and interior air film resistance. The standard design construction is based on JA4-10 Table 4.3.3 and assumes an exterior air film of R-0.17 and an interior air film of R-0.68.</p> <p>For metal framed walls, the standard design construction shall include the following layers:</p> <table border="1"> <tr> <td>Layer 1</td> <td>Stucco – 7/8 in.</td> <td>R - 0.18</td> </tr> <tr> <td>Layer 2</td> <td>Building Paper</td> <td>R – 0.06</td> </tr> <tr> <td>Layer 3</td> <td>Continuous Insulation</td> <td>R - Based on Climate Zone</td> </tr> <tr> <td>Layer 4</td> <td>Closed Framing and No Ins.</td> <td>R – 0.65</td> </tr> </table>	Layer 1	Stucco – 7/8 in.	R - 0.18	Layer 2	Building Paper	R – 0.06	Layer 3	Continuous Insulation	R - Based on Climate Zone	Layer 4	Closed Framing and No Ins.	R – 0.65								
Layer 1	Stucco – 7/8 in.	R - 0.18																			
Layer 2	Building Paper	R – 0.06																			
Layer 3	Continuous Insulation	R - Based on Climate Zone																			
Layer 4	Closed Framing and No Ins.	R – 0.65																			

	Layer 5	Gypsum Board – 1/2 in.	R – 0.45
<i>Standard Design: Existing Buildings</i>	The value of the continuous insulation layer entirely outside framing shall be set to achieve the following R-values:		
	Nonresidential Buildings: Continuous Insulation		
	Climate Zones 1, 6, and 7	R – 12.30	
	Climate Zones 2, 4, 5, and 8 – 16	R – 13.94	
	Climate Zones 3	R – 10.01	
	High-Rise Residential Buildings and Hotel/Motel Guestrooms: Continuous Insulation		
	Climate Zones 1 - 6, and 8-16	R – 12.30	
	Climate Zone 7	R – 7.33	
	For mixed-use buildings that contain both nonresidential and residential spaces, walls adjacent to nonresidential spaces shall use the Nonresidential Buildings standard design construction, and walls adjacent to residential and high-rise residential spaces shall use the high-rise residential standard design construction.		
	For relocatable classroom buildings, the standard design shall use the construction assembly corresponding to the most stringent of requirements in any climate zone, or R-13.94 continuous insulation.		

5.5.5 Exterior Floors

Floor Name	
<i>Applicability</i>	All floor surfaces
<i>Definition</i>	A unique name or code that relates the exposed floor to the design documents. Exposed floors include floors exposed to the outdoors and floors over unconditioned spaces, but do not include slab-on-grade floors, below grade floors, or interior floors.
<i>Units</i>	Text
<i>Input Restrictions</i>	Must be unique
<i>Standard Design</i>	None
<i>Standard Design: Existing Buildings</i>	None

Floor Type	
<i>Applicability</i>	All exterior floor surfaces, optional
<i>Definition</i>	The category that defines the standard design prescriptive floor requirements
<i>Units</i>	List: mass or other
<i>Input Restrictions</i>	
<i>Standard Design</i>	The standard design building floors shall be of type “other”
<i>Standard Design: Existing Buildings</i>	Same as proposed

Floor Geometry	
<i>Applicability</i>	All exterior floors
<i>Definition</i>	Floor geometry defines the position, orientation, azimuth, and tilt of the floor surface. The details of how the coordinate system is implemented may vary between software programs. The data structure for surfaces is described in the reference section of this chapter.
<i>Units</i>	Data structure: surface
<i>Input Restrictions</i>	As designed. Required input.
<i>Standard Design</i>	Standard design building floor geometry is identical to the proposed design.
<i>Standard Design: Existing Buildings</i>	Same as proposed.

Floor Construction	
<i>Applicability</i>	All floors
<i>Definition</i>	A specification containing a series of layers that result in a construction assembly for the proposed design. The first layer in the series represents the outside (or exterior) layer and the last layer represents the inside (or interior) layer. See the building descriptors above for floor construction type.
<i>Units</i>	List: Layers
<i>Input Restrictions</i>	The area weighted-average of the floor construction assembly U-factors, defined by a series of layers, must be equal to or more efficient than the mandatory U-factor requirements of Section 120.7 of the standards for new construction, and Section 141.0 of the standards for alterations. Note that these U-factor requirements assume an exterior air film of R-

	<p>0.17 and an interior air film of R-0.92. Each layer specified must be listed in the materials database in the ACM Appendix 5.5A.</p> <p>New Construction</p> <table border="1" data-bbox="467 306 1404 464"> <tr> <td>Raised Mass Floors</td> <td>U – 0.269</td> </tr> <tr> <td>Other Floors</td> <td>U – 0.071</td> </tr> <tr> <td>Heated Slab Floors</td> <td>Climate Zone (see Section 120.7)</td> </tr> </table> <p>Additions and Alterations</p> <table border="1" data-bbox="467 512 1404 747"> <tr> <td>Metal Building</td> <td>U – 0.113</td> </tr> <tr> <td>Metal Framed</td> <td>U – 0.217</td> </tr> <tr> <td>Wood Framed and Others</td> <td>U – 0.110</td> </tr> <tr> <td>Spandrel Panels / Glass Curtain Walls</td> <td>U – 0.280</td> </tr> </table> <p>Appropriate R-values for insulation can be calculated using the formula below.</p> $R_{insulation} = (1/UFactor) - R_{Layer(1)} - R_{Layer(2)} - R_{Layer(3)} - R_{Layer(n)}$ $R_{insulation} = R_{inscontinuous} + R_{insframing}$ <p>Floors that form the boundary between the modeled building of an addition and alteration project and the existing, un-modeled portion of the building may be modeled as adiabatic floors (no heat transfer).</p>	Raised Mass Floors	U – 0.269	Other Floors	U – 0.071	Heated Slab Floors	Climate Zone (see Section 120.7)	Metal Building	U – 0.113	Metal Framed	U – 0.217	Wood Framed and Others	U – 0.110	Spandrel Panels / Glass Curtain Walls	U – 0.280
Raised Mass Floors	U – 0.269														
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Metal Building	U – 0.113														
Metal Framed	U – 0.217														
Wood Framed and Others	U – 0.110														
Spandrel Panels / Glass Curtain Walls	U – 0.280														
<p><i>Standard Design</i></p>	<p>The U-factor required for floor construction is defined in Table 140.3-B, C or D of the standards. Programs that model a U-factor shall use an exterior and interior air film resistance. The standard design construction is based on JA4-10 Table 4.4.5 and assumes an exterior air film of R-0.17 and an interior air film of R-0.92.</p> <p>For metal framed floors, the standard design construction shall include the following layers:</p> <table border="1" data-bbox="467 1329 1404 1535"> <tr> <td>Layer 1</td> <td>Open Framing + No Ins.</td> <td>R – 0.00</td> </tr> <tr> <td>Layer 2</td> <td>Continuous Insulation</td> <td>R – Based on Climate Zone</td> </tr> <tr> <td>Layer 3</td> <td>Plywood – 5/8 in.</td> <td>R – 0.78</td> </tr> <tr> <td>Layer 4</td> <td>Carpet and Pad – 3/4 in.</td> <td>R – 1.30</td> </tr> </table>	Layer 1	Open Framing + No Ins.	R – 0.00	Layer 2	Continuous Insulation	R – Based on Climate Zone	Layer 3	Plywood – 5/8 in.	R – 0.78	Layer 4	Carpet and Pad – 3/4 in.	R – 1.30		
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Layer 4	Carpet and Pad – 3/4 in.	R – 1.30													
<p><i>Standard Design: Existing Buildings</i></p>	<p>The value of the continuous insulation layer entirely above or below framing shall be set to achieve the following R-values:</p> <p>Nonresidential Buildings: Continuous Insulation</p> <table border="1" data-bbox="467 1713 1404 1854"> <tr> <td>Climate Zones 1</td> <td>R – 17.66</td> </tr> <tr> <td>Climate Zones 2, 11, and 14 -16</td> <td>R – 22.47</td> </tr> <tr> <td>Climate Zones 3 – 10, 12, and 13</td> <td>R – 10.91</td> </tr> </table>	Climate Zones 1	R – 17.66	Climate Zones 2, 11, and 14 -16	R – 22.47	Climate Zones 3 – 10, 12, and 13	R – 10.91								
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Climate Zones 3 – 10, 12, and 13	R – 10.91														

	High-Rise Residential Buildings and Hotel/Motel Guestrooms: Continuous Insulation	
	Climate Zones 1, 2, 14, and 16	R – 26.24
	Climate Zones 3 – 6, 8 – 13, and 15	R – 22.47
	Climate Zones 7	R – 10.91

5.5.6 Doors

Door Name	
<i>Applicability</i>	All exterior doors, optional input
<i>Definition</i>	A unique name or code that relates the door to the design documents submitted. Doors that are more than 50 percent glass are treated as windows and must be determined and entered by using the Fenestration building descriptors.
<i>Units</i>	Text: unique
<i>Input Restrictions</i>	None
<i>Standard Design</i>	None
<i>Standard Design: Existing Buildings</i>	None

Door Type	
<i>Applicability</i>	All exterior doors, required input
<i>Definition</i>	One of two door classifications of either: swinging or non-swinging. Non-swinging are generally roll-up doors. The prescriptive U-factor requirements depend on door type and climate. This building descriptor may be derived from other building descriptors, in which case a specific input is not necessary.
<i>Units</i>	List: swinging or non-swinging
<i>Input Restrictions</i>	The door type shall be consistent with the type of door represented on the construction documents or as-built drawings.
<i>Standard Design</i>	The standard design building door type shall be the same as the proposed design.
<i>Standard Design: Existing Buildings</i>	Same as new construction

Door Geometry

<i>Applicability</i>	All exterior doors
<i>Definition</i>	Door geometry defines the position and dimensions of the door surface relative to its parent wall surface. The azimuth and tilt (if any) of the door is inherited from the parent surface. The position of the door within the parent surface is specified through (X,Y) coordinates. The size is specified as a height and width (all doors are generally assumed to be rectangular in shape). The details of how the geometry of doors is specified may vary for each energy simulation program.
<i>Units</i>	Data structure: opening
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Door geometry in the standard design building is identical to the proposed design.
<i>Standard Design: Existing Buildings</i>	Same as new construction

Door Construction																			
<i>Applicability</i>	All exterior doors																		
<i>Definition</i>	The thermal transmittance of the door, including the frame.																		
<i>Units</i>	Btu/h·ft ² ·°F																		
<i>Input Restrictions</i>	The construction assembly must be equal to or more efficient than the mandatory U-factor requirements of Section 110.6 of the standards for new construction. There are no restrictions for alterations.																		
<i>Standard Design</i>	<p>For new construction, the U-factor required for door construction is defined in Table 140.3-B, C or D of the standards.</p> <p>Nonresidential Buildings – U Factor:</p> <table border="1"> <tr> <td colspan="2"><i>Non-Swinging Doors:</i></td> </tr> <tr> <td>Climate Zones 1, and 16</td> <td>U – 0.50</td> </tr> <tr> <td>Climate Zones 2 – 15</td> <td>U – 1.45</td> </tr> <tr> <td colspan="2"><i>Swinging Doors:</i></td> </tr> <tr> <td>Climate Zones 1 – 16</td> <td>U – 0.70</td> </tr> </table> <p>High-Rise Residential Buildings and Hotel/Motel Guestrooms – U Factor:</p> <table border="1"> <tr> <td colspan="2"><i>Non-Swinging Doors:</i></td> </tr> <tr> <td>Climate Zones 1, and 16</td> <td>U – 0.50</td> </tr> <tr> <td>Climate Zones 2 – 15</td> <td>U – 1.45</td> </tr> <tr> <td colspan="2"><i>Swinging Doors:</i></td> </tr> </table>	<i>Non-Swinging Doors:</i>		Climate Zones 1, and 16	U – 0.50	Climate Zones 2 – 15	U – 1.45	<i>Swinging Doors:</i>		Climate Zones 1 – 16	U – 0.70	<i>Non-Swinging Doors:</i>		Climate Zones 1, and 16	U – 0.50	Climate Zones 2 – 15	U – 1.45	<i>Swinging Doors:</i>	
<i>Non-Swinging Doors:</i>																			
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<i>Swinging Doors:</i>																			

	Climate Zones 1 – 16	U – 0.70
<i>Standard Design: Existing Buildings</i>	For alterations, the U-factor in the standard design is either the same standard design as the new construction standard design if the door is replaced, or the equal to the existing door construction, if the door is unaltered. Where applicable, selection shall be based on building type, assembly, and climate zone.	

Operable Door Opening Type	
<i>Applicability</i>	All exterior doors
<i>Definition</i>	The opening type that determines whether interlocks with mechanical cooling and heating are required, per Section 140.4(n). If manual, then interlocks are required when operable windows are present in <u>the any nonresidential space, excluding highrise residential and healthcare spaces and facilities buildings</u> . If self-closing or a glazed door, interlocks are not required and are not present in the standard design.
<i>Units</i>	Btu/h·ft ² ·°F
<i>Input Restrictions</i>	List: Self-Closing, Manual, Glazed Door
<i>Standard Design</i>	Same as Proposed
<i>Standard Design: Existing Buildings</i>	

5.5.7 Fenestration

Note that fenestration includes windows, doors that have more than 2550 percent glazed area, and skylights. A skylight is fenestration that has a tilt of less than 60 degrees from horizontal.

Fenestration Name	
<i>Applicability</i>	All fenestration, optional input
<i>Definition</i>	A unique name or code that relates the fenestration to the design documents and a parent surface.
<i>Units</i>	Text: unique
<i>Input Restrictions</i>	None
<i>Standard Design</i>	None
<i>Standard Design: Existing Buildings</i>	None

Fenestration Type (Vertical Fenestration)	
<i>Applicability</i>	All vertical fenestration
<i>Definition</i>	This is a classification of vertical fenestration that determines the thermal performance and solar performance requirement for vertical fenestration
<i>Units</i>	List: Fixed, Operable, Curtain Wall, or Glazed Doors
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	Same as new construction

Fenestration Type (Skylights)	
<i>Applicability</i>	All skylights
<i>Definition</i>	This is a classification of skylights that determines the thermal performance and solar performance requirement for vertical fenestration
<i>Units</i>	List: Glass, Curb-mounted, Glass, Deck-mounted, or Plastic, <u>or Tubular Daylighting Devices (TDD)</u>
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	Same as new construction

Default Fenestration Type	
<i>Applicability</i>	All fenestration that uses default thermal performance factors
<i>Definition</i>	This is a classification of fenestration that determines the thermal performance for fenestration using defaults from standards section 110.6, Table 110.6-A. This is used for fenestration without National Fenestration Rating Council (NFRC) ratings or for fenestration for altered buildings that includes window films.
<i>Units</i>	List: fixed, operable, greenhouse/garden, doors, or skylight
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

Default Glazing Type	
<i>Applicability</i>	All fenestration that uses default thermal performance factors
<i>Definition</i>	This is a classification of fenestration that determines the thermal performance for fenestration using defaults from standards section 110.6, Table 110.6-A. This is used for fenestration without NFRC ratings or for fenestration for altered buildings that includes window films.
<i>Units</i>	List: single pane, double pane, glass block
<i>Input Restrictions</i>	As designed Glass block is only allowed if the default fenestration type is operable or fixed.
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

Default Framing Type	
<i>Applicability</i>	All fenestration that uses default thermal performance factors and window films for altered fenestration
<i>Definition</i>	This is a classification of fenestration that determines the thermal performance for fenestration using defaults from standards section 110.6, Table 110.6-A. This is used for fenestration without NFRC ratings or for fenestration for altered buildings that includes window films.
<i>Units</i>	List: metal, metal with thermal break, or nonmetal
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

Fenestration Geometry	
<i>Applicability</i>	All fenestration
<i>Definition</i>	Fenestration geometry defines the position and dimensions of the fenestration surface within its parent surface and the identification of the parent surface. The orientation and tilt is inherited from the parent surface. The details of how the coordinate system is implemented may vary between compliance software programs. Display Perimeter: Display perimeter is the length of an exterior wall in a B-2 occupancy that

	<p>immediately abuts a public sidewalk, measured at the sidewalk level for each story that abuts a public sidewalk. The compliance software shall allow the user to specify a value for the length of display perimeter, in feet, for each floor or story of the building. The user entry for display perimeter shall have a default value of zero. Note: Any non-zero input for display perimeter is an exceptional condition that shall be reported on the PRF-1 exceptional condition list and shall be reported on the ENV forms. The value for display perimeter is used as an alternate means of establishing maximum wall fenestration area in the standard design (§140.3 of the standards).</p> <p>The display perimeter shall be calculated separately for west-facing fenestration, and for non-west facing fenestration.</p> <p>Floor Number:</p> <p>The compliance software shall also allow the user to specify the display perimeter associated with each floor (story) of the building.</p>
<i>Units</i>	<p>Data structure: opening</p> <p>Geometry is defined relative to the parent surface and can include setbacks.</p> <p>Inputs include:</p> <p>Geometry of opening (window or skylight), parent surface, display perimeter (optional), percent of roof area exempt from skylight requirements §140.3 of the standards.</p>
<i>Input Restrictions</i>	<p>As designed</p> <p>Specification of the fenestration position within its parent surface is required for the following conditions:</p> <ol style="list-style-type: none"> 1) Exterior shading is modeled from buildings, vegetation, or other objects; or 2) If daylighting is modeled within the adjacent space.
<i>Standard Design</i>	<p>The standard design calculates the window wall ratio (WWR) for each orientation and the overall window wall ratio for the building. The window wall ratio is the total fenestration area (including framing) divided by the gross exterior wall area (excluding wall area that is underground). Note that exterior wall area that is below grade, but has exposure to ambient conditions, and any associated fenestration, is included in the WWR calculation.</p> <p>The standard design vertical fenestration area and horizontal fenestration area for spaces that are specified as computer rooms or data centers (a server process load of 20W/ft² or higher) shall be zero.</p> <p>For all other buildings, the geometry of the fenestration in the standard design shall be identical to the proposed design with the following exceptions:</p> <p>Exception 1: Either the whole building window wall ratio or west window wall ratio exceeds 40 percent.</p>

	<p>Exception 2: If display perimeter is entered, the fenestration area exceeds the greater of 40 percent of the gross wall area (excluding adiabatic walls) and six times the display perimeter.</p> <p>Exception 1: The fenestration is adjusted based on the following conditions:</p> <p>Case 1. $WWR_o > 0.40$, $WWR_w \leq 0.40$</p> <p>In this case, the fenestration area of all windows is reduced by multiplying the fenestration area by the ratio $0.40/WWR_o$. The dimensions of each window are reduced by increasing the sill height so that the window height is modified by the multiplier $(0.40/WWR_o)$ so that the same window width is maintained.</p> <p>Case 2: $WWR_o < 0.40$. $WWR_w > 0.40$</p> <p>In this case, the fenestration area of all windows on the west orientation is reduced by multiplying the fenestration area by the ratio $0.40/WWR_o$. The dimensions of each window are reduced by multiplying the proposed window dimension by increasing the sill height so that the window height is modified by the multiplier $(0.40/WWR_o)$, so that the window width is maintained.</p> <p>Case 3: $WWR_o > 0.40$. $WWR_w > 0.40$</p> <p>If both the west window wall ratio and the overall window wall ratio exceed the prescriptive limit of 0.40, the fenestration areas must be reduced by:</p> <ol style="list-style-type: none"> 1) Adjust the west window area multiplying the west window area by the ratio $0.4/WWR_w$. 2) Calculate the WWR of the north, east and south facades: $WWR_{nes} = \text{Window Area}_{nes} / \text{Gross Wall Area}_{nes}$ 3) Adjust the window area of the windows on the north, east and south facades by the following ratio: $\text{WindowArea}_{N,std} = \text{WindowArea}_{N,prop} \times 0.4 / WWR_{nes}$ $\text{WindowArea}_{E,std} = \text{WindowArea}_{E,prop} \times 0.4 / WWR_{nes}$ $\text{WindowArea}_{S,std} = \text{WindowArea}_{S,prop} \times 0.4 / WWR_{nes}$ 4) Adjust each window geometry for the west façade by multiplying the window height by $(0.4/WWR_w)$ by adjusting the sill height and by maintaining the same window width. 5) Adjust each window geometry for the north, east and south façade by multiplying the window height by $(0.4/WWR_{nes})$ by adjusting the sill height and by maintaining the same window width. <p>Exception 2: If the display perimeter is entered and the window area exceeds the prescriptive limit, the window area for the standard design is calculated by multiplying the proposed window area by the following ratio: $\text{WindowArea}_{std} = 6 \times \text{DisplayPerimeter}$</p>
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	<p>The geometry of each window is modified by the following, and by modifying the sill height but not the head height position relative to the floor:</p> $\text{WindowHeight}_{\text{std}} = \text{WindowHeight}_{\text{prop}} \times (\text{WindowArea}_{\text{std}} / \text{WindowArea}_{\text{prop}})$ $\text{WindowWidth}_{\text{std}} = \text{WindowWidth}_{\text{prop}}$ <p>The following rules apply for calculating geometry of skylights. For the calculation of the standard design skylight area, the gross roof area is defined as the total roof area, including skylights, that is directly over conditioned space.</p> <p>The skylight area of the standard design is set:</p> <ol style="list-style-type: none"> 1) For buildings without atria or with atria having a height less than 55 feet over conditioned space, the smaller of the proposed skylight area and 5 percent of the gross roof area that is over conditioned space. 2) For buildings with atria at a height of 55 ft or greater over conditioned spaces, the smaller of the proposed skylight area and 10 percent of the gross roof area that is over conditioned space. 3) For buildings with atria or other roof area directly over unconditioned spaces, the smaller of the proposed skylight area or 5 percent of the roof area excluding the atria area and excluding any adiabatic walls, if present in the modeled building. The skylight area of the atria or roof area directly over unconditioned space is not included in the skylight area limit in this case. <p>The skylight area for atria over unconditioned space is not included in determining the skylight to roof ratio (SRR) for the building.</p> <p>Depending on the following condition, adjustments to the SRR as described shall be made.</p> <ol style="list-style-type: none"> i. For open spaces other than auditoriums, churches, movie theaters, museums and refrigerated warehouses, for buildings in climate zones 2 through 15, and when spaces have ceiling heights greater than 15 ft and floor areas greater than 5000 ft², the skylight area shall be the greater of 3 percent or the area required to provide daylight coverage through skylights or primary sidelighting to 75 percent of the floor area in the space. See 5.4.5 for detail description on primary daylit area and skylit daylit area. ii. If the above condition is met and $\text{SRR} \leq 0.05$, no adjustments are needed. iii. If the condition is met and $\text{SRR} > 0.05$, skylight dimensions = Existing Dimension $\times [1 - \sqrt{(0.05/\text{SRR of Proposed Building})}]$ iv. If the condition is not met triggering the need for additional skylights, the <u>standard design</u> baseline case shall be modeled with new skylights irrespective of the skylight location of the proposed case. The new skylights shall be distributed uniformly such that there is no overlapping of primary daylit areas from skylights or sidelights. The dimension of the new skylights shall be the same as the proposed design
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	<p>if calculated new SRR ≤ 0.05. If SRR > 0.05, skylight dimensions = existing dimension x $[1 - \sqrt{(0.05/SRR \text{ of proposed building})}]$.</p> <p>Note that the adjustments to SRR are done after adjustments to WWR, if any are completed.</p>
<p><i>Standard Design: Existing Buildings</i></p>	<p>For alterations of existing vertical fenestration or skylights, where no fenestration area is added, the fenestration geometry of the standard design shall be the same as the proposed for the existing building.</p> <p>For additions of vertical fenestration or skylights, where the additional fenestration causes the fenestration area to exceed the limit of 40 percent window to wall ratio (WWR) for the building, 40 percent WWR for the west orientation of the building, 5 percent skylight to roof ratio (SRR) for existing buildings without atria 55 feet or higher, or 10 percent SRR for existing buildings with atria 55 feet or higher, the fenestration geometry for the standard design shall be adjusted from the proposed design according to the rules set forth under the standard design rules.</p> <p>For additions of vertical fenestration and/or skylights that do not cause the fenestration area to exceed any of these limits, the fenestration geometry of the standard design shall be the same as the proposed design.</p>

Skylight Requirement Exception Fraction	
<i>Applicability</i>	All buildings with interior ceiling heights greater than 15 feet
<i>Definition</i>	<p>The fraction of floor area that is exempt from the minimum skylight area requirement for spaces with high ceilings.</p> <p>Identifying areas subject to §140.3 of the standards:</p> <p>When a proposed space has ceiling heights greater than 15 ft, with exterior surfaces having a tilt angle less than 60 degrees (roofs) and no more than three stories above grade, the user shall enter the fraction of the modeled space that is exempt from requirements of §140.3 of the standards. If the proposed design has skylights, the user shall also indicate the area of the proposed design daylight area under skylights in this space. When the user enters a value greater than zero percent for the fraction of the space area exempt to §140.3 of the standards, the compliance software shall require that the user indicate at least one of the following exceptions:</p> <ol style="list-style-type: none"> 1. The building is not located in climate zone 1 or climate zone 16 2. Designed general lighting is less than 0.5 W/ft² 3. Existing walls on plans result in enclosed spaces less than 5,000 ft² 4. Future walls or ceilings on plans result in enclosed spaces less than 5,000 ft² or ceiling heights less than 15 ft 5. Plans or documents show that space is an auditorium, religious building of worship, movie theater, museum, or refrigerated warehouses

<i>Units</i>	List four exceptions listed above (specified if fraction > 0)
<i>Input Restrictions</i>	No restrictions, other than that the vertical fenestration type must agree with the type specified on the construction documents or the as-built drawings
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	Not applicable

Fenestration Construction	
<i>Applicability</i>	All fenestration
<i>Definition</i>	<p>A collection of values that together describe the performance of a fenestration system.</p> <p>The values that are used to specify the criteria are U-factor, SHGC, and VT. U-factor and SHGC inputs are whole-window values.</p>
<i>Units</i>	<p>Data structure: shall include at a minimum the following properties as specified by NFRC ratings:</p> <p>U-factor: whole window U-factor</p> <p>SHGC: whole window solar heat gain coefficient</p> <p>VT: visible transmittance</p>
<i>Input Restrictions</i>	<p>For new construction, performance information for fenestration shall be obtained from NFRC test results or shall be developed from procedures outlined in section 110.6 of the standards, as specified below. Values entered shall be consistent with the specifications and the construction documents.</p> <p>For manufactured products:</p> <ul style="list-style-type: none"> • U-factor, SHGC, and VT shall be equivalent to NFRC rated values. • For products not rated by NFRC, U-factor, SHGC and VT shall be determined from Energy Commission default tables (110.6 A and B). <p>For site built products:</p> <ul style="list-style-type: none"> • U-factor, SHGC, and VT shall be equivalent to NFRC rated values. • For products not rated by NFRC, <u>up to 200 ft² of fenestration (total of new and altered) may use center of glass properties and Reference Appendix NA6 equations to calculate U-factor, SHGC, and VT.</u> up to 1,000 square feet of new, altered or additional fenestration can be modeled using center of glass properties and equations in Reference Appendix NA6. Any site-built fenestration in excess of <u>2004,000 ft² square feet</u> square feet must use the default values in Table 110.6-A and 110.6-B. <p>For buildings with fenestration area that meets requirements for use of</p>

	<p>center-of-glass U-factor and SHGC, the fenestration overall U-factor, SHGC, and VT shall be determined by the following equations from the Reference Appendix NA6:</p> $U_T = C_1 + (C_2 \cdot U_c)$ $SHGC_T = 0.08 + (0.86 \cdot SHGC_c)$ $VT_T = VT_F \cdot VT_C$ <p>Where,</p> <p>U_T = U-factor is the total performance of the fenestration including glass and frame</p> <p>C_1 = Coefficient selected from Table NA6-5 in Reference Appendix NA6</p> <p>C_2 = Coefficient selected from Table NA6-5 in Reference Appendix NA6</p> <p>U_c = Center of glass U-factor calculated in accordance with NFRC 100 Section 4.5.3.1</p> <p>$SHGC_T$ = Total SHGC performance including glass and frame $SHGC_c$ = Center of glass SHGC calculated in accordance with NFRC 200 Section 4.5.1.1</p> <p>VT_T = Is the total performance of the fenestration including glass and frame</p> <p>$VT_F = 0.53$ for projecting windows, such as casement and awning windows</p> <p>$VT_F = 0.67$ for operable or sliding windows</p> <p>$VT_F = 0.77$ for fixed or non-operable windows</p> <p>$VT_F = 0.88$ for curtain wall/storefront, site-built and manufactured non-curb mounted skylights</p> <p>$VT_F = 1.0$ for curb mounted manufactured skylights</p> <p>VT_C = Center of glass VT is calculated in accordance with NFRC 200 Section 4.5.1.1 or NFRC 202 for Translucent Products or NFRC 203 for Tubular Daylighting Devices and Hybrid Tubular Daylighting Devices or ASTM E972</p> <p>For skylights, the default values shall be the alternate default U-factor and SHGC using default calculations specified above and in Reference Appendix NA6 or the U-factor and SHGC listed in Table 110.6-A and Table 110.6-B in the standards.</p> <p>For alterations of existing fenestration using window films, the thermal performance for solar heat gain coefficient shall be calculated from two user inputs:</p> <ol style="list-style-type: none"> 1) Default Glazing Reference: unique identifier for the glazing without film, that determines the Energy Commission default value (column 3 in table below) and the SHGC ratio multiplier (column 4), and 2) NFRC window film SHGC: the solar heat gain coefficient multiplier
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	<p>for the window film, from NFRC ratings.</p> <p>The adjusted SHGC for the glazing with the window film is:</p> $SHGC_{adj} = DefaultSHGCValue \times SHGCRatio \times NFRCfilm$ <p>Where:</p> <p>DefaultSHGCValue = the Energy Commission default value from 110.6, given in column 3 in the table below</p> <p>SHGCRatio = multiplier based on the default glazing reference, given in column 4 in the table below</p> <p>NFRCfilm = the NFRC SHGC rating of the window film</p>
<i>Standard Design</i>	<p>For new construction, the requirements for vertical fenestration U factor, SHGC, and visible light transmission by window <u>or skylight</u> type and framing type are specified in Table 140.3-B, C, or D of the standards. For plastic skylights, SHGC of 0.50 is assumed.</p>
<i>Standard Design: Existing Buildings</i>	<p>The U-factor, SHGC, and VT in the standard design shall be modeled as design if unchanged, as the values stated in Table 141.0-A of the standards when the existing window area is unchanged (different than the new construction performance requirement), or Table 140.3-B, C, or D of the standards for all other cases.</p> <p>The standard design does not include window films.</p>

1	2	3		4		5		6
Operator Type	Default Glazing Reference	Energy Commission Default SHGC Value Table 110.6-B X		SHGC Ratio X _y		NFRC Window Film SHGC ^{1,2,3} Z		New Adjusted Total Value T
Residential or Commercial								
Fixed	3 mm (1/8in.) clear (Single Pane)	0.83	x	1.1528	x		=	
Fixed	3 mm (1/8in.) clear 3 mm (1/8in.) clear (Double Pane - Clear)	0.73	x	1.1406	X		=	

1	2	3		4		5		6
Operator Type	Default Glazing Reference	Energy Commission Default SHGC Value		SHGC Ratio X _y		NFRC Window Film SHGC ^{1,2,3} Z		New Adjusted Total Value T

		Table 110.6-B X					
Residential or Commercial							
Fixed	3 mm (1/8in.) clear (Single Pane)	0.83	x	1.1528	X	=	
Fixed	3 mm (1/8in.) clear 3 mm (1/8in.) clear (Double Pane - Clear)	0.73	x	1.1406	X	=	

External Shading Devices	
<i>Applicability</i>	All fenestration
<i>Definition</i>	Devices or building features that are documented in the construction documents and shade the glazing, such as overhangs, fins, shading screens, and setbacks of windows from the exterior face of the wall. The Title 24 compliance software shall be capable of modeling vertical fins and overhangs. Recessed windows may also be modeled with side fins and overhangs.
<i>Units</i>	Data structure: opening shade
<i>Input Restrictions</i>	No restrictions other than that the inputs must match the construction documents
<i>Standard Design</i>	The standard design building is modeled without external shading devices.
<i>Standard Design: Existing Buildings</i>	<u>No shading devices</u>

Internal Shading Devices	
<i>Applicability</i>	All fenestration
<i>Definition</i>	Curtains, blinds, louvers, or other devices that are applied on the room side of the glazing material Glazing systems that use blinds between the glazing layers are also considered internal shading devices. Glass coatings, components, or treatments of the glazing materials are addressed through the fenestration construction building descriptor.
<i>Units</i>	Data structure: indicates the type of control, or blind schedule if applicable
<i>Input Restrictions</i>	Internal shading shall not be modeled in the proposed design, unless it is automatically controlled, based on input from an astronomical time clock, an exterior pyronometer, or other sensors. The control algorithm shall be documented on the construction documents. Interior shades without

	automatic controls shall not be modeled Interior shades shall only be modeled when automatic controls are present.
<i>Standard Design</i>	The <u>standard design</u> baseline building shall be modeled without interior shades. None (not applicable)
<i>Standard Design:</i> <i>Existing Buildings</i>	<u>No interior shades</u>

Dynamic Glazing Present	
<i>Applicability</i>	All fenestration that has dynamic glazing
<i>Definition</i>	This is a flag used for reporting purposes only. Dynamic glazing is not modeled directly in compliance software.
<i>Units</i>	Boolean
<i>Input Restrictions</i>	None
<i>Standard Design</i>	False (not present)
<i>Standard Design:</i> <i>Existing Buildings</i>	<u>Not Applicable</u>

5.5.8 Below-Grade Walls

Below-Grade Wall Name	
<i>Applicability</i>	All projects, optional input
<i>Definition</i>	A unique name that keys the below-grade wall to the construction documents
<i>Units</i>	Text: unique
<i>Input Restrictions</i>	None
<i>Standard Design</i>	Not applicable
<i>Standard Design:</i> <i>Existing Buildings</i>	<u>Same as proposed</u>

Below-Grade Wall Geometry	
<i>Applicability</i>	All projects
<i>Definition</i>	A geometric construct that describes the dimensions and placement of walls located below grade. Below-grade walls have soil or crushed rock

	on one side and interior space on the other side. Some simulation models take the depth below grade into account when estimating heat transfer so the geometry may include height and width.
<i>Units</i>	Data structure: below-grade wall geometry
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	The geometry of below-grade walls in the standard design building is identical to the below-grade walls in the proposed design.
<i>Standard Design: Existing Buildings</i>	<u>Same as proposed</u>

Below-Grade Wall Construction	
<i>Applicability</i>	All projects, required input
<i>Definition</i>	A specification containing a series of layers that result in a construction assembly for the proposed design. The first layer in the series represents the outside (or exterior) layer and the last layer represents the inside (or interior) layer. See the building descriptors above for below-grade wall construction type.
<i>Units</i>	Data structure: construction assembly The construction can be described as a C-factor which is similar to a U-factor, except that the outside air film is excluded, or the construction can be represented as a series of layers, like exterior constructions.
<i>Input Restrictions</i>	The construction assembly, defined by a series of layers, must be equal to or more efficient than the mandatory R-value and C-factor requirements of Section 120.7 of the standards for new construction, and Section 141.0 of the standards for alterations. Note that these requirements only apply when the slab floor connected to the below-grade wall is heated. For new construction, the inputs shall be in agreement with the construction documents. Values for the C-factor shall be taken from Table 4.3.5, 4.3.6, or 4.3.7 of Reference Appendix JA4. For alterations there are no restrictions.
<i>Standard Design</i>	For new construction, see Table 13. The standard design building shall use default values for C-factor. The height shall be the same as specified in the proposed design. For below-grade walls, the standard design construction shall include the layers described in Table 9 Table 12 Table 13 <u>Table 12</u> . For alterations, the C-factor in the standard design shall be modeled as the more efficient of either the existing conditions, or the values stated above for new construction standard design. For below-grade walls, the alteration standard design assembly shall include the appropriate existing layers.

<i>Standard Design:</i>	<u>Same as proposed</u>
<i>Existing Buildings</i>	

Table 912: Standard Design Building Below-Grade Wall Construction Assemblies

Construction	Layer	Thickness (inch)	Conductivity (Btu/h ft ² F)	Density (lb/ft ³)	Specific Heat (Btu/lb°F)	R-value (ft ² ·°F/h/Btu)	C-factor (Btu/ft ² ·°F·h)
NR	115 lb/ft ³ CMU, solid grout	8	0.45	115	0.20	0.87	1.140
R-7.5 c.i.	115 lb/ft ³ CMU, solid grout	8	0.45	115	0.20	0.87	
	R-10 continuous insulation	1.8	0.02	1.8	0.29	7.50	
	Total assembly					8.37	0.119
R-10 c.i.	115 lb/ft ³ CMU, solid grout	8	0.45	115	0.20	0.87	
	R-10 continuous insulation	2.4	0.02	1.8	0.29	10.00	
	Total assembly					10.87	0.092
R-12.5 c.i.	115 lb/ft ³ CMU, solid grout	8	0.45	115	0.20	0.87	
	R-10 continuous insulation	3.0	0.02	1.8	0.29	12.50	
	Total assembly					13.37	0.075

5.5.9 Slab Floors in Contact with Ground

These building descriptors apply to slab-on-grade or below-grade floors that are in direct contact with the ground.

Slab Floor Name	
<i>Applicability</i>	All slab floors, optional
<i>Definition</i>	A unique name or code that relates the exposed floor to the construction documents
<i>Units</i>	Text: unique
<i>Input Restrictions</i>	None
<i>Standard Design</i>	Not applicable
<i>Standard Design:</i> <i>Existing Buildings</i>	<u>Not applicable</u>

Slab Floor Type	
<i>Applicability</i>	All slab floors, required
<i>Definition</i>	One of two types and two subtypes of floors in contact with ground: 1) Heated slab-on-grade floors, 2) Unheated slab-on-grade floors

	<p>3) Heated below-grade floors</p> <p>4) Unheated below-grade floors.</p> <p>Heated slab-on-grade floors include all floors that are heated directly in order to provide heating to the space. Unheated slab-on-grade floors are all other floors in contact with ground.</p>
<i>Units</i>	List: restricted to the four selections listed above
<i>Input Restrictions</i>	None
<i>Standard Design</i>	The slab for type is unheated (either unheated slab-on-grade for slab-on-grade floors or unheated below-grade for below-grade floors).
<i>Standard Design: Existing Buildings</i>	<u>Same as proposed</u>

Slab Floor Geometry	
<i>Applicability</i>	All slab floors, required
<i>Definition</i>	<p>A geometric construct representing a slab floor in contact with the earth.</p> <p>The geometric representation can vary depending on how the energy simulation software models slabs-on-grade. Some models require that only the perimeter of the slab be entered. Other models divide the slab into a perimeter band within 2 ft of the edge and the interior portion or core area, such that the perimeter area and the core area sum to the total area of the slab.</p>
<i>Units</i>	<p>Data structure: as appropriate for the simulation tool</p> <p>This may include: area, perimeter exposed</p>
<i>Input Restrictions</i>	None
<i>Standard Design</i>	The geometry of the slab floor in the standard design building is identical to the slab floor in the proposed design.
<i>Standard Design: Existing Buildings</i>	<u>Same as proposed</u>

Slab Floor Construction	
<i>Applicability</i>	All slab floors, required
<i>Definition</i>	<p>A specification containing a series of layers that result in a construction assembly for the proposed design.</p> <p>The first layer in the series represents the outside (or exterior) layer and the last layer represents the inside (or interior) layer. See the building descriptors above for slab floor construction type.</p>

	<p>A description of how the slab is insulated (or not)</p> <p>How the construction is described will depend on the energy simulation model. The construction can be represented by an F-factor that represents the entire construction (floor and insulation).</p> <p>Simple models may include just an F-factor, representing an instantaneous heat loss/gain to outside air. The F-factor could be related to the configuration of insulation in the proposed design. Other slab loss models may require that the surface area of the slab floor be divided between the perimeter and the interior. The insulation conditions then define heat transfer between both outside air and ground temperature.</p> <p>The insulation condition for slabs includes the R-value of the insulation and the distance it extends into the earth at the slab edge and how far it extends underneath the slab.</p>																									
<p><i>Units</i></p>	<p>F-factor from Reference Appendix JA4; this is one selection from list 1 and one selection from list 2. Note that some combinations from list 1 and list 2 are not allowed, see Reference Appendix JA4 Table 4.4.8 and Table 4.4.7 for details.</p> <p>List 1:</p> <table data-bbox="479 909 1047 1140"> <tr> <td>None</td> <td>12 in vertical</td> </tr> <tr> <td>12 in horizontal</td> <td>24 in vertical</td> </tr> <tr> <td>24 in horizontal</td> <td>36 in vertical</td> </tr> <tr> <td>36 in horizontal</td> <td>48 in vertical</td> </tr> <tr> <td>48 in horizontal</td> <td>Fully insulated slab</td> </tr> </table> <p>List 2:</p> <table data-bbox="479 1203 1177 1434"> <tr> <td>R-0</td> <td>R-20</td> <td>R-45</td> </tr> <tr> <td>R-5</td> <td>R-25</td> <td>R-50</td> </tr> <tr> <td>R-7.5</td> <td>R-30</td> <td>R-55</td> </tr> <tr> <td>R-10</td> <td>R-35</td> <td></td> </tr> <tr> <td>R-15</td> <td>R-40</td> <td></td> </tr> </table> <p>The software shall also provide the following slab insulation options:</p> <ol style="list-style-type: none"> 1) Horizontal+Vertical, R-5 vertical down to the horizontal insulation and R-5 horizontal insulation extending 4 feet inwards from the perimeter 2) Horizontal+Vertical, R-10 vertical down to the horizontal insulation and R-7 horizontal insulation extending 4 feet inwards from the perimeter <p>These two combinations of slab insulation are mapped to an F-factor in Appendix 5.4B.</p>	None	12 in vertical	12 in horizontal	24 in vertical	24 in horizontal	36 in vertical	36 in horizontal	48 in vertical	48 in horizontal	Fully insulated slab	R-0	R-20	R-45	R-5	R-25	R-50	R-7.5	R-30	R-55	R-10	R-35		R-15	R-40	
None	12 in vertical																									
12 in horizontal	24 in vertical																									
24 in horizontal	36 in vertical																									
36 in horizontal	48 in vertical																									
48 in horizontal	Fully insulated slab																									
R-0	R-20	R-45																								
R-5	R-25	R-50																								
R-7.5	R-30	R-55																								
R-10	R-35																									
R-15	R-40																									
<p><i>Input Restrictions</i></p>	<p>The construction assembly, defined by an F-factor, must be equal to or more efficient than the mandatory F-factor requirements of Section 120.7 of the standards for new construction, and Section 141.0 of the standards for alterations.</p>																									

	For new construction, F-factors shall be taken from Table 4.4.8 of Reference Appendix JA4 for heated slab floors and Table 4.4.7 for unheated slab floors. For all methods, inputs shall be consistent with the construction documents. For heated slab floors, the F-factor shall be determined by the mandatory R-value and installation requirements in Section 110.8 of the standards. That information is used in Table 4.4.8 of Reference Appendix JA4 to determine the required F-factor. The same requirements apply for alterations.
<i>Standard Design</i>	Slab loss shall be modeled with the simple method (F-factor). The standard design construction shall include the following layer: Layer 1: Concrete 140lb/ft ³ – 6 in. (R - 0.44) The standard design shall include no insulation, equivalent to an F-factor of 0.73. For alterations, the F-factor in the standard design shall be modeled as the more efficient of either the existing conditions, or the values stated above for new construction standard design.
<i>Standard Design: Existing Buildings</i>	<u>Same as proposed</u>

5.5.10 Heat Transfer between Thermal zones

Partition Name	
<i>Applicability</i>	All partitions, optional
<i>Definition</i>	A unique name or code that relates the partition to the construction documents
<i>Units</i>	Text: unique
<i>Input Restrictions</i>	The text should provide a key to the construction documents.
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	Not applicable

Partition Geometry	
<i>Applicability</i>	All partitions
<i>Definition</i>	A geometric construct that defines the position and size of partitions that separate one thermal zone from another. The construct shall identify the thermal zones on each side of the partition. Since solar gains are not generally significant for interior

	partitions, the geometry of partitions is sometimes specified as- an area along with identification of the thermal zones on each side.
<i>Units</i>	Data structure: surface with additional information identifying the two thermal zones that the partition separates
<i>Input Restrictions</i>	No restrictions other than agreement with the construction documents
<i>Standard Design</i>	The geometry of partitions in the standard design building shall be identical to the proposed design.
<i>Standard Design: Existing Buildings</i>	<u>Same as proposed</u>

Partition Construction	
<i>Applicability</i>	All partitions
<i>Definition</i>	A description of the construction assembly for the partition
<i>Units</i>	Data structure: construction assembly
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Partitions in the standard design <u>baseline building</u> shall be steel framed walls with 5/8-inch gypsum board on each side. For walls, partitions in the standard design building shall be steel-framed walls with 5/8-inch gypsum board on each side. For interior floors and ceilings, <u>standard design</u> baseline construction shall be 5/8-inch gypsum board, 4 inches of heavyweight concrete, and 5/8-inch gypsum board.
<i>Standard Design: Existing Buildings</i>	<u>Same as proposed</u>

Demising Partition Construction	
<i>Applicability</i>	All demising walls and demising partitions (ceilings, floors) that separate conditioned spaces from unconditioned spaces
<i>Definition</i>	A description of the construction assembly for the partition
<i>Units</i>	Data structure: construction assembly
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	For walls, <u>when the proposed design demising partition is metal-framed or other, partitions in the standard design shall be a metal-framed wall meeting the mandatory U-factor requirements of section 120.7 (b) of the Standards.</u> s with 5/8 inch gypsum board on each side. For walls, when the proposed design demising partition is wood-framed, the standard design shall <u>will</u> be a wood-framed wall meeting the mandatory U-factor requirements of section 120.7 (b) of the s <u>Standards.</u>

	<p>For windows in demising walls, the fenestration area shall be equal the fenestration area of the proposed design. The window U-factor for fenestration in demising walls shall equal the fixed window prescriptive U-factor requirement of 5.5.7.</p> <p>Demising ceiling partitions, separating conditioned space from unconditioned space and attics, shall be insulated to the same levels as exterior roofs in section 5.5.3. Demising floor partitions shall be insulated to the same levels as exterior floors in section 5.5.5.</p>
<p><i>Standard Design: Existing Buildings</i></p>	<p><u>Demising ceiling partitions, separating conditioned space from unconditioned space and attics, shall be insulated to the same levels as exterior roofs in section 5.5.3. Demising floor partitions shall be insulated to the same levels as exterior floors in section 5.5.5.</u></p>

5.5.11 Simplified Geometry Simulation Option

The compliance software may have an option to model a building with simplified two-dimensional (2D) geometry. This is an optional capability as an alternative to modeling the three-dimensional (3D) geometry of a building. If the compliance software only provides a 2D building model, the following features cannot be modeled:

- Daylighting controls and dimming
- Exterior shading or self-shading

All mandatory and prescriptive daylight controls must be present when submitting a compliance project using software that only models a building with 2D geometry.

The compliance software must pass all reference method tests corresponding to 2D geometry to meet certification requirements as compliance software. Consult Appendix 3B of the *ACM Reference Manual* for additional information. The software must pass the rule set implementation tests, and for the sensitivity tests that verify simulation accuracy, there are 2D tests specified for building envelope, but for other building components such as lighting and HVAC, the software is compared against the results of the reference method, which uses a 3D geometry model.

The compliance software must have sufficient information to specify each exterior surface when modeling a building with 2D geometry. At a minimum, building surface azimuth, elevation, and area are required and the tilt, azimuth and area is specified for roof components. The model must use only vertical walls for the analysis. The model follows all other ACM requirements for space and zone definitions, lighting and HVAC specifications, and follows the same rules for the standard design and proposed design constraints.

The model also requires the following explicit inputs from the user:

- Total Building Story Count – the total number of stories
- Total Above Grade Stories – the total number of stories above grade, used in determination of high-rise residential classification

5.6 HVAC Zone Level Systems

This group of building descriptors relate to HVAC systems at the zone level. There is not a one-to-one relationship between HVAC components in the proposed design and the standard design since the standard design system is determined from building type, size, and heating source. The applicability of each building descriptor for each of the 11 standard design systems is indicated in tables under the building descriptor standard design rules. Additions and alterations should follow the same requirements stated for new construction proposed designs and new construction standard designs; unless otherwise noted in the descriptor.

5.6.1 Space Temperature Control

Space Thermostat Throttling Range	
<i>Applicability</i>	All thermal zones
<i>Definition</i>	The number of degrees that the room temperature must change to cause the HVAC system to go from no heating or cooling (i.e., space temperatures floating) to full heating or cooling
<i>Units</i>	Degrees Fahrenheit (°F)
<i>Input Restrictions</i>	The prescribed value is 2°F. (This is equivalent to a +/- 1°F temperature tolerance around the heating and cooling setpoint.) No input is needed and the prescribed value may not be overridden.
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	

Space Temperature Schedule	
<i>Applicability</i>	All thermal zones
<i>Definition</i>	An hourly space thermostat schedule
<i>Units</i>	Data structure: temperature schedule
<i>Input Restrictions</i>	Prescribed The schedule group is specified for the given space type in Appendix 5.4A, and the schedule values are specified in Appendix 5.4B.
<i>Standard Design</i>	Schedules in the standard design shall be identical to the proposed design, unless operable windows are present in the thermal zone. If operable windows are present, the heating schedule shall be overridden to 50°F and the cooling schedule shall be overridden to 90°F whenever the operable window schedule for that hour indicates that the window(s) are open.

<i>Standard Design: Existing Buildings</i>	
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5.6.2 Terminal Device Data

Terminal Type	
<i>Applicability</i>	All thermal zones
<i>Definition</i>	A terminal unit includes any device serving a zone (or group of zones collected in a thermal zone) that has the ability to re-heat or re-cool in response to the zone thermostat. This includes: <ul style="list-style-type: none"> • None (the case for single zone units) • VAV box • Series fan-powered VAV box • Parallel fan-powered VAV box • Dual-duct mixing box (constant volume and VAV) • Reheat coil (constant volume systems)
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Table 16 specifies the HVAC terminal device for each of the standard design systems. See Section 5.1.2 for a summary of the HVAC mapping. <u>For healthcare facilities, same as the Proposed Design.</u>
<i>Standard Design: Existing Buildings</i>	

Table 1043: Standard Design HVAC Terminal Devices

Standard Design System	Terminal Type
System 1 – <u>PTAGSZAC</u>	None
System 2 – FPFC	None
System 3 – PSZ-AC	None
System 5 – Packaged VAV with Reheat	VAV Box
System 6 – VAV with Reheat	VAV Box
System 7 – SZVAV	None
System 9 – Heating and Ventilation	None
System 10 – CRAH	None
System 11 – CRAC	None
System 12 – LAB	None
System 13 – Kitchen	None

5.6.3 Terminal Heating

This group of building descriptors applies to proposed design systems that have reheat coils at the zone level. The building descriptors are applicable for standard design systems 5 and 6.

Terminal Heat Type	
<i>Applicability</i>	Systems that have reheat coils at the zone level
<i>Definition</i>	The heating source for the terminal unit. This includes: <ul style="list-style-type: none"> • Electric resistance • Gas furnace • Oil furnace • Hot water • Steam
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Table 17 shows the terminal heat type for each standard design system. <u>For healthcare facilities, same as the Proposed Design.</u>
<i>Standard Design: Existing Buildings</i>	

Table 1144: Standard Design Baseline Building Terminal Heat Type

Standard Baseline Building Design System	Terminal Heat Type
System 1 – PTAC	None
System 2 – PTHP	None
System 3 – PSZ-AC	None
System 4 – PSZ-HP	None
System 5 – Packaged VAV with Reheat	Hot Water
System 6 – VAV with Reheat	Hot Water
System 7 – SZVAV	None
System 8 – SZVAV HP	None
System 9 – Heating and Ventilation	None
System 10 – CRAH Unit	None
System 11 – CRAC Unit	None
System 12 – LAB	None
System 13 – Kitchen	None

Terminal Heat Capacity	
<i>Applicability</i>	Systems that have reheat coils at the zone level
<i>Definition</i>	The heating capacity of the terminal heating source
<i>Units</i>	Btu/h
<i>Input Restrictions</i>	As designed

	If the unmet load hours exceed 150, the energy analyst and design team may have to increase the size of the equipment so that the unmet load hours are less than 150.
<i>Standard Design</i>	The software shall automatically size the terminal heating capacity to be 25 percent greater than the design loads.
<i>Standard Design: Existing Buildings</i>	

Reheat Delta T (ΔT_{reheat})											
<i>Applicability</i>	Systems that have reheat coils at the zone level										
<i>Definition</i>	<p>This is an alternate method to enter the terminal heat capacity, which can be calculated as follows:</p> $\Delta T_{reheat} = T_{reheat} - T_{cool_supply}$ $\Delta T_{reheat} = Q_{coil} / 1.1 \times CFM$ <p>Where:</p> <table border="1"> <tr> <td>ΔT_{reheat}</td> <td>Heat rise across the terminal unit heating coil (F)</td> </tr> <tr> <td>T_{reheat}</td> <td>Heating air temperature at design (F)</td> </tr> <tr> <td>T_{cool_supply}</td> <td>Supply air temperature at the heating coil (F)</td> </tr> <tr> <td>Q_{coil}</td> <td>Heating coil load (Btu/h)</td> </tr> <tr> <td>CFM</td> <td>Airflow (ft³/min)</td> </tr> </table>	ΔT_{reheat}	Heat rise across the terminal unit heating coil (F)	T_{reheat}	Heating air temperature at design (F)	T_{cool_supply}	Supply air temperature at the heating coil (F)	Q_{coil}	Heating coil load (Btu/h)	CFM	Airflow (ft ³ /min)
ΔT_{reheat}	Heat rise across the terminal unit heating coil (F)										
T_{reheat}	Heating air temperature at design (F)										
T_{cool_supply}	Supply air temperature at the heating coil (F)										
Q_{coil}	Heating coil load (Btu/h)										
CFM	Airflow (ft ³ /min)										
<i>Units</i>	Degrees Fahrenheit (°F)										
<i>Input Restrictions</i>	As designed but may need to be increased if zone unmet load hours are greater than 150										
<i>Standard Design</i>	Method not used for standard design. The temperature difference shall be no more than 40°F. See heat capacity.										
<i>Standard Design: Existing Buildings</i>											

5.6.4 Baseboard Heat

Note that a furnace zone system can be specified as a variation of baseboard heating.

Baseboard Capacity	
<i>Applicability</i>	All thermal zones

<i>Definition</i>	Total heating capacity of the baseboard unit(s)
<i>Units</i>	Btu/h
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

Baseboard Heat Control	
<i>Applicability</i>	All thermal zones
<i>Definition</i>	Defines the control scheme of base board heating as controlled by a space thermostat
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	Controlled by space thermostat is the only type allowed if baseboard heating is used.
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

Baseboard Heat Source	
<i>Applicability</i>	All thermal zones with furnaces or baseboard heating at the zone
<i>Definition</i>	Heating source
<i>Units</i>	List <ul style="list-style-type: none"> • Electric heat • Gas furnace • Hot water
<i>Input Restrictions</i>	
<i>Standard Design</i>	Not applicable, <u>except for healthcare facilities, same as the Proposed Design.</u>
<i>Standard Design: Existing Buildings</i>	

5.6.5 VRF Zone Systems (Indoor Units)

The following inputs are required when zone systems are connected to a VRF system (condensing unit).

Acceptance Test Required	
<i>Applicability</i>	<u>VRF</u>
<i>Definition</i>	<u>Flag if acceptance test is required</u>
<i>Units</i>	<u>Boolean</u>
<i>Input Restrictions</i>	<u>None</u>
<i>Standard Design</i>	<u>Not applicable</u>

VRF Zone System Type	
<i>Applicability</i>	<u>VRF</u>
<i>Definition</i>	<u>Type of zone system</u>
<i>Units</i>	<u>VRF</u>
<i>Input Restrictions</i>	<u>VRF</u>
<i>Standard Design</i>	<u>Not applicable</u>

Count	
<i>Applicability</i>	<u>VRF</u>
<i>Definition</i>	<u>The number of duplicate systems represented by the current system. All system attributes must be identical for multiple system assignment.</u>
<i>Units</i>	<u>None</u>
<i>Input Restrictions</i>	<u>None</u>
<i>Standard Design</i>	<u>Not applicable</u>

Indoor Unit Type	
<i>Applicability</i>	<u>VRF</u>
<i>Definition</i>	<u>Ducted or unducted</u>
<i>Units</i>	<u>List – ducted, unducted</u>
<i>Input Restrictions</i>	<u>None</u>
<i>Standard Design</i>	<u>Not applicable</u>

Design Supply Air Temperature (Cooling)	
<i>Applicability</i>	<u>VRF</u>

<u>Definition</u>	<u>Design SAT in cooling for the zone</u>
<u>Units</u>	<u>Deg F</u>
<u>Input Restrictions</u>	<u>As Designed</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Design Supply Air Temperature (Heating)</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>Design SAT in heating for the zone</u>
<u>Units</u>	<u>Deg F</u>
<u>Input Restrictions</u>	<u>As Designed</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Net Cooling Capacity</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>Net cooling capacity of the zone system (one system if count>1), which includes all cooling to the zone but excludes any fan motor heat</u>
<u>Units</u>	<u>Btu/h</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Net Heating Capacity</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>Net heating capacity of the zone system (one system if count>1), which includes all heating to the zone but excludes any fan motor heat</u>
<u>Units</u>	<u>Btu/h</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Supply Fan Capacity for Cooling</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>The supply fan flow rate when the zone requires cooling</u>
<u>Units</u>	<u>cfm (for each mode)</u>

<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Supply Fan Capacity for Heating</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>The supply fan flow rate</u>
<u>Units</u>	<u>cfm (for each mode)</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Supply Fan Capacity for Deadband</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>Identify the supply fan airflow rate in cooling mode, heating mode and deadband (floating) mode</u>
<u>Units</u>	<u>cfm (for each mode)</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Supply Temp Control</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>The method of controlling the system supply air temperature</u>
<u>Units</u>	<u>List (Constant, reset by outside air, reset by demand)</u>
<u>Input Restrictions</u>	<u>No Supply Air Temperature Control</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Auxiliary Power When On</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>The parasitic electrical energy use of the zone terminal unit when either</u>

	<u>terminal unit coil is operating</u>
<u>Units</u>	<u>Watts or Btu/h</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

Auxiliary Power When Off	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>The parasitic electrical energy use of the zone terminal unit when the terminal unit coils are off</u>
<u>Units</u>	<u>Watts or Btu/h</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

5.6.55.6.6 Zone Level Air Flow

5.6.5.15.6.6.1 VAV Air Flow

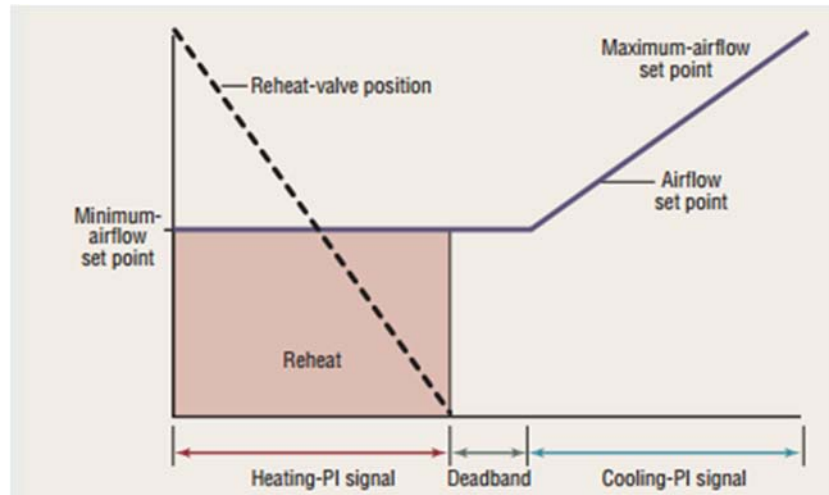
This group of building descriptors applies to proposed design systems that vary the volume of air at the zone level. The building descriptors are applicable for standard design systems 5 and 6.

Design Airflow	
<u>Applicability</u>	Systems that vary the volume of air at the zone level
<u>Definition</u>	The air delivery rate at design conditions
<u>Units</u>	Cfm
<u>Input Restrictions</u>	As designed If the unmet load hours in the proposed design are greater than 150, the user may have to modify the design airflow value manually.
<u>Standard Design</u>	For systems 5 and 6, the software shall automatically size the system airflow to meet both: a) the standard design peak cooling load based on a supply-air-to-room-air temperature difference of 20°F for exterior zones or 15°F for interior zones, the required ventilation air from Table 120.1-A of the standards, or makeup air, whichever is greater; and b) the standard design peak heating load based on 50 percent zone flow and 95°F supply air temperature. An exterior zone is defined as a type of thermal zone that has any exterior walls, and that has a non-zero amount of vertical fenestration (windows). Any zone that does not meet the definition of an exterior zone

	is an interior zone. For kitchen MAU systems, the design airflow is the greater of the airflow required to meet the design cooling loads and the total exhaust cfm.
<i>Standard Design: Existing Buildings</i>	

Terminal Minimum Stop	
<i>Applicability</i>	Systems that vary the volume of air at the zone level
<i>Definition</i>	The minimum airflow that will be delivered by a terminal unit before reheating occurs
<i>Units</i>	Unitless fraction of airflow (cfm) or specific airflow (cfm/ft ²)
<i>Input Restrictions</i>	Input must be greater than or equal to the outside air ventilation rate
<i>Standard Design</i>	For systems 5 and 6, set the minimum airflow to be the greater of 20 percent of the peak supply air volume to the zone or the minimum outside air ventilation rate. For laboratories, the minimum airflow fraction shall be fixed at a value equivalent to <u>the greater of the proposed design minimum exhaust requirements and 20% of the design supply airflow.</u> for VAV systems, <u>minimum air flow for single-zone VAV systems is the greater of proposed design minimum exhaust requirements and 50% of the design supply airflow rate.</u> minimum and 1 cfm/sf, 6 ACH.
<i>Standard Design: Existing Buildings</i>	

Terminal Heating Control Type	
<i>Applicability</i>	VAV boxes with reheat
<i>Definition</i>	The control strategy for the heating mode. Single Maximum: In the single maximum control mode, the airflow is set to a minimum constant value in both the deadband and heating mode. This airflow can vary but is typically 30 to 50 percent of maximum. This control mode typically has a higher minimum airflow than the minimum used in the dual maximum below, resulting in more frequent reheat. Figure 99: Single Maximum VAV Box Control



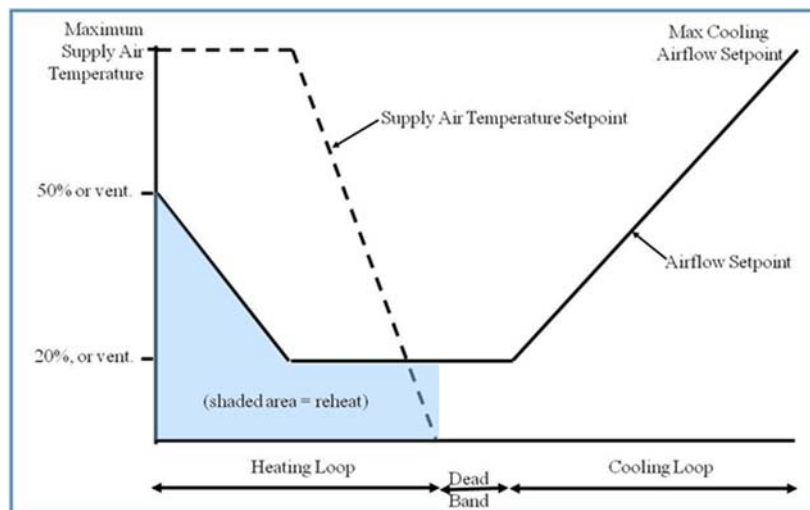
Source: Taylor Engineering

Dual Maximum:

Raises the supply air temperature (SAT) as the first stage of heating, and increases the airflow to the zone as the second stage of heating.

1. The first stage of heating consists of modulating the zone supply air temperature setpoint up to a maximum setpoint no larger than 95°F while the airflow is maintained at the dead band flow rate.
2. The second stage of heating consists of modulating the airflow rate from the dead band flow rate up to the heating maximum flow rate (50 percent of design flow rate).

Figure 1040: Dual Maximum Control Sequence



Units

- List:
- Single maximum
 - Dual maximum

<i>Input Restrictions</i>	Fixed at single maximum if control system type is not direct digital control (DDC) control to the zone level
<i>Standard Design</i>	Dual maximum <u>For healthcare facilities, same as the Proposed Design.</u>
<i>Standard Design: Existing Buildings</i>	

5.6.5.25.6.6.2 Fan Powered Boxes

Fan Powered Box Type	
<i>Applicability</i>	Thermal zones that have fan powered boxes
<i>Definition</i>	Defines the type of fan-powered induction box
<i>Units</i>	List : <ul style="list-style-type: none"> • series • parallel
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable <u>For healthcare facilities, same as the Proposed Design.</u>
<i>Standard Design: Existing Buildings</i>	

<u>Terminal</u> Fan Power	
<i>Applicability</i>	Thermal zones that have fan powered boxes
<i>Definition</i>	Rated power input of the fan in a fan-powered box
<i>Units</i>	W or W/cfm
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable <u>For healthcare facilities, same as the Proposed Design.</u>
<i>Standard Design: Existing Buildings</i>	

Fan Powered Box Induced Air Zone	
<i>Applicability</i>	Thermal zones that have fan powered boxes

<i>Definition</i>	Zone from which a series or parallel fan-powered box draws its air
<i>Units</i>	List (of zones)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable For healthcare facilities, same as the Proposed Design.
<i>Standard Design: Existing Buildings</i>	

Parallel PIU Induction Ratio	
<i>Applicability</i>	Thermal zones that have fan-powered boxes
<i>Definition</i>	Ratio of induction-side airflow of a fan-powered box at design heating conditions to the primary airflow
<i>Units</i>	Ratio
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable For healthcare facilities, same as the Proposed Design.
<i>Standard Design: Existing Buildings</i>	

Parallel Fan Box Thermostat Setpoint	
<i>Applicability</i>	Thermal zones that have fan powered boxes
<i>Definition</i>	Temperature difference above the heating setpoint at which the parallel fan is turned on
<i>Units</i>	Degrees Fahrenheit (°F)
<i>Input Restrictions</i>	2°F above the heating setpoint schedule
<i>Standard Design</i>	Not applicable For healthcare facilities, same as the Proposed Design.
<i>Standard Design: Existing Buildings</i>	

5.6.5.35.6.6.3 Zone Exhaust

This group of building descriptors describes the rate of exhaust and the schedule or control for this exhaust. An exhaust system can serve one thermal zone or multiple thermal zones. Energy is summed for the exhaust system level, not the thermal zone level.

This section also contains unique inputs for kitchen exhaust systems that must meet requirements of section 140.9 of the ~~standards~~ Standards.

Kitchen Exhaust Hood Length	
<i>Applicability</i>	Exhaust fans in spaces of type kitchen, commercial food preparation
<i>Definition</i>	The exhaust hood length
<i>Units</i>	ft
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	

Kitchen Exhaust Hood Style	
<i>Applicability</i>	Exhaust fans in spaces of type kitchen, commercial food preparation
<i>Definition</i>	The hood style as defined in Table 140.9-A of the standards
<i>Units</i>	List: <ul style="list-style-type: none"> • Wall-mounted canopy • Single island • Double island • Eyebrow • Backshelf/passover
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	

Kitchen Exhaust Hood Cooking Duty	
<i>Applicability</i>	Exhaust fans in spaces of type kitchen, commercial food preparation
<i>Definition</i>	The hood cooking duty as defined in Table 140.9-A of the standards
<i>Units</i>	List: <ul style="list-style-type: none"> • Light-duty • Medium-duty • Heavy-duty • Extra heavy-duty
<i>Input Restrictions</i>	As designed

<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	

Exhaust Fan Name	
<i>Applicability</i>	All thermal zones
<i>Definition</i>	A reference to an exhaust fan system that serves the thermal zone
<i>Units</i>	Text or other unique reference to an exhaust fan system defined in the secondary systems section
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	

Laboratory Exhaust Rate Type	
<i>Applicability</i>	All zones comprised of laboratory, scientific or laboratory, equipment room space
<i>Definition</i>	The type of load that dictates lab exhaust rate requirements
<i>Units</i>	List: <ul style="list-style-type: none"> • Hood-dominated • Load-dominated
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	

Exhaust Air Flow Rate	
<i>Applicability</i>	All thermal zones
<i>Definition</i>	Rate of exhaust from a thermal zone
<i>Units</i>	cfm /sfcfm/ft ²
<i>Input Restrictions</i>	As designed for non-process zones As designed for kitchen exhaust systems For laboratory zones, the exhaust air flow rate is defaulted to:

	<p>15 ACH for hood dominated spaces, or 6 ACH for load dominated spaces</p> <p>The design exhaust air flow rate may be specified by the user, but a warning shall be posted if less than 6 ACH <u>1 cfm/ft² cfm/sf</u>.</p>
<i>Standard Design</i>	<p>Same as the proposed design but not above the maximum standard design ventilation rates listed in Appendix 5.4A for spaces that do not include covered processes. Exception for buildings with over 5,000 cfm of kitchen exhaust; the <u>standard design baseline</u> is a function of the kitchen exhaust hood length, kitchen exhaust hood style, and kitchen exhaust hood cooking duty, and is determined by Title 24 standards, Table 140.9-A.</p> <p>For lab exhaust systems, the standard design is the same as the proposed.</p> <p><u>For healthcare facilities, same as the Proposed Design.</u></p>
<i>Standard Design: Existing Buildings</i>	

Exhaust Minimum Air Flow Rate	
<i>Applicability</i>	All laboratory zones
<i>Definition</i>	Rate of exhaust from a zone
<i>Units</i>	Cfm/sf
<i>Input Restrictions</i>	<p>As designed for non-process zones</p> <p>For laboratory zones, the exhaust air flow rate is the maximum of the hood scheduled exhaust air flow rate and the minimum ventilation rate. A warning is posted if the <u>minimum design</u> exhaust rate is 2 ACH or less.</p>
<i>Standard Design</i>	<p>For laboratory systems with constant flow exhaust fan control, the exhaust minimum air flow rate is 6 ACH <u>the greater of the proposed design minimum and 1 cfm/ft² cfm/sf and 50% of the design exhaust air flow rate</u>. For laboratory systems with variable flow or variable speed drive exhaust fan control, the exhaust minimum air flow rate is 4 ACH <u>the greater of the proposed design minimum exhaust air flow rate at 20% of the design exhaust air flow rate</u>.</p> <p><u>For healthcare facilities, same as the Proposed Design.</u></p>
<i>Standard Design: Existing Buildings</i>	

Exhaust Fan Schedule	
<i>Applicability</i>	All thermal zones

<i>Definition</i>	Schedule indicating the pattern of use for exhaust air from the thermal zone
<i>Units</i>	Data structure: schedule, fraction
<i>Input Restrictions</i>	<p>For non-process spaces, schedule is prescribed based on occupancy type and specified in Appendix 5.4B. As designed for non-covered process spaces</p> <p>Exhaust schedules for kitchen exhaust hoods are in the building with total flow greater than or less than 5,000 cfm are prescribed and specified in Appendix 5.4B.</p> <p>For laboratory spaces, if the exhaust is constant flow, the schedule shall be fixed at 0.90 for all hours of the year (24/7).</p> <p>For laboratory spaces, if the exhaust is variable flow, all the schedule shall be prescribed at the values in Appendix 5.4B <u>the compliance software shall automatically select use either the standard laboratory exhaust schedule or, the modified laboratory variable exhaust schedule for hoods with sash control, or an interpolated average if only a fraction of the exhaust hoods have sash control.</u></p>
<i>Standard Design</i>	<p>Same as the proposed design for non-covered process spaces</p> <p>Exhaust schedules for kitchen exhaust hoods for flow less than 5,000 cfm and for flow greater than 5,000 cfm are prescribed and specified in Appendix 5.4B.</p> <p>For laboratory spaces, if the proposed design exhaust air flow rate is greater than or equal to 10 ACH, the schedule shall be prescribed at 0.90 for all hours of the year (24/7).</p> <p>For laboratory spaces, if the proposed design minimum exhaust air flow rate is 10 ACH or less, the exhaust fan schedule shall be prescribed at the values defined in Appendix 5.4B for variable flow. <u>For laboratory exhaust systems with VAV control and fume hood dominated airflow, the standard design shall use the Appendix 5.4B modified laboratory variable exhaust schedule for hoods with sash controls.</u></p> <p><u>For healthcare facilities, same as the Proposed Design.</u></p>
<i>Standard Design: Existing Buildings</i>	

<u>Exhaust Fan Fraction Sash Control</u>	
<i>Applicability</i>	<u>Zones with laboratory exhaust hoods and sash controls</u>
<i>Definition</i>	<u>The fraction of exhaust hoods that contain sash flow controls. This input is needed to appropriately model cases where only a fraction of the exhaust hoods that have automatic sash controls.</u>
<i>Units</i>	<u>Fraction</u>

<i>Input Restrictions</i>	<u>As Designed (between 0 and 1)</u>
<i>Standard Design</i>	<u>1 if sash controls are required for the laboratory space (per Table 140.9-B of the Standards)-</u> <u>For healthcare facilities, same as the Proposed Design.</u>
<i>Standard Design:</i> <i>Existing Buildings</i>	<u>As Designed (between 0 and 1)</u>

5.6.5.45.6.6.4 Outdoor Air Ventilation

Ventilation Source	
<i>Applicability</i>	All thermal zones
<i>Definition</i>	The source of ventilation for an HVAC system. The choices are: <ul style="list-style-type: none"> • Natural (by operable openings) • Mixed-mode (primary ventilation by HVAC fan, supplemented by natural ventilation when operable windows are open) • Forced (by fan) – <u>can be supply fan or dedicated exhaust fan operating continuously</u>
<i>Units</i>	List natural or forced
<i>Input Restrictions</i>	For residential units and hotel/motel guest rooms, as designed. For all other occupancies, forced or mixed mode.
<i>Standard Design</i>	For residential units, the same modeling method as forced ventilation. Ventilation shall be provided by exhaust fans in each zone. <u>For healthcare facilities, same as the Proposed Design.</u> For all other occupancies, set to forced ventilation if the window type is fixed for all windows in the zone. If operable windows are specified, set to mixed mode.
<i>Standard Design:</i> <i>Existing Buildings</i>	

Ventilation Standard Source	
<i>Applicability</i>	Thermal zones with special ventilation requirements, such as a process space, which have no defined requirements in Title 24
<i>Definition</i>	Minimum ventilation rates for: <ul style="list-style-type: none"> • Title 24 (default) • Other
<i>Units</i>	List: See above
<i>Input Restrictions</i>	None

	User should be prepared to show justification for not using Title 24 ventilation source. <u>If 'Other' is used, the user must enter a description of which standard applies, such as OSHPD3, Animal Shelter, etc.</u>
<i>Standard Design</i>	Same as proposed
<i>Standard Design: Existing Buildings</i>	<u>Same as proposed</u>

Design Ventilation Rate	
<i>Applicability</i>	All thermal zones
<i>Definition</i>	<p>The quantity of ventilation air that is provided to the space for the specified thermal zone at maximum occupancy.</p> <p>This is adjusted by the occupancy fraction to determine the design ventilation rate for the space.</p>
<i>Units</i>	cfm
<i>Input Restrictions</i>	<p>To accommodate transfer air requirements for makeup air for exhaust from other zones, the design ventilation rate may be <u>up to between 95 percent and 120 percent of the proposed design exhaust code required ventilation rate for all high-rise residential spaces and thermal zones, or between 95 percent and 110 percent of code required ventilation rates for all other buildings</u> on a building story without penalty.</p> <p>The minimum required ventilation rate for each space shall be the larger of 15 cfm times the design occupancy from Appendix 5.4A or the conditioned floor area times the applicable ventilation rate from Appendix 5.4A. As defined by the user, provided that the total outside air ventilation rate to the building story matches the standard design outside air ventilation rate for the building story within a specified tolerance (within 20%).</p> <p><u>Ventilation rates below 95% of the code required ventilation rate for a building story are not allowed.</u></p> <p>If the ventilation source is natural for residential dwelling spaces, then the proposed design design-ventilation rate for compliance model purposes shall be zero.</p>
<i>Standard Design</i>	<p><u>For healthcare facilities, same as the Proposed Design.</u></p> <p>If the total exhaust airflow requirement on the building floor does not exceed the total ventilation requirement, then the standard design outside air ventilation rate shall be the same as the proposed, <u>if the design ventilation rate is between the limits defined above (95% and 120% of code minimum for high-rise residential spaces, and 95% and 110% for all other buildings.</u> If the proposed ventilation rate exceeds the <u>limits above,</u> default minimum, the standard design ventilation rate for each space</p>

	<p>shall be the proposed rate uniformly reduced such that the total ventilation air delivered to the building story is equal to the total minimum <u>maximum allowed</u> ventilation air rate:</p> $\text{Design Ventilation Rate}_{\text{std}} = \text{Design Ventilation Rate}_{\text{prop}} \times \left(\frac{\text{BFVent}_{\text{std}}}{\text{BFVent}_{\text{prop}}} \right)$ <p>Where:</p> <p>BFVent_{min} is the building floor design minimum required ventilation flow, as specified by the standards, and</p> <p>BFVent_{prop} is the building floor design ventilation flow for the proposed design.</p> <p>If the total exhaust airflow requirement on the building floor exceeds the minimum required ventilation flow requirements:</p> <p>The standard design outside air ventilation rate shall be the same as the proposed up to 120 percent of the design exhaust air flow. If the proposed ventilation rate exceeds 120 percent of the design exhaust flow, the standard design ventilation rate for each space shall be the proposed rate uniformly reduced such that the total ventilation air delivered to the building story is equal to 120 percent of the design exhaust flow rate:</p> $\text{Design Ventilation Rate}_{\text{std}} = \text{Design Ventilation Rate}_{\text{prop}} \times \left(\frac{\text{BFExh}_{\text{prop}} \times 1.2}{\text{BFVent}_{\text{prop}}} \right)$ <p>Where:</p> <p>BFExh_{prop} is the building floor design exhaust flow, and</p> <p>BFVent_{prop} is the building floor design ventilation flow for the proposed design</p> <p>If the ventilation source is “natural” for residential dwelling spaces, then the standard design (baseline) design ventilation rate for compliance model purposes shall be zero.</p>
<p><i>Standard Design: Existing Buildings</i></p>	<p><u>Same as the proposed, if unaltered. If space type is altered such that different ventilation rate requirements apply, the outside air ventilation rate should follow the same rules as for new construction.</u></p>

Occupancy Fraction	
<i>Applicability</i>	All thermal zones
<i>Definition</i>	<p>The space design requirements for ventilation at design occupancy that are compliant with space egress requirements and design ventilation requirements.</p> <p>The occupancy fraction also affects hot water use requirements for the space.</p>
<i>Units</i>	Unitless fraction
<i>Input Restrictions</i>	Default of 0.5 as designed with a minimum value of 0.5 and a maximum

	value of 5
<i>Standard Design</i>	0.5
<i>Standard Design: Existing Buildings</i>	<u>0.5</u>

Building Floor Ventilation Requirement	
<i>Applicability</i>	Internal variable, calculated for each building story (floor)
<i>Definition</i>	<p>The total outside air ventilation airflow requirement for all spaces on a building story or floor.</p> <p>This is calculated by summing the ventilation levels for each space, and comparing it to the minimum required ventilation rate and the design exhaust air flow requirements.</p>
<i>Units</i>	cfm (ft ³ /min)
<i>Input Restrictions</i>	Not a user input; derived by summing the ventilation and exhaust air flows from all spaces on the building floor
<i>Standard Design</i>	<p><u>For healthcare facilities, same as the Proposed Design.</u></p> <p><u>For all other spaces:</u></p> <p>This is calculated by the following procedure:</p> <ol style="list-style-type: none"> 1. Calculate the proposed ventilation for the building story as the sum of design ventilation flow for each space included on a building story, including all spaces except space designated as lab space. 2. Calculate the proposed exhaust for the building story as the sum of design exhaust flow for each space on the building story, including all spaces except spaces designated as lab space. 3. Calculate the code minimum ventilation requirement as the sum of all minimum required ventilation airflows, as defined by Appendix 5.4A, for all spaces in the building story. 4. If the proposed exhaust is greater than the code minimum ventilation rate, then: <ol style="list-style-type: none"> a. Total standard design building story ventilation requirement shall be: Standard ventilation = Min (proposed ventilation, proposed exhaust x 1.2) <p>Otherwise: Standard ventilation = Min (code minimum ventilation, proposed ventilation)</p>
<i>Standard Design: Existing Buildings</i>	

Minimum Ventilation Rate	
<i>Applicability</i>	All thermal zones, excluding unconditioned spaces
<i>Definition</i>	The minimum quantity of ventilation air that must be provided to the space when it is occupied
<i>Units</i>	cfm (ft ³ /min)
<i>Input Restrictions</i>	<p>As designed but not lower than code minimum (default value)</p> <p>The default value shall be the larger of 15 cfm times the design occupancy times the occupancy fraction or the conditioned floor area times the applicable ventilation rate from Appendix 5.4A.</p> <p>For spaces where demand control ventilation is installed, the minimum ventilation rate is specified by the greater of the rate in Table 120.1-A or 15 cfm times the scheduled occupancy for that hour.</p> <p>For residential spaces where the proposed design ventilation source is natural ventilation, the minimum ventilation rate shall be zero.</p>
<i>Standard Design</i>	<p><u>For healthcare facilities, same as the Proposed Design.</u></p> <p>For spaces where demand control ventilation is required, the minimum ventilation rate is specified by the greater of the rate in Appendix 5.4A or 15 cfm times the scheduled occupancy for that hour.</p> <p>For systems serving laboratories, the airflow minimum for each lab space shall be 6 ACH for constant flow systems, or 4 ACH for variable flow systems.</p> <p>For residential spaces where the proposed design ventilation source is natural ventilation, the standard design minimum ventilation rate shall be zero.</p>
<i>Standard Design: Existing Buildings</i>	

Ventilation Control Method	
<i>Applicability</i>	All thermal zones
<i>Definition</i>	<p>The method used to determine outside air ventilation needed for each hour in the simulation.</p> <p>This information is reported to the system serving the zone. The method of controlling outside air at the system level in response to this information is discussed under secondary systems. Options at the zone level are:</p> <ul style="list-style-type: none"> • Occupant sensors: When the space is occupied according to the prescribed occupancy schedule, the outside air requirement is equal to the design ventilation rate. Otherwise, the outside air requirement is either 25 percent of the minimum ventilation rate or scheduled off (0).

	<ul style="list-style-type: none"> • CO2 sensors in the space: The outside air is varied to maintain a maximum CO2 concentration in the space. This shall be approximated by multiplying the ventilation rate per occupant times the number of occupants for that hour. (When turnstile counts are used to automatically adjust ventilation levels based on occupancy, this method may also be used.) • Fixed ventilation rate: Outside air is delivered to the zone at a constant rate and is equal to the design ventilation rate (see above).
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	<p>As designed</p> <p>If the space (other than a classroom space, which is exempted) includes a design occupant density greater than or equal to 25 persons per 1,000 ft², and the system includes an airside economizer, <u>or if the design airflow rate for the system exceeds 3,000 cfm</u>, the input is restricted to CO2 sensors in the space (a mandatory requirement).</p> <p><i>Note: a classroom space greater than 750 ft² must have an occupancy sensor for ventilation control and setback to meet the mandatory Title 24 Standards requirements of 120.2(e)3. This requirement should be indicated on the appropriate compliance form submittal.</i></p>
<i>Standard Design</i>	<p><u>For healthcare facilities, same as the Proposed Design.</u></p> <p>For the federal tax credit Title 24 compliance, if <u>If the default occupancy for the specified space function</u> from Appendix 5.4B is greater than or equal to 25 persons per 1,000 ft² and the system includes and airside economizer and the space is not a classroom space, set control method to CO2 sensors in the space. Otherwise, set to fixed ventilation rate.</p>
<i>Standard Design: Existing Buildings</i>	

NNNN

<u>Rated COP</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>Full load heating efficiency per AHRI rating conditions</u>
<u>Units</u>	<u>none</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Rated COP</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>Full load heating efficiency per AHRI rating conditions</u>

<u>Units</u>	<u>none</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Rated COP</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>Full load heating efficiency per AHRI rating conditions</u>
<u>Units</u>	<u>none</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Rated COP</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>Full load heating efficiency per AHRI rating conditions</u>
<u>Units</u>	<u>none</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Rated COP</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>Full load heating efficiency per AHRI rating conditions</u>
<u>Units</u>	<u>none</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Rated COP</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>Full load heating efficiency per AHRI rating conditions</u>
<u>Units</u>	<u>none</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

Rated COP	
<i>Applicability</i>	<u>VRF</u>
<i>Definition</i>	<u>Full load heating efficiency per AHRI rating conditions</u>
<i>Units</i>	<u>none</u>
<i>Input Restrictions</i>	<u>None</u>
<i>Standard Design</i>	<u>Not applicable</u>

5.7 HVAC Secondary Systems

This group of building descriptors relate to the secondary HVAC systems. There is not a one-to-one relationship between secondary HVAC system components in the proposed and standard design and the ~~baseline building~~ since the standard design~~baseline building~~ system is determined from building type, size, and number of floors. Where the standard design for a building descriptor varies with the standard design HVAC system selection, the baseline is specified as a table with the applicable value for each of the 11 standard design~~baseline~~ systems.

The HVAC standard design (baseline) systems are described in the summary tables below for reference. The details of individual building descriptor definitions can be found in section 5.7.1 and in subsections under 5.7.

Table 15: System #1 Description

System Description:	Packaged Terminal Air Conditioner (#1)
Supply Fan Power:	N/A (fan power integral to unit efficiency), ventilation provided naturally through operable windows
Supply Fan Control:	Constant volume
Min Supply Temp:	$50 \leq T \leq 60$ — DEFAULT: 20°F below return air temperature
Cooling System:	Direct expansion (DX)
Cooling Efficiency:	Minimum seasonal energy efficiency ratio (SEER) or energy efficiency ratio (EER) based on equipment type and output capacity of standard design unit(s). Adjusted EER is calculated to account for supply fan energy.
Maximum Supply Temp:	$85 \leq T \leq 110$ — DEFAULT: 100
Heating System:	Gas furnace (#3) or heat pump (#4)
Heating Efficiency:	Minimum annual fuel utilization efficiency (AFUE), thermal efficiency, coefficient of performance (COP) or heating season performance factor (HSPF) based on equipment type and output capacity of standard design unit(s).
Economizer:	None
Ducts:	N/A (unducted)

Table 16: System #2 Description

System Description:	Four Pipe Fan Coil (#2)
Supply Fan Power:	0.35 W/cfm
Supply Fan Control:	Cycles with load
Min Supply Temp:	$50 \leq T \leq 60$ — DEFAULT: 20°F below return air temperature
Space Temp Control:	SAT is fixed at 55°F. Fan cycles to meet the load.
Cooling System:	Chilled water
Cooling Efficiency:	Minimum kW/ton and IPLV per Path B chiller requirements in Title 24 Section 110.2
Maximum Supply Temp:	$85 \leq T \leq 110$ — DEFAULT: 100
Heating System:	Boiler
Heating Efficiency:	Minimum AFUE, thermal efficiency per Section 110.2 of Title 24 Part 6 for the applicable heating capacity
Economizer:	None
Ducts:	N/A (unducted)

Table 17: System #3 Description

System Description:	Packaged Single Zone with Gas Furnace/Electric Air Conditioning (#3)
Supply Fan Power:	See Section 5.7.3
Supply Fan Control:	Constant volume
Min Supply Temp:	$50 \leq T \leq 60$ — DEFAULT: 20°F below return air temperature
Cooling System:	Direct expansion (DX)
Cooling Efficiency:	Minimum SEER or EER based on equipment type and output capacity of standard design unit(s). Adjusted EER is calculated to account for supply fan energy.
Maximum Supply Temp:	$85 \leq T \leq 110$ — DEFAULT: 100
Heating System:	Gas furnace (#3)
Heating Efficiency:	Minimum AFUE, Thermal Efficiency, COP or HSPF based on equipment type and output capacity of standard design unit(s)
Economizer:	Integrated economizer with differential dry bulb high limit, when mechanical cooling output capacity of the standard design as modeled in the compliance run by the compliance software is over 54,000 Btu/hr
Ducts:	For ducts installed in unconditioned buffer spaces or outdoors as specified in §140.4(l), the duct system efficiency shall be as modified by accounting for duct leakage rate

Table 18: System #5 Description

System Description:	Packaged VAV with Boiler and Reheat
Supply Fan Power:	See Section 5.7.3
Supply Fan Control:	VAV—variable speed drive
Relief Fan Control:	See fan section
Minimum Supply Temp:	$50 \leq T \leq 60$ —DEFAULT: 20°F below return air temperature
Cooling System:	Direct expansion (DX)
Cooling Efficiency:	Minimum efficiency based on average standard design output capacity of equipment unit(s)
Maximum Supply Temp:	$90 \leq T \leq 110$ —DEFAULT: 105
Heating System:	Gas boiler
Hot Water Pumping System	Variable flow (two-way valves) riding the pump curve
Heating Efficiency:	Minimum efficiency based on average standard design output capacity of equipment unit(s)
Economizer:	Integrated dry bulb economizer with differential dry bulb limit

Table 19: System #6 Description

System Description:	Chilled Water VAV with Reheat
Supply Fan Power:	See Section 5.7.3
Supply Fan Control:	VAV—variable speed drive
Return Fan Control:	Same as supply fan
Minimum Supply Temp:	$50 \leq T \leq 60$ —DEFAULT: 20°F below return air temperature
Cooling System:	Chilled water
Chilled Water Pumping System	Variable flow (two-way valves) with a variable speed drive (VSD) on the pump if three or more fan coils or air handlers; constant volume flow with water temperature reset control if less than three fan coils or air handlers; reset supply pressure by demand if standard system has DDC controls
Cooling Efficiency:	Minimum efficiency based on standard design output capacity of equipment unit(s)
Maximum Supply Temp:	$90 \leq T \leq 110$ —DEFAULT: 105
Heating System:	Gas boiler
Hot Water Pumping System	Variable flow (two-way valves) riding the pump curve if three or more fan coils or air handlers; constant volume flow with water temperature reset control if less than three fan coils or air handlers; reset supply pressure by demand
Heating Efficiency:	Minimum efficiency based on standard design output capacity of equipment unit(s)
Economizer:	Integrated dry bulb economizer with differential dry bulb limit

Table 20: System #7 Description

System Description:	Single-Zone VAV System
Supply Fan Power:	See Section 5.7.3
Supply Fan Control	Variable speed drive
Minimum Supply Temp:	$50 \leq T \leq 60$ — DEFAULT: 20°F below return air temperature
Supply Temp Control:	Supply air temperature setpoint shall be linearly reset from minimum at 50 percent cooling load and above to maximum at 0 percent cooling load. Fan volume shall be linearly reset from 100 percent air flow at 100 percent cooling load to minimum air flow at 50 percent cooling load and below. Minimum fan volume setpoint shall be 50 percent. (This is effectively an "airflow first" sequence.)
Cooling System:	Direct expansion
Cooling Efficiency:	Minimum efficiency based on the standard output capacity of specific equipment unit(s)
Compressor Stages:	Two to four (See minimum unloading ratio requirement)
Maximum Supply Temp:	$90 \leq T \leq 110$ — DEFAULT: 100
Heating System:	Gas furnace
Hot Water Pumping System	Variable flow (two-way valves) riding the pump curve if three or more fan coils; constant volume flow with water temperature reset control if less than three fan coils; reset supply pressure by demand if standard system has DDC controls
Heating Efficiency:	Minimum efficiency based on the standard output capacity of specific equipment unit(s)
Economizer:	Integrated dry bulb economizer with differential dry-bulb high limit

Table 21: System #9 Description

System Description:	Heating and ventilation only system
Supply Fan Power:	See fan power details
Supply Fan Control	Constant volume
Minimum Supply Temp:	N/A
Cooling System:	None
Cooling Efficiency:	N/A
Maximum Supply Temp:	$90 \leq T \leq 110$ — DEFAULT: 100
Heating System:	Gas furnace
Hot Water Pumping System	N/A
Heating Efficiency:	Minimum efficiency based on the standard output capacity of specific equipment unit(s)
Economizer:	TBD

Table 22: System #10 Description

System Description:	Computer Room Air Handler (CRAH)
Supply Fan Power:	0.49 W/cfm at design air flow where economizer is required, 0.39 W/cfm where economizer is not required;
Supply Fan Control	variable speed drive; fan power part load curve is for "VSD with static pressure reset"
Minimum Supply Temp:	60°F
Cooling System:	Single-zone VAV with chilled water cooling source
Cooling Efficiency:	Same as System #6 (Built-up VAV)
Maximum Supply Temp:	80°F
Heating System:	None
Economizer:	Integrated 100 percent outside air economizer with differential dry-bulb limit
Supply Temp Control:	VAV: Supply air temperature setpoint shall be reset between 60°F and 80°F; fan volume shall be reset between 100 percent and 50 percent flow, depending on cooling load; minimum fan volume setpoint shall be 50 percent

Table 23: System #11 Description

System Description:	Computer Room Air Conditioner (CRAC)
Supply Fan Power:	0.49 W/cfm at design flow (see equipment sizing) where economizer is required; 0.39 W/cfm where economizer is not required
Supply Fan Control:	Constant speed if the computer room receptacle load is less than 17.5 kW; otherwise, variable speed drive. Fan power curve for VSD is 'VSD with static pressure reset'.
Relief Fan Control:	No relief fan
Minimum Supply Temp:	60°F
Cooling System:	Single-zone Air-cooled DX
Cooling Capacity:	Equipment design CFM and cooling capacity sized at 115 percent of the capacities generated from a system sizing run; one system per zone
Cooling Efficiency:	Appliance Standards Table C-9
Maximum Supply Temp:	80°F
Heating System:	None
Economizer:	Integrated 100 percent outside air economizer with differential dry-bulb limit if the net cooling capacity is greater than 54,000 Btu/h
Supply Temp Control:	VAV: Supply air temperature setpoint shall be between 60 F and 80 F; fan supply air volume shall be reset between 100 percent and 50 percent of the rated fan condition, depending on the cooling load; minimum fan volume setpoint shall be 50 percent CV: supply air temperature setpoint modulates to meet the load

Table 24: System #12 Description

System Description:	Laboratory HVAC System
Supply Fan Power:	See supply fan static pressure, supply fan efficiency, and supply fan motor efficiency for standard design specifications
Supply Fan Control:	Variable speed drive
Return Fan Control:	No return fans
Exhaust Fan Control:	Variable volume, variable speed drive if total building laboratory exhaust air flow rate is greater than 2,000 cfm and 10 ACH or greater; constant volume otherwise.
Ventilation:	Same as proposed but subject to a minimum 6 ACH; system is 100 percent outside air
Minimum Supply Temp:	55°F
Cooling System:	PVAV with air-cooled DX if total lab floor area <150,000 ft ² ; water-cooled chiller if greater than 150,000 ft ² floor area
Cooling Capacity:	Equipment fan CFM and cooling capacity sized at 115 percent of the capacities determined using a sizing run; one laboratory system per floor when total building laboratory floor area is 10,000 ft ² or greater, otherwise one laboratory system per zone
Cooling Efficiency:	Minimum efficiency requirements per Section 110.2
Maximum Supply Temp:	95
Heating System:	Gas furnace if building total floor area is 10,000 ft ² or less; hot water coils/boiler if greater than 10,000 ft ²
Economizer:	integrated 100 percent outside air economizer with differential dry bulb limit
Supply Temp Control:	VAV: Supply air temperature setpoint shall be reset upwards by 5 F based on warmest zone; fan volume shall be reset between 100 percent and 50 percent flow, depending on cooling load; minimum fan volume setpoint shall be 50 percent CV: supply air temperature setpoint modulates to meet the load

Table 25: System #13 Description

System Description:	Kitchen HVAC system
Supply Fan Power:	Fixed supply fan static pressure, see Section 5.7.3
Supply Fan Control:	Constant speed if total exhaust airflow rate is greater than or equal to 2,000 cfm; otherwise, variable speed drive; fan power ratio at part load is determined by the part load ratio and the static pressure reset system curve
Return Fan Control:	No return fans
Exhaust Fan Control:	Variable volume, variable speed drive if total exhaust air flow rate > 5,000 cfm; constant volume otherwise
Minimum Supply Temp:	20°F below space temperature setpoint
Cooling System:	PVAV with air-cooled DX if floor area < 50,000 ft ² ; water-cooled chiller if greater 50,000 ft ² floor area
Cooling Capacity:	Cooling capacity sized at 115 percent of the calculated room load; fan airflow sizing based on exhaust airflow requirements; one fan system per room
Cooling Efficiency:	Minimum efficiency requirements per Section 110.2
Maximum Supply Temp:	95°F
Heating System:	Gas furnace if building total floor area is 10,000 ft ² or less; hot water coils/boiler if greater than 10,000 ft ²
Economizer:	None
Supply Temp Control:	VAV: Supply air temperature setpoint shall be linearly reset from minimum at 50 percent cooling load and above to maximum at 0 percent cooling load; fan volume shall be linearly reset from 100 percent air flow at 100 percent cooling load to minimum air flow at 50 percent cooling load and below; minimum fan volume setpoint shall be 50 percent (effectively an "airflow first" sequence) CV: supply air temperature modulates to meet the load

5.7.1 Basic System Information

HVAC System Name	
Applicability	All system types
Definition	A unique descriptor for each HVAC system
Units	Text, unique
Input Restrictions	When applicable, input should match the tags that are used on the plans
Standard Design	None
Standard Design: Existing Buildings	<u>None</u>

System Type	
Applicability	All system types
Definition	A unique descriptor which identifies the following attributes of an HVAC system: <ul style="list-style-type: none"> • Number of air decks (one to three) • Constant or variable air flow • Type of terminal device • Fan configuration for multiple deck systems
Units	List from the choices below
Input Restrictions	PTAC – Packaged Terminal Air Conditioner <u>SZAC – Zonal air conditioning unit</u>

	<p>PTHP – Packaged Terminal Heat Pump PSZ-AC – Packaged Single Zone <u>(three-phase unit)</u> PSZ-HP – Packaged Single Zone Heat Pump <u>(three-phase unit)</u> PVAV – Packaged VAV with Reheat VAV* – VAV with Reheat PSZVAV* – Single Zone VAV PSZVAVHP – Single Zone VAV Heat Pump HV – Heating and Ventilation Only CRAC – Computer Room Air Conditioner CRAH – Computer Room Air Handler FPFC – Four-pipe fan coil DFDD – Dual fan dual duct RADFLR – Radiant floor heating and cooling WSHP – water-source heat pump SPVAC – Single package vertical air conditioner (minimum efficiency defined in Section 110.2) SPVHP – Single package vertical heat pump (minimum efficiency defined in Section 110.2) <u>SZAC – single zone, single-phase split air conditioner for residential-type spaces with DX coil, furnace and no economizer</u> * Choice includes series and parallel fan-powered boxes as zone terminal units</p>
<p><i>Standard Design</i></p>	<p>Based on the prescribed system type in the HVAC system map (see Section 5.1.2 Table 4); <u>standard design</u> baseline system types are shown in Table 27</p>
<p><i>Standard Design: Existing Buildings</i></p>	

Table 26: Baseline Building System Type

Baseline Building System Type
System 1 — PTAC
System 2 — FPFC
System 3 — PSZ-AC
System 5 — Packaged VAV with Reheat
System 6 — VAV with Reheat
System 7 — PSZ-SZVAV
System 9 — Heating and Ventilation
System 10 — CRAC Unit
System 11 — CRAH Unit
System 12 — LAB
System 13 — Kitchen

Air Distribution Type	
<i>Applicability</i>	All system types
<i>Definition</i>	Type of air distribution system that is coupled with the HVAC system. The choices are (overhead) mixing ventilation system, <u>underfloor air distribution system (UFAD)</u> , and displacement ventilation system (DV).
<i>Units</i>	List mixing, UFAD, DV
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Mixing
<i>Standard Design: Existing Buildings</i>	

Thermal zone List	
<i>Applicability</i>	All system types
<i>Definition</i>	Comprehensive list of all thermal zones served by a given HVAC system
<i>Units</i>	None
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	

Total Cooling Capacity	
<i>Applicability</i>	All system types

<i>Definition</i>	The installed cooling capacity <u>at rated conditions</u> of the project. This includes all: <ul style="list-style-type: none"> • Chillers • Built-up DX • Packaged cooling units
<i>Units</i>	Cooling tons (12,000 Btu/h per ton)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Autosize The cooling capacity shall be oversized by 15 percent. If the number of unmet load hours exceeds 150, increase the cooling capacity according to the procedures in Chapter 2.
<i>Standard Design: Existing Buildings</i>	

5.7.2 System Controls

5.7.2.1 Control System Type

Control System Type	
<i>Applicability</i>	All HVAC systems that serve more than one control zone, as well as the hydronic systems that serve building HVAC systems
<i>Definition</i>	The type of control system for multi-zone HVAC systems and their related equipment. This input affects the proposed design system specification for zone level controls, supply air temperature reset controls, ventilation controls and fan and pump static pressure part-load curves. See the following building descriptors: <ul style="list-style-type: none"> • Ventilation control method • Terminal heating control type • Pump part-load curve • Fan part-load curve • Optimal start
<i>Units</i>	None
<i>Input Restrictions</i>	List one of the following inputs: Direct digital control (DDC) control to the zone level – DDC systems with control to the zone level Other – other control systems, including pneumatic and DDC systems without control to the zone level
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> DDC control to the zone level

<i>Standard Design: Existing Buildings</i>	
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5.7.2.2 Schedules

Cooling Schedule	
<i>Applicability</i>	All cooling systems
<i>Definition</i>	A schedule that represents the availability of cooling
<i>Units</i>	Data structure: schedule, on/off
<i>Input Restrictions</i>	Schedule group is prescribed in Appendix 5.4A and schedule values are prescribed in Appendix 5.4B. See Section 2.3.3 on how software shall assign schedules when the spaces served by the system are assigned to different schedule groups in Appendix 5.4A.
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	

Heating Schedule	
<i>Applicability</i>	All systems
<i>Definition</i>	A schedule that represents the availability of heating
<i>Units</i>	Data structure: schedule, on/off
<i>Input Restrictions</i>	Schedule group is prescribed in Appendix 5.4A and schedule values are prescribed in Appendix 5.4B. See Section 2.3.3 on how software shall assign schedules when the spaces served by the system are assigned to different schedule groups in Appendix 5.4A.
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	

Air Handler Schedule	
<i>Applicability</i>	All systems that do not cycle with loads
<i>Definition</i>	A schedule that indicates when the air handler operates continuously
<i>Units</i>	Data structure: schedule, on/off
<i>Input Restrictions</i>	For healthcare facilities, same as the Proposed Design. For all others,

	<p>Schedule group is prescribed in Appendix 5.4A and schedule values are prescribed in Appendix 5.4B. See Section 2.3.3 on how software shall assign schedules when the spaces served by the system are assigned to different schedule groups in Appendix 5.4A.</p> <p><u>The fan schedules and HVAC operations are defined so that the air handlers provide the necessary outside air 1 hour prior to scheduled occupancy.</u></p> <p>When a fan system serves several occupancies, the fan schedule must remains ON to serve the operating hours of each occupancy.</p>
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	

Air Handler Fan Cycling	
<i>Applicability</i>	All fan systems
<i>Definition</i>	This building descriptor indicates whether the system supply fan operates continuously or cycles with building loads when the HVAC schedule indicates the building is occupied. (See night cycle control input for fan operation during unoccupied hours.) The fan systems in most commercial buildings operate continuously.
<i>Units</i>	List continuous or cycles with loads
<i>Input Restrictions</i>	<p>For four-pipe fan coil systems, aAs designed if the HVAC system serving the spaces includes a dedicated outside air source for ventilation and if the system is a zonal system; otherwise, fixed at continuous.</p> <p>For all other systems, fixed at continuous.</p>
<i>Standard Design</i>	<p><u>For healthcare facilities, same as the Proposed Design. For all others,</u></p> <p>Cycles with loads for PTAC or FPFC systems; continuous for all other standard design system types</p>
<i>Standard Design: Existing Buildings</i>	

Optimal Start Control	
<i>Applicability</i>	Systems with the control capability for flexible scheduling of system start time based on building loads
<i>Definition</i>	Optimal start control adjusts the start time of the HVAC unit such that the space is brought to setpoint just prior to occupancy. This control strategy modifies the heating, cooling, and fan schedules.
<i>Units</i>	Boolean (Yes/No)

<i>Input Restrictions</i>	Fixed at yes if control system type is DDC to the zone level; otherwise, as designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Fixed at yes, if control system type is DDC to the zone level
<i>Standard Design: Existing Buildings</i>	

Night-Cycle HVAC Fan Control	
<i>Applicability</i>	All <u>air systems</u> – not applicable for zone systems
<i>Definition</i>	The control of an HVAC system that is triggered by the heating or cooling temperature setpoint for thermal zones during periods when the heating, cooling and fan systems are scheduled to be off. For this control, the space is controlled to the setback or setup temperature only; this control is not equivalent to a night purge control. The choices are: <ul style="list-style-type: none"> • Cycle on call from any zone • Cycle on call from the primary control zone • Stay off • Cycle zone fans only (for systems with fan-powered boxes) Restart fans below given ambient temperature.
<i>Units</i>	None
<i>Input Restrictions</i>	Cycle on call from any zone, except for systems with fan-powered boxes, where either cycle on call from any zone or cycle zone fans only is allowed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Cycle on call from any zone
<i>Standard Design: Existing Buildings</i>	

5.7.2.3 Cooling Control

Cooling Supply Air Temperature	
<i>Applicability</i>	Applicable to all systems
<i>Definition</i>	The supply air temperature setpoint at design cooling conditions
<i>Units</i>	Degrees Fahrenheit (°F)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> 15°F below the space temperature setpoint for interior zones that are

	served by multiple zone systems; for all other zones, 20°F below the space temperature setpoint
<i>Standard Design: Existing Buildings</i>	

Cooling Supply Air Temperature Control	
<i>Applicability</i>	Any cooling system
<i>Definition</i>	The method of controlling the supply air temperature. Choices are: <ul style="list-style-type: none"> • No control – for this scheme the cooling coils are energized whenever there is a call for cooling • Fixed (constant) • Reset by warmest zone, airflow first • Reset by warmest zone, temperature first • Reset by outside air dry-bulb temperature • Scheduled setpoint • Staged setpoint (for single zone VAV and DX with multiple stages)
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	For healthcare facilities, same as the Proposed Design. For all others, For <u>standard design</u> baseline building systems 1 through 4, the SAT control is fixed. For systems 5 through 8, 10, and 11, the SAT control shall be reset by warmest zone, airflow first. For system 9 (heating and ventilation), this input is not applicable.
<i>Standard Design: Existing Buildings</i>	

Cooling Reset Schedule by OSA	
<i>Applicability</i>	When the proposed design resets SAT by outside air dry-bulb temperature
<i>Definition</i>	A linear reset schedule that represents the SAT setpoint as a function of outdoor air dry-bulb temperature This schedule is defined by the following data points (see Figure 9): <ul style="list-style-type: none"> • The coldest cooling supply air temperature • The corresponding (hot) outdoor air dry-bulb setpoint • The warmest cooling supply air temperature • The corresponding (cool) outdoor air dry-bulb setpoint <p>Figure 1144: SAT Cooling Setpoint Reset Based On Outdoor Air</p>

Temperature (OAT)																	
	<table border="1" style="display: none;"> <caption>Data points for Temperature (OAT) graph</caption> <thead> <tr> <th>OAT</th> <th>SAT Setpoint</th> </tr> </thead> <tbody> <tr><td>40</td><td>60</td></tr> <tr><td>50</td><td>60</td></tr> <tr><td>60</td><td>60</td></tr> <tr><td>70</td><td>57.5</td></tr> <tr><td>80</td><td>55</td></tr> <tr><td>90</td><td>55</td></tr> <tr><td>100</td><td>55</td></tr> </tbody> </table>	OAT	SAT Setpoint	40	60	50	60	60	60	70	57.5	80	55	90	55	100	55
OAT	SAT Setpoint																
40	60																
50	60																
60	60																
70	57.5																
80	55																
90	55																
100	55																
<i>Units</i>	Data structure (two matched pairs of SAT and OAT, see above)																
<i>Input Restrictions</i>	As designed																
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable																
<i>Standard Design: Existing Buildings</i>																	

5.7.2.4 Heating Control

Preheat Setpoint	
<i>Applicability</i>	Systems with a preheat coil located in the outside air stream
<i>Definition</i>	The control temperature leaving the preheat coil
<i>Units</i>	Degrees Fahrenheit (°F)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Heating Supply Air Temperature	
<i>Applicability</i>	All systems
<i>Definition</i>	The supply air temperature leaving the air handler when the system is in a heating mode (not the air temperature leaving the reheat coils in VAV boxes)
<i>Units</i>	Degrees Fahrenheit (°F)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, 95°F for system types 1-4; 60°F for multiple zone systems; no heating for data centers and computer rooms</u>
<i>Standard Design: Existing Buildings</i>	

Heating Supply Air Temperature Control	
<i>Applicability</i>	Systems with the capability to vary heating SAT
<i>Definition</i>	The method of controlling heating SAT Choices are: <ul style="list-style-type: none"> • No control – the heating coil is energized on a call for heating but the supply air temperature is not directly controlled, instead it is dependent on the entering air temperature, the heating capacity and the airflow rate • Fixed (constant) • Reset by coldest zone, airflow first • Reset by coldest zone, temperature first • Reset by outside air dry-bulb temperature • Staged setpoint • Scheduled setpoint
<i>Units</i>	Degrees Fahrenheit (°F)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Fixed (constant)</u>
<i>Standard Design: Existing Buildings</i>	

Heating Reset Schedule by OSA	
<i>Applicability</i>	Systems that reset the heating SAT by outside dry-bulb temperature (this

	typically applies to dual-duct systems or to single zone systems with hydronic heating coils)																
<i>Definition</i>	<p>A linear reset schedule that represents the heating supply air temperature or hot deck supply air temperature (for dual duct systems) as a function of outdoor air dry-bulb temperature</p> <p>This schedule is defined by the following data points (see Figure 10):</p> <ul style="list-style-type: none"> • The hottest heating supply air temperature • The corresponding (cold) outdoor air dry-bulb threshold • The coolest heating supply air temperature • The corresponding (mild) outdoor air dry-bulb threshold <p>Figure 1242: SAT Heating Setpoint Reset Based on Outdoor Air Temperature (OAT)</p> <table border="1"> <caption>Data points for Figure 1242</caption> <thead> <tr> <th>OAT</th> <th>SAT Setpoint (F°)</th> </tr> </thead> <tbody> <tr> <td>20</td> <td>110</td> </tr> <tr> <td>30</td> <td>110</td> </tr> <tr> <td>40</td> <td>110</td> </tr> <tr> <td>50</td> <td>100</td> </tr> <tr> <td>60</td> <td>90</td> </tr> <tr> <td>70</td> <td>90</td> </tr> <tr> <td>80</td> <td>90</td> </tr> </tbody> </table>	OAT	SAT Setpoint (F°)	20	110	30	110	40	110	50	100	60	90	70	90	80	90
OAT	SAT Setpoint (F°)																
20	110																
30	110																
40	110																
50	100																
60	90																
70	90																
80	90																
<i>Units</i>	Degrees Fahrenheit (F°)																
<i>Input Restrictions</i>	As designed																
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable																
<i>Standard Design: Existing Buildings</i>																	

5.7.3 Fan and Duct Systems

5.7.3.1 ~~Standard Design~~ ~~Baseline Building~~ Fan System Summary

The ~~standard design~~ ~~baseline building~~ fan system is summarized in this section. See Section 5.7.1, Table 27 for the HVAC ~~standard design~~ ~~baseline building~~ system mapping.

When the proposed design has exhaust fans (toilets or kitchens) or fume hood exhaust systems, the ~~standard design~~ ~~baseline building~~ has the same systems.

5.7.3.2 Supply Fans

Fan System Modeling Method	
<i>Applicability</i>	All fan systems
<i>Definition</i>	Software commonly models fans in one of three ways. The simple method is for the user to enter the electric power per unit of flow (W/cfm). This method is commonly used for unitary equipment and other small fan systems. A more detailed method is to model the fan as a system whereby the static pressure, fan efficiency, part-load curve, and motor efficiency are specified at design conditions. A third method is to specify brake horsepower at design conditions instead of fan efficiency and static pressure. This is a variation of the second method whereby brake horsepower is specified in lieu of static pressure and fan efficiency. The latter two methods are commonly used for VAV and other larger fan systems.
<i>Units</i>	List power-per-unit-flow, static pressure, or brake horsepower
<i>Input Restrictions</i>	As designed Either the static pressure or brake horsepower method shall be used. The user is required to enter the brake horsepower and motor horsepower of all fans.
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, The standard design baseline building shall use the static pressure brake horsepower method for all HVAC systems except the four-pipe fan coil system, which shall use the power-per-unit-flow method.</u>
<i>Standard Design: Existing Buildings</i>	

Supply Fan Power Ratio	
<i>Applicability</i>	<u>All fan systems</u>
<i>Definition</i>	<u>The standard design fan power requirements apply to all fans that operate at design conditions. To apportion the fan power to the supply fan and exhaust fans, a ratio is defined that is the the ratio of supply fan power to total system fan power. Supply fan power includes all supply fans and any series-powered fans, if present.</u>

<u>Units</u>	<u>List power per unit flow, static pressure, or brake horsepower</u> Unitless ratio
<u>Input Restrictions</u>	As designed, not a user input. <u>This is the ratio of the supply fan power to total system fan power, which includes supply fans, exhaust fans greater than 1 HP nameplate, any return fans, and any series-powered fans.</u>
<u>Standard Design</u>	<u>Same as proposed</u>
<u>Standard Design:</u> <u>Existing Buildings</u>	<u>Same as proposed</u>

Supply Fan Design Airflow	
<u>Applicability</u>	All fan systems
<u>Definition</u>	The air flow rate of the supply fan(s) at design conditions. This building descriptor sets the 100 percent point for the fan part-load curve.
<u>Units</u>	CFM (ft ³ /min)
<u>Input Restrictions</u>	As designed* For multiple deck systems, a separate entry should be made for each deck. <i>*The airflow is typically between 250 cfm/ton and 500 cfm/ton; values well outside of this range may cause simulation engine runtime efforts that must be addressed by the user (currently there are no input restrictions on this).</i>
<u>Standard Design</u>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> The program shall automatically size the air flow at each thermal zone to meet the loads. The design air flow rate calculation shall be based on a 20°F temperature differential between supply air and the room air 20°F temperature differential between the supply air and the return air for exterior zones and a 15°F temperature differential for interior zones served by multiple zone systems. The design supply air flow rate is the larger of the flow rate required to meet space conditioning requirements and the required ventilation flow rate. <u>The</u> For equipment with DX cooling coils, the program applies the results of a sizing run, and oversizes both the cooling coil capacity and airflows by 15% to obtain the appropriate airflow. The supply fan design air flow rate shall be the system airflow rate that satisfies that coincident peak of all thermal zones at the design supply air temperature (55°F).

<i>Standard Design: Existing Buildings</i>	
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Fan Control Method	
<i>Applicability</i>	All fan systems
<i>Definition</i>	<p>A description of how the supply (and return/relief) fan(s) are controlled</p> <p>The options include:</p> <ul style="list-style-type: none"> • Constant volume • Variable-flow, inlet, or discharge dampers • Variable-flow, inlet guide vanes • Variable-flow, variable speed drive (VSD) • Variable-flow, variable pitch blades • Variable-flow, other • Two-speed • Constant volume, cycling (fan cycles with heating and cooling)
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<p><u>For healthcare facilities, same as the Proposed Design. For all others,</u></p> <p>Applicable to variable air volume systems</p> <p>Based on the prescribed system type. Refer to the HVAC System Map in 5.1.2.</p>
<i>Standard Design: Existing Buildings</i>	

Table 124527: Standard Design Baseline Building Fan Control Method

<u>Standard Design Baseline</u> building System	Fan Control Method
System 1 – <u>PTAGSZAC</u>	Constant volume, cycling <u>or continuous</u>
System 2 – FPFC	Constant volume, cycling
System 3 – PSZ-AC	Constant volume
System 5 – Packaged VAV with Reheat	Variable-flow, variable speed drive (VSD)
System 6 – VAV with Reheat	Variable-flow, variable speed drive (VSD)
System 7 – PSZ, Single Zone VAV	Variable-flow, variable-speed drive (VSD)
System 9 – Heating and Ventilation	Constant volume
System 10 – CRAH Units	Variable-flow, variable speed drive (VSD)*
System 11 – CRAC Units	Variable-flow, variable speed drive (VSD)*

* For CRAH Units, fan volume shall be linearly reset from 100 percent air flow at 100 percent cooling load to minimum airflow at 50 percent cooling load and below.

Supply Fan Brake Horsepower	
<i>Applicability</i>	All fan systems, except those specified using the power-per-unit-flow method
<i>Definition</i>	The design shaft brake horsepower of each supply fan. This input does not need to be supplied if the supply fan kW is supplied.
<i>Units</i>	Horsepower (hp)
<i>Input Restrictions</i>	<p>As designed</p> <p>If this building descriptor is specified for the proposed design, then the static pressure and fan efficiency are not.</p> <p>The compliance software shall apply the following rule to specify the proposed design bhp, based on user input:</p> <p>A standard motor size table (hp) is defined as: 1/12, 1/8, 1/4, 1/2, 3/4, 1, 1.5, 2, 3, 5, 7.5, 10, 15, 20, 25, 30, 40, 50, 60, 75, 100, 125, 150, 200.</p> <p>The user entered brake horsepower for the proposed design is compared against the next smaller motor size from the user entered supply fan motor horsepower. The proposed design supply fan brake horsepower (bhp) is set to the maximum of the user entered bhp and 95 percent of the next smaller motor horsepower:</p> <p>Proposed bhp = max(user bhp, 95 percent × MHP_{i-1})</p> <p>Where User bhp is the user entered supply fan brake horsepower:</p> <p>MHP_i is the proposed (nameplate) motor horsepower</p> <p>MHP_{i-1} is the next smaller motor horsepower from the Standard Motor Size table above. For example, if the proposed motor horsepower is 25, the next smaller motor horsepower from the table above is 20, and 95 percent of the next smaller motor horsepower is 19.</p>
<i>Standard Design</i>	<p>Not applicable</p> <p><u>For healthcare facilities, same as the Proposed Design. For all others, For FPFC, Not Applicable.</u></p> <p><u>For PVAV and built-up VAV systems:</u></p> <p><u>Supply Fan BHP = (0.0013 x cfm_{max} + A) x Supply Fan Ratio,</u></p> <p><u>For other systems,</u></p> <p><u>Supply Fan BHP = (0.00094 x cfm + A) x SupplyFanRatio, where</u></p> <p><u>cfm = the design supply air flow, and</u></p> <p><u>A = the fan power adjustment (see separate building descriptor)</u></p> <p><u>SupplyFanRatio is the ratio of supply fan brake horse power at design conditions to total system brake horsepower at design conditions</u></p> <p><u>SFbhp = (0.0013 x cfm_{max} + A) / RFbhp, EFbhp</u></p>

	<p><u>For constant volume systems, $0.00094 \times \text{cfm} + A$, where</u></p> <p><u>$\text{Cfm}_{\text{max}}$ = the design supply air flow, and</u></p> <p><u>A = the fan power adjustment (see separate building descriptor)</u></p> <p><u>RFbhp = the relief fan brakehorsepower at minimum OA damper position</u></p> <p><u>Assume fan static pressure of 0.5" w.g. for relief fan.</u></p> <p><u>EFbhp = the total exhaust fan brake horsepower of all exhaust fans greater than 1 HP that are not manually switched. For exhaust fans that serve multiple spaces, the exhaust fan power shall be distributed in a manner consistent with the ASHRAE PRM.</u></p>
Standard Design: Existing Buildings	<p><u>Same as proposed if existing and unaltered; otherwise, same as new construction</u></p>

Supply Fan Motor Horsepower	
Applicability	All fan systems, except those specified using the power-per-unit-flow method
Definition	The motor nameplate horsepower of the supply fan
Units	List: choose from standard motor sizes: 1/12, 1/8, 1/4, 1/2, 3/4, 1, 1.5, 2, 3, 5, 7.5, 10, 15, 20, 25, 30, 40, 50, 60, 75, 100, 125, 150, 200
Input Restrictions	As designed. This building descriptor is required for the static pressure or the brake horsepower methods.
Standard Design	The brake horsepower for the supply fan is this value times the supply fan ratio (see above). <u>Set to the standard motor efficiency for the next larger nominal HP size, from NEMA standards</u>
Standard Design: Existing Buildings	<u>Set to the standard motor efficiency for the next larger nominal HP size, from NEMA standards</u>

Supply Fan Static Pressure	
Applicability	All fan systems using the static pressure method
Definition	The design static pressure for the supply fan. This is important for both fan electric energy usage and duct heat gain calculations.
Units	Inches of water column (in. H ₂ O)
Input Restrictions	As designed The design static pressure for the supply fan does not need to be specified if the supply fan brake horsepower (bhp) is specified.

<p><i>Standard Design</i></p>	<p>The standard design for all systems except four pipe fan coil (FPFC) and PTAC is defined by the following table:</p> <table border="1" data-bbox="467 289 1226 661"> <thead> <tr> <th data-bbox="467 289 657 443">Airflow</th> <th data-bbox="657 289 847 443">Single zone, six stories or less</th> <th data-bbox="847 289 1037 443">Multiple zone, less than six stories</th> <th data-bbox="1037 289 1226 443">Multiple zone, greater than six stories</th> </tr> </thead> <tbody> <tr> <td data-bbox="467 443 657 493"><2000 cfm</td> <td data-bbox="657 443 847 493">2.5"</td> <td data-bbox="847 443 1037 493">3.0"</td> <td data-bbox="1037 443 1226 493">3.5"</td> </tr> <tr> <td data-bbox="467 493 657 577">2000 cfm—10,000 cfm</td> <td data-bbox="657 493 847 577">3.0"</td> <td data-bbox="847 493 1037 577">3.5"</td> <td data-bbox="1037 493 1226 577">4.0"</td> </tr> <tr> <td data-bbox="467 577 657 661">>10,000 cfm</td> <td data-bbox="657 577 847 661">3.5"</td> <td data-bbox="847 577 1037 661">4.0"</td> <td data-bbox="1037 577 1226 661">4.5"</td> </tr> </tbody> </table> <p>An additional pressure drop allowance is available for special filtration requirements only for specific processes such as clean rooms. See process and filtration pressure drop for details.</p> <p><u>Not applicable. Defined in the software as a function of brake horsepower and supply fan efficiency.</u></p> <p>Not applicable for the four pipe fan coil system.</p>	Airflow	Single zone, six stories or less	Multiple zone, less than six stories	Multiple zone, greater than six stories	<2000 cfm	2.5"	3.0"	3.5"	2000 cfm—10,000 cfm	3.0"	3.5"	4.0"	>10,000 cfm	3.5"	4.0"	4.5"
Airflow	Single zone, six stories or less	Multiple zone, less than six stories	Multiple zone, greater than six stories														
<2000 cfm	2.5"	3.0"	3.5"														
2000 cfm—10,000 cfm	3.0"	3.5"	4.0"														
>10,000 cfm	3.5"	4.0"	4.5"														
<p><i>Standard Design: Existing Buildings</i></p>	<p><u>Not applicable</u></p>																

<p>Supply Fan Efficiency</p>	
<p><i>Applicability</i></p>	<p>All fan systems using the static pressure method</p>
<p><i>Definition</i></p>	<p>The efficiency of the fan at design conditions; this is the static efficiency and does not include motor losses</p>
<p><i>Units</i></p>	<p>Unitless</p>
<p><i>Input Restrictions</i></p>	<p>As designed</p> <p>The supply fan efficiency does not need to be specified if the supply fan brake horsepower (bhp) is specified.</p>
<p><i>Standard Design</i></p>	<p>For all standard design systems except the four pipe fan coil:</p> <p>The baseline supply fan efficiency shall be 50 percent if the design supply air flow is less than 2000 cfm; 60 percent if the design supply air flow is between 2000 cfm and 10,000 cfm; or 62 percent if the design supply airflow is greater than 10,000 cfm.</p> <p><u>65%</u></p> <p>Not applicable for the four-pipe fan coil system.</p>
<p><i>Standard Design: Existing Buildings</i></p>	<p><u>Not applicable</u></p>

Supply Motor Efficiency	
<i>Applicability</i>	All supply fans, except those specified using the power-per-unit-flow method
<i>Definition</i>	The full-load efficiency of the motor serving the supply fan
<i>Units</i>	Unitless
<i>Input Restrictions</i>	As designed Not applicable when the power-per-unit-flow method is used.
<i>Standard Design</i>	The motor efficiency is determined from Table 29 for the next motor size greater than the bhp.
<i>Standard Design: Existing Buildings</i>	<u>Same as proposed</u>

Fan Power Adjustment							
<i>Applicability</i>	<u>Any system with special requirements for filtration or other process requirements</u>						
<i>Definition</i>	<u>Additional system brake horsepower related to application-specific filtration requirements or other process requirements</u> <u>Special documentation requirements may apply.</u>						
<i>Units</i>	<u>List</u>						
<i>Input Restrictions</i>	<p>The user chooses one or more fan power adjustment credits from the list below. If the adjustment credit is specified in terms of static pressure, the static pressure adjustment is fixed. For the credits that depend on the design, the user enters the pressure drop for each device.</p> <p>The proposed design fan power adjustment (bhp) is given by the equation:</p> $FPA = \sum SP_i \times CFM_i / 4131$ <p>Where SP_i = the static pressure through the device, inches w.c. CFM_i = the design airflow through the device</p> <table border="1" style="width: 100%; border-collapse: collapse;"> <thead> <tr> <th style="width: 50%;"><u>Device</u></th> <th style="width: 50%;"><u>Adjustment Credits</u></th> </tr> </thead> <tbody> <tr> <td><u>Return of exhaust systems required by code to be fully ducted</u></td> <td><u>0.5" w.g.</u></td> </tr> <tr> <td><u>Exhaust filters, scrubbers, or other</u></td> <td><u>The pressure drop of the</u></td> </tr> </tbody> </table>	<u>Device</u>	<u>Adjustment Credits</u>	<u>Return of exhaust systems required by code to be fully ducted</u>	<u>0.5" w.g.</u>	<u>Exhaust filters, scrubbers, or other</u>	<u>The pressure drop of the</u>
<u>Device</u>	<u>Adjustment Credits</u>						
<u>Return of exhaust systems required by code to be fully ducted</u>	<u>0.5" w.g.</u>						
<u>Exhaust filters, scrubbers, or other</u>	<u>The pressure drop of the</u>						

	<u>exhaust treatment</u>	<u>proposed design</u>
	<u>Particulate filtration credit: MERV 16 or greater and electronically enhanced filters</u>	<u>Pressure drop calculated at 2 x clean filter pressure drop at fan system design condition</u>
	<u>Carbon and other gas-phase air cleaners</u>	<u>Clean filter pressure drop at fan system design condition</u>
	<u>Biosafety cabinet</u>	<u>Pressure drop of device at fan system design condition</u>
<u>Standard Design</u>	<u>Same as proposed</u>	
<u>Standard Design: Existing Buildings</u>	<u>Same as proposed for new HVAC equipment; not applicable for existing, unaltered systems.</u>	

Table 131628: Minimum Nominal Efficiency for Electric Motors (Percent)

Motor Horse Power	
1	85.5
1.5	86.5
2	86.5
3	89.5
5	89.5
7.5	91.7
10	91.7
15	92.4
20	93.0
25	93.6
30	93.6
40	94.1
50	94.5
60	95.0
75	95.4
100	95.4
125	95.4
150	95.8
200	96.2
250	96.2
300	96.2
350	96.2
400	96.2
450	96.2
500	96.2

Fan Position	
<i>Applicability</i>	All supply fans
<i>Definition</i>	The position of the supply fan relative to the cooling coil. The configuration is either draw through (fan is downstream of the coil) or blow through (fan is upstream of the coil).
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Draw through

<i>Standard Design: Existing Buildings</i>	
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Motor Position	
<i>Applicability</i>	All supply fans
<i>Definition</i>	The position of the supply fan motor relative to the cooling air stream. The choices are in the air stream or out of the air stream.
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	In the air stream
<i>Standard Design: Existing Buildings</i>	

Fan Part-Flow Power Curve									
<i>Applicability</i>	All variable flow fan systems								
<i>Definition</i>	A part-load power curve that represents the percentage full-load power draw of the supply fan as a function of the percentage full-load air flow. The curve is typically represented as a quadratic equation with an absolute minimum power draw specified.								
<i>Units</i>	Unitless ratio								
<i>Input Restrictions</i>	<p>As designed</p> <p>The user shall not be able to select VSD with static pressure reset if the building does not have DDC controls to the zone level. The default fan curve shall be selected from Equation 3 and Table 30 for the type of fan specified in the proposed design.</p> $PLR = (a) + (b \times FanRatio) + (c \times FanRatio^2) + (d \times FanRatio^3)$ $PLR = PowerMin$ <p>Where:</p> <table border="1" style="width: 100%; border-collapse: collapse;"> <tr> <td style="text-align: center;"><i>PLR</i></td> <td>Ratio of fan power at part load conditions to full load fan power</td> </tr> <tr> <td style="text-align: center;"><i>PowerMn</i></td> <td>Minimum fan power ratio</td> </tr> <tr> <td style="text-align: center;"><i>FanRatio</i></td> <td>Ratio of cfm at part-load to full-load cfm</td> </tr> <tr> <td style="text-align: center;"><i>a, b, c, and d</i></td> <td>Constants from the table below</td> </tr> </table>	<i>PLR</i>	Ratio of fan power at part load conditions to full load fan power	<i>PowerMn</i>	Minimum fan power ratio	<i>FanRatio</i>	Ratio of cfm at part-load to full-load cfm	<i>a, b, c, and d</i>	Constants from the table below
<i>PLR</i>	Ratio of fan power at part load conditions to full load fan power								
<i>PowerMn</i>	Minimum fan power ratio								
<i>FanRatio</i>	Ratio of cfm at part-load to full-load cfm								
<i>a, b, c, and d</i>	Constants from the table below								

<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Not applicable for standard design baseline building systems constant volume systems. The curve VSD with static pressure reset fans shall be used for variable volume systems.</u>
<i>Standard Design: Existing Buildings</i>	

	<i>a</i>	<i>b</i>	<i>c</i>	<i>d</i>	Percent Power _{min}
Fan Type – Control Type	0.1631	1.5901	-0.8817	0.1281	70%
AF or BI riding the curve ^a	0.9977	-0.659	0.9547	-0.2936	50%
AF or BI with inlet vanes or discharge dampers ^a	0.1224	0.612	0.5983	-0.3334	50%
FC riding the curve ^a	0.3038	-0.7608	2.2729	-0.8169	50%
FC with inlet vanes ^a	0.1639	-0.4016	1.9909	-0.7541	40%
Vane-axial with variable pitch blades ^a	0.070428852	0.385330201	-0.460864118	1.00920344	10%
Any fan with VSD	0.040759894	0.08804497	-0.07292612	0.943739823	10%
VSD with static pressure reset					

a: *Advanced VAV System Design Guide*, California Energy Commission, CEC Publication 500-03-082 A-11, 2007

Supply Fan Power Index	
<i>Applicability</i>	Fan systems that use the power-per-unit-flow method
<i>Definition</i>	The supply fan power (at the motor) per unit of flow
<i>Units</i>	W/cfm
<i>Input Restrictions</i>	As designed or specified in the manufacturers' literature; may only be used for four-pipe fan coil systems
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, For FPFC systems, 0.35 W/cfm; for heating and ventilation only systems, 0.53 W/cfm for HV systems, 0.53 W/cfm;</u> <u>For CRAC and CRAH systems, 27 W/kBtu-h of sensible cooling capacity, which approximately equals 0.81 W/cfm assuming 400 cfm/ton.</u> for other systems, not applicable.
<i>Standard Design: Existing Buildings</i>	

Process and Filtration Pressure Drop	
<i>Applicability</i>	Any system with special requirements for filtration or other process requirements
<i>Definition</i>	Additional system pressure drop related to application-specific filtration requirements or other process requirements Special documentation requirements may apply.
<i>Units</i>	List
<i>Input Restrictions</i>	As designed Default is zero. Special documentation is required to claim any credit for filtration in excess of 1 inch wc. Filtration shall be associated with process requirements (such as clean room or hospital areas).
<i>Standard Design</i>	Same as proposed but subject to a maximum of 1 inch wc
<i>Standard Design: Existing Buildings</i>	

5.7.3.3 Return/Relief Fans

The baseline standard design building has no return fan. The standard design system has a relief fan only if the standard design system has an economizer.

Plenum Zone	
<i>Applicability</i>	Any system with return ducts or return air plenum
<i>Definition</i>	A reference to the thermal zone that serves as return plenum or where the return ducts are located
<i>Units</i>	Text, unique
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

Return Air Path	
<i>Applicability</i>	Any system with return ducts or return air plenum
<i>Definition</i>	Describes the return path for air. Can be ducted return, plenum return, or direct-to-unit.
<i>Units</i>	List (see above)

<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Applicable when the standard design <u>baseline building</u> has a relief fan. For standard design <u>baseline building</u> systems 1 and 2, the return air path shall be direct-to-unit. For baseline building <u>standard design</u> systems 3 through 11, the standard design <u>baseline building</u> shall be ducted return.
<i>Standard Design: Existing Buildings</i>	

Return/Relief Fan Design Airflow	
<i>Applicability</i>	All systems with a return or relief fan
<i>Definition</i>	The design air flow fan capacity of the return or relief fan(s). This sets the 100 percent fan flow point for the part-load curve (see below).
<i>Units</i>	Cfm
<i>Input Restrictions</i>	<u>For return fans, a</u> As designed For relief fans, the return/relief fan design airflow is set equal to the proposed design outside air ventilation rate minus the proposed design exhaust fan design airflow and minus 0.05 cfm/ft² for pressurization.
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, not modeled.</u> The relief design airflow is equal to the design outside airflow minus the exhaust design air flow rate and minus 0.05 cfm/ft² for pressurization.
<i>Standard Design: Existing Buildings</i>	

Return/Relief Fan Brake Horsepower	
<i>Applicability</i>	Any system with return or relief fans that uses the brake horsepower method
<i>Definition</i>	The design shaft brake horsepower of the return/relief fan(s)
<i>Units</i>	Brake horsepower (bhp)
<i>Input Restrictions</i>	As designed The compliance software shall apply the following pre-processing rule to specify the proposed design return/relief fan brake horsepower, based on user input: A standard motor size table (hp) is defined as: 1/12, 1/8, 1/4, 1/2, 3/4, 1, 1.5, 2, 3, 5, 7.5, 10, 15, 20, 25, 30, 40, 50, 60, 75, 100, 125, 150, 200.

	<p>The return/relief fan brake horsepower is determined from user inputs of brake horsepower and motor horsepower for the proposed design, in the same manner as the supply fan brake horsepower.</p> <p>Proposed bhp = min (user bhp, 95 percent x MHP_{i-1})</p> <p>Where:</p> <p>Proposed bhp is the return/relief fan brake horsepower used in the simulation;</p> <p>User bhp is the actual fan bhp as entered by the user; and</p> <p>MHP_{i-1} is the motor horsepower of the next smaller motor size from the standard motor size table above; MHP_i is the motor size that the user enters for the return/relief fan.</p> <p>See the supply fan brake horsepower descriptor for further details.</p>
<i>Standard Design</i>	<p><u>For healthcare facilities, same as the Proposed Design. For all others, not modeled.</u></p> <p>Standard design systems with an economizer shall use relief fans and shall use the static pressure and fan efficiency method.</p>
<i>Standard Design: Existing Buildings</i>	

Return/Relief Fan Motor Horsepower	
<i>Applicability</i>	All fan systems, except those specified using the power-per-unit-flow method
<i>Definition</i>	The motor nameplate horsepower of the supply fan
<i>Units</i>	List choose from standard motor sizes: 1/12, 1/8, 1/4, 1/2, 3/4, 1, 1.5, 2, 3, 5, 7.5, 10, 15, 20, 25, 30, 40, 50, 60, 75, 100, 125, 150, 200
<i>Input Restrictions</i>	<p>As designed</p> <p>This building descriptor is required for the static pressure or the brake horsepower methods.</p>
<i>Standard Design</i>	The brake horsepower for the supply fan is this value times the supply fan ratio (see above). <u>Not modeled</u>
<i>Standard Design: Existing Buildings</i>	

Return/Relief Design Static Pressure	
<i>Applicability</i>	Any system with return or relief fans that uses the static pressure method
<i>Definition</i>	The design static pressure for return fan system.

	This is important for both fan electric energy usage and duct heat gain calculations.
<i>Units</i>	Inches of water column (in. H2O gauge)
<i>Input Restrictions</i>	As designed. The design static pressure for the return fan does not need to be specified if the return fan brake horsepower (bhp) is specified.
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, not applicable (there are no return fans, and the relief fan is not modeled.</u> For fans with design airflow less than 10,000 cfm, the static pressure is 0.75". For fans with design airflow rate 10,000 cfm or greater, the static pressure is 1.0".
<i>Standard Design: Existing Buildings</i>	

Return/Relief Fan Efficiency	
<i>Applicability</i>	Any system with return or relief fans that uses the static pressure method
<i>Definition</i>	The efficiency of the fan at design conditions. This is the static efficiency and does not include the efficiency loss of the motor.
<i>Units</i>	Unitless
<i>Input Restrictions</i>	As designed. The return/relief fan efficiency does not need to be specified if the return fan brake horsepower (bhp) is specified.
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, not applicable.</u> For design airflow less than 10,000 cfm, 40 percent. For design airflow 10,000 cfm or greater, 50 percent.
<i>Standard Design: Existing Buildings</i>	

Return/Relief Motor Efficiency	
<i>Applicability</i>	All return fans, except those specified using the power-per-unit-flow method
<i>Definition</i>	The full-load efficiency of the motor serving the supply fan
<i>Units</i>	Unitless
<i>Input Restrictions</i>	As designed. Not applicable when the power-per-unit-flow method is used.
<i>Standard Design</i>	From ACM Table 29 <u>Not applicable</u>

<i>Standard Design: Existing Buildings</i>	
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Motor Position	
<i>Applicability</i>	All return fans
<i>Definition</i>	The position of the supply fan motor relative to the cooling air stream. The choices are in the air stream or out of the air stream.
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed.
<i>Standard Design</i>	In the air stream
<i>Standard Design: Existing Buildings</i>	

Fan Part-Flow Power Curve	
<i>Applicability</i>	All return fans for variable flow fan systems.
<i>Definition</i>	A part-load power curve which represents the percentage full-load power draw of the supply fan as a function of the percentage full-load air flow.
<i>Units</i>	Unitless ratio
<i>Input Restrictions</i>	As designed. The default fan curve shall be selected from Equation 3 and Table 30 for the type of fan specified in the proposed design.
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable for baseline building systems 1-4. The curve for VSD fans shall be used for baseline building systems that have a return/relief fan.
<i>Standard Design: Existing Buildings</i>	

5.7.3.4 Exhaust Fan Systems

The standard design shall track the proposed design exempt process exhaust flow rate up to the prescribed outside air ventilation rate by space type (see Appendix 5.4A for the standard design~~baseline~~ maximum exhaust rate). Exempt process exhaust includes exhaust from toilets, break rooms, copy rooms and kitchens. Covered process exhaust includes garage ventilation, lab exhaust and exhaust from kitchens with over 5,000 cfm of exhaust. Rules for the standard design~~baseline~~ covered process exhaust rate and fan power are discussed in the following sections.

Exhaust fan flow is specified and scheduled for each thermal zone. An exhaust fan system may serve multiple thermal zones. For the standard design, total outside air ventilation supply airflow may need to be adjusted so that the design supply airflow for each floor of the building matches the total design exhaust airflow for that floor.

If an exhaust fan system is used in the proposed design as a ventilation system, then an equivalent baseline exhaust fan system is NOT defined, since the baseline has its own specification of a ventilation system that is independent of the proposed.

Exhaust Fan Name	
<i>Applicability</i>	All exhaust systems serving multiple thermal zones
<i>Definition</i>	A unique descriptor for each exhaust fan. This should be keyed to the construction documents, if possible, to facilitate plan checking. Exhaust rates and schedules at the thermal zone level refer to this name.
<i>Units</i>	Text, unique
<i>Input Restrictions</i>	Where applicable, this should match the tags that are used on the plans.
<i>Standard Design</i>	The standard design <u>baseline building will shall</u> have an exhaust system that corresponds to the proposed design. However, if the user has specified an exhaust system as the ventilation system an equivalent standard design <u>baseline</u> system will not be modeled since the standard design <u>baseline</u> has its own definition for ventilation systems. The name can be identical to that used for the proposed design or some other appropriate name may be used.
<i>Standard Design: Existing Buildings</i>	

Exhaust Fan System Modeling Method	
<i>Applicability</i>	All exhaust fan systems
<i>Definition</i>	Compliance software can model fans in three ways. See definition for supply system modeling method.
<i>Units</i>	List: power-per-unit-flow, static pressure or brake horsepower
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	The standard design <u>baseline building</u> shall use the static pressure method.
<i>Standard Design: Existing Buildings</i>	

<u>Exhaust Fan Power Ratio</u>

<u>Applicability</u>	<u>All fan systems</u>
<u>Definition</u>	<u>The standard design fan power requirements apply to all fans that operate at design conditions. To apportion the fan power to the supply fan and exhaust fans, a ratio is defined that is the the ratio of non-exempt exhaust fan power to total system fan power.</u>
<u>Units</u>	<u>List power-per-unit-flow, static pressure, or brake horsepower</u>
<u>Input Restrictions</u>	<u>As designed, not a user input.</u> <u>This is the ratio of the exhaust fan power to total system fan power, which includes: supply fans, exhaust fans, return fans, greater than 1 HP nameplate, and any series-powered fans. Return and relief fans are modeled, but are not included in calculating this ratio.</u>
<u>Standard Design</u>	<u>Same as proposed</u>
<u>Standard Design:</u> <u>Existing Buildings</u>	<u>Same as proposed</u>

Exhaust Fan Design Airflow	
<u>Applicability</u>	All exhaust fan systems
<u>Definition</u>	The rated design air flow rate of the exhaust fan system. This building descriptor defines the 100 percent flow point of the part-flow curve. Actual air flow is the sum of the flow specified for each thermal zone, as modified by the schedule for each thermal zone.
<u>Units</u>	Cfm
<u>Input Restrictions</u>	As designed. The total design exhaust flow capacity for building (conditioned space) shall not exceed the sum of building <u>story</u> minimum ventilation (outdoor) air flow. Exhaust makeup can be transferred from other zones in the building provided that the total building exhaust rate does not exceed the total minimum outside air flow rate.
<u>Standard Design</u>	<u>For healthcare facilities, same as the Proposed Design. For all others, Same as proposed design, but with the same limitations described under input restrictions. The design supply air ventilation rate for zone(s) may need to be adjusted by the software, so that the total design outside air ventilation rate supplied to all zones on a floor equals the total exhaust fan design airflow for all zones on the floor.</u>
<u>Standard Design:</u> <u>Existing Buildings</u>	

Exhaust Fan Control Method

<i>Applicability</i>	All exhaust fan systems
<i>Definition</i>	A description of how the exhaust fan(s) are controlled. The options include: <ul style="list-style-type: none"> • Constant volume • Variable-flow, variable speed drive (VSD)
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed, when exhaust fan flow at the thermal zone level is varied through a schedule, one of the variable-flow options shall be specified.
<i>Standard Design</i>	<p><u>For healthcare facilities, same as the Proposed Design. For all others,</u></p> <p>The <u>standard design</u>baseline building exhaust fan control shall be the same as the proposed design, but subject to the conditions described above.</p> <p>For exhaust fans serving kitchen spaces, the fan control method is constant volume for fans with flow rate 5,000 cfm and below, and variable flow, variable speed drive for fans with flow rate greater than 5,000 cfm.</p> <p>For exhaust fans serving laboratory spaces, the fan control method is variable-flow, variable speed drive when the <u>minimum</u> exhaust flow is 10 ACH or less. If the lab exhaust flow <u>minimum</u> is greater than 10 ACH, the control method is constant volume.</p>
<i>Standard Design: Existing Buildings</i>	

Sash Control Fraction	
<i>Applicability</i>	<u>Laboratory exhaust fan systems</u>
<i>Definition</i>	<u>A fraction that indicates the fraction of laboratory exhaust hoods that have sash controls</u>
<i>Units</i>	<u>Unitless fraction</u>
<i>Input Restrictions</i>	<u>Between 0 and 1 inclusive</u>
<i>Standard Design</i>	<u>1</u>
<i>Standard Design: Existing Buildings</i>	

Exhaust Fan Schedule	
<i>Applicability</i>	All exhaust fan systems
<i>Definition</i>	A schedule that indicates when the exhaust fan system is available for operation. Exhaust fan flow is specified at the thermal zone level.

<i>Units</i>	Data structure: schedule, on/off
<i>Input Restrictions</i>	For exhaust fans not serving kitchen and lab spaces, the schedule is fixed to match the HVAC availability schedule for the specified occupancy in Appendix 5.4B. For kitchen and lab spaces, the schedule is defined in Appendix 5.4B.
<i>Standard Design</i>	Specified in Appendix 5.4B for the specified occupancy. <u>For laboratory exhaust systems with a mix of fume hoods with and without sash controls, the laboratory exhaust schedule shall be prorated between the schedule value for no sash controls and the schedule with sash controls, adjusted by the sash control fraction.</u>
<i>Standard Design: Existing Buildings</i>	

Exhaust Fan Brake Horsepower	
<i>Applicability</i>	All exhaust fan systems
<i>Definition</i>	The design shaft brake horsepower of the exhaust fan(s).
<i>Units</i>	Brake horsepower (bhp)
<i>Input Restrictions</i>	<p>As designed</p> <p><u>The compliance software implements a pre-processing rule to specify the proposed design exhaust fan brake horsepower (bhp), based on user input:</u></p> <p><u>A standard motor size table (hp) is defined as: 1/12, 1/8, 1/4, 1/2, 3/4, 1, 1.5, 2, 3, 5, 7.5, 10, 15, 20, 25, 30, 40, 50, 60, 75, 100, 125, 150, 200</u></p> <p><u>The exhaust fan brake horsepower is determined from user inputs of brake horsepower and motor horsepower for the proposed design, in the same manner as the supply fan brake horsepower.</u></p> <p><u>Proposed bhp = max (user bhp, 95 percent x MHP_{i-1})</u></p> <p><u>Where:</u></p> <p><u>Proposed bhp is the return/relief fan brake horsepower used in the simulation.</u></p> <p><u>User bhp is the actual fan bhp as entered by the user</u></p> <p><u>MHP_{i-1} is the motor horsepower of the next smaller motor size from the standard motor size table above; MHP_i is the motor size that the user enters for the exhaust fan</u></p> <p><u>See the supply fan brake horsepower descriptor for further details.</u></p>
<i>Standard Design</i>	<p><u>For healthcare facilities, same as the Proposed Design.</u></p> <p><u>For laboratory exhaust, equivalent to 0.65 W/cfm. If the user designates that the system includes scrubbers or other air treatment devices, the</u></p>

	<p><u>standard design process exhaust fan power shall be equivalent to 0.85 W/cfm.</u></p> <p><u>For all others,</u></p> <p><u>The standard design exhaust fan brake horsepower is equal to the fan system power allowance times the exhaust fan power ratio, where the fan system power allowance is equal to:</u></p> <p><u>For constant volume systems, $(0.00094 \times \text{cfm}_{\text{max}} + A) \times \text{ExhaustFanPowerRatio}$</u></p> <p><u>For variable volume systems, $0.000795 \times \text{cfm}_{\text{max}} + A) \times \text{ExhaustFanPowerRatio}$</u></p> <p><u>For variable volume systems, $0.0013 \times \text{cfm}_{\text{max}} + A) \times \text{ExhaustFanPowerRatio}$</u></p> <p>The compliance software implements a pre-processing rule to specify the proposed design exhaust fan brake horsepower (bhp), based on user input:</p> <p>A standard motor size table (hp) is defined as: 1/12, 1/8, 1/4, 1/2, 3/4, 1, 1.5, 2, 3, 5, 7.5, 10, 15, 20, 25, 30, 40, 50, 60, 75, 100, 125, 150, 200</p> <p>The exhaust fan brake horsepower is determined from user inputs of brake horsepower and motor horsepower for the proposed design, in the same manner as the supply fan brake horsepower.</p> <p>Proposed bhp = max (user bhp, 95 percent x MHP_{i-1})</p> <p>Where:</p> <p>Proposed bhp is the return/relief fan brake horsepower used in the simulation,</p> <p>User bhp is the actual fan bhp as entered by the user</p> <p>MHP_{i-1} is the motor horsepower of the next smaller motor size from the standard motor size table above; MHP_i is the motor size that the user enters for the exhaust fan</p> <p>See the supply fan brake horsepower descriptor for further details.</p>
<p><i>Standard Design: Existing Buildings</i></p>	

<p>Exhaust Fan Motor Horsepower</p>	
<p><i>Applicability</i></p>	<p>All fan systems, except those specified using the power-per-unit-flow method</p>
<p><i>Definition</i></p>	<p>The motor nameplate horsepower of the supply fan</p>
<p><i>Units</i></p>	<p>List - choose from standard motor sizes: 1/12, 1/8, 1/4, 1/2, 3/4, 1, 1.5, 2, 3, 5, 7.5, 10, 15, 20, 25, 30, 40, 50, 60, 75, 100, 125, 150, 200</p>

<i>Input Restrictions</i>	As designed This building descriptor is required for the static pressure or the brake horsepower methods.
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

Exhaust Fan Design Static Pressure	
<i>Applicability</i>	Any system with return or relief fans that uses the static pressure method
<i>Definition</i>	The design static pressure for exhaust fan system. This is important for both fan electric energy usage and duct heat gain calculations.
<i>Units</i>	Inches of water column (in. H ₂ O)
<i>Input Restrictions</i>	As designed for exhaust fans not serving kitchens. The design static pressure for the exhaust fan does not need to be specified if the exhaust fan brake horsepower (bhp) is specified.
<i>Standard Design</i>	<p>For healthcare facilities, same as the Proposed Design. For all others,</p> <p>For kitchen exhaust fans, the static pressure is fixed at 2.5" wc. For lab exhaust, 4" if six stories or less; or 4.5" if greater than six stories.</p> <p>When the proposed design has an non-exempt exhaust fans greater than 1-HP that operates at design conditions, the standard design bhp / static pressure shall be 0.75" w.g. the same as the proposed design, up to a limit of 2.0" w.g.</p> <p>For all other exhaust fans, the standard design fan W/cfm shall be the same as the proposed design W/cfm.</p>
<i>Standard Design: Existing Buildings</i>	

Exhaust Fan Efficiency	
<i>Applicability</i>	Any exhaust fan system that uses the static pressure method
<i>Definition</i>	<p>The efficiency of the exhaust fan at rated capacity.</p> <p>This is the static efficiency and does not include losses through the motor.</p>
<i>Units</i>	Unitless
<i>Input Restrictions</i>	<p>For kitchen exhaust fans, the fan efficiency is prescribed at 50 percent.</p> <p>For all other exhaust fans, as designed. The exhaust fan efficiency does</p>

	not need to be specified if the return fan brake horsepower (bhp) is specified.
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design.</u> For all others, For kitchen exhaust fans, the fan efficiency is 50 percent, while for lab exhaust it is 62 percent. For all other exhaust fans, the standard design efficiency (and resulting W/cfm) shall be the same as the proposed design efficiency (and resulting W/cfm). <u>is 65%</u>
<i>Standard Design: Existing Buildings</i>	

Exhaust Fan Motor Efficiency	
<i>Applicability</i>	All exhaust fan systems
<i>Definition</i>	The full-load efficiency of the motor serving the exhaust fan
<i>Units</i>	Unitless
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design.</u> For all others, For exempt process fans other than lab, kitchen, and garage exhaust fans, same as proposed. For process fans, the value is taken. Otherwise, from Table 30.
<i>Standard Design: Existing Buildings</i>	

Fan Part-Flow Power Curve	
<i>Applicability</i>	All variable flow exhaust fan systems
<i>Definition</i>	A part-load power curve that represents the ratio full-load power draw of the exhaust fan as a function of the ratio full-load air flow.
<i>Units</i>	Unitless ratio
<i>Input Restrictions</i>	As designed The default fan curve shall be selected from Equation 3 and Table 30 for the type of fan specified in the proposed design.
<i>Standard Design</i>	<u>For healthcare facilities, same as the pProposed dDesign.</u> For all others, The standard design <u>baseline-building</u> fan curve shall be selected from Equation 3 and Table 30 for the type of fan specified in the proposed design.

<i>Standard Design: Existing Buildings</i>	
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Exhaust Fan Power Index	
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<i>Applicability</i>	Exhaust systems serving high-rise residential units and hotel/motel guestrooms
<i>Definition</i>	The fan power of the exhaust fan per unit of flow. This building descriptor is applicable only with the power-per-unit-flow method.
<i>Units</i>	W/cfm
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	For healthcare facilities, same as the Proposed Design. For all others, For high-rise residential units and hotel/motel guestrooms, 0.58 W/cfm
<i>Standard Design: Existing Buildings</i>	

<u>Process Exhaust Fan Name</u>	
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<u><i>Applicability</i></u>	<u>All exhaust systems serving multiple thermal zones</u>
<u><i>Definition</i></u>	<u>A unique descriptor for each exhaust fan. This should be keyed to the construction documents, if possible, to facilitate plan checking. Exhaust rates and schedules at the thermal zone level refer to this name.</u>
<u><i>Units</i></u>	<u>Text, unique</u>
<u><i>Input Restrictions</i></u>	<u>Where applicable, this should match the tags that are used on the plans.</u>
<u><i>Standard Design</i></u>	<u>The baseline building will have an exhaust system that corresponds to the proposed design. However, if the user has specified an exhaust system as the ventilation system an equivalent baseline system will not be modeled since the baseline has its own definition for ventilation systems. The name can be identical to that used for the proposed design or some other appropriate name may be used.</u>
<u><i>Standard Design: Existing Buildings</i></u>	

<u>Process Exhaust Fan System Modeling Method</u>	
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<u><i>Applicability</i></u>	<u>All exhaust fan systems</u>
<u><i>Definition</i></u>	<u>Compliance software can model fans in three ways. See definition for supply system modeling method.</u>

<u>Units</u>	<u>List: power per unit flow, static pressure or brake horsepower</u>
<u>Input Restrictions</u>	<u>Brake horsepower</u>
<u>Standard Design</u>	<u>Power per unit flow</u>
<u>Standard Design:</u> <u>Existing Buildings</u>	

<u>Process Exhaust Fan Design Airflow</u>	
<u>Applicability</u>	<u>All exhaust fan systems</u>
<u>Definition</u>	<u>The rated design air flow rate of the exhaust fan system. This building descriptor defines the 100 percent flow point of the part flow curve. Actual air flow is the sum of the flow specified for each thermal zone, as modified by the schedule for each thermal zone.</u>
<u>Units</u>	<u>Cfm</u>
<u>Input Restrictions</u>	<u>As designed.</u>
<u>Standard Design</u>	<u>Same as proposed design</u>
<u>Standard Design:</u> <u>Existing Buildings</u>	

<u>Process Exhaust Fan Control Method</u>	
<u>Applicability</u>	<u>All exhaust fan systems</u>
<u>Definition</u>	<u>A description of how the exhaust fan(s) are controlled. The options include:</u> <ul style="list-style-type: none"> — <u>Constant volume</u> — <u>Variable flow, variable speed drive (VSD)</u>
<u>Units</u>	<u>List (see above)</u>
<u>Input Restrictions</u>	<u>Constant Volume</u>
<u>Standard Design</u>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> <u>Constant Volume</u>
<u>Standard Design:</u> <u>Existing Buildings</u>	

<u>Process Exhaust Fan Schedule</u>	
<u>Applicability</u>	<u>All process exhaust fan systems</u>
<u>Definition</u>	<u>A schedule that indicates when the exhaust fan system is available for</u>

	<u>operation. Exhaust fan flow is specified at the thermal zone level.</u>
<u>Units</u>	<u>Data structure: schedule, on/off</u>
<u>Input Restrictions</u>	<u>Specified in Appendix 5.4B.</u>
<u>Standard Design</u>	<u>Specified in Appendix 5.4B</u>
<u>Standard Design:</u> <u>Existing Buildings</u>	

<u>Process Exhaust Fan Brake Horsepower</u>	
<u>Applicability</u>	<u>All exhaust fan systems</u>
<u>Definition</u>	<u>The design shaft brake horsepower of the exhaust fan(s).</u>
<u>Units</u>	<u>Brake horsepower (bhp)</u>
<u>Input Restrictions</u>	<u>As designed.</u> <u>For systems qualifying for anemometer based control or contaminant-based control, the input shall be fixed at 0.65 W/cfm.</u>
<u>Standard Design</u>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> <u>Equivalent to 0.65 W/cfm. If the user designates that the system includes scrubbers or other air treatment devices, the standard design process exhaust fan power shall be equivalent to 0.85 W/cfm.</u>
<u>Standard Design:</u> <u>Existing Buildings</u>	

<u>Process Exhaust Fan Motor Horsepower</u>	
<u>Applicability</u>	<u>All fan systems, except those specified using the power per unit flow method</u>
<u>Definition</u>	<u>The motor nameplate horsepower of the supply fan</u>
<u>Units</u>	<u>List – choose from standard motor sizes: 1/12, 1/8, 1/4, 1/2, 3/4, 1, 1.5, 2, 3, 5, 7.5, 10, 15, 20, 25, 30, 40, 50, 60, 75, 100, 125, 150, 200</u>
<u>Input Restrictions</u>	<u>As designed</u> <u>This building descriptor is required for the static pressure or the brake horsepower methods.</u>
<u>Standard Design</u>	<u>Not applicable</u>
<u>Standard Design:</u> <u>Existing Buildings</u>	

<u>Process Exhaust Fan Efficiency</u>	
<u>Applicability</u>	<u>Any process exhaust fan system that uses the static pressure method</u>
<u>Definition</u>	<u>The efficiency of the exhaust fan at rated capacity.</u> <u>This is the static efficiency and does not include losses through the motor.</u>
<u>Units</u>	<u>Unitless</u>
<u>Input Restrictions</u>	<u>As designed</u>
<u>Standard Design</u>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> <u>Not applicable</u>
<u>Standard Design:</u> <u>Existing Buildings</u>	

<u>Process Exhaust Fan Motor Efficiency</u>	
<u>Applicability</u>	<u>All exhaust fan systems</u>
<u>Definition</u>	<u>The full-load efficiency of the motor serving the exhaust fan</u>
<u>Units</u>	<u>Unitless</u>
<u>Input Restrictions</u>	<u>As designed</u>
<u>Standard Design</u>	<u>Not applicable</u>
<u>Standard Design:</u> <u>Existing Buildings</u>	

5.7.3.5 Garage Exhaust Fan Systems

When garage exhaust fan systems are modeled the fans shall be modeled as constant volume fans, with the fan power determined by whether or not the fan has CO controls.

Garage Exhaust Fan Name	
<i>Applicability</i>	All garage exhaust systems
<i>Definition</i>	A unique descriptor for each garage exhaust fan or fan system Fans with equivalent efficiency and motor efficiencies may be combined and modeled as one fan.
<i>Units</i>	Text, unique

<i>Input Restrictions</i>	Where applicable, this should match the tags that are used on the plans.
<i>Standard Design</i>	The <u>standard design</u> baseline building will <u>shall</u> have an exhaust system that corresponds to the proposed design. The name can be identical to that used for the proposed design or some other appropriate name may be used.
<i>Standard Design: Existing Buildings</i>	

Garage Exhaust Fan System Modeling Method	
<i>Applicability</i>	All exhaust fan systems
<i>Definition</i>	Software commonly models fans in three ways, see definition for supply system modeling method.
<i>Units</i>	List power-per-unit-flow, static pressure, or brake horsepower
<i>Input Restrictions</i>	Brake horsepower method (fixed value)
<i>Standard Design</i>	The <u>standard design</u> baseline building shall use the power-per-unit-flow method.
<i>Standard Design: Existing Buildings</i>	

Garage Exhaust Fan Rated Capacity	
<i>Applicability</i>	All exhaust systems
<i>Definition</i>	The rated design air flow rate of the garage exhaust fan system
<i>Units</i>	Cfm
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Same as proposed design
<i>Standard Design: Existing Buildings</i>	

Garage Exhaust Fan Control Method	
<i>Applicability</i>	All exhaust fan systems
<i>Definition</i>	The control method for the garage exhaust fan. This input determines the fan power for the exhaust fan. No other fan inputs are required.

<i>Units</i>	List constant volume or CO control
<i>Input Restrictions</i>	For systems with fan capacity below 10,000 cfm, either constant volume or CO control. For systems with fan capacity above 10,000 cfm, CO control. If constant volume is selected, proposed fan power is as designed. If CO control is selected, proposed fan power is 12.5 percent of the design fan power.
<i>Standard Design</i>	Same as proposed
<i>Standard Design: Existing Buildings</i>	

5.7.3.6 Duct Systems in Unconditioned Space

The details of these modeling rule requirements are under development.

Duct Leakage Rate	
<i>Applicability</i>	Any single-zone systems with ducts in unconditioned space serving zones of 5,000 ft ² or less
<i>Definition</i>	The leakage rate from the duct system into unconditioned space. All leakage is assumed to occur to unconditioned space (not to outdoors).
<i>Units</i>	Percentage of design airflow (%)
<i>Input Restrictions</i>	For new systems: If duct leakage testing is performed as per instructions in the Reference Appendices and certified by a Home Energy Rating System (HERS) rater or <u>Acceptance Test Technician (ATT)</u> , as designed. If not tested, 15 percent. For existing, altered systems: 15 percent if tested and verified by the HERS procedures in Reference Appendix NA2. If untested or if failed test, 20 percent.
<i>Standard Design</i>	6 percent for new construction 15 percent for existing, altered systems <u>Not applicable</u>
<i>Standard Design: Existing Buildings</i>	

Duct Location	
<i>Applicability</i>	Single zone, constant volume systems with ducts in unconditioned space, serving 5000 ft²
<i>Definition</i>	The duct location is the fraction that is in unconditioned space, in conditioned space, and outdoors

<i>Units</i>	Three unitless fractions
<i>Input Restrictions</i>	All in unconditioned space
<i>Standard Design</i>	All in unconditioned space
<i>Standard Design: Existing Buildings</i>	

Duct Leakage HERS Fan Power Adjustment	
<i>Applicability</i>	Single zone, constant volume systems with ducts in unconditioned space, serving 5000 ft ²
<i>Definition</i>	A fan power penalty or credit based on the testing performed when ducts are in unconditioned spaces
<i>Units</i>	List: Penalty, No Change Credit
<i>Input Restrictions</i>	Not a user input Penalty: if the HERS duct leakage testing isn't done when required, or if the testing fails the duct leakage rate criteria No Change: testing not required Credit: testing not required, but HERS testing performed and leakage rates are verified
<i>Standard Design</i>	All in conditioned space
<i>Standard Design: Existing Buildings</i>	

5.7.4 Outdoor Air Controls and Economizers

5.7.4.1 Outside Air Controls

Maximum Outside Air Ratio	
<i>Applicability</i>	All systems with modulating outside air dampers
<i>Definition</i>	The descriptor is used to limit the maximum amount of outside air that a system can provide as a percentage of the design supply air. It is used where the installation has a restricted intake capacity.
<i>Units</i>	Ratio
<i>Input Restrictions</i>	For systems with capacity under 54,000 Btu/h without FDD, the maximum allowed value is 0.9. For all other systems the maximum allowed value is 1.
<i>Standard Design</i>	1.0 for all systems above 54,000 Btu/h cooling capacity; 0.9 for other systems

<i>Standard Design: Existing Buildings</i>	
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Design Outside Air Flow	
<i>Applicability</i>	All systems with outside air dampers
<i>Definition</i>	The rate of outside air that needs to be delivered by the system at design conditions. This input may be derived from the sum of the design outside air flow for each of the zones served by the system.
<i>Units</i>	Cfm
<i>Input Restrictions</i>	As designed but no lower than the ventilation rate of the standard design
<i>Standard Design</i>	<p><u>For healthcare facilities, same as the Proposed Design. For all others,</u></p> <p>Minimum ventilation requirements specified by standard 120(b)2 as the greater of 15 cfm/person and the minimum ventilation rates specified in Appendix 5.4</p> <p>For systems serving laboratory spaces, the system shall be 100 percent outside air, with ventilation rates as specified by the user, but not less than 6 <u>ACH</u>1 cfm/sf.</p> <p>See ventilation control method at the zone level.</p>
<i>Standard Design: Existing Buildings</i>	

Outdoor Air Control Method	
<i>Applicability</i>	All HVAC systems that deliver outside air to zones
<i>Definition</i>	<p>The method of determining the amount of outside air that needs to be delivered by the system</p> <p>Each of the zones served by the system report their outside air requirements on an hourly basis. The options for determining the outside air at the zone level are discussed above. This control method addresses how the system responds to this information on an hourly basis. Options include:</p> <ul style="list-style-type: none"> • Average Flow - The outside air delivered by the system is the sum of the outside air requirement for each zone, without taking into account the position of the VAV damper in each zone. The assumption is that there is mixing between zones through the return air path. • Critical Zone - The critical zone is the zone with the highest ratio of outside air to supply air. The assumption is that there is no mixing between zones. This method will provide greater outside air than the average flow method because when the critical zone

	sets the outside air fraction at the system, the other zones are getting greater outside air than required.
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Average flow
<i>Standard Design: Existing Buildings</i>	

5.7.4.2 Air Side Economizers

Economizer Control Type	
<i>Applicability</i>	All systems with an air-side economizer
<i>Definition</i>	<p>An air-side economizer increases outside air ventilation during periods when refrigeration loads can be reduced from increased outside air flow. The control types include:</p> <ul style="list-style-type: none"> • No economizer. • Fixed dry-bulb. The economizer is enabled when the temperature of the outside air is equal to or lower than temperature fixed setpoint (e.g., 75°F). • Differential dry-bulb. The economizer is enabled when the temperature of the outside air is lower than the return air temperature. • Differential enthalpy. The economizer is enabled when the enthalpy of the outside air is lower than the return air enthalpy. • Differential dry-bulb and enthalpy. The system shifts to 100 percent outside air or the maximum outside air position needed to maintain the cooling SAT setpoint, when the outside air dry-bulb is less than the return air dry-bulb AND the outside air enthalpy is less than the return air enthalpy. This control option requires additional sensors. • Fixed or dry-bulb enthalpy. The economizer is enabled when the outside air dry-bulb and enthalpy are both below the fixed setpoints for the return air. • Fixed dewpoint and dry-bulb. The system shifts to 100 percent outside air, or the maximum outside air position needed to maintain the SAT setpoint, when the dewpoint of the air and dry-bulb are below the specified setpoints.
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> The control should be no economizer when the baseline standard design

	<p><u>total</u> cooling capacity < 54,000 Btu/h and when the standard design cooling system is not a computer room air handling unit (CRAH). Otherwise, the <u>standard design</u>baseline building shall assume an integrated differential dry-bulb economizer.</p> <p>An exception is that economizers shall not be modeled for systems serving high-rise residential or hotel/motel guestroom occupancies.</p>
<p><i>Standard Design:</i> <i>Existing Buildings</i></p>	

Economizer Integration Level	
<i>Applicability</i>	Airside economizers
<i>Definition</i>	<p>This input specifies whether or not the economizer is integrated with mechanical cooling. It is up to the modeling software to translate this into software-specific inputs to model this feature. The input could take the following values:</p> <ul style="list-style-type: none"> • Non-integrated - The system runs the economizer as the first stage of cooling. When the economizer is unable to meet the load, the economizer returns the outside air damper to the minimum position and the compressor turns on as the second stage of cooling. • Integrated - The system can operate with the economizer fully open to outside air and mechanical cooling active (compressor running) simultaneously, even on the lowest cooling stage.
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	List non-integrated or integrated
<i>Standard Design</i>	<u>I For healthcare facilities, same as the Proposed Design. For all others,</u> integrated for systems above capacity 54,000 Btu/h at Air-Conditioning, Heating, and Refrigeration Institute (AHRI) conditions
<p><i>Standard Design:</i> <i>Existing Buildings</i></p>	

Economizer High Temperature Lockout	
<i>Applicability</i>	Systems with fixed dry-bulb economizer
<i>Definition</i>	The outside air setpoint temperature above which the economizer will return to minimum position
<i>Units</i>	Degrees Fahrenheit (°F)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u>

	Not applicable
<i>Standard Design: Existing Buildings</i>	

Economizer Low Temperature Lockout	
<i>Applicability</i>	Systems with air-side economizers
<i>Definition</i>	A feature that permits the lockout of economizer operation (return to minimum outside air position) when the outside air temperature is below the lockout setpoint.
<i>Units</i>	Degrees Fahrenheit (F°)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Economizer High Enthalpy Lockout	
<i>Applicability</i>	Systems with differential enthalpy economizers
<i>Definition</i>	The outside air enthalpy above which the economizer will return to minimum position
<i>Units</i>	Btu/lb
<i>Input Restrictions</i>	As designed The default is 28 Btu/lb (high altitude locations may require different setpoints.) The compliance software shall apply a fixed offset and add 2 Btu/lb to the user-entered value.
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> No lockout limit
<i>Standard Design: Existing Buildings</i>	

5.7.5 Cooling Systems

5.7.5.1 General

This group of building descriptors applies to all cooling systems.

Cooling Source	
<i>Applicability</i>	All systems
<i>Definition</i>	The source of cooling for the system; either chilled water, direct expansion (DX), or other
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	The <u>standard design</u> baseline building cooling source is shown in Table 33
<i>Standard Design: Existing Buildings</i>	

Table 141729: Cooling Source for Standard Design ~~Baseline Building~~ System

<u>Standard Design</u> Baseline building System	Cooling Source
System 1 – <u>PTAC</u> SZAC	Direct expansion (DX)
System 2 – FPFC	Chilled water
System 3 – PSZ-AC	Direct expansion (DX)
System 5 – Packaged VAV with Reheat	Direct expansion (DX)
System 6 – VAV with Reheat	Chilled water
System 7 – PSZ, Single Zone VAV	Direct expansion (DX)
System 9 – Heating and Ventilation	None
System 10 – CRAH Unit for Data Centers	Chilled water
System 11 – CRAC Unit for Data Centers	Direct expansion (DX)

Gross Total Cooling Capacity	
<i>Applicability</i>	All cooling systems
<i>Definition</i>	The total gross cooling capacity (both sensible and latent) of a cooling coil or packaged DX system at AHRI conditions. The building descriptors defined in this chapter assume that the fan is modeled separately, including any heat it adds to the air stream. The cooling capacity specified by this building descriptor should not consider the heat of the fan.
<i>Units</i>	Btu/h
<i>Input Restrictions</i>	<p>NOT A USER INPUT</p> <p>For packaged equipment that has the fan motor in the air stream such that it adds heat to the cooled air, the software shall calculate the net total cooling capacity as follows:</p> $Q_{t,net,rated} = Q_{t,gross,rated} - Q_{fan,rated}$ <p>Where:</p>

	<table border="1"> <tr> <td>$Q_{t,net,rated}$</td> <td>The net total cooling capacity of a packaged unit as rated by AHRI (Btu/h)</td> </tr> <tr> <td>$Q_{t,gross,rated}$</td> <td>The AHRI rated total cooling capacity of a packaged unit (Btu/h) $Q_{fan,rated}$; the heat generated by the fan and fan motor (if fan motor is in airstream) at AHRI rated conditions</td> </tr> </table> <p>If the gross and net total cooling capacities at AHRI conditions are known, the fan heat at rated conditions is the difference between the two values. If the either the gross or net total cooling capacity is unknown, the fan heat at rated conditions shall be accounted for by using Equation 4:</p> $Q_{fan,rated} = Q_{t,gross,rated} \times 0.0415$ <p>Equation 4 is based on an AHRI rated fan power of 0.365 W/cfm, and a cooling airflow of 400 cfm/ton.</p> <p>If the number of UMLH in the proposed design exceeds 150, the software shall warn the user to resize the equipment.</p>	$Q_{t,net,rated}$	The net total cooling capacity of a packaged unit as rated by AHRI (Btu/h)	$Q_{t,gross,rated}$	The AHRI rated total cooling capacity of a packaged unit (Btu/h) $Q_{fan,rated}$; the heat generated by the fan and fan motor (if fan motor is in airstream) at AHRI rated conditions
$Q_{t,net,rated}$	The net total cooling capacity of a packaged unit as rated by AHRI (Btu/h)				
$Q_{t,gross,rated}$	The AHRI rated total cooling capacity of a packaged unit (Btu/h) $Q_{fan,rated}$; the heat generated by the fan and fan motor (if fan motor is in airstream) at AHRI rated conditions				
<i>Standard Design</i>	The gross total cooling capacity of the systems in the <u>standard design baseline building</u> is determined from the standard design net cooling capacity, and from applying the fan power rules above for adjusting for fan heat.				
<i>Standard Design: Existing Buildings</i>					

Gross Total Cooling Capacity	
<i>Applicability</i>	All cooling systems
<i>Definition</i>	<p>The gross sensible cooling capacity of the coil or packaged equipment at AHRI conditions. The building descriptors defined in this chapter assume that the fan is modeled separately, including any heat it adds to the air stream. The cooling capacity specified by this building descriptor should be adjusted to calculate the net sensible cooling capacity, which includes the effect of fan motor heat.</p> <p>The sensible heat ratio (SHR) used by some energy simulation tools can be calculated from the sensible cooling capacity and total cooling capacity:</p> <p>SHR = sensible cooling capacity/total cooling capacity</p>
<i>Units</i>	Btu/h
<i>Input Restrictions</i>	<p>As designed.</p> <p>For packaged equipment, the compliance software adjusts the user input of gross sensible cooling capacity to account for the effect of fan motor heat as follows:</p> $Q_{s,net,rated} = Q_{s,gross,rated} - Q_{fan,rated}$

	<p>Where:</p> <p>$Q_{s,net,rated}$ The AHRI rated (from manufacturers' literature) or net sensible cooling capacity of a packaged unit (Btu/h)</p> <p>$Q_{t,gross,rated}$ The AHRI rated (from manufacturers' literature) or gross sensible cooling capacity of a packaged unit (Btu/h)</p> <p>$Q_{fan,rated}$ The heat generated by the fan and fan motor (if fan motor is in air stream) at AHRI rated or hourly conditions (Btu/h). See gross total cooling capacity building descriptor.</p> <p>If the number of UMLH in the proposed design exceeds 150, the software shall warn the user to resize the equipment.</p>
<i>Standard Design</i>	The gross total cooling capacity of the systems serving the <u>standard design</u> baseline building is autosized by the compliance software, and then oversized by 15 percent. Sizing calculations shall be based on 0.5 percent design dry-bulb and mean coincident wet-bulb.
<i>Standard Design: Existing Buildings</i>	

Gross Total Cooling Capacity							
<i>Applicability</i>	All cooling systems						
<i>Definition</i>	<p>A curve that represents the available total cooling capacity as a function of cooling coil and/or condenser conditions. The common form of these curves is given as follows:</p> $Q_{t,available} = CAP_FT \times Q_{t,adj}$ <p>For air-cooled direct expansion:</p> $CAP_FT = a + b(t_{wb}) + c(t_{wb})^2 + d(t_{odb}) + e(t_{odb})^2 + f(t_{wb} \times t_{odb})$ <p>For water-cooled direct expansion:</p> $CAP_FT = a + b(t_{wb}) + c(t_{wb})^2 + d(t_{wt}) + e(t_{wt})^2 + f(t_{wb} \times t_{wt})$ <p>For chilled water coils:</p> $CAP_FT = a + b(t_{wb}) + c(t_{wb})^2 + d(t_{db}) + e(t_{db})^2 + f(t_{wb} \times t_{db})$ <p>Where:</p> <table border="1"> <tr> <td>$Q_{t,available}$</td> <td>Available cooling capacity at specified evaporator and/or condenser conditions (MBH)</td> </tr> <tr> <td>$Q_{t,adj}$</td> <td>Adjusted capacity at AHRI conditions (Btu/h)</td> </tr> <tr> <td>CAP_FT</td> <td>A multiplier to adjust $Q_{t,adj}$</td> </tr> </table>	$Q_{t,available}$	Available cooling capacity at specified evaporator and/or condenser conditions (MBH)	$Q_{t,adj}$	Adjusted capacity at AHRI conditions (Btu/h)	CAP_FT	A multiplier to adjust $Q_{t,adj}$
$Q_{t,available}$	Available cooling capacity at specified evaporator and/or condenser conditions (MBH)						
$Q_{t,adj}$	Adjusted capacity at AHRI conditions (Btu/h)						
CAP_FT	A multiplier to adjust $Q_{t,adj}$						

	<table border="1"> <tr> <td>t_{wb}</td> <td>The entering coil wet-bulb temperature (°F)</td> </tr> <tr> <td>t_{db}</td> <td>The entering coil dry-bulb temperature (°F)</td> </tr> <tr> <td>t_{wt}</td> <td>The water supply temperature (°F)</td> </tr> <tr> <td>t_{odb}</td> <td>The outside air dry-bulb temperature (°F)</td> </tr> </table> <p>Note: If an air-cooled unit employs an evaporative condenser, t_{odb} is the effective dry-bulb temperature of the air leaving the evaporative cooling unit.</p> <p>Software may represent the relationship between cooling capacity and temperature in ways other than the equations given above.</p>	t_{wb}	The entering coil wet-bulb temperature (°F)	t_{db}	The entering coil dry-bulb temperature (°F)	t_{wt}	The water supply temperature (°F)	t_{odb}	The outside air dry-bulb temperature (°F)
t_{wb}	The entering coil wet-bulb temperature (°F)								
t_{db}	The entering coil dry-bulb temperature (°F)								
t_{wt}	The water supply temperature (°F)								
t_{odb}	The outside air dry-bulb temperature (°F)								
<i>Units</i>	Data structure								
<i>Input Restrictions</i>	As designed The equations and coefficients given above are the default.								
<i>Standard Design</i>	Use the default curves or equivalent data for other models								
<i>Standard Design: Existing Buildings</i>									

Table 151830: Cooling Capacity Curve Coefficients

Coefficient	Air-Cooled Direct Expansion		Water-Cooled Direct Expansion		Chilled Water Coils	
	Air-Source (PTAC)	Air-Source (Other DX)	Water-Source (Heat Pump)	Water-Source (Other DX)	Fan-Coil	Other Chilled Water
a	1.1839345	0.8740302	-0.2780377	0.9452633	0.5038866	2.5882585
b	-0.0081087	-0.0011416	0.0248307	-0.0094199	-0.0869176	-0.2305879
c	0.0002110	0.0001711	-0.0000095	0.0002270	0.0016847	0.0038359
d	-0.0061435	-0.0029570	-0.0032731	0.0004805	0.0336304	0.1025812
e	0.0000016	0.0000102	0.0000070	-0.0000045	0.0002478	0.0005984
f	-0.0000030	-0.0000592	-0.0000272	-0.0000599	-0.0010297	-0.0028721

Note: These curves are the DOE-2.1E defaults, except for Water-Source (Other DX), which is taken from the "ECB Compliance Supplement", public review draft prepared by the SSPC 90.1 ECB Panel, Version 1.2, March 1996.

Coil Latent Modeling Method	
<i>Applicability</i>	All DX cooling systems
<i>Definition</i>	The method of modeling coil latent performance at part-load conditions
<i>Units</i>	List
<i>Input Restrictions</i>	One of the following values: Bypass factor – used by DOE-2 based programs NTU-effectiveness – used by EnergyPlus

<i>Standard Design</i>	Same as proposed
<i>Standard Design: Existing Buildings</i>	

Coil Bypass Factor	
<i>Applicability</i>	All DX cooling systems using the bypass factor coil latent modeling method
<i>Definition</i>	The ratio of air that bypasses the cooling coil at design conditions to the total system airflow
<i>Units</i>	Ratio
<i>Input Restrictions</i>	Prescribed values as shown in Table 33.
<i>Standard Design</i>	Defaults
<i>Standard Design: Existing Buildings</i>	

Table 161931: Default Coil Bypass Factors

System Type	Default Bypass Factor
Packaged Terminal Air Conditioner and Heat Pumps	0.241
Other Packaged Equipment	0.190
Multi-Zone Systems	0.078
All Other	0.037

Coil Bypass Factor Adjustment Curve	
<i>Applicability</i>	All DX cooling systems using the bypass factor coil latent modeling method
<i>Definition</i>	Adjustments for the amount of coil bypass due to the following factors: <ul style="list-style-type: none"> • Coil airflow rate as a percentage of rated system airflow • Entering air wet-bulb temperature • Entering air dry-bulb temperature • Part load ratio
<i>Units</i>	Data structure
<i>Input Restrictions</i>	Where applicable, prescribed (fixed) simulation engine defaults based on HVAC system type. The following default values shall be used for the adjustment curves:

	$CBF_{adj} = CBF_{rated}(Coil_{BF,Flow})(Coil_{BF,FT})(Coil_{BF,FPLR})$ $Coil_{BF,Flow} = a + b(CFMR) + c(CFMR)^2 + d(CFMR)^3$ $Coil_{BF,FT} = a + b(T_{wb}) + c(T_{wb})^2 + d(T_{db}) + e(T_{db})^2 + f(T_{wb})(T_{db})$ $Coil_{BF,FPLR} = a + b(PLR)$ <p>Where:</p> <table border="1"> <tr> <td data-bbox="467 453 646 506">CBF_{rated}</td> <td data-bbox="646 453 1419 506">The coil bypass factor at AHRI rating conditions</td> </tr> <tr> <td data-bbox="467 506 646 585">CBF_{adj}</td> <td data-bbox="646 506 1419 585">The coil bypass factor adjusted for airflow and coil conditions</td> </tr> <tr> <td data-bbox="467 585 646 642">$CFMR$</td> <td data-bbox="646 585 1419 642">The ratio of airflow to design airflow</td> </tr> <tr> <td data-bbox="467 642 646 758">$Coil_{BF,Flow}$</td> <td data-bbox="646 642 1419 758">A multiplier on the rated coil bypass factor to account for variation in air flow across the coil (take coefficients from Table 34)</td> </tr> <tr> <td data-bbox="467 758 646 873">$Coil_{F,FT}$</td> <td data-bbox="646 758 1419 873">A multiplier on the rated coil bypass factor to account for a variation in coil entering conditions (take coefficients from Table 35)</td> </tr> <tr> <td data-bbox="467 873 646 961">$Coil_{BF,FPLR}$</td> <td data-bbox="646 873 1419 961">A multiplier on the rated coil bypass factor to account for the part load ratio (take coefficients from Table 36)</td> </tr> <tr> <td data-bbox="467 961 646 1014">T_{wb}</td> <td data-bbox="646 961 1419 1014">The entering coil wet-bulb temperature (F)</td> </tr> <tr> <td data-bbox="467 1014 646 1066">T_{db}</td> <td data-bbox="646 1014 1419 1066">The entering coil dry-bulb temperature (F)</td> </tr> <tr> <td data-bbox="467 1066 646 1115">PLR</td> <td data-bbox="646 1066 1419 1115">Part load ratio</td> </tr> </table>	CBF_{rated}	The coil bypass factor at AHRI rating conditions	CBF_{adj}	The coil bypass factor adjusted for airflow and coil conditions	$CFMR$	The ratio of airflow to design airflow	$Coil_{BF,Flow}$	A multiplier on the rated coil bypass factor to account for variation in air flow across the coil (take coefficients from Table 34)	$Coil_{F,FT}$	A multiplier on the rated coil bypass factor to account for a variation in coil entering conditions (take coefficients from Table 35)	$Coil_{BF,FPLR}$	A multiplier on the rated coil bypass factor to account for the part load ratio (take coefficients from Table 36)	T_{wb}	The entering coil wet-bulb temperature (F)	T_{db}	The entering coil dry-bulb temperature (F)	PLR	Part load ratio
CBF_{rated}	The coil bypass factor at AHRI rating conditions																		
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$Coil_{BF,FPLR}$	A multiplier on the rated coil bypass factor to account for the part load ratio (take coefficients from Table 36)																		
T_{wb}	The entering coil wet-bulb temperature (F)																		
T_{db}	The entering coil dry-bulb temperature (F)																		
PLR	Part load ratio																		
<i>Standard Design</i>	Defaults																		
<i>Standard Design: Existing Buildings</i>																			

Table 172032: Coil Bypass Factor Airflow Adjustment Factor

Coefficient	COIL-BF-FFLOW (PTAC)	COIL-BF-FFLOW (HP)	COIL-BF-FFLOW (PSZ/other)
a	-2.277	-0.8281602	-0.2542341
b	5.21140	14.3179150	1.2182558
c	-1.93440	-21.8894405	0.0359784
d		9.3996897	

Table 182433: Coil Bypass Factor Temperature Adjustment Factor

Coefficient	COIL-BF-FT (PTAC)	COIL-BF-FT (HP)	COIL-BF-FT (PSZ, other)
a	-1.5713691	-29.9391098	1.0660053
b	0.0469633	0.8753455	-0.0005170
c	0.0003125	-0.0057055	0.0000567
d	-0.0065347	0.1614450	-0.0129181
e	0.0001105	0.0002907	-0.0000017
f	-0.0003719	-0.0031523	0.0001503

Table 192234: Coil Bypass Factor Part Load Adjustment Factor

Coefficient	COIL-BF-FPLR (All Systems)
a	0.00
b	1.00

Cooling Capacity Airflow Adjustment Curve							
<i>Applicability</i>	All DX cooling systems using the NTU effectiveness coil latent modeling method						
<i>Definition</i>	Normalized curve that varies cooling capacity as a function of airflow, which affects system latent capacity						
<i>Units</i>	Data structure						
<i>Input Restrictions</i>	<p>Where applicable, prescribed (fixed) simulation engine defaults based on HVAC system type. The following default values shall be used for the adjustment curves:</p> $CoolCap_{adj} = CoolCap_{rated} \times CoolCap_{FFLOW} \times CoolCap_{FT}$ <p>Where:</p> <table border="1"> <tbody> <tr> <td>$CFMR$</td> <td>The ratio of airflow to design airflow</td> </tr> <tr> <td>$CoolCap_{FFLOW}$</td> <td>A multiplier on the rated coil capacity to account for variation in air flow across the coil (take coefficients from Table 33)</td> </tr> <tr> <td>$CoolCap_{FT}$</td> <td>A multiplier on the rated coil bypass factor to account for a variation in coil entering conditions (take coefficients from Table 34)</td> </tr> </tbody> </table> <p>The curve takes the form:</p> $CoolCap_{FFLOW} = a + b(CFMR) + c(CFMR)^2 + d(CFR)^3$ <p>With the following coefficients:</p>	$CFMR$	The ratio of airflow to design airflow	$CoolCap_{FFLOW}$	A multiplier on the rated coil capacity to account for variation in air flow across the coil (take coefficients from Table 33)	$CoolCap_{FT}$	A multiplier on the rated coil bypass factor to account for a variation in coil entering conditions (take coefficients from Table 34)
$CFMR$	The ratio of airflow to design airflow						
$CoolCap_{FFLOW}$	A multiplier on the rated coil capacity to account for variation in air flow across the coil (take coefficients from Table 33)						
$CoolCap_{FT}$	A multiplier on the rated coil bypass factor to account for a variation in coil entering conditions (take coefficients from Table 34)						

	<table border="1"> <tr> <td>Coefficient</td> <td>$CoolCap_{FELLOW}$</td> </tr> <tr> <td>a</td> <td>0.47278589</td> </tr> <tr> <td>b</td> <td>1.2433415</td> </tr> <tr> <td>c</td> <td>-1.0387055</td> </tr> <tr> <td>d</td> <td>0.32257813</td> </tr> </table>	Coefficient	$CoolCap_{FELLOW}$	a	0.47278589	b	1.2433415	c	-1.0387055	d	0.32257813
Coefficient	$CoolCap_{FELLOW}$										
a	0.47278589										
b	1.2433415										
c	-1.0387055										
d	0.32257813										
<i>Standard Design</i>	Use defaults as described above										
<i>Standard Design: Existing Buildings</i>											

5.7.5.2 Direct Expansion

Direct Expansion Cooling Efficiency									
<i>Applicability</i>	Packaged DX equipment								
<i>Definition</i>	<p>The cooling efficiency of a direct expansion (DX) cooling system at AHRI rated conditions as a ratio of output over input in Btu/h per W, excluding fan energy.</p> <p>The abbreviation used for this full-load efficiency is Energy Efficiency Ratio (EER).</p> <p>For all unitary and applied equipment where the fan energy is part of the equipment efficiency rating, the EER shall be adjusted as follows:</p> $EER_{adj} = \frac{Q_{t,net,rated} + Q_{fan,rated}}{\frac{Q_{t,net,rated}}{EER} - \frac{Q_{fan,rated}}{3.413}}$ <p>Where:</p> <table border="1"> <tr> <td>EER_{adj}</td> <td>The adjusted EER for simulation purposes</td> </tr> <tr> <td>EER</td> <td>The rated EER</td> </tr> <tr> <td>$Q_{t,net,rated}$</td> <td>The AHRI rated total gross cooling capacity of a packaged unit (kBtu/h)</td> </tr> <tr> <td>$Q_{fan,rated}$</td> <td>The AHRI rated fan energy, specified in Equation 7 for the gross total cooling capacity building descriptor</td> </tr> </table>	EER_{adj}	The adjusted EER for simulation purposes	EER	The rated EER	$Q_{t,net,rated}$	The AHRI rated total gross cooling capacity of a packaged unit (kBtu/h)	$Q_{fan,rated}$	The AHRI rated fan energy, specified in Equation 7 for the gross total cooling capacity building descriptor
EER_{adj}	The adjusted EER for simulation purposes								
EER	The rated EER								
$Q_{t,net,rated}$	The AHRI rated total gross cooling capacity of a packaged unit (kBtu/h)								
$Q_{fan,rated}$	The AHRI rated fan energy, specified in Equation 7 for the gross total cooling capacity building descriptor								
<i>Units</i>	Btu/h-W								
<i>Input Restrictions</i>	As designed, except that the user-entered value must meet mandatory minimum requirements of Table 110.2-A, Table 110.2-B, or Table 110.2-C, or Table 110.2-E for the applicable equipment type. When possible,								

	<p>specify the SEER and EER for packaged equipment with cooling capacity less than 65,000 Btu/h from manufacturer’s literature. For equipment with capacity above 65,000 Btu/h, specify EER.</p> <p><u>Equipment subject to minimum efficiency requirements under the Appliance Efficiency Regulations (small packaged air conditioners and heat pumps with SEER ratings) must meet the applicable mandatory minimum efficiency requirements.</u></p> <p>When EER is not available for packaged equipment with SEER ratings (AHRI cooling capacity of 65,000 Btu/h or smaller), it shall be calculated as follows:</p> $EER = MIN(-0.0194 \times SEER^2 + 1.0864 \times SEER, 13)$ <p>The default EER shall be calculated by the equation above, but constrained to be no greater than 13.</p> <p>Evaporative cooling systems that pass the requirements of the Western Cooling Challenge may be modeled with an EER as if the equipment were packaged unitary equipment. See Section 5.7.5.4.</p>
<i>Standard Design</i>	Use the minimum cooling efficiency (EER) from tables in Tables 110.2-A, 110.2-B, and 110.2-E in Section 110.2 of the standards.
<i>Standard Design: Existing Buildings</i>	

Seasonal Energy Efficiency Ratio	
<i>Applicability</i>	Packaged DX equipment with AHRI cooling capacity of 65,000 Btu/h or smaller
<i>Definition</i>	The seasonal cooling efficiency of a direct expansion (DX) cooling system at AHRI rated conditions as a ratio of output over input in Btu/h per W, excluding fan energy. The software must accommodate user input in terms of either the EER or the SEER. For equipment with SEER ratings, EER shall be taken from manufacturers’ data when it is available.
<i>Units</i>	Btu/h-W
<i>Input Restrictions</i>	As designed This input is required for small DX systems. The Direct Expansion Cooling Efficiency input is optional for these systems.
<i>Standard Design</i>	Use the minimum SEER from the 2015 Appliance Efficiency Standards.
<i>Standard Design: Existing Buildings</i>	

Integrated Energy Efficiency Ratio

<i>Applicability</i>	Packaged DX equipment with AHRI cooling capacity of 65,000 Btu/h or greater
<i>Definition</i>	<p>Integrated Energy Efficiency Ratio</p> <p>This is a SEER that is a composite rating for a range of part-load conditions and different ambient conditions. The rating is determined according to AHRI procedures. Equipment with this rating is subject to mandatory minimum requirements.</p> <p>This input is currently only used for mandatory minimum efficiency checks.</p>
<i>Units</i>	Btu/h-W
<i>Input Restrictions</i>	<p>As designed</p> <p>If the IEER rating is below mandatory minimum required levels specified in Section 110.2 of the standards, the compliance run shall fail.</p>
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

Direct Expansion Cooling Efficiency Temperature Adjustment Curve											
<i>Applicability</i>	Packaged DX equipment										
<i>Definition</i>	<p>A curve that varies the cooling efficiency of a direct expansion (DX) coil as a function of evaporator conditions, condenser conditions, and for small packaged equipment, part-load ratio.</p> <p>For air-cooled DX systems:</p> $EIR_{FT} = a + b(t_{wb}) + c(t_{wb})^2 + d(t_{odb}) + e(t_{odb})^2 + f(t_{wb})(t_{odb})$ <p>For water-cooled DX systems:</p> $EIR_{FT} = a + b(t_{wb}) + c(t_{wb})^2 + d(t_{wt}) + e(t_{wt})^2 + f(t_{wb})(t_{wt})$ $P_{operating} = P_{rated}(EIR_{FPLR})(EIR_{FT})(CAP_{FT})$ <p>Where:</p> <table border="1"> <tr> <td>EIR_{FPLR}</td> <td>Part-load ratio based on available capacity (not rated capacity)</td> </tr> <tr> <td>EIR_{FT}</td> <td>A multiplier on the EIR to account for the wet-bulb temperature entering the coil and the outdoor dry-bulb temperature</td> </tr> <tr> <td>t_{wb}</td> <td>The entering coil wet-bulb temperature (F)</td> </tr> <tr> <td>t_{wt}</td> <td>The water supply temperature (F)</td> </tr> <tr> <td>t_{odb}</td> <td>The outside-air dry-bulb temperature (F)</td> </tr> </table>	EIR_{FPLR}	Part-load ratio based on available capacity (not rated capacity)	EIR_{FT}	A multiplier on the EIR to account for the wet-bulb temperature entering the coil and the outdoor dry-bulb temperature	t_{wb}	The entering coil wet-bulb temperature (F)	t_{wt}	The water supply temperature (F)	t_{odb}	The outside-air dry-bulb temperature (F)
EIR_{FPLR}	Part-load ratio based on available capacity (not rated capacity)										
EIR_{FT}	A multiplier on the EIR to account for the wet-bulb temperature entering the coil and the outdoor dry-bulb temperature										
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t_{wt}	The water supply temperature (F)										
t_{odb}	The outside-air dry-bulb temperature (F)										

	<table border="1"> <tr> <td>P_{rated}</td> <td>Rated power draw at AHRI conditions (kW)</td> </tr> <tr> <td>$P_{operating}$</td> <td>Power draw at specified operating conditions (kW)</td> </tr> </table>	P_{rated}	Rated power draw at AHRI conditions (kW)	$P_{operating}$	Power draw at specified operating conditions (kW)
P_{rated}	Rated power draw at AHRI conditions (kW)				
$P_{operating}$	Power draw at specified operating conditions (kW)				
<i>Units</i>	Data structure				
<i>Input Restrictions</i>	<p>User may input curves or use default curves. If defaults are overridden, the software must indicate that supporting documentation is required on the output forms.</p> <p>For DX equipment with a capacity greater than 65,000 Btu/h, the user may not enter data on the temperature dependent equipment performance. However, the ACM compliance software vendor may work with manufacturers to collect such data and build this data into the ACM compliance software. The user may either select equipment for which the ACM compliance software vendor has collected or use the defaults.</p>				
<i>Standard Design</i>	For all systems except packaged DX units with cooling capacity \leq 65,000 Btu/h, use default curves from Appendix 5.7. For packaged DX units with cooling capacity less than or equal to 65,000 Btu/h that have SEER ratings, the user inputs EER and SEER. The software generates the equipment performance curve based on the pre-defined performance curves specified in Appendix 5.7.				
<i>Standard Design: Existing Buildings</i>					

Table 202335: Cooling System Coefficients for EIR-FT

Coefficient	Water-Source (Heat Pump)	Water-Source (Other)	Air-Source (PTAC)	Air-Source (PSZ with Cool Cap \leq 65,000 Btu/h)	Air-Source (Other)
a	2.0280385	-1.8394760	-0.6550461	n/a (see Standard Design)	-1.0639310
b	-0.0423091	0.0751363	0.0388910	n/a	0.0306584
c	0.0003054	-0.0005686	-0.0001925	n/a	-0.0001269
d	0.0149672	0.0047090	0.0013046	n/a	0.0154213
e	0.0000244	0.0000901	0.0001352	n/a	0.0000497
f	-0.0001640	-0.0001218	-0.0002247	n/a	-0.0002096

Direct Expansion Part-Load Efficiency Adjustment Curve															
<i>Applicability</i>	Packaged systems with DX cooling														
<i>Definition</i>	<p>A normalized performance adjustment curve to the rated efficiency (energy input ratio) that describes how the efficiency varies at part-load conditions. At a value of 1 (full load), the normalized efficiency is 1 (same as part-load conditions).</p> <p>The default curves are given as follows as adjustments to the energy input ratio (EIR) :</p> $PLR = \frac{Q_{operating}}{Q_{available}(t_{wb}, t_{odb}/wt)}$ $EIR_{FPLR} = a + b(PLR) + c(PLR)^2 + d(PLR)^3$ $PLF_{FPLR} = a + b(PLR) + c(PLR)^2 + d(PLR)^3$ <p>This curve may take the form of a part-load factor (PLF) or EIR-FLPR, which is the fraction of time that the unit must run to meet the part-load for that hour. For example, at 40 percent of full load, the equipment might need to run 50 percent of the hour for cycling losses.</p> <p>Note: For small packaged equipment with SEER ratings <65,000 Btu/h, the part-load efficiency curve is set to no degradation, since the part-load degradation is built into the direct expansion cooling efficiency temperature adjustment curve.</p> <p>Default curves are provided for the different major classes of equipment. Where:</p> <table border="1" style="width: 100%; border-collapse: collapse;"> <tbody> <tr> <td style="text-align: center;">EIR_{FPLR}</td> <td>Part-load ratio based on available capacity (not rated capacity)</td> </tr> <tr> <td style="text-align: center;">PLF_{FPLR}</td> <td>A multiplier on the EIR to account for the wet-bulb temperature entering the coil and the outdoor dry-bulb temperature</td> </tr> <tr> <td style="text-align: center;">t_{wb}</td> <td>The entering coil wet-bulb temperature (°F)</td> </tr> <tr> <td style="text-align: center;">t_{wt}</td> <td>The water supply temperature (°F)</td> </tr> <tr> <td style="text-align: center;">t_{odb}</td> <td>The outside-air dry-bulb temperature (°F)</td> </tr> <tr> <td style="text-align: center;">$Q_{operating}$</td> <td>Present load on heat pump (Btu/h)</td> </tr> <tr> <td style="text-align: center;">$Q_{available}$</td> <td>Heat pump available capacity at present evaporator and condenser conditions (Btu/h)</td> </tr> </tbody> </table>	EIR_{FPLR}	Part-load ratio based on available capacity (not rated capacity)	PLF_{FPLR}	A multiplier on the EIR to account for the wet-bulb temperature entering the coil and the outdoor dry-bulb temperature	t_{wb}	The entering coil wet-bulb temperature (°F)	t_{wt}	The water supply temperature (°F)	t_{odb}	The outside-air dry-bulb temperature (°F)	$Q_{operating}$	Present load on heat pump (Btu/h)	$Q_{available}$	Heat pump available capacity at present evaporator and condenser conditions (Btu/h)
EIR_{FPLR}	Part-load ratio based on available capacity (not rated capacity)														
PLF_{FPLR}	A multiplier on the EIR to account for the wet-bulb temperature entering the coil and the outdoor dry-bulb temperature														
t_{wb}	The entering coil wet-bulb temperature (°F)														
t_{wt}	The water supply temperature (°F)														
t_{odb}	The outside-air dry-bulb temperature (°F)														
$Q_{operating}$	Present load on heat pump (Btu/h)														
$Q_{available}$	Heat pump available capacity at present evaporator and condenser conditions (Btu/h)														
<i>Units</i>	Coefficients (three for a quadratic, or up to four for a cubic)														
<i>Input Restrictions</i>	The coefficients should sum to 1 (within a small tolerance). This corresponds to a curve output of 1 for an input of 1.														

<i>Standard Design</i>	The <u>standard design</u> baseline part-load efficiency adjustment curves are shown in the tables below.
<i>Standard Design: Existing Buildings</i>	

Table 212436: Cooling System Coefficients for EIR-FPLR

Coefficient	Water-Source (Heat Pump)	Water-Source (Other)	Air-Source (PTAC)	Air-Source (PSZ with Cap<65,000 Btu/h)	Air-Source (Other)
a	0.1250000	0.2012301	0.1250000	0	0.2012301
b	0.8750000	-0.0312175	0.8750000	1	-0.0312175
c	0.0000000	1.9504979	0.0000000	0	1.9504979
d	0.0000000	-1.1205105	0.0000000	0	-1.1205105

Table 222537: Cooling System Coefficients for Part-Load Factor (PLF) Correlation (EnergyPlus)

Coefficient	Water-Source (Heat Pump)	Water-Source (Other)	Air-Source (PTAC)	Air-Source (PSZ with Cap<65,000 Btu/h)	Air-Source (Other)
a	0.85	0	0.85	1	0
b	0.15	5.1091	0.15	0	5.1091
c	0	-8.5515	0	0	-8.5515
d	0	4.4744	0	0	4.4744

Number of Cooling Stages	
<i>Applicability</i>	Single zone VAV systems and DX systems with multiple stages
<i>Definition</i>	This applies to single zone VAV and any HVAC systems with multiple compressors or multiple discrete stages of cooling. This system is a packaged unit with multiple compressors and a two-speed or variable-speed fan. Systems with unequally sized compressors may have additional cooling stages.
<i>Units</i>	None (Integer)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	The standard design shall be two for the single zone VAV baseline and packaged VAV baseline.
<i>Standard Design: Existing Buildings</i>	

Total Cooling Capacity by Stage	
<i>Applicability</i>	Single zone VAV systems and DX systems with multiple stages

<i>Definition</i>	This provides the total cooling capacity of each cooling stage, at AHRI rated conditions. The capacity is expressed as an array, with each entry a fraction of the total rated cooling capacity for the unit. For example, if the stage cooling capacity is 4 tons (48,000 Btu/h) and the total cooling capacity is 8 tons (96,000 Btu/h), the capacity is expressed as "0.50" for that stage.
<i>Units</i>	Array of fractions
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> The default shall be (0.50,1) for the single zone VAV baseline.
<i>Standard Design: Existing Buildings</i>	

Sensible Cooling Capacity by Stage	
<i>Applicability</i>	Single zone VAV systems and DX systems with multiple stages
<i>Definition</i>	This provides the sensible cooling capacity of each cooling stage, at AHRI rated conditions. The capacity is expressed as an array, with each entry a fraction of the total rated sensible cooling capacity for the unit. For example, if the stage sensible cooling capacity is 3.5 <u>4.5</u> tons (42,000 000 <u>54,000</u> Btu/h) and the total sensible cooling capacity is 7 tons (72,000 Btu/h), the capacity is expressed as "0.5" for that stage."
<i>Units</i>	Array of fractions
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> The default shall be (0.50, 1) for the single zone VAV baseline.
<i>Standard Design: Existing Buildings</i>	

Supply Air Temperature Reset by Stage	
<i>Applicability</i>	Single zone VAV systems with supply air temperature control method set to staged control
<i>Definition</i>	This provides the cooling supply air temperature setpoint deviation from the cooling design supply air temperature, specified in the building descriptor cooling supply air temperature. The temperature reset is expressed as an array, with each entry a deviation from the design supply air temperature. For example, an entry of "5" for a stage would indicate a 5°F reset (for example, 60°F from 55°F).

<i>Units</i>	Array of temperature differences, in °F
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

Number of Heating Stages	
<i>Applicability</i>	Single zone VAV systems and DX systems with multiple stages
<i>Definition</i>	The number of heating stages provided by the system. Multiple stages could be provided via a heat pump or via a multiple-stage gas furnace.
<i>Units</i>	Integer
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> 1
<i>Standard Design: Existing Buildings</i>	

Heating Capacity by Stage	
<i>Applicability</i>	Single zone VAV systems and DX systems with multiple stages
<i>Definition</i>	This provides the total heating capacity of each heating stage, at AHRI rated conditions. The capacity is expressed as an array, with each entry a fraction of the total rated cooling capacity for the unit. For example, if the stage heating capacity is 48,000 Btu/h and the heating capacity is 96,000 Btu/h, the capacity is expressed as “0.50” for that stage.
<i>Units</i>	Array of fractions
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Heating Supply Air Temperature by Stage	
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<i>Applicability</i>	Single zone VAV systems and DX systems with multiple stages
<i>Definition</i>	<p>This provides the heating supply air temperature setpoint deviation from the design heating supply air temperature, specified in the building descriptor heating supply air temperature.</p> <p>The temperature reset is expressed as an array, with each entry a deviation from the design supply air temperature. For example, an entry of “-10” for a stage would indicate a 10°F reset (for example, 95°F from 105°F).</p>
<i>Units</i>	Array of temperature differences, in °F
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Supply Fan Low Speed Ratio	
<i>Applicability</i>	Single zone VAV systems and DX systems with multiple stages and two-speed fans
<i>Definition</i>	This specifies the low fan speed setting on a single zone VAV system or DX system with multiple cooling stages.
<i>Units</i>	None (fraction)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> The default shall be the greater of 0.50 or the minimum outside air fraction for the single zone VAV baseline.
<i>Standard Design: Existing Buildings</i>	

Supply Fan Low Power Ratio	
<i>Applicability</i>	Single zone VAV systems and DX systems with multiple stages and two-speed fans
<i>Definition</i>	This specifies the fraction of full load fan power corresponding to low fan speed operation on a single zone VAV system or DX system with multiple cooling stages.
<i>Units</i>	None (fraction)
<i>Input Restrictions</i>	As designed

<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> The default shall be 0.30, or the minimum power ratio calculated by applying the minimum outside air fraction to the “any fan with VSD” curve, for the single zone VAV baseline.
<i>Standard Design: Existing Buildings</i>	

Minimum Unloading Ratio	
<i>Applicability</i>	Packaged DX systems
<i>Definition</i>	The fraction of total cooling capacity below which compressor(s) must cycle on and off to meet the cooling load. Below the minimum unloading ratio, part-load efficiency is reduced. Also, below the minimum unloading ratio, the economizer will not operate in a fully integrated mode with compressor cooling.
<i>Units</i>	Ratio
<i>Input Restrictions</i>	As designed The user must enter this descriptor for each DX cooling system. If hot-gas bypass is not employed, a value of 0 may be entered.
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> 0.25 for units with a peak total cooling capacity greater than or equal to 240 kBtu/h; 0.35 for units with a peak cooling capacity greater than or equal to 65 kBtu/h and less than 240 kBtu/h
<i>Standard Design: Existing Buildings</i>	

Minimum Hot-Gas Bypass Ratio	
<i>Applicability</i>	Packaged systems which use hot-gas bypass during low load conditions
<i>Definition</i>	The lower end of the hot-gas bypass operating range. The percentage of peak cooling capacity below which hot-gas bypass will no longer operate (i.e., the compressor will cycle).
<i>Units</i>	Ratio
<i>Input Restrictions</i>	0
<i>Standard Design</i>	0
<i>Standard Design: Existing Buildings</i>	

Condenser Type	
<i>Applicability</i>	All direct expansion systems including heat pumps
<i>Definition</i>	The type of condenser for a DX cooling system The choices are: <ul style="list-style-type: none"> • Air-cooled • Water-cooled • Air-cooled with evaporative pre-cooler
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Based on the prescribed system type Refer to the HVAC System Map in section 5.7.1. Air-cooled for systems 1 (PTAGSZAC), 3, (PSZ), 5 (PVAV) and 11 (CRAC). Not applicable for other standard design systems.
<i>Standard Design: Existing Buildings</i>	

Condenser Flow Type	
<i>Applicability</i>	All direct expansion systems including heat pumps
<i>Definition</i>	Describes water flow control for a water-cooled condenser. The choices are: <ul style="list-style-type: none"> • Fixed flow • Two-position • Variable flow
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	Default to fixed flow If the variable-flow is selected, the software must indicate that supporting documentation is required on the output forms.
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Two-position
<i>Standard Design: Existing Buildings</i>	

Supplementary DX Cooling Unit	
<i>Applicability</i>	Required when user-defined natural ventilation or evaporative cooling

	systems have excessive unmet load hours
<i>Definition</i>	The specification of a supplementary DX cooling system that must be used when the user-defined cooling system results in unmet load hours exceeding 150 for any zone.
<i>Units</i>	List
<i>Input Restrictions</i>	<p>The user shall input the following system characteristics, with one unit for each zone:</p> <p>Total cooling capacity (Btu/h)</p> <p>Total heating capacity (Btu/h)</p> <p>The compliance software shall define the following prescribed system characteristics:</p> <p>Efficiency - minimum efficiency from Table 110.2-A, based on cooling capacity</p> <p>System airflow - 350 cfm/ton cooling</p> <p>Economizer - none</p> <p>Design supply air temperature - 55°F</p> <p>Supply air temperature control - fixed</p> <p>Design heating supply air temperature - 105°F</p>
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

5.7.5.3 Evaporative Cooler

This is equipment that pre-cools the outside air that is brought into the building. It may be used with any type of cooling system that brings in outside air. This equipment is not applicable for the standard design ~~baseline~~ building.

Evaporative Cooling Type	
<i>Applicability</i>	Systems with evaporative cooling
<i>Definition</i>	<p>The type of evaporative pre-cooler, including:</p> <ul style="list-style-type: none"> • None • Non-integrated direct • Non-integrated indirect • Non-integrated direct/indirect • Integrated direct • Integrated indirect • Integrated direct/indirect

	An integrated cooler can operate together with compressor or CHW cooling. A non-integrated cooler will shut down the evaporative cooling whenever it is unable to provide 100 percent of the cooling required. Direct evaporative cooling can only be applied to the outside air. Indirect evaporative cooling can be applied to outside air or return air.
<i>Units</i>	None
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Evaporative Cooling System Capacity	
<i>Applicability</i>	Systems with evaporative cooling
<i>Definition</i>	The total sensible cooling capacity of the evaporative cooling system at design outdoor dry-bulb conditions. This value may be derived from other inputs of supply fan design air rated capacity (Section 5.7.3), direct stage effectiveness, indirect stage effectiveness, and design outdoor conditions.
<i>Units</i>	None
<i>Input Restrictions</i>	Not applicable Derived input. If there are excessive unmet load hours in any zone served by the evaporative cooling system, a supplementary DX cooling unit must be defined by the user. See Section 5.7.5.2.
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Direct Stage Effectiveness	
<i>Applicability</i>	Systems with evaporative cooling
<i>Definition</i>	The effectiveness of the direct stage of an evaporative cooling system. Effectiveness is defined as: $Direct_{EFF} = \frac{T_{db} + T_{direct}}{T_{db} - T_{wb}}$ Where:

	<table border="1"> <tr> <td>$Direct_{EFF}$</td> <td>The direct stage effectiveness</td> </tr> <tr> <td>T_{db}</td> <td>The entering air dry-bulb temperature</td> </tr> <tr> <td>T_{wb}</td> <td>The entering air wet-bulb temperature</td> </tr> <tr> <td>T_{direct}</td> <td>The direct stage leaving dry-bulb temperature</td> </tr> </table>	$Direct_{EFF}$	The direct stage effectiveness	T_{db}	The entering air dry-bulb temperature	T_{wb}	The entering air wet-bulb temperature	T_{direct}	The direct stage leaving dry-bulb temperature
$Direct_{EFF}$	The direct stage effectiveness								
T_{db}	The entering air dry-bulb temperature								
T_{wb}	The entering air wet-bulb temperature								
T_{direct}	The direct stage leaving dry-bulb temperature								
Units	Numeric ($0 \leq EFF \leq 1$)								
Input Restrictions	As designed								
Standard Design	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable								
Standard Design: Existing Buildings									

Indirect Stage Effectiveness									
Applicability	Systems with evaporative cooling								
Definition	<p>The effectiveness of the indirect stage of an evaporative cooling system. Effectiveness is defined as:</p> $Indirect_{EFF} = \frac{T_{db} + T_{indirect}}{T_{db} - T_{wb}}$ <p>Where:</p> <table border="1"> <tr> <td>$Indirect_{EFF}$</td> <td>The indirect stage effectiveness</td> </tr> <tr> <td>T_{db}</td> <td>The entering air dry-bulb temperature</td> </tr> <tr> <td>T_{wb}</td> <td>The entering air wet-bulb temperature</td> </tr> <tr> <td>$T_{indirect}$</td> <td>The indirect stage leaving dry-bulb temperature</td> </tr> </table>	$Indirect_{EFF}$	The indirect stage effectiveness	T_{db}	The entering air dry-bulb temperature	T_{wb}	The entering air wet-bulb temperature	$T_{indirect}$	The indirect stage leaving dry-bulb temperature
$Indirect_{EFF}$	The indirect stage effectiveness								
T_{db}	The entering air dry-bulb temperature								
T_{wb}	The entering air wet-bulb temperature								
$T_{indirect}$	The indirect stage leaving dry-bulb temperature								
Units	Numeric ($0 \leq EFF \leq 1$)								
Input Restrictions	As designed								
Standard Design	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable								
Standard Design: Existing Buildings									

Evaporative Cooling Performance Curves	
Applicability	Systems with evaporative cooling

<i>Definition</i>	A curve that varies the evaporative cooling effectiveness as a function of primary air stream airflow. The default curves are given as:	
	$PLR = \frac{CFM_{operating}}{CFM_{design}}$ $EFF_{FFLOW} = a + b(PLR) + c(PLR)^2$	
	Where:	
	<i>PLR</i>	Part load ratio of airflow based on design airflow
	<i>EFF_{FFLOW}</i>	A multiplier on the evaporative cooler effectiveness to account for variations in part load
	<i>CFM_{operating}</i>	Operating primary air stream airflow (cfm)
	<i>CFM_{design}</i>	Design primary air stream airflow (cfm)
<i>Units</i>	Data structure	
<i>Input Restrictions</i>	User may input curves or use default curves. If defaults are overridden, the software must indicate that supporting documentation is required on the output forms.	
<i>Standard Design</i>	For healthcare facilities, same as the Proposed Design. For all others, Not applicable	
<i>Standard Design: Existing Buildings</i>		

Table 232638: Part-Load Curve Coefficients – Evaporative Cooler Effectiveness

Coefficient	Direct	Indirect
A	1.1833000	1.0970000
B	-0.2575300	-0.1650600
C	0.0742450	0.0680690

Auxiliary Evaporative Cooling Power	
<i>Applicability</i>	Systems with evaporative cooling
<i>Definition</i>	The auxiliary energy of the indirect evaporative cooler fan, and the pumps for both direct and indirect stages
<i>Units</i>	Watts
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable

<i>Standard Design:</i> <i>Existing Buildings</i>	
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Evaporative Cooling Scavenger Air Source	
<i>Applicability</i>	Systems with evaporative cooling
<i>Definition</i>	The source of scavenger air for an indirect section of an evaporative cooler. Options include: <ul style="list-style-type: none"> • Return air • Outside air
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable
<i>Standard Design:</i> <i>Existing Buildings</i>	

5.7.5.4 Western Cooling Challenge Equipment

A special credit is available in the *ACM Reference Manual* for equipment (including evaporative equipment) that meets efficiency and water use requirements of the Western Cooling Challenge (WCC), a program of the Western Cooling Efficiency Center. This compliance option triggers an exceptional condition and documentation requirements. See Chapter 3 of the *ACM Reference Manual* for more information.

This equipment is modeled as high-efficiency DX equipment, with a constant volume fan, integrated economizer, and fixed performance curve. Unlike the input model for a packaged DX equipment, the WCC equipment has many building descriptor inputs fixed (prescribed). Rather than listing the WCC input restrictions as an option within each building descriptor, all requirements for WCC equipment are contained in this section.

Heating equipment must be specified separately, as the WCC test procedure does not include heating conditions. The heating system may be any system other than a heat pump.

Building Descriptor	Section	Proposed Input
System Type	5.7.1	PSZ-AC and PSZVAV-AC only
Cooling Supply Air Temperature	5.7.2.2	20°F below return air temperature
Cooling Supply Air Temperature Control	5.7.2.2	Fixed (single zone system)
Heating Supply Air temperature	5.7.2.3	105°F
Heating Supply Air	5.7.2.3	Fixed (constant)

Temperature Control		
Supply Fan Design Air Rated Capacity	5.7.3	400 cfm/ton
Fan Control Method	5.7.3	Constant Volume
Supply Fan Power Index	5.7.3.2	0.365 W/cfm
Maximum Outside Air Ratio	5.7.4.1	1
Design Outside Air Flow	5.7.4.1	(as designed)
Economizer Control Type	5.7.4.2	Differential dry-bulb
Economizer Integration Level	5.7.4.2	Integrated
Economizer High Temperature Lockout	5.7.4.2	n/a
Economizer low temperature lockout	5.7.4.2	n/a
Cooling Source	5.7.5	DX
Total Cooling Capacity	5.7.5	As designed (NOTE: If there are UMLH, the user will have to enter a larger capacity system, or add a supplemental DX unit with standard efficiency (EER~11) to meet cooling load.)
Sensible Cooling Capacity	5.7.5	As designed
Cooling Capacity Adjustment Curves	5.7.5	Use air-cooled DX defaults
Coil Bypass Factor (if used)	5.7.5	0.190
Direct Expansion Cooling Efficiency	5.7.5.2	As designed (User enters EER from WCC test and software makes adjustment for EER, removing fan power from efficiency as needed.)
Seasonal Energy Efficiency Ratio	5.7.5.2	n/a
Direct Expansion Cooling Efficiency Temperature Adjustment Curve	5.7.5.2	Use default curve
Direct Expansion Part-Load Efficiency Adjustment Curve	5.7.5.2	Use default curve
Refrigerant Charge Factor	5.7.5.2	1
Airflow Adjustment Factor	5.7.5.2	1
Duct Leakage Rate	5.7.5.2	As designed (for qualifying spaces)
Minimum Unloading Ratio	5.7.5.2	0.25 (units > 240 kBtu/h) 0.35 (units >= 65 kBtu/h and < 120 kBtu/h)
Cooling Setpoint Schedule	5.4	Use default schedule for space type (this will be 75°F when occupied, in most cases)
Heating		
Heating Source	5.7.6.1	As designed (but heat pump not allowed)
Preheat Coil	5.7.6.2	No preheat coil

Heating Coil Capacity	5.7.6.3	As designed
Furnace Capacity	5.7.6.4	As designed
Furnace Fuel Heating Efficiency	5.7.6.3	As designed
Furnace Fuel Heating Part Load Efficiency Curve	5.7.6.4	Fixed default curve
Furnace Fuel Heating Pilot	5.7.6.4	0
Furnace Fuel Heating Fan/Auxiliary	5.7.6.4	n/a
Heat recovery	5.7.6.6	Not allowed for this credit
Humidity Controls and Devices	5.7.7	None

5.7.5.5 Evaporative Condenser

Evaporative Condenser Power	
<i>Applicability</i>	Direct expansion systems with an evaporatively cooled condenser
<i>Definition</i>	The power of the evaporative precooling unit; includes any pump(s) and/or fans that are part of the precooling unit
<i>Units</i>	Watts
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Evaporative Condenser Effectiveness									
<i>Applicability</i>	Direct expansion systems with an evaporatively cooled condenser								
<i>Definition</i>	<p>The effectiveness of the evaporative precooling unit for a condenser. Effectiveness is defined as:</p> $DirectEFF = \frac{T_{db} - T_{direct}}{T_{db} - T_{wb}}$ <p>Where:</p> <table border="1"> <tr> <td><i>DirectEFF</i></td> <td>The direct stage effectiveness</td> </tr> <tr> <td><i>T_{db}</i></td> <td>The outside air dry-bulb temperature</td> </tr> <tr> <td><i>T_{wb}</i></td> <td>The outside air wet-bulb temperature</td> </tr> <tr> <td><i>T_{direct}</i></td> <td>The direct stage leaving dry-bulb temperature (at the condenser inlet)</td> </tr> </table>	<i>DirectEFF</i>	The direct stage effectiveness	<i>T_{db}</i>	The outside air dry-bulb temperature	<i>T_{wb}</i>	The outside air wet-bulb temperature	<i>T_{direct}</i>	The direct stage leaving dry-bulb temperature (at the condenser inlet)
<i>DirectEFF</i>	The direct stage effectiveness								
<i>T_{db}</i>	The outside air dry-bulb temperature								
<i>T_{wb}</i>	The outside air wet-bulb temperature								
<i>T_{direct}</i>	The direct stage leaving dry-bulb temperature (at the condenser inlet)								

<i>Units</i>	Ratio
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Evaporative Condenser Operation Range					
<i>Applicability</i>	Direct expansion systems with an evaporatively cooled condenser				
<i>Definition</i>	The temperature range within which the evaporative condenser operates. Two values are provided:				
	<table border="1"> <tr> <td>$T_{maximum}$</td> <td>The threshold outside air dry-bulb temperature below which evaporative condenser operates</td> </tr> <tr> <td>$T_{minimum}$</td> <td>The threshold outside air dry-bulb temperature above which evaporative condenser operates</td> </tr> </table>	$T_{maximum}$	The threshold outside air dry-bulb temperature below which evaporative condenser operates	$T_{minimum}$	The threshold outside air dry-bulb temperature above which evaporative condenser operates
	$T_{maximum}$	The threshold outside air dry-bulb temperature below which evaporative condenser operates			
$T_{minimum}$	The threshold outside air dry-bulb temperature above which evaporative condenser operates				
<i>Units</i>	Degrees Fahrenheit (°F)				
<i>Input Restrictions</i>	As designed				
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable				
<i>Standard Design: Existing Buildings</i>					

5.7.5.6 Four-Pipe Fan Coil Systems

This section contains building descriptors required to model four-pipe fan coil systems. Note that this system requires an outside air ventilation source to serve the zones and that an airside economizer is not available.

The fan coil fans shall be modeled with the power-per-unit-flow method. The standard design fan power shall be 0.35 W/cfm when the four-pipe fan coil is the standard design system. See the supply fan information, which can be found in the *ACM Reference Manual*, section 5.7.3.2.

Supply air flow rates are set at the zone level. Chilled water flow rates are set according to the rules in section 5.8.5 on pumps.

Additional HVAC components (chiller, boiler, pumps) are needed to fully define this system. If a water-side economizer is specified with this system, refer to section 5.8.4 for a list of applicable building descriptors.

Capacity Control Method	
<i>Applicability</i>	Four-pipe fan coil systems
<i>Definition</i>	The control method for the fan coil unit at the zone. The following choices are available: <ul style="list-style-type: none"> • Constant Fan Variable Flow • Cycling Fan • Variable Fan Constant Flow • Variable Fan Variable Flow
<i>Units</i>	List (with choices above)
<i>Input Restrictions</i>	Not a user input. It comes from building descriptors for fan control and chiller loop flow control
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Cycling Fan
<i>Standard Design: Existing Buildings</i>	

Rated Gross Capacity	
<i>Applicability</i>	Four-pipe fan coil systems and chilled beams
<i>Definition</i>	The gross cooling capacity of the cooling coil
<i>Units</i>	Btu/h
<i>Input Restrictions</i>	None
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Cooling Coil Design Flow rate	
<i>Applicability</i>	Four-pipe fan coil systems and chilled beams
<i>Definition</i>	The design flow rate of the cooling coil
<i>Units</i>	Gallons per minute (gpm)
<i>Input Restrictions</i>	None
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable

<i>Standard Design: Existing Buildings</i>	
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Pressure Drop	
<i>Applicability</i>	Four-pipe fan coil systems and chilled beams
<i>Definition</i>	The pressure drop across the cooling coil
<i>Units</i>	ft. wg
<i>Input Restrictions</i>	None
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Chilled Beam Induced Air Flow	
<i>Applicability</i>	Active and passive chilled beams
<i>Definition</i>	The induced air flow rate across the coil
<i>Units</i>	Cfm
<i>Input Restrictions</i>	None
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Capacity Control Method	
<i>Applicability</i>	Four-pipe fan coil systems
<i>Definition</i>	The control method for the fan coil unit at the zone. The following choices are available: <ul style="list-style-type: none"> • Constant Fan Variable Flow • Cycling Fan • Variable Fan Constant Flow • Variable Fan Variable Flow
<i>Units</i>	List (with choices above)
<i>Input Restrictions</i>	Not a user input – derived from building descriptors for fan control and

	chiller loop flow control.
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Cycling Fan
<i>Standard Design: Existing Buildings</i>	

5.7.5.7 Radiant Cooling

This section describes a floor-based radiant cooling system and the inputs required for Title 24 compliance evaluation.

Hydronic Tubing Length	
<i>Applicability</i>	Floor-based radiant cooling systems
<i>Definition</i>	The length of the hydronic tubing in the slab
<i>Units</i>	ft
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Hydronic Tubing Inside Diameter	
<i>Applicability</i>	Floor-based radiant cooling systems
<i>Definition</i>	The inside diameter of the hydronic tubing in the slab
<i>Units</i>	Ft
<i>Input Restrictions</i>	As designed, between a minimum of 1/2" and a maximum of 3/4"
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Temperature Control Type	
<i>Applicability</i>	Floor-based radiant cooling systems
<i>Definition</i>	The temperature used for control (operative temperature, mean air

	temperature, mean radiant temperature, Outdoor Dry Bulb (ODB), or Outdoor Wet Bulb (OWB)
<i>Units</i>	None
<i>Input Restrictions</i>	Fixed at mean air temperature for compliance calculations
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Cooling Control Temperature	
<i>Applicability</i>	Variable flow systems
<i>Definition</i>	The temperature used for control (operative temperature, mean air temperature, mean radiant temperature, ODB, or OWB)
<i>Units</i>	None
<i>Input Restrictions</i>	Fixed at mean air temperature for compliance calculations
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Condensation Control Type	
<i>Applicability</i>	Floor-based radiant cooling systems
<i>Definition</i>	The temperature used for control (operative temperature, mean air temperature, mean radiant temperature, ODB, or OWB)
<i>Units</i>	None
<i>Input Restrictions</i>	Fixed at mean air temperature for compliance calculations
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Condensation Control Dewpoint Offset	
<i>Applicability</i>	Floor-based radiant cooling systems

<i>Definition</i>	The temperature difference above dewpoint that is the minimum cold water supply temperature
<i>Units</i>	None
<i>Input Restrictions</i>	Fixed at 2°F above dewpoint
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Rated Pump Power Consumption	
<i>Applicability</i>	Floor-based radiant cooling systems
<i>Definition</i>	The rated pump power at design conditions
<i>Units</i>	Watts
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Motor Efficiency	
<i>Applicability</i>	Floor-based radiant cooling systems
<i>Definition</i>	The pump motor efficiency
<i>Units</i>	Decimal fraction
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Default motor efficiency from Table N2-20 (table numbering may change) based on motor nameplate hp
<i>Standard Design: Existing Buildings</i>	

Fraction of Motor Heat to Fluid	
<i>Applicability</i>	Floor-based radiant cooling systems
<i>Definition</i>	Fraction of the heat from the motor inefficiencies that enters the fluid stream

<i>Units</i>	None
<i>Input Restrictions</i>	As designed; default is 0
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Cooling High Water Temperature	
<i>Applicability</i>	Floor-based radiant cooling systems
<i>Definition</i>	The high temperature used for control. If the water temperature is above the high temperature, the control temperature is set to the low control temperature.
<i>Units</i>	°F
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Cooling Low Water Temperature	
<i>Applicability</i>	Floor-based radiant cooling systems
<i>Definition</i>	The temperature used for control of the water temperature. If the water temperature of the radiant cooling is below this temperature, cooling is disabled.
<i>Units</i>	°F
<i>Input Restrictions</i>	Fixed at 55°F
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

Condensation Control Type	
<i>Applicability</i>	Floor-based radiant cooling systems
<i>Definition</i>	The simulation program may have a means of detecting when

	condensation is likely to occur on floor surfaces in the space. When this occurs, the simulation can shut off the system to prevent condensation from occurring.
<i>Units</i>	List none, simple, or variable
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

5.7.5.8 Chilled Beams

Active and passive chilled beams can be modeled, in a similar manner to four-pipe fan coil systems. Active chilled beams are modeled as a four-pipe fan coil object. The zone fan for the beam is not specified by the user, but is internally added to the compliance model with no fan power.

Chilled Beam Name	
<i>Applicability</i>	Chilled beams
<i>Definition</i>	A unique name designating the chilled beam
<i>Units</i>	None
<i>Input Restrictions</i>	None
<i>Standard Design</i>	<u>For healthcare facilities, same as the pProposed dDesign. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Chilled Beam Type	
<i>Applicability</i>	Chilled beams
<i>Definition</i>	Specification of the beam as active or passive
<i>Units</i>	List: <ul style="list-style-type: none"> • Active • Passive
<i>Input Restrictions</i>	None
<i>Standard Design</i>	<u>For healthcare facilities, same as the pProposed dDesign. For all others,</u>

	Not applicable
<i>Standard Design: Existing Buildings</i>	

Design Cooling Capacity	
<i>Applicability</i>	Chilled beams
<i>Definition</i>	The designed cooling capacity of the chilled beam
<i>Units</i>	Btu/h
<i>Input Restrictions</i>	None
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Design Chilled Water Temperature	
<i>Applicability</i>	Chilled beams
<i>Definition</i>	The minimum supplied chilled water temperature to the beam. This is typically at least 2°F higher than the space dewpoint temperature at design conditions, to prevent condensation.
<i>Units</i>	°F
<i>Input Restrictions</i>	None
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Maximum Chilled Water Temperature	
<i>Applicability</i>	Chilled beams
<i>Definition</i>	The maximum supplied chilled water temperature to the beam. This allows for chilled water temperature reset at the source.
<i>Units</i>	°F
<i>Input Restrictions</i>	Should be equal to or greater than the design chilled water temperature
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u>

	Not applicable
<i>Standard Design: Existing Buildings</i>	

Active Beam Maximum Primary Flow Rate	
<i>Applicability</i>	Chilled beams
<i>Definition</i>	The design flow rate of the active fan
<i>Units</i>	Cfm
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Active Beam Induced Air Rate	
<i>Applicability</i>	Active chilled beams
<i>Definition</i>	The rate at which induced air is drawn through the chilled beam. The total airflow across the beam is the sum of the maximum primary flow rate and the active beam induced air flow rate.
<i>Units</i>	Cfm
<i>Input Restrictions</i>	None
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Active Fan Static Pressure	
<i>Applicability</i>	Chilled beams
<i>Definition</i>	The design status of the active fan
<i>Units</i>	in. wc
<i>Input Restrictions</i>	None
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable

<i>Standard Design: Existing Buildings</i>	
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Active Fan Static Efficiency	
<i>Applicability</i>	Chilled beams
<i>Definition</i>	The fan static efficiency
<i>Units</i>	in. wc
<i>Input Restrictions</i>	Between 0 and 1
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Not applicable</u>
<i>Standard Design: Existing Buildings</i>	

Active Fan Motor Efficiency	
<i>Applicability</i>	Chilled beams
<i>Definition</i>	The motor efficiency of the fan
<i>Units</i>	in. wc
<i>Input Restrictions</i>	None
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Not applicable</u>
<i>Standard Design: Existing Buildings</i>	

Chilled Beam Heating Capacity	
<i>Applicability</i>	Chilled beams
<i>Definition</i>	The heating capacity of the chilled beam
<i>Units</i>	Btu/h
<i>Input Restrictions</i>	None; defaults to 1 if no heating
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Not applicable</u>
<i>Standard Design: Existing Buildings</i>	

Chilled Beam Heating Source	
<i>Applicability</i>	Chilled beams
<i>Definition</i>	Defaults to electric resistance, whether there is heating provided by the beam or not
<i>Units</i>	None
<i>Input Restrictions</i>	Electric resistance
<i>Standard Design</i>	For healthcare facilities, same as the Proposed Design. For all others, Not applicable
<i>Standard Design: Existing Buildings</i>	

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5.7.6 Heating Systems

5.7.6.1 General

Heating Source	
<i>Applicability</i>	All systems that provide heating
<i>Definition</i>	The source of heating for the heating and preheat coils The choices are: <ul style="list-style-type: none"> • Hot water • Steam • Electric resistance • Electric heat pump • Gas furnace • Gas heat pump (optional feature) • Oil furnace • Heat recovery
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Based on the prescribed system type. Refer to the HVAC system map in Section 5.1.2.
<i>Standard Design: Existing Buildings</i>	

Table 242739: Heating Source for Standard Design ~~Baseline Building~~

Standard Baseline Building Design System	Heating Source
System 1 – <u>PTAGSZAC</u>	Hot water Furnace
System 2 – FPFC	Hot water
System 3 – PSZ-AC	Gas or Oil Furnace
System 5 – Packaged VAV with Reheat	Hot water
System 6 – VAV with Reheat	Hot water
System 7 – Single Zone VAV	Gas Furnace
System 9 – Heating and Ventilation	Gas Furnace
System 10 – CRAH Unit, Data Center	None
System 11 – CRAC Unit, Data Center	None

5.7.6.2 Preheat Coil

Preheat Coil Capacity	
<i>Applicability</i>	Systems with a preheat coil located in the outside air stream
<i>Definition</i>	The heating capacity of a preheating coil at design conditions
<i>Units</i>	Btu/h
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Not applicable</u>
<i>Standard Design: Existing Buildings</i>	

Preheat Coil Efficiency	
<i>Applicability</i>	Systems with a preheat coil with gas heating
<i>Definition</i>	The heating efficiency of a preheating coil at design conditions
<i>Units</i>	Percentage
<i>Input Restrictions</i>	As designed; default is 80 percent
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Not applicable</u>
<i>Standard Design: Existing Buildings</i>	

5.7.6.3 Hydronic/Steam Heating Coils

Systems with boilers have heating coils, including standard design ~~baseline building~~ systems with hot water heating.

Heating Coil Capacity	
<i>Applicability</i>	All systems with a heating coil
<i>Definition</i>	The heating capacity of a heating coil at AHRI conditions
<i>Units</i>	Btu/h
<i>Input Restrictions</i>	As designed The user may need to manually adjust the capacity if the number of unmet load hours exceeds 150.
<i>Standard Design</i>	Autosize with a heating oversizing factor of 25 percent. If the number of unmet load hours for the <u>standard design</u> baseline exceeds <u>150</u> , reduce <u>increase</u> the heating coil capacity as indicated in Section 2.6.2.
<i>Standard Design: Existing Buildings</i>	

5.7.6.4 Furnace

Furnace Capacity	
<i>Applicability</i>	Systems with a furnace
<i>Definition</i>	The full load heating capacity of the unit
<i>Units</i>	Btu/h
<i>Input Restrictions</i>	As designed The user may need to manually adjust the capacity if the number of unmet load hours exceeds 150.
<i>Standard Design</i>	Autosize with an oversizing factor of 25 percent (let the software determine heating capacity based on the building loads). If the number of unmet load hours for the <u>standard design</u> baseline exceeds 150, reduce <u>increase</u> the furnace capacity as indicated in Section 2.4 and 2.6.2
<i>Standard Design: Existing Buildings</i>	

Furnace Fuel Heating Efficiency	
<i>Applicability</i>	Systems with a furnace
<i>Definition</i>	The full load thermal efficiency of either a gas or oil furnace at design conditions. The software must accommodate input in either thermal efficiency (E_t) or annual fuel utilization efficiency (AFUE). Where AFUE is provided, E_t shall be calculated as: $E_t = 0.005163(AFUE + 0.4033)$

	<p>Where:</p> <p>$AFUE$ The annual fuel utilization efficiency (%)</p> <p>E_t The thermal efficiency (fraction)</p>
<i>Units</i>	Fraction
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Look up the requirement from the equipment efficiency tables in Table 6.8.1E of the standards. The <u>standard design</u> baseline efficiency requirement is located in Table E-3 or Table E-4 of the <i>2010 Appliance Efficiency Standards</i> . Use the heating input of the standard design system to determine the size category.
<i>Standard Design: Existing Buildings</i>	

Furnace Fuel Heating Part Load Efficiency Curve											
<i>Applicability</i>	Systems with a furnace										
<i>Definition</i>	<p>An adjustment factor that represents the percentage of full load fuel consumption as a function of the percentage full load capacity. This curve shall take the form of a quadratic equation as follows:</p> $Fuel_{partload} = Fuel_{rated} \times FHeatPLC$ $FHeatPLC = a + b(Q_{partload}/Q_{rated}) + c(Q_{partload}/Q_{rated})^2$ <p>Where:</p> <table border="1"> <tr> <td>$FHeatPLC$</td> <td>The fuel heating part load efficiency curve</td> </tr> <tr> <td>$Fuel_{partload}$</td> <td>The fuel consumption at part load conditions (Btu/h)</td> </tr> <tr> <td>$Fuel_{rated}$</td> <td>The fuel consumption at full load (Btu/h)</td> </tr> <tr> <td>$Q_{partload}$</td> <td>The capacity at part load conditions (Btu/h)</td> </tr> <tr> <td>Q_{rated}</td> <td>The capacity at rated conditions (Btu/h)</td> </tr> </table>	$FHeatPLC$	The fuel heating part load efficiency curve	$Fuel_{partload}$	The fuel consumption at part load conditions (Btu/h)	$Fuel_{rated}$	The fuel consumption at full load (Btu/h)	$Q_{partload}$	The capacity at part load conditions (Btu/h)	Q_{rated}	The capacity at rated conditions (Btu/h)
$FHeatPLC$	The fuel heating part load efficiency curve										
$Fuel_{partload}$	The fuel consumption at part load conditions (Btu/h)										
$Fuel_{rated}$	The fuel consumption at full load (Btu/h)										
$Q_{partload}$	The capacity at part load conditions (Btu/h)										
Q_{rated}	The capacity at rated conditions (Btu/h)										
<i>Units</i>	Data structure										
<i>Input Restrictions</i>	Fixed										
<i>Standard Design</i>	Fixed										
<i>Standard Design: Existing Buildings</i>											

Table 252840: Furnace Efficiency Curve Coefficients

Coefficient	Furnace
a	0.0186100
b	1.0942090
c	-0.1128190

Furnace Fuel Heating Pilot	
<i>Applicability</i>	Systems that use a furnace for heating
<i>Definition</i>	The fuel input for a pilot light on a furnace
<i>Units</i>	Btu/h
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Zero (pilotless ignition)
<i>Standard Design: Existing Buildings</i>	

Furnace Fuel Heating Fan/Auxiliary	
<i>Applicability</i>	Systems that use a furnace for heating
<i>Definition</i>	The fan energy in forced draft furnaces and the auxiliary (pumps and outdoor fan) energy in fuel-fired heat pumps
<i>Units</i>	Kilowatts (kW)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

5.7.6.5 Electric Heat Pump

Electric Heat Pump Heating Capacity	
<i>Applicability</i>	All heat pumps
<i>Definition</i>	The full load heating capacity of the unit, excluding supplemental heating capacity at AHRI rated conditions
<i>Units</i>	Btu/h
<i>Input Restrictions</i>	As designed

<i>Standard Design</i>	Autosize and use an oversizing factor of 25 percent. The software determines heating capacity based on the building loads.
<i>Standard Design: Existing Buildings</i>	

Electric Heat Pump Supplemental Heating Source	
<i>Applicability</i>	All heat pumps
<i>Definition</i>	<p>The auxiliary heating source for a heat pump heating system</p> <p>The common control sequence is to lock out the heat pump compressor when the supplemental heat is activated. Other building descriptors may be needed if this is not the case. Choices for supplemental heat include:</p> <ul style="list-style-type: none"> • Electric resistance • Gas furnace • Oil furnace • Hot water • Other
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	For healthcare facilities, same as the Proposed Design. For all others, Electric resistance
<i>Standard Design: Existing Buildings</i>	

Electric Heat Pump Heating Efficiency	
<i>Applicability</i>	All heat pumps
<i>Definition</i>	<p>The heating efficiency of a heat pump at AHRI rated conditions as a dimensionless ratio of output over input. The software must accommodate user input of either the coefficient of performance (COP) or the heating season performance factor (HSPF). Where HSPF is provided, COP shall be calculated as:</p> $COP = (0.2778 \times HSPF) + 0.9667$ <p>For all unitary and applied equipment where the fan energy is part of the equipment efficiency rating, the COP shall be adjusted as follows to remove the fan energy:</p> $COP_{adj} = \frac{\frac{HCAP_{rated} - Q_{fan,rated}}{3.413}}{\frac{HCAP_{rated}}{COP \times 3.413} - \frac{Q_{fan,rated}}{3.413}}$

	<p>Where:</p> <table border="1"> <tr> <td>COP_{adj}</td> <td>The adjusted coefficient of performance for simulation purposes</td> </tr> <tr> <td>COP</td> <td>The AHRI rated coefficient of performance</td> </tr> <tr> <td>$HCAP_{rated}$</td> <td>The AHRI rated heating capacity of a packaged unit (kBtu/h)</td> </tr> <tr> <td>$Q_{fan,rated}$</td> <td>ARI rated fan power, equal to the gross rated cooling capacity times 0.04.</td> </tr> </table>	COP_{adj}	The adjusted coefficient of performance for simulation purposes	COP	The AHRI rated coefficient of performance	$HCAP_{rated}$	The AHRI rated heating capacity of a packaged unit (kBtu/h)	$Q_{fan,rated}$	ARI rated fan power, equal to the gross rated cooling capacity times 0.04.
COP_{adj}	The adjusted coefficient of performance for simulation purposes								
COP	The AHRI rated coefficient of performance								
$HCAP_{rated}$	The AHRI rated heating capacity of a packaged unit (kBtu/h)								
$Q_{fan,rated}$	ARI rated fan power, equal to the gross rated cooling capacity times 0.04.								
Units	Unitless								
Input Restrictions	As designed								
Standard Design	<u>For healthcare facilities, same as the Proposed Design. For all others, Not applicable</u>								
Standard Design: Existing Buildings									

Electric Heat Pump Heating Capacity Adjustment Curve(s)											
Applicability	All heat pumps										
Definition	<p>A curve or group of curves that represent the available heat-pump heating capacity as a function of evaporator and condenser conditions. The default curves are given as:</p> $Q_{available} = CAP_{FT} \times Q_{rated}$ <p>For air-cooled heat pumps:</p> $CAP_{FT} = a + b(t_{odb}) + c(t_{odb})^2 + d(t_{odb})^3$ <p>For water-cooled heat pumps:</p> $CAP_{FT} = a + b(t_{db}) + d(t_{wt})$ <p>Where:</p> <table border="1"> <tr> <td>$Q_{available}$</td> <td>Available heating capacity at present evaporator and condenser conditions (kBtu/h)</td> </tr> <tr> <td>t_{db}</td> <td>The entering coil dry-bulb temperature (°F)</td> </tr> <tr> <td>t_{wt}</td> <td>The water supply temperature (°F)</td> </tr> <tr> <td>t_{odb}</td> <td>The outside-air dry-bulb temperature (°F)</td> </tr> <tr> <td>Q_{rated}</td> <td>Rated capacity at AHRI conditions (in kBtu/h)</td> </tr> </table>	$Q_{available}$	Available heating capacity at present evaporator and condenser conditions (kBtu/h)	t_{db}	The entering coil dry-bulb temperature (°F)	t_{wt}	The water supply temperature (°F)	t_{odb}	The outside-air dry-bulb temperature (°F)	Q_{rated}	Rated capacity at AHRI conditions (in kBtu/h)
$Q_{available}$	Available heating capacity at present evaporator and condenser conditions (kBtu/h)										
t_{db}	The entering coil dry-bulb temperature (°F)										
t_{wt}	The water supply temperature (°F)										
t_{odb}	The outside-air dry-bulb temperature (°F)										
Q_{rated}	Rated capacity at AHRI conditions (in kBtu/h)										

<i>Units</i>	Data structure
<i>Input Restrictions</i>	Fixed. Use curves in Table 45 for water-source or air-source heat pumps as appropriate.
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Table 262941: Heat Pump Capacity Adjustment Curves (CAP-FT)

Coefficient	Water-Source	Air-Source
a	0.4886534	0.2536714
b	-0.0067774	0.0104351
c	N/A	0.0001861
d	0.0140823	-0.0000015

Electric Heat Pump Heating Efficiency Adjustment Curve(s)					
<i>Applicability</i>	All heat pumps				
<i>Definition</i>	<p>A curve or group of curves that varies the heat pump heating efficiency as a function of evaporator conditions, condenser conditions and part-load ratio. The default curves are given as:</p> $PLR = \frac{Q_{operating}}{Q_{available}(t_{db}, t_{odb/wt})}$ $EIR_{FPLR} = a + b(PLR) + c(PLR)^2 + d(PLR)^3$ <p>Air-Source Heat Pumps:</p> $EIR_{FT} = a + b(t_{odb}) + c(t_{odb})^2 + d(t_{odb})^3$ <p>Water-Source Heat Pumps:</p> $EIR_{FT} = a + b(t_{wt}) + d(t_{db})$ $P_{operating} = P_{rated}(EIR_{FPLR})(EIR_{FT})(CAP_{FT})$ <p>Where:</p> <table border="1"> <tr> <td>PLR</td> <td>Part-load ratio based on available capacity (not rated capacity)</td> </tr> <tr> <td>EIR_{FPLR}</td> <td>A multiplier on the EIR of the heat pump as a function of part-load ratio</td> </tr> </table>	PLR	Part-load ratio based on available capacity (not rated capacity)	EIR_{FPLR}	A multiplier on the EIR of the heat pump as a function of part-load ratio
PLR	Part-load ratio based on available capacity (not rated capacity)				
EIR_{FPLR}	A multiplier on the EIR of the heat pump as a function of part-load ratio				

	EIR_{FT}	A multiplier on the EIR of the heat pump as a function of the wet-bulb temperature entering the coil and the outdoor dry-bulb temperature
	$Q_{operating}$	Present load on heat pump (Btu/h)
	$Q_{available}$	Heat pump available capacity at present evaporator and condenser conditions (Btu/h)
	t_{db}	The entering coil dry-bulb temperature (°F)
	t_{wt}	The water supply temperature (°F)
	t_{odb}	The outside air dry-bulb temperature (°F)
	P_{rated}	Rated power draw at AHRI conditions (kW)
	$P_{operating}$	Power draw at specified operating conditions (kW)
Units	None	
Input Restrictions	Fixed. Use appropriate curve from Table 46.	
Standard Design	For healthcare facilities, same as the Proposed Design. For all others, Not applicable	
Standard Design: Existing Buildings		

Table 273042: Heat Pump Heating Efficiency Curves

Coefficient	Air/Water-Source	Water-Source	Air-Source
	EIR_{FPLR}	EIR_{FT}	EIR_{FT}
a	0.0856522	1.3876102	2.4600298
b	0.9388137	0.0060479	-0.0622539
c	-0.1834361	N/A	0.0008800
d	0.1589702	-0.0115852	-0.0000046

Electric Heat Pump Supplemental Heating Capacity	
Applicability	All heat pumps
Definition	The design heating capacity of a heat pump supplemental heating coil at AHRI conditions
Units	Btu/h
Input Restrictions	As designed
Standard Design	For healthcare facilities, same as the Proposed Design. For all others,

	Not applicable
<i>Standard Design: Existing Buildings</i>	

Electric Supplemental Heating Control Temp	
<i>Applicability</i>	All heat pumps
<i>Definition</i>	The outside dry-bulb temperature below which the heat pump supplemental heating is allowed to operate
<i>Units</i>	Degrees Fahrenheit (°F)
<i>Input Restrictions</i>	As designed; default to 40°F
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Heat Pump Compressor Minimum Operating Temp	
<i>Applicability</i>	All heat pumps
<i>Definition</i>	The outside dry-bulb temperature below which the heat pump compressor is disabled
<i>Units</i>	Degrees Fahrenheit (°F)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Coil Defrost	
<i>Applicability</i>	Air-cooled electric heat pump
<i>Definition</i>	The defrost control mechanism for an air-cooled heat pump The choices are: <ul style="list-style-type: none"> • Hot-gas defrost, on-demand • Hot-gas defrost, timed 3.5 minute cycle • Electric resistance defrost, on-demand • Electric resistance defrost, timed 3.5 minute cycle

	Defrost shall be enabled whenever the outside air dry-bulb temperature drops below 40°F.
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	Default to use hot-gas defrost, timed 3.5 minute cycle. User may select any of the above.
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Coil Defrost kW	
<i>Applicability</i>	Heat pumps with electric resistance defrost
<i>Definition</i>	The capacity of the electric resistance defrost heater
<i>Units</i>	Kilowatts (kW)
<i>Input Restrictions</i>	As designed; defaults to 0 if nothing is entered
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Crank Case Heater kW	
<i>Applicability</i>	All heat pumps
<i>Definition</i>	The capacity of the electric resistance heater in the crank case of a direct expansion (DX) compressor. The crank case heater operates only when the compressor is off.
<i>Units</i>	Kilowatts (kW)
<i>Input Restrictions</i>	As designed; defaults to 0.1 if nothing is entered
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Crank Case Heater Shutoff Temperature	
<i>Applicability</i>	All heat pumps

<i>Definition</i>	The outdoor air dry-bulb temperature above which the crank case heater is not permitted to operate
<i>Units</i>	Degrees Fahrenheit (°F)
<i>Input Restrictions</i>	As designed; defaults to 50°F
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Not applicable</u>
<i>Standard Design: Existing Buildings</i>	

5.7.6.6 Heat Recovery

Recovery Type	
<i>Applicability</i>	All systems with airside heat recovery
<i>Definition</i>	The type of heat recovery system
<i>Units</i>	List: sensible, latent, or total (sensible and latent)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Not applicable</u>
<i>Standard Design: Existing Buildings</i>	

Exhaust Air Sensible Heat Recovery Effectiveness									
<i>Applicability</i>	Any system with outside air heat recovery								
<i>Definition</i>	<p>The effectiveness of an air-to-air heat exchanger between the building exhaust and entering outside air streams. Effectiveness is defined as:</p> $HREFF = \frac{EEA_{db} - ELA_{db}}{EEA_{db} - OSA_{db}}$ <p>Where:</p> <table border="1"> <tr> <td><i>HREFF</i></td> <td>The air-to-air heat exchanger effectiveness</td> </tr> <tr> <td><i>EEA_{db}</i></td> <td>The exhaust air dry-bulb temperature entering the heat exchanger</td> </tr> <tr> <td><i>ELA_{db}</i></td> <td>The exhaust air dry-bulb temperature leaving the heat exchanger</td> </tr> <tr> <td><i>OSA_{db}</i></td> <td>The outside air dry-bulb temperature</td> </tr> </table>	<i>HREFF</i>	The air-to-air heat exchanger effectiveness	<i>EEA_{db}</i>	The exhaust air dry-bulb temperature entering the heat exchanger	<i>ELA_{db}</i>	The exhaust air dry-bulb temperature leaving the heat exchanger	<i>OSA_{db}</i>	The outside air dry-bulb temperature
<i>HREFF</i>	The air-to-air heat exchanger effectiveness								
<i>EEA_{db}</i>	The exhaust air dry-bulb temperature entering the heat exchanger								
<i>ELA_{db}</i>	The exhaust air dry-bulb temperature leaving the heat exchanger								
<i>OSA_{db}</i>	The outside air dry-bulb temperature								
<i>Units</i>	Two unitless numbers (ratio between 0 and 1), separate for cooling and								

	heating
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	For healthcare facilities, same as the Proposed Design. For all others, Not applicable
<i>Standard Design: Existing Buildings</i>	

Exhaust Air Sensible Part-Load Effectiveness									
<i>Applicability</i>	Any system with outside air heat recovery								
<i>Definition</i>	<p>The effectiveness of an air-to-air heat exchanger between the building exhaust and entering outside air streams at 75 percent of design airflow. Effectiveness is defined as:</p> $HREFF = \frac{EEA_{db} - ELA_{db}}{EEA_{db} - OSA_{db}}$ <p>Where:</p> <table border="1"> <tr> <td><i>HREFF</i></td> <td>The air-to-air heat exchanger effectiveness</td> </tr> <tr> <td><i>EEA_{db}</i></td> <td>The exhaust air dry-bulb temperature entering the heat exchanger</td> </tr> <tr> <td><i>ELA_{db}</i></td> <td>The exhaust air dry-bulb temperature leaving the heat exchanger</td> </tr> <tr> <td><i>OSA_{db}</i></td> <td>The outside air dry-bulb temperature</td> </tr> </table>	<i>HREFF</i>	The air-to-air heat exchanger effectiveness	<i>EEA_{db}</i>	The exhaust air dry-bulb temperature entering the heat exchanger	<i>ELA_{db}</i>	The exhaust air dry-bulb temperature leaving the heat exchanger	<i>OSA_{db}</i>	The outside air dry-bulb temperature
<i>HREFF</i>	The air-to-air heat exchanger effectiveness								
<i>EEA_{db}</i>	The exhaust air dry-bulb temperature entering the heat exchanger								
<i>ELA_{db}</i>	The exhaust air dry-bulb temperature leaving the heat exchanger								
<i>OSA_{db}</i>	The outside air dry-bulb temperature								
<i>Units</i>	Two unitless numbers (ratio between 0 and 1), separate for cooling and heating								
<i>Input Restrictions</i>	As designed								
<i>Standard Design</i>	For healthcare facilities, same as the Proposed Design. For all others, Not applicable								
<i>Standard Design: Existing Buildings</i>									

Exhaust Air Latent Heat Recovery Effectiveness	
<i>Applicability</i>	Any system with outside air enthalpy heat recovery
<i>Definition</i>	The latent heat recovery effectiveness of an air-to-air heat exchanger between the building exhaust and entering outside air streams. Effectiveness is defined as:

	$HREFF = \frac{EEA_w - ELA_w}{EEA_w - OSA_w}$ <p>Where:</p> <table border="1"> <tr> <td><i>HREFF</i></td> <td>The air-to-air heat exchanger effectiveness</td> </tr> <tr> <td><i>EEA_w</i></td> <td>The exhaust air humidity ratio (fraction of mass of moisture in air to mass of dry air) entering the heat exchanger</td> </tr> <tr> <td><i>ELA_w</i></td> <td>The exhaust air humidity ratio leaving the heat exchanger</td> </tr> <tr> <td><i>OSA_w</i></td> <td>The outside air humidity ratio</td> </tr> </table> <p>Note: For sensible heat exchangers, this term is not applicable.</p>	<i>HREFF</i>	The air-to-air heat exchanger effectiveness	<i>EEA_w</i>	The exhaust air humidity ratio (fraction of mass of moisture in air to mass of dry air) entering the heat exchanger	<i>ELA_w</i>	The exhaust air humidity ratio leaving the heat exchanger	<i>OSA_w</i>	The outside air humidity ratio
<i>HREFF</i>	The air-to-air heat exchanger effectiveness								
<i>EEA_w</i>	The exhaust air humidity ratio (fraction of mass of moisture in air to mass of dry air) entering the heat exchanger								
<i>ELA_w</i>	The exhaust air humidity ratio leaving the heat exchanger								
<i>OSA_w</i>	The outside air humidity ratio								
<i>Units</i>	Two unitless numbers (ratio between 0 and 1), separate for cooling and heating								
<i>Input Restrictions</i>	As designed								
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Not applicable</u>								
<i>Standard Design: Existing Buildings</i>									

Exhaust Air Latent Part-Load Effectiveness									
<i>Applicability</i>	Any system with outside air enthalpy heat recovery								
<i>Definition</i>	<p>The latent heat recovery effectiveness of an air-to-air heat exchanger between the building exhaust and entering outside air streams at 75 percent of design airflow. Effectiveness is defined as:</p> $HREFF = \frac{EEA_w - ELA_w}{EEA_w - OSA_w}$ <p>Where:</p> <table border="1"> <tr> <td><i>HREFF</i></td> <td>The air-to-air heat exchanger effectiveness</td> </tr> <tr> <td><i>EEA_w</i></td> <td>The exhaust air humidity ratio (fraction of mass of moisture in air to mass of dry air) entering the heat exchanger</td> </tr> <tr> <td><i>ELA_w</i></td> <td>The exhaust air humidity ratio leaving the heat exchanger</td> </tr> <tr> <td><i>OSA_w</i></td> <td>The outside air humidity ratio</td> </tr> </table> <p>Note: For sensible heat exchangers, this term is not applicable.</p>	<i>HREFF</i>	The air-to-air heat exchanger effectiveness	<i>EEA_w</i>	The exhaust air humidity ratio (fraction of mass of moisture in air to mass of dry air) entering the heat exchanger	<i>ELA_w</i>	The exhaust air humidity ratio leaving the heat exchanger	<i>OSA_w</i>	The outside air humidity ratio
<i>HREFF</i>	The air-to-air heat exchanger effectiveness								
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<i>ELA_w</i>	The exhaust air humidity ratio leaving the heat exchanger								
<i>OSA_w</i>	The outside air humidity ratio								
<i>Units</i>	Two unitless numbers (ratio between 0 and 1), separate for cooling and heating								

<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Not applicable</u>
<i>Standard Design: Existing Buildings</i>	

Economizer Enabled during Heat Recovery	
<i>Applicability</i>	All systems with airside heat recovery
<i>Definition</i>	A flag to indicate whether or not the economizer is enabled when heat recovery is active
<i>Units</i>	Boolean
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Not applicable</u>
<i>Standard Design: Existing Buildings</i>	

5.8 HVAC Primary Systems

5.8.1 Boilers

Boiler Name	
<i>Applicability</i>	All boilers
<i>Definition</i>	A unique descriptor for each boiler, heat pump, central heating heat-exchanger, or heat recovery device
<i>Units</i>	None
<i>Input Restrictions</i>	User entry Where applicable, this should match the tags that are used on the plans for the proposed design.
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Boilers are only designated in the <u>standard design</u> baseline model if the baseline system type uses hot water for space heating.</u>
<i>Standard Design: Existing Buildings</i>	

Boiler Fuel Source

<i>Applicability</i>	All boilers														
<i>Definition</i>	The fuel source for the central heating equipment The choices are: <ul style="list-style-type: none"> • Gas • Oil • Electricity 														
<i>Units</i>	List (see above)														
<i>Input Restrictions</i>	As designed This input is restricted, based on the choice of boiler type, according to the following rules: <table border="1" style="width: 100%; border-collapse: collapse; text-align: center;"> <tr> <td style="width: 33%;"></td> <td style="width: 33%;">Electricity</td> <td style="width: 33%;">Gas</td> <td style="width: 33%;">Steam</td> </tr> <tr> <td>Steam Boiler</td> <td></td> <td></td> <td>Allowed</td> </tr> <tr> <td>Hot Water Boiler</td> <td></td> <td>Allowed</td> <td></td> </tr> </table>				Electricity	Gas	Steam	Steam Boiler			Allowed	Hot Water Boiler		Allowed	
	Electricity	Gas	Steam												
Steam Boiler			Allowed												
Hot Water Boiler		Allowed													
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Gas</u>														
<i>Standard Design: Existing Buildings</i>															

Boiler Type	
<i>Applicability</i>	All boilers
<i>Definition</i>	The fuel source for the central heating equipment The choices are: <ul style="list-style-type: none"> • Gas • Oil • Electricity
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Hot water boiler</u>
<i>Standard Design: Existing Buildings</i>	

Boiler Draft Type	
<i>Applicability</i>	All boilers

<i>Definition</i>	<p>How combustion airflow is drawn through the boiler.</p> <p>The choices are natural (sometimes called atmospheric) or mechanical.</p> <p>Natural draft boilers use natural convection to draw air for combustion through the boiler. Natural draft boilers are subject to outside air conditions and the temperature of the flue gases.</p> <p>Mechanical draft boilers enhance the air flow in one of three ways: 1) induced draft, which uses ambient air, a steam jet, or a fan to induce a negative pressure which pulls flow through the exhaust stack; 2) forced draft, which uses a fan and ductwork to create a positive pressure that forces air into the furnace, or 3) balanced draft, which uses both induced draft and forced draft methods to bring air through the furnace, usually keeping the pressure slightly below atmospheric.</p>
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Mechanical (forced)</u>
<i>Standard Design: Existing Buildings</i>	

Number of Identical Boiler Units	
<i>Applicability</i>	All boilers
<i>Definition</i>	The number of identical units for staging
<i>Units</i>	Numeric: integer
<i>Input Restrictions</i>	As designed; default is 1
<i>Standard Design</i>	The <u>standard design</u> baseline building shall have one boiler for a when the <u>standard design</u> baseline plant serves a conditioned floor area of 15,000 ft ² or less and have two equally size boilers for plants serving more than 15,000 ft ² .
<i>Standard Design: Existing Buildings</i>	

Boiler Design Capacity	
<i>Applicability</i>	All boilers
<i>Definition</i>	The heating capacity at design conditions
<i>Units</i>	Btu/h
<i>Input Restrictions</i>	As designed

	If unmet load hours exceed 150, the user may need to manually adjust boiler design capacity
<i>Standard Design</i>	The boiler is sized to be 25 percent larger than the peak loads of the standard design <u>baseline building</u> . Standard design <u>Baseline</u> boilers shall be sized using weather files containing 99.6 percent heating design temperatures and 0.5 percent dry-bulb and 1 percent wet-bulb cooling design temperatures.
<i>Standard Design: Existing Buildings</i>	

Boiler Efficiency Type	
<i>Applicability</i>	All boilers
<i>Definition</i>	The full load efficiency of a boiler is expressed as one of the following: <ul style="list-style-type: none"> • Annual fuel utilization efficiency (AFUE) is a measure of the boiler’s efficiency over a predefined heating season. • Thermal efficiency (E_t) is the ratio of the heat transferred to the water divided by the heat input of the fuel. • Combustion efficiency (E_c) is the measure of how much energy is extracted from the fuel and is the ratio of heat transferred to the combustion air divided by the heat input of the fuel.
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	None
<i>Standard Design</i>	AFUE for all gas and oil-fired boilers with less than 300,000 Btu/h capacity. Thermal efficiency (E_t) for all gas and oil-fired boilers with capacities between 225,000 and 2,500,000 Btu/h. Combustion efficiency (E_c), for all gas and oil-fired boilers with capacities above 2,500,000 Btu/h.
<i>Standard Design: Existing Buildings</i>	

Boiler Efficiency	
<i>Applicability</i>	All boilers
<i>Definition</i>	The full load efficiency of a boiler at rated conditions (see efficiency type above) expressed as a dimensionless ratio of output over input. The software must accommodate input in either thermal efficiency (E_t), combustion efficiency (E_c), or AFUE. The software shall make appropriate conversions to thermal efficiency if either AFUE or combustion efficiency is entered as the rated efficiency.

	<p>Where AFUE is provided, E_t shall be calculated as follows:</p> <table border="1"> <tr> <td>$75\% \leq AFUE < 80\%$</td> <td>$E_t = 0.1(AFUE + 72.5\%)$</td> </tr> <tr> <td>$80\% \leq AFUE \leq 100\%$</td> <td>$E_t = 0.875(AFUE + 10.5\%)$</td> </tr> </table> <p>If combustion efficiency is entered, the compliance software shall convert the efficiency to thermal efficiency by the relation:</p> $E_t = E_c - 0.015$ <p>All electric boilers will have an efficiency of 98 percent.</p>	$75\% \leq AFUE < 80\%$	$E_t = 0.1(AFUE + 72.5\%)$	$80\% \leq AFUE \leq 100\%$	$E_t = 0.875(AFUE + 10.5\%)$
$75\% \leq AFUE < 80\%$	$E_t = 0.1(AFUE + 72.5\%)$				
$80\% \leq AFUE \leq 100\%$	$E_t = 0.875(AFUE + 10.5\%)$				
<i>Units</i>	Ratio				
<i>Input Restrictions</i>	As designed				
<i>Standard Design</i>	Boilers for the standard design baseline design are assumed to have the minimum efficiency as listed in Table E-4 of the 2009 California Appliance Efficiency Regulations Standards .				
<i>Standard Design: Existing Buildings</i>					

Boiler Part-Load Performance Curve																	
<i>Applicability</i>	All boilers																
<i>Definition</i>	<p>An adjustment factor that represents the percentage full load fuel consumption as a function of the percentage full load capacity. This curve shall take the form of a quadratic equation as follows:</p> $Fuel_{partload} = Fuel_{design} [FHeatPLC(Q_{partload}, Q_{rated})]$ $FHeatPLC = \left(a + b \left(\frac{Q_{partload}}{Q_{rated}} \right) + c \left(\frac{Q_{partload}}{Q_{rated}} \right)^2 \right)$ <p>Where:</p> <table border="1"> <tr> <td>$FHeatPLC$</td> <td>The fuel heating part-load efficiency curve</td> </tr> <tr> <td>$Fuel_{partload}$</td> <td>The fuel consumption at part-load conditions (Btu/h)</td> </tr> <tr> <td>$Fuel_{design}$</td> <td>The fuel consumption at design conditions (Btu/h)</td> </tr> <tr> <td>$Q_{partload}$</td> <td>The boiler capacity at part-load conditions (Btu/h)</td> </tr> <tr> <td>Q_{rated}</td> <td>The boiler capacity at design conditions (Btu/h)</td> </tr> <tr> <td>a</td> <td>Constant</td> </tr> <tr> <td>b</td> <td>Constant</td> </tr> <tr> <td>c</td> <td>Constant</td> </tr> </table>	$FHeatPLC$	The fuel heating part-load efficiency curve	$Fuel_{partload}$	The fuel consumption at part-load conditions (Btu/h)	$Fuel_{design}$	The fuel consumption at design conditions (Btu/h)	$Q_{partload}$	The boiler capacity at part-load conditions (Btu/h)	Q_{rated}	The boiler capacity at design conditions (Btu/h)	a	Constant	b	Constant	c	Constant
$FHeatPLC$	The fuel heating part-load efficiency curve																
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b	Constant																
c	Constant																

<i>Units</i>	Ratio
<i>Input Restrictions</i>	Prescribed to the part-load performance curve in the ACM Appendix 5.7, based on the boiler draft type.
<i>Standard Design</i>	The <u>standard design</u> baseline building uses the mechanical draft fan curve in Appendix 5.7.
<i>Standard Design: Existing Buildings</i>	

Boiler Forced Draft Fan Power	
<i>Applicability</i>	All mechanical draft boilers
<i>Definition</i>	The fan power of the mechanical draft fan at design conditions.
<i>Units</i>	Nameplate horsepower
<i>Input Restrictions</i>	As designed The software shall convert the user entry of motor horsepower to fan power in watts by the following equation: $Fan\ Power = Motor\ HP(7.46)(0.5)$
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Sized for an energy input ratio of 0.001018 (0.2984 W per kBtu/h heat input)
<i>Standard Design: Existing Buildings</i>	

Boiler Minimum Unloading Ratio	
<i>Applicability</i>	All boilers
<i>Definition</i>	The minimum unloading capacity of a boiler expressed as a percentage of the rated capacity. Below this level, the boiler must cycle to meet the load.
<i>Units</i>	Percent (%)
<i>Input Restrictions</i>	As designed If the user does not use the default value, the software must indicate that supporting documentation is required on the output forms. Fixed at 1 percent (this accounts for jacket losses and start/stop losses).
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> 1 percent

<i>Standard Design: Existing Buildings</i>	
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Boiler Minimum Flow Rate	
<i>Applicability</i>	All boilers
<i>Definition</i>	The minimum flow rate recommended by the boiler manufacturer for stable and reliable operation of the boiler
<i>Units</i>	Gpm
<i>Input Restrictions</i>	<p>As designed</p> <p>If the boiler(s) is piped in a primary only configuration in a variable flow system then the software shall assume there is a minimum flow bypass valve that allows the hot water pump to bypass water from the boiler outlet back to the boiler inlet to maintain the minimum flow rate when boiler is enabled.</p> <p>Note: The boiler entering water temperature must accurately reflect the mixed temperature (colder water returning from the coil(s) and hotter bypass water) to accurately model boiler efficiency as a function of boiler entering water temperature.</p>
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, 0 gpm</u>
<i>Standard Design: Existing Buildings</i>	

Hot Water Supply Temperature	
<i>Applicability</i>	All boilers
<i>Definition</i>	The temperature of the water produced by the boiler and supplied to the hot water loop
<i>Units</i>	Degrees Fahrenheit (°F)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Use 180°F for standard design <u>baseline</u> boiler</u>
<i>Standard Design: Existing Buildings</i>	

Hot Water Return Temperature	
<i>Applicability</i>	All boilers
<i>Definition</i>	The temperature of the water returning to the boiler from the hot water loop
<i>Units</i>	Degrees Fahrenheit (°F)
<i>Input Restrictions</i>	As designed.
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Use 140°F for standard design</u> baseline boiler design
<i>Standard Design: Existing Buildings</i>	

Hot Water Supply Temperature Reset	
<i>Applicability</i>	All boilers
<i>Definition</i>	Variation of the hot water supply temperature with outdoor air temperature
<i>Units</i>	Degrees Fahrenheit (°F)
<i>Input Restrictions</i>	As designed (not allowed for non-condensing boilers)
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, The hot water supply temperature should vary according to the following:</u> <ul style="list-style-type: none"> • 180°F when outside air is < 20°F • Ramp linearly between 180°F and 150°F when outdoor air is between 20°F and 50°F • 150°F when outdoor air is > 50°F
<i>Standard Design: Existing Buildings</i>	

5.8.2 Chillers

Chiller Name	
<i>Applicability</i>	All chillers
<i>Definition</i>	A unique descriptor for each chiller
<i>Units</i>	Text, unique
<i>Input Restrictions</i>	User entry; where applicable, this should match the tags that are used on the plans
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Chillers are only designated when the standard design</u> baseline system

	uses chilled water
<i>Standard Design: Existing Buildings</i>	

Chiller Type									
<i>Applicability</i>	All chillers								
<i>Definition</i>	<p>The type of chiller, either a vapor-compression chiller or an absorption chiller.</p> <p>Vapor compression chillers operate on the reverse Rankine cycle, using mechanical energy to compress the refrigerant, and include:</p> <ul style="list-style-type: none"> • Reciprocating* • Scroll* • Screw* • Centrifugal – uses rotating impeller blades to compress the air and impart velocity • Direct-Fired Single Effect Absorption – uses a single generator and condenser • Direct-Fired Double Effect Absorption – uses two generators/ concentrators and condensers, one at a lower temperature and the other at a higher temperature. It is more efficient than the single effect but it must use a higher temperature heat source. • Indirect-Fired Double Effect Absorption • Gas Engine-Driven <p>*Positive displacement – includes reciprocating (piston-style), scroll and screw compressors</p>								
<i>Units</i>	List (see above). The software shall support all chiller types listed above.								
<i>Input Restrictions</i>	As designed								
<i>Standard Design</i>	<p>For healthcare facilities, same as the Proposed Design. For all others, The <u>standard design</u> baseline building chiller is based on the design capacity of the standard design (baseline) as follows:</p> <p style="text-align: center;">Table 283143: Type and Number of Chillers</p> <table border="1" style="width: 100%;"> <thead> <tr> <th>Building Peak Cooling Load</th> <th>Number and type of chiller(s)</th> </tr> </thead> <tbody> <tr> <td>≤ 300 tons</td> <td>One water-cooled screw chiller</td> </tr> <tr> <td>300 < Load < 600</td> <td>Two water-cooled screw chillers, sized equally</td> </tr> <tr> <td>≥ 600 tons</td> <td>A minimum of two water-cooled centrifugal chillers, sized to keep the unit size below 800 tons</td> </tr> </tbody> </table>	Building Peak Cooling Load	Number and type of chiller(s)	≤ 300 tons	One water-cooled screw chiller	300 < Load < 600	Two water-cooled screw chillers, sized equally	≥ 600 tons	A minimum of two water-cooled centrifugal chillers, sized to keep the unit size below 800 tons
Building Peak Cooling Load	Number and type of chiller(s)								
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300 < Load < 600	Two water-cooled screw chillers, sized equally								
≥ 600 tons	A minimum of two water-cooled centrifugal chillers, sized to keep the unit size below 800 tons								

<i>Standard Design: Existing Buildings</i>	
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Number of Identical Chiller Units	
<i>Applicability</i>	All chillers
<i>Definition</i>	The number of identical units for staging
<i>Units</i>	None
<i>Input Restrictions</i>	As designed; default is 1
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, From Table 46 above</u>
<i>Standard Design: Existing Buildings</i>	

Number of Identical Chiller Units																																				
<i>Applicability</i>	All chillers																																			
<i>Definition</i>	<p>The fuel source for the chiller</p> <p>The choices are:</p> <ul style="list-style-type: none"> • Electricity (for all vapor-compression chillers) • Gas (absorption units only, designated as direct-fired units) • Hot water (absorption units only, designated as indirect-fired units) • Steam (absorption units only, designated as indirect-fired units) 																																			
<i>Units</i>	List (see above)																																			
<i>Input Restrictions</i>	<p>As designed</p> <p>This input is restricted, based on the choice of chiller type, according to the following rules:</p> <table border="1" style="width: 100%; border-collapse: collapse;"> <thead> <tr> <th></th> <th>Electricity</th> <th>Gas</th> <th>Hot Water</th> <th>Steam</th> </tr> </thead> <tbody> <tr> <td>Reciprocating</td> <td>Allowed</td> <td></td> <td></td> <td></td> </tr> <tr> <td>Scroll</td> <td>Allowed</td> <td></td> <td></td> <td></td> </tr> <tr> <td>Screw</td> <td>Allowed</td> <td></td> <td></td> <td></td> </tr> <tr> <td>Centrifugal</td> <td>Allowed</td> <td></td> <td></td> <td></td> </tr> <tr> <td>Single Effect Absorption</td> <td></td> <td>Allowed</td> <td>Allowed</td> <td>Allowed</td> </tr> <tr> <td>Direct-Fired Double</td> <td></td> <td>Allowed</td> <td>Allowed</td> <td>Allowed</td> </tr> </tbody> </table>		Electricity	Gas	Hot Water	Steam	Reciprocating	Allowed				Scroll	Allowed				Screw	Allowed				Centrifugal	Allowed				Single Effect Absorption		Allowed	Allowed	Allowed	Direct-Fired Double		Allowed	Allowed	Allowed
	Electricity	Gas	Hot Water	Steam																																
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Centrifugal	Allowed																																			
Single Effect Absorption		Allowed	Allowed	Allowed																																
Direct-Fired Double		Allowed	Allowed	Allowed																																

	Effect Absorption				
	Indirect-Fired Absorption		Allowed	Allowed	Allowed
<i>Standard Design</i>	For healthcare facilities, same as the Proposed Design. For all others, Electricity				
<i>Standard Design: Existing Buildings</i>					

Chiller Rated Capacity	
<i>Applicability</i>	All chillers
<i>Definition</i>	The cooling capacity of a piece of heating equipment at rated conditions
<i>Units</i>	Btu/h or tons
<i>Input Restrictions</i>	As designed The user may need to manually adjust the capacity if the number of unmet load hours exceeds 150.
<i>Standard Design</i>	Determine loads for <u>standard design</u> baseline building and oversize by 15 percent
<i>Standard Design: Existing Buildings</i>	

Chiller Rated Efficiency	
<i>Applicability</i>	All chillers
<i>Definition</i>	The efficiency of the chiller (EER for air-cooled chillers, kW/ton for water-cooled electric chillers, and COP for fuel-fired and heat-driven chillers) at AHRI 550/590 rated full-load conditions
<i>Units</i>	Ratio (kW/ton, COP, EER, depending on chiller type and condenser type) Water-cooled electric chiller - kW/ton Air-cooled or evaporatively-cooled electric chiller - EER All non-electric chillers – COP
<i>Input Restrictions</i>	As designed Must meet the minimum requirements of Table 110.2-D.
<i>Standard Design</i>	Use the minimum efficiency requirements from Tables 110.2-D Path B. If chiller type is reciprocating, scroll, or screw, use the efficiency for positive displacement chillers from Table 110.2-D.

<i>Standard Design: Existing Buildings</i>	
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Integrated Part-Load Value	
<i>Applicability</i>	All chillers
<i>Definition</i>	The part-load efficiency of a chiller developed from a weighted average of four rating conditions, according to AHRI Standard 550
<i>Units</i>	Ratio (kW/ton, COP, EER, depending on chiller type and condenser type) Water-cooled electric chiller - kW/ton Air-cooled or evaporatively-cooled electric chiller - EER All non-electric chillers – COP
<i>Input Restrictions</i>	As designed; must meet the minimum requirements of Table 110.2-D
<i>Standard Design</i>	Not used When the standard design system has a chiller, the standard design will always use Path B performance curves.
<i>Standard Design: Existing Buildings</i>	

Chiller Minimum Unloading Ratio															
<i>Applicability</i>	All chillers														
<i>Definition</i>	<p>The minimum unloading capacity of a chiller expressed as a fraction of the rated capacity</p> <p>Below this level the chiller must either cycle to meet the load or false-load the compressor (such as with hot gas bypass).</p> <p style="text-align: center;">Table 293244: Default Minimum Unloading Ratios</p> <table border="1" style="margin-left: auto; margin-right: auto;"> <thead> <tr> <th>Chiller Type</th> <th>Default Unloading Ratio</th> </tr> </thead> <tbody> <tr> <td>Reciprocating</td> <td>25%</td> </tr> <tr> <td>Screw</td> <td>15%</td> </tr> <tr> <td>Centrifugal</td> <td>10%</td> </tr> <tr> <td>Scroll</td> <td>25%</td> </tr> <tr> <td>Single Effect Absorption</td> <td>10%</td> </tr> <tr> <td>Double Effect Absorption</td> <td>10%</td> </tr> </tbody> </table>	Chiller Type	Default Unloading Ratio	Reciprocating	25%	Screw	15%	Centrifugal	10%	Scroll	25%	Single Effect Absorption	10%	Double Effect Absorption	10%
Chiller Type	Default Unloading Ratio														
Reciprocating	25%														
Screw	15%														
Centrifugal	10%														
Scroll	25%														
Single Effect Absorption	10%														
Double Effect Absorption	10%														
<i>Units</i>	Percent (%)														

<i>Input Restrictions</i>	As designed but constrained to a minimum value of 10 percent. If the user does not employ the default values, supporting documentation is required.
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Use defaults listed above</u>
<i>Standard Design: Existing Buildings</i>	

Chiller Minimum Part Load Ratio	
<i>Applicability</i>	All chillers
<i>Definition</i>	<p>The minimum unloading capacity of a chiller expressed as a fraction of the rated capacity</p> <p>Below this level the chiller must cycle to meet the load. If the chiller minimum part-load ratio (PLR) is less than the chiller minimum unloading ratio, then the compliance software shall assume hot gas bypass operation between the minimum PLR and the minimum unloading ratio.</p>
<i>Units</i>	Percent (%)
<i>Input Restrictions</i>	As designed, but constrained to a minimum value of 10 percent. If the user does not employ the default values, supporting documentation is required.
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, When the standard design has a screw chiller, the minimum PLR is 15 percent. When the standard design has a centrifugal chiller, the minimum PLR is 10 percent.</u>
<i>Standard Design: Existing Buildings</i>	

Chiller Cooling Capacity Adjustment Curve	
<i>Applicability</i>	All chillers
<i>Definition</i>	<p>A curve or group of curves or other functions that represent the available total cooling capacity as a function of evaporator and condenser conditions and perhaps other operating conditions. The default curves are given as:</p> $Q_{available} = CAP_{FT}(Q_{rated})$ <p>For air-cooled chillers:</p> $CAP_{FT} = a + b(t_{chws}) + c(t_{chws})^2 + d(t_{odb}) + e(t_{odb})^2 + f(t_{chws})(t_{odb})$ <p>For water-cooled chillers:</p>

	$CAP_{FT} = a + b(t_{chws}) + c(t_{chws})^2 + d(t_{cws}) + e(t_{cws})^2 + f(t_{chws})(t_{cws})$ <p>Where:</p> <table border="1"> <tr> <td>$Q_{available}$</td> <td>Available cooling capacity at present evaporator and condenser conditions (MBH)</td> </tr> <tr> <td>t_{chws}</td> <td>The chilled water supply temperature (°F)</td> </tr> <tr> <td>t_{cws}</td> <td>The condenser water supply temperature (°F)</td> </tr> <tr> <td>t_{odb}</td> <td>The outside air dry-bulb temperature (°F)</td> </tr> <tr> <td>Q_{rated}</td> <td>Rated capacity at AHRI conditions (MBH)</td> </tr> </table> <p>Note: If an air-cooled unit employs an evaporative condenser, t_{odb} is the effective dry-bulb temperature of the air leaving the evaporative cooling unit.</p> <p>Separate curves are provided for Path A and Path B chillers in Appendix 5.7.</p>	$Q_{available}$	Available cooling capacity at present evaporator and condenser conditions (MBH)	t_{chws}	The chilled water supply temperature (°F)	t_{cws}	The condenser water supply temperature (°F)	t_{odb}	The outside air dry-bulb temperature (°F)	Q_{rated}	Rated capacity at AHRI conditions (MBH)
$Q_{available}$	Available cooling capacity at present evaporator and condenser conditions (MBH)										
t_{chws}	The chilled water supply temperature (°F)										
t_{cws}	The condenser water supply temperature (°F)										
t_{odb}	The outside air dry-bulb temperature (°F)										
Q_{rated}	Rated capacity at AHRI conditions (MBH)										
Units	Data structure										
Input Restrictions	Prescribed curves are provided in Appendix 5.7 for the proposed design chiller type and the compliance path (A or B). If the default curves are overridden, supporting documentation is required.										
Standard Design	For healthcare facilities, same as the Proposed Design (choose from available performance curves, no custom curves). For all others, Use prescribed curve for Path B chiller as applicable to the standard design chiller type.										
Standard Design: Existing Buildings											

Electric Chiller Cooling Efficiency Adjustment Curves	
Applicability	All chillers
Definition	<p>A curve or group of curves that varies the cooling efficiency of an electric chiller as a function of evaporator conditions, condenser conditions and part-load ratio.</p> <p>Note: For variable-speed chillers, the part-load cooling efficiency curve is a function of both part-load ratio and leaving condenser water temperature. The default curves are given as:</p> $PLR = \frac{Q_{operating}}{Q_{available}(t_{chws}, t_{cws/odb})}$ $EIR_{FPLR} = a + b(PLR) + c(PLR)^2$ <p>Variable Speed:</p> $EIR_{FPLR} = a + b(PLR) + c(PLR)^2$

	<p>Air-Cooled:</p> $EIR_{FT} = a + b(t_{chws}) + c(t_{chws})^2 + d(t_{odb}) + e(t_{odb})^2 + f(t_{chws})(t_{odb})$ <p>Water-Cooled:</p> $EIR_{FT} = a + b(t_{chws}) + c(t_{chws})^2 + d(t_{cws}) + e(t_{cws})^2 + f(t_{chws})(t_{cws})$ $P_{operating} = P_{rated}(EIR_{FPLR})(EIR_{FT})(CAP_{FT})$ <p>Where:</p> <table border="1"> <tr> <td>PLR</td> <td>Part-load ratio based on available capacity (not rated capacity)</td> </tr> <tr> <td>$Q_{operating}$</td> <td>Present load on chiller (Btu/h)</td> </tr> <tr> <td>$Q_{available}$</td> <td>Chiller available capacity at present evaporator and condenser conditions (Btu/h)</td> </tr> <tr> <td>t_{chws}</td> <td>The chilled water supply temperature (°F)</td> </tr> <tr> <td>t_{cws}</td> <td>The condenser water supply temperature (°F)</td> </tr> <tr> <td>t_{odb}</td> <td>The outside air dry-bulb temperature (°F)</td> </tr> <tr> <td>P_{rated}</td> <td>Rated power draw at AHRI conditions (kW)</td> </tr> <tr> <td>$P_{operating}$</td> <td>Power draw at specified operating conditions (kW)</td> </tr> </table> <p>Note: If an air-cooled chiller employs an evaporative condenser, t_{odb} is the effective dry-bulb temperature of the air leaving the evaporative cooling unit.</p>	PLR	Part-load ratio based on available capacity (not rated capacity)	$Q_{operating}$	Present load on chiller (Btu/h)	$Q_{available}$	Chiller available capacity at present evaporator and condenser conditions (Btu/h)	t_{chws}	The chilled water supply temperature (°F)	t_{cws}	The condenser water supply temperature (°F)	t_{odb}	The outside air dry-bulb temperature (°F)	P_{rated}	Rated power draw at AHRI conditions (kW)	$P_{operating}$	Power draw at specified operating conditions (kW)
PLR	Part-load ratio based on available capacity (not rated capacity)																
$Q_{operating}$	Present load on chiller (Btu/h)																
$Q_{available}$	Chiller available capacity at present evaporator and condenser conditions (Btu/h)																
t_{chws}	The chilled water supply temperature (°F)																
t_{cws}	The condenser water supply temperature (°F)																
t_{odb}	The outside air dry-bulb temperature (°F)																
P_{rated}	Rated power draw at AHRI conditions (kW)																
$P_{operating}$	Power draw at specified operating conditions (kW)																
<i>Units</i>	Data structure																
<i>Input Restrictions</i>	Curves are prescribed in Appendix 5.7 given the chiller capacity and type. A separate set of curves are provided for Path A chillers and Path B chillers. The path is determined by comparing software inputs of full-load efficiency and integrated part-load value with the requirements of standards Table 110.2-D.																
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Use Path B curves specified in Appendix 5.7</u>																
<i>Standard Design: Existing Buildings</i>																	

Fuel and Steam Chiller Cooling Efficiency Adjustment Curves	
<i>Applicability</i>	All chillers
<i>Definition</i>	A curve or group of curves that varies the cooling efficiency of a fuel-fired or steam chiller as a function of evaporator conditions, condenser conditions, and part-load ratio. The default curves are given as follows:

Default curves for steam-driven single and double effect absorption chillers:

$$PLR = \frac{Q_{operating}}{Q_{available}(t_{chws}, t_{cws/odb})}$$

$$FIR_{FPLR} = a + b(PLR) + c(PLR)^2$$

$$FIR_{FT} = a + b(t_{chws}) + c(t_{chws})^2 + d(t_{cws}) + e(t_{cws})^2 + f(t_{chws})(t_{cws})$$

$$Fuel_{partload} = (Fuel_{rated})(FIR_{FPLR})(FIR_{FT})(CAP_{FT})$$

Default curves for direct-fired double effect absorption chillers:

$$PLR = \frac{Q_{operating}}{Q_{available}(t_{chws}, t_{cws/odb})}$$

$$FIR_{FPLR} = a + b(PLR) + c(PLR)^2$$

$$FIR_{FT1} = a + b(t_{chws}) + c(t_{chws})^2$$

$$FIR_{FT2} = d + e(t_{cws}) + f(t_{cws})^2$$

$$Fuel_{partload} = (Fuel_{rated})(FIR_{FPLR})(FIR_{FT1})(FIR_{FT2})(CAP_{FT})$$

The default curves for engine driven chillers are the same format as those for the steam-driven single and double effect absorption chillers but there are three sets of curves for different ranges of operation based on the engine speed.

Where:

PLR	Part-load ratio based on available capacity (not rated capacity)
FIR_{FPLR}	A multiplier on the fuel input ratio (FIR) to account for part-load conditions
FIR_{FT}	A multiplier on the fuel input ratio (FIR) to account for the chiller water supply temperature and the condenser water temperature
FIR_{FT1}	A multiplier on the fuel input ratio (FIR) to account for chilled water supply temperature

	FIR_{FT2}	A multiplier on the fuel input ratio (FIR) to account for condenser water supply temperature
	CAP_{FT}	A multiplier on the capacity of the chiller (Equation 45)
	$Q_{operating}$	Present load on chiller (in Btu/h)
	$Q_{available}$	Chiller available capacity at present evaporator and condenser conditions (in Btu/h)
	t_{chws}	The chilled water supply temperature (in °F)
	t_{cws}	The condenser water supply temperature (in °F)
	t_{odb}	The outside air dry-bulb temperature (°F)
	$Fuel_{rated}$	Rated fuel consumption at AHRI conditions (in Btu/h)
	$Fuel_{partload}$	Fuel consumption at specified operating conditions (in Btu/h)
Units	Data structure	
Input Restrictions	Restricted to curves specified in Appendix 5.7	
Standard Design	Use prescribed curves specified in Appendix 5.7	
Standard Design: Existing Buildings		

Chilled Water Supply Temperature	
Applicability	All chillers
Definition	The chilled water supply temperature of the chiller at design conditions
Units	Degrees Fahrenheit (°F)
Input Restrictions	As designed
Standard Design	For healthcare facilities, same as the Proposed Design. For all others, The <u>standard design</u> baseline chilled water supply temperature is set to 44°F.
Standard Design: Existing Buildings	

Chilled Water Return Temperature	
Applicability	All chillers
Definition	The chilled water return temperature setpoint at design conditions
Units	Degrees Fahrenheit (°F)

<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> The standard design chilled water return temperature is set to 64°F.
<i>Standard Design: Existing Buildings</i>	

Chilled Water Supply Temperature Control Type	
<i>Applicability</i>	All chillers
<i>Definition</i>	The method by which the chilled water setpoint temperature is reset The chilled water setpoint may be reset based on demand or outdoor air temperature.
<i>Units</i>	List none, outside air-based reset, or demand-based reset
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Outside air-based reset
<i>Standard Design: Existing Buildings</i>	

Chilled Water Supply Temperature Reset	
<i>Applicability</i>	All chillers
<i>Definition</i>	The reset schedule for the chilled water supply temperature. The chilled water setpoint may be reset based on demand or outdoor air temperature.
<i>Units</i>	Degrees Fahrenheit (°F)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> 10°F from design chilled water supply temperature The chilled water supply temperature reset follows an outside air reset scheme, where the setpoint is 44°F at outside air conditions of 80°F dry-bulb and above; the setpoint is 54°F at outside air conditions of 60°F dry-bulb and below; and ramps linearly from 44°F to 54°F as the outside air dry-bulb temperature varies between 80°F and 60°F.
<i>Standard Design: Existing Buildings</i>	

Condenser Type	
<i>Applicability</i>	All chillers
<i>Definition</i>	<p>The type of condenser for a chiller</p> <p>The choices are:</p> <ul style="list-style-type: none"> • Air-cooled • Water-cooled • Evaporatively-cooled <p>Air-cooled chillers use air to cool the condenser coils. Water-cooled chillers use cold water to cool the condenser and additionally need either a cooling tower or a local source of cold water. Evaporatively-cooled chillers are similar to air-cooled chillers, except a water mist is used to cool the condenser coil, making them more efficient.</p>
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, the standard design</u> baseline chiller is always assumed to have a water-cooled condenser, although the chiller type will change depending on the design capacity.
<i>Standard Design: Existing Buildings</i>	

5.8.3 Cooling Towers

Standard Design Summary. Standard design system 6 has one or more cooling towers. One tower is assumed to be matched to each standard design chiller. Each standard design chiller has its own condenser water pump that operates when the chiller is brought into service. The range between the condenser water return (CWR) and condenser water supply (CWS) is 10°F. The standard design ~~baseline~~ building condenser pumping energy is assumed to be 12 W/gpm. The cooling tower is assumed to have a variable-speed fan that is controlled to provide a CWS equal to the design wet-bulb temperature when weather permits. The design approach shall be 10°F.

Cooling Tower Name	
<i>Applicability</i>	All cooling towers
<i>Definition</i>	A unique descriptor for each cooling tower
<i>Units</i>	Text, unique
<i>Input Restrictions</i>	User entry; where applicable, this should match the tags that are used on the plans
<i>Standard Design</i>	Descriptive name that keys the <u>standard design</u> baseline building plant

<i>Standard Design: Existing Buildings</i>	
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Cooling Tower Type	
<i>Applicability</i>	All cooling towers
<i>Definition</i>	Type of cooling tower employed. The choices are: <ul style="list-style-type: none"> • Open tower, centrifugal fan • Open tower, axial fan Open cooling towers collect the cooled water from the tower and pump it directly back to the cooling system. Closed towers circulate the evaporated water over a heat exchanger to indirectly cool the system fluid.
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, The standard design</u> baseline cooling tower is an open tower axial fan device
<i>Standard Design: Existing Buildings</i>	

Cooling Tower Capacity	
<i>Applicability</i>	All cooling towers
<i>Definition</i>	The tower thermal capacity per cell adjusted to Cooling Technology Institute (CTI) rated conditions of 95°F condenser water return, 85°F condenser water supply, and 78°F wet-bulb with a 3 gpm/nominal ton water flow. The default cooling tower curves below are at unity at these conditions.
<i>Units</i>	Btu/h
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	The <u>standard design</u> baseline building chiller is autosized and increased by 15 percent. The tower is sized to supply 85°F condenser water at design conditions for the oversized chiller.

<i>Standard Design: Existing Buildings</i>	
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Cooling Tower Number of Cells	
<i>Applicability</i>	All cooling towers
<i>Definition</i>	The number of cells in the cooling tower Each cell will be modeled as equal size. Cells are subdivisions in cooling towers into individual cells, each with their own fan and water flow, that allow the cooling system to respond more efficiently to lower load conditions.
<i>Units</i>	Numeric: integer
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	One cell per tower and one tower per chiller
<i>Standard Design: Existing Buildings</i>	

Cooling Tower Total Fan HorsepowerPower	
<i>Applicability</i>	All cooling towers
<i>Definition</i>	The sum of the nameplate rated horsepower (hp) of all fan motors on the cooling tower. Pony motors should not be included.
<i>Units</i>	Gpm/hp or unitless if energy input ratio (EIR) is specified (if the nominal tons but not the condenser water flow is specified, the condenser design water flow shall be 3.0 gpm per nominal cooling ton).
<i>Input Restrictions</i>	As designed, but the cooling towers shall meet minimum performance requirements in Table 110.2-G
<i>Standard Design</i>	For healthcare facilities, same as the Proposed Design. For all others, The cooling tower fan horsepower is 60 gpm/hp, with the following exceptions: <ol style="list-style-type: none"> 1) <u>Cooling towers in climate zones 1 or 16 shall set the standard design to the mandatory minimum, 42.1 gpm/hp</u> 2) <u>Cooling towers with a design condenser water flow of 900 gpm or less shall set the standard design to the mandatory minimum, 42.1 gpm/hp</u>
<i>Standard Design: Existing Buildings</i>	<u>42.1 gpm/hp</u>

Cooling Tower Design Wet-Bulb	
<i>Applicability</i>	All cooling towers
<i>Definition</i>	The design wet-bulb temperature that was used for selection and sizing of the cooling tower
<i>Units</i>	Degrees Fahrenheit (°F)
<i>Input Restrictions</i>	Specified from design wet-bulb conditions from Reference Appendix JA2 for the city where the building is located, or the city closest to where the building is located
<i>Standard Design</i>	Specified from design wet-bulb conditions from Reference Appendix JA2 for the city where the building is located, or from the city closest to where the building is located
<i>Standard Design: Existing Buildings</i>	

Cooling Tower Design Entering Water Temperature	
<i>Applicability</i>	All cooling towers
<i>Definition</i>	The design condenser water supply temperature (leaving tower) that was used for selection and sizing of the cooling tower
<i>Units</i>	Degrees Fahrenheit (°F)
<i>Input Restrictions</i>	As designed; default to 85°F
<i>Standard Design</i>	85°F or 10°F above the design wet-bulb temperature, whichever is lower
<i>Standard Design: Existing Buildings</i>	

Cooling Tower Design Return Water Temperature	
<i>Applicability</i>	All cooling towers
<i>Definition</i>	The design condenser water return temperature (entering tower) that was used for selection and sizing of the cooling tower
<i>Units</i>	Degrees Fahrenheit (°F)
<i>Input Restrictions</i>	As designed; default to 95°F
<i>Standard Design</i>	Set to a range of 10°F (10°F above the cooling tower design entering water temperature)
<i>Standard Design: Existing Buildings</i>	

Cooling Tower Capacity Adjustment Curve	
<i>Applicability</i>	All cooling towers
<i>Definition</i>	<p>A curve or group of curves that represent the available total cooling capacity as a function of outdoor air wet-bulb, condenser water supply, and condenser water return temperatures.</p> <p>The default curves are given as follows:</p> $\begin{aligned} \text{Approach} = & \text{Coeff}(1) + \text{Coeff}(2) \cdot \text{FRair} + \text{Coeff}(3) \cdot (\text{FRair})^2 \\ & + \text{Coeff}(4) \cdot (\text{FRair})^3 + \text{Coeff}(5) \cdot \text{FRwater} + \text{Coeff}(6) \cdot \text{FRair} \cdot \text{FRwater} + \\ & \text{Coeff}(7) \cdot (\text{FRair})^2 \cdot \text{FRwater} + \text{Coeff}(8) \cdot (\text{FRwater})^2 + \\ & \text{Coeff}(9) \cdot \text{FRair} \cdot (\text{FRwater})^2 + \\ & \text{Coeff}(10) \cdot (\text{FRwater})^3 + \text{Coeff}(11) \cdot \text{Twb} + \text{Coeff}(12) \cdot \text{FRair} \cdot \text{Twb} + \\ & \text{Coeff}(13) \cdot (\text{FRair})^2 \cdot \text{Twb} + \text{Coeff}(14) \cdot \text{FRwater} \cdot \text{Twb} + \\ & \text{Coeff}(15) \cdot \text{FRair} \cdot \text{FRwater} \cdot \text{Twb} + \text{Coeff}(16) \cdot (\text{FRwater})^2 \cdot \text{Twb} + \\ & \text{Coeff}(17) \cdot (\text{Twb})^2 + \text{Coeff}(18) \cdot \text{FRair} \cdot (\text{Twb})^2 + \text{Coeff}(19) \cdot \text{FRwater} \cdot (\text{Twb})^2 \\ & + \text{Coeff}(20) \cdot (\text{Twb})^3 + \text{Coeff}(21) \cdot \text{Tr} + \text{Coeff}(22) \cdot \text{FRair} \cdot \text{Tr} + \\ & \text{Coeff}(23) \cdot \text{FRair} \cdot \text{FRair} \cdot \text{Tr} + \text{Coeff}(24) \cdot \text{FRwater} \cdot \text{Tr} + \\ & \text{Coeff}(25) \cdot \text{FRair} \cdot \text{FRwater} \cdot \text{Tr} + \\ & \text{Coeff}(26) \cdot (\text{FRwater})^2 \cdot \text{Tr} + \text{Coeff}(27) \cdot \text{Twb} \cdot \text{Tr} + \text{Coeff}(28) \cdot \text{FRair} \cdot \text{Twb} \cdot \text{Tr} + \\ & \text{Coeff}(29) \cdot \text{FRwater} \cdot \text{Twb} \cdot \text{Tr} + \\ & \text{Coeff}(30) \cdot (\text{Twb})^2 \cdot \text{Tr} + \text{Coeff}(31) \cdot (\text{Tr})^2 + \text{Coeff}(32) \cdot \text{FRair} \cdot (\text{Tr})^2 + \\ & \text{Coeff}(33) \cdot \text{FRwater} \cdot (\text{Tr})^2 + \text{Coeff}(34) \cdot \text{Twb} \cdot (\text{Tr})^2 + \text{Coeff}(35) \cdot (\text{Tr})^3 \end{aligned}$ <p>Where:</p> <p>FRair – ratio of airflow to airflow at design conditions</p> <p>FRwater – ratio of water flow to water flow at design conditions</p> <p>Tr – tower range (°F)</p> <p>Twb – wet-bulb temperature</p> <p>Coefficients for this performance curve are provided in Appendix 5.7.</p>
<i>Units</i>	Data structure
<i>Input Restrictions</i>	User must use one of the prescribed curves defined in Appendix 5.7
<i>Standard Design</i>	Use one of the prescribed curves defined in Appendix 5.7
<i>Standard Design: Existing Buildings</i>	

Cooling Tower Set Point Control

<i>Applicability</i>	All cooling towers
<i>Definition</i>	<p>The type of control for the condenser water supply.</p> <p>The choices are fixed or wet-bulb reset.</p> <p>A fixed control will modulate the tower fans to provide the design condenser water supply temperature at all times when possible. A wet-bulb reset control will reset the condenser water setpoint to a fixed approach to outside air wet-bulb temperature. The approach defaults to 10°F. A lower approach may be used with appropriate documentation.</p>
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed; default to 95°F
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Fixed at the 0.4 percent design wet-bulb temperature, which is prescribed and specified for each of the 86 weather data files</u>
<i>Standard Design: Existing Buildings</i>	

Cooling Tower Capacity Control	
<i>Applicability</i>	All cooling towers
<i>Definition</i>	<p>Describes the modulation control employed in the cooling tower.</p> <p>Choices include:</p> <ul style="list-style-type: none"> • Fluid Bypass provides a parallel path to divert some of the condenser water around the cooling tower at part-load conditions. • Fan Cycling is a simple method of capacity control where the tower fan is cycled on and off. This is often used on multiple-cell installations. • Two-Speed Fan/Pony Motor are the same from an energy perspective. A lower horsepower pony motor is an alternative to a two-speed motor. The pony motor runs at part-load conditions (instead of the full-sized motor) and saves fan energy when the tower load is reduced. Additional building descriptors are triggered when this method of capacity control is selected. • Variable-Speed Fan is a variable frequency drive is installed for the tower fan so that the speed can be modulated.
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Variable-speed fan</u>

<i>Standard Design: Existing Buildings</i>	
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Cooling Tower Low-Speed Airflow Ratio	
<i>Applicability</i>	All cooling towers with two-speed or pony motors
<i>Definition</i>	The percentage full-load airflow that the tower has at low speed or with the pony motor operating; equivalent to the percentage full-load capacity when operating at low speed
<i>Units</i>	Ratio
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Not applicable</u>
<i>Standard Design: Existing Buildings</i>	

Cooling Tower Low-Speed kW Ratio	
<i>Applicability</i>	All cooling towers with two-speed or pony motors
<i>Definition</i>	The percentage full-load power that the tower fans draw at low speed or with the pony motor operating
<i>Units</i>	Ratio
<i>Input Restrictions</i>	Calculated, using the as-designed flow ratio and the cooling tower power adjustment curve below
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Not applicable</u>
<i>Standard Design: Existing Buildings</i>	

Cooling Tower Power Adjustment Curve	
<i>Applicability</i>	All cooling towers with VSD control
<i>Definition</i>	<p>A curve that varies the cooling tower fan energy usage as a function of part-load ratio for cooling towers with variable speed fan control. The default curve is given as:</p> $PLR = \frac{Q_{operating}}{Q_{available}(t_R, t_A, t_{OWB})}$ $TWR_{Fan-FPLR} = a + b(PLR) + c(PLR)^2 + d(PLR)^3$

$P_{operating} = P_{rated}(TWR_{Fan-FPLR})$	
Where:	
<i>PLR</i>	Part-load ratio based on available capacity (not rated capacity)
<i>Q_{operating}</i>	Present load on tower (in Btu/h)
<i>Q_{available}</i>	Tower available capacity at present range, approach, and outside wet-bulb conditions (in Btu/h)
<i>t_{OWB}</i>	The outside air wet-bulb temperature (°F)
<i>t_R</i>	The tower range (°F)
<i>t_A</i>	The tower approach (°F)
<i>P_{rated}</i>	Rated power draw at CTI conditions (kW)
<i>P_{operating}</i>	Power draw at specified operating conditions (kW)
Table 303345: Default Efficiency TWR_{Fan-FPLR} Coefficients - VSD on Cooling Tower Fan	
Coefficient	<i>TWR_{Fan-FPLR}</i>
<i>a</i>	0.33162901
<i>b</i>	-0.88567609
<i>c</i>	0.60556507
<i>d</i>	0.9484823
<i>Units</i>	Data structure
<i>Input Restrictions</i>	User shall use only default curves
<i>Standard Design</i>	Use default curves given above
<i>Standard Design: Existing Buildings</i>	

Cooling Tower Minimum Speed	
<i>Applicability</i>	All cooling towers with a VSD control
<i>Definition</i>	The minimum fan speed setting of a VSD controlling a cooling tower fan expressed as a ratio of full load speed
<i>Units</i>	Ratio
<i>Input Restrictions</i>	As designed; default is 0.50

<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, 0.5</u>
<i>Standard Design: Existing Buildings</i>	

5.8.4 Water-side Economizers

Standard Design~~Baseline Building Summary~~ - None of the standard design~~baseline~~ building systems use a water-side economizer.

Water-Side Economizer Name	
<i>Applicability</i>	All water-side economizers
<i>Definition</i>	The name of a water-side economizer for a cooling system
<i>Units</i>	Text, unique
<i>Input Restrictions</i>	Descriptive reference to the construction documents; default is no water-side economizer
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, No water economizer</u>
<i>Standard Design: Existing Buildings</i>	

Water Economizer Type	
<i>Applicability</i>	All water-side economizers
<i>Definition</i>	<p>The type of water-side economizer. Choices include:</p> <ul style="list-style-type: none"> • None • Heat exchanger in parallel with chillers. This would be used with an open cooling tower and is often referred to as a non-integrated economizer because the chillers are locked out when the plant is in economizer mode. • Heat exchanger in series with chillers. This would be used with an open cooling tower and is often referred to as integrated because the chillers can operate simultaneously with water economizer operation. • Direct water economizer. This would be used with a closed cooling tower. In this case, a heat exchanger is not needed. This type works only as a non-integrated economizer (also known as strainer-cycle).
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed

<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, No water economizer</u>
<i>Standard Design: Existing Buildings</i>	

Water-Side Economizer HX Effectiveness

<i>Applicability</i>	Water-side economizers with an open cooling tower																		
<i>Definition</i>	<p>The effectiveness of a water-side heat exchanger at design conditions This is defined as:</p> $Q_{econ} = (m_{CHW})(Cp_{CHW})(\epsilon)(T_{CHW,R} - T_{CW,S})$ <p>Where:</p> <table border="1"> <tr> <td>Q_{econ}</td> <td>The maximum load that the economizer can handle</td> </tr> <tr> <td>m_{CHW}</td> <td>The chilled water flow rate</td> </tr> <tr> <td>Cp_{CHW}</td> <td>The chilled water specific heat</td> </tr> <tr> <td>$T_{CHW,R}$</td> <td>The chilled water return temperature</td> </tr> <tr> <td>$T_{CW,S}$</td> <td>The condenser water supply temperature</td> </tr> <tr> <td>WSEff</td> <td>The effectiveness of the water-side economizer coil</td> </tr> <tr> <td>tea</td> <td>The entering coil air dry-bulb temperature (°F)</td> </tr> <tr> <td>tla</td> <td>The leaving coil air dry-bulb temperature (°F)</td> </tr> <tr> <td>tea</td> <td>The entering coil water temperature (°F)</td> </tr> </table>	Q_{econ}	The maximum load that the economizer can handle	m_{CHW}	The chilled water flow rate	Cp_{CHW}	The chilled water specific heat	$T_{CHW,R}$	The chilled water return temperature	$T_{CW,S}$	The condenser water supply temperature	WSEff	The effectiveness of the water-side economizer coil	tea	The entering coil air dry-bulb temperature (°F)	tla	The leaving coil air dry-bulb temperature (°F)	tea	The entering coil water temperature (°F)
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tla	The leaving coil air dry-bulb temperature (°F)																		
tea	The entering coil water temperature (°F)																		
<i>Units</i>	Ratio																		
<i>Input Restrictions</i>	As designed; default is 60 percent																		
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, No water economizer</u>																		
<i>Standard Design: Existing Buildings</i>																			

Water-Side Economizer Heat Exchanger Heat Transfer Coefficient Value

<i>Applicability</i>	Water-side economizers with an open cooling tower
<i>Definition</i>	The heat transfer coefficient of the plate-and-frame heat exchanger with the waterside economizer

<i>Units</i>	Btu/h-°F
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Not applicable</u>
<i>Standard Design: Existing Buildings</i>	

Water-Side Economizer Approach	
<i>Applicability</i>	All water-side economizers
<i>Definition</i>	The design temperature difference between the chilled water temperature leaving the heat exchanger and the condenser water inlet to the heat exchanger
<i>Units</i>	Degrees Fahrenheit (°F)
<i>Input Restrictions</i>	As designed; default is 34°F
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, No water economizer</u>
<i>Standard Design: Existing Buildings</i>	

Water-Side Economizer Maximum CWS	
<i>Applicability</i>	All water-side economizers
<i>Definition</i>	The control temperature (condenser water supply temperature) above which the water-side economizer is disabled
<i>Units</i>	Degrees Fahrenheit (°F)
<i>Input Restrictions</i>	As designed; default is 50°F
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, No water economizer</u>
<i>Standard Design: Existing Buildings</i>	

Water-Side Economizer Availability Schedule	
<i>Applicability</i>	All water-side economizers
<i>Definition</i>	A schedule which represents the availability of the water-side economizer

<i>Units</i>	Data structure: schedule, on/off
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> No water economizer
<i>Standard Design: Existing Buildings</i>	

Water-Side Economizer Auxiliary kW	
<i>Applicability</i>	Water-side economizers with an open tower
<i>Definition</i>	The electrical input (pumps and auxiliaries) for a dedicated pump for the chilled water side of the heat exchanger This power is in excess of the condenser water pumps and cooling tower fans for the system during water-side economizer operation.
<i>Units</i>	KW or kW/ton
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> No water economizer
<i>Standard Design: Existing Buildings</i>	

5.8.5 Pumps

Standard Design Baseline Building Summary - Hot water pumping in the standard design baseline building shall be modeled as a variable flow, primary only system. Two-way valves are assumed at the heating coils.

Chilled water pumping in the standard design baseline building (system 6) is a primary system. Each chiller has its own primary and condenser water pumps that operate when the chiller is activated.

General Notes - The building descriptors in this section are repeated for each pumping system. See the pump service building descriptor for a list of common pump services.

Pump Name	
<i>Applicability</i>	All pumps
<i>Definition</i>	A unique descriptor for each pump
<i>Units</i>	Text, unique
<i>Input Restrictions</i>	User entry; were applicable, should match the tags that are used on the plans

<i>Standard Design</i>	Same as proposed design If there is no equivalent in the proposed design, assign a sequential tag to each piece of equipment. The sequential tags should indicate the pump service as part of the descriptor (e.g., CW for condenser water, CHW for chilled water, or HHW for heating hot water).
<i>Standard Design: Existing Buildings</i>	

Pump Service	
<i>Applicability</i>	All pumps
<i>Definition</i>	The service for each pump. Choices include: <ul style="list-style-type: none"> • Chilled water • Chilled water (primary) • Chilled water (secondary) • Heating water • Heating water (primary) • Heating water (secondary) • Service hot water • Condenser water • Loop water (for hydronic heat pumps)
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	As needed by the <u>standard design</u> baseline building system
<i>Standard Design: Existing Buildings</i>	

Number of Pumps	
<i>Applicability</i>	All pumps
<i>Definition</i>	The number of identical pumps in service in a particular loop, e.g., the heating hot water loop, chilled water loop, or condenser water loop
<i>Units</i>	Numeric: integer
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	There will be one heating hot water pump for each boiler, one chilled water pump, and one condenser water pump for each chiller.

<i>Standard Design: Existing Buildings</i>	
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Water Loop Design	
<i>Applicability</i>	All pumps
<i>Definition</i>	The heating and cooling delivery systems can consist of a simple primary loop system, or more complicated primary/secondary loops or primary/secondary/tertiary loops
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, Assume primary loops only for heating hot water; for chilled water loops, a primary loop design is assumed.</u>
<i>Standard Design: Existing Buildings</i>	

Pump Motor Modeling Method	
<i>Applicability</i>	All pumps
<i>Definition</i>	Software commonly models pumps in one of two ways. The simple method is for the user to enter the electric power per unit of flow (W/gpm). This method is commonly used for smaller systems. A more detailed method requires a specification of the pump head, design flow, impeller, and motor efficiency.
<i>Units</i>	List power-per-unit-flow or detailed
<i>Input Restrictions</i>	Detailed
<i>Standard Design</i>	Detailed for chilled water and condenser water pumps; power-per-unit-flow for heating hot water and service hot water pumps
<i>Standard Design: Existing Buildings</i>	

Pump Motor Power-Per-Unit-Flow	
<i>Applicability</i>	All proposed design pumps that use the power-per-unit-flow method
<i>Definition</i>	The electric power of the pump divided by the flow at design conditions
<i>Units</i>	W/gpm
<i>Input Restrictions</i>	As designed

<i>Standard Design</i>	For healthcare facilities, same as the Proposed Design. For all others, Not applicable for chilled water and condenser water pumps; 19 W/gpm for heating hot water and service hot water pumps
<i>Standard Design: Existing Buildings</i>	

Pump Motor Horsepower	
<i>Applicability</i>	All pumps
<i>Definition</i>	The nameplate motor horsepower
<i>Units</i>	Horsepower (hp)
<i>Input Restrictions</i>	Constrained to be a value from the following standard motor sizes: 1/12, 1/8, 1/4, 1/2, 3/4, 1, 1.5, 2, 3, 5, 7.5, 10, 15, 20, 25, 30, 40, 50, 60, 75, 100, 125, 150, 200
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

Pump Design Head					
<i>Applicability</i>	All standard and proposed design baseline building pumps and proposed design pumps that use the detailed method				
<i>Definition</i>	The head of the pump at design flow conditions				
<i>Units</i>	ft of wc				
<i>Input Restrictions</i>	<p>As designed but subject to an input restriction. The user inputs of pump design head, impeller efficiency, cooling tower design entering water temperature, and cooling tower design return water temperature shall be used to calculate the proposed brake horsepower. This shall be compared to the pump motor horsepower for the next smaller motor size (MHP_{i-1}) than the one specified by the user (MHP_i).</p> <p>The proposed design for the pump design head shall be constrained so that the resulting brake horsepower is no smaller than 95 percent of the next smaller motor size:</p> $design\ bhp_{prop} = \max[design\ bhp_{prop-user-head}, 0.95(MHP_{i-1})]$ <p>Where:</p> <table border="1" style="width: 100%;"> <tr> <td><i>design bhp_{prop}</i></td> <td>The brake horsepower used in the simulation</td> </tr> <tr> <td><i>design bhp_{prop-user-head}</i></td> <td>The brake horsepower resulting from the user input of design head</td> </tr> </table>	<i>design bhp_{prop}</i>	The brake horsepower used in the simulation	<i>design bhp_{prop-user-head}</i>	The brake horsepower resulting from the user input of design head
<i>design bhp_{prop}</i>	The brake horsepower used in the simulation				
<i>design bhp_{prop-user-head}</i>	The brake horsepower resulting from the user input of design head				

	MHP_i	The pump motor horsepower specified by the user
	i	The index into the standard motor size table for the user motor horsepower
	MHP_{i-1}	The motor horsepower for the next smaller motor size. For example, if the user-specified pump motor horsepower is 25, the next smaller motor size in the table above is 20
<p>Since all other user inputs that affect the proposed design brake horsepower are not modified, the proposed design pump design head is adjusted in the same proportion as the pump brake horsepower in the equation above. If the user-entered pump design head results in a brake horsepower that is at least 95 percent of the horsepower of the next smaller motor size, no modification of the user input is required.</p>		
<i>Standard Design</i>	<p><u>For healthcare facilities, same as the Proposed Design. For all others,</u> For chilled water pumps: $(40 \text{ ft}) + (0.03 \text{ ft/ton}) \times [\text{chiller plant nominal capacity (tons)}]$ (not to exceed 100 ft) For condenser water pumps: 45 ft</p>	
<i>Standard Design: Existing Buildings</i>		

Impeller Efficiency	
<i>Applicability</i>	All pumps in proposed design that use the detailed modeling method
<i>Definition</i>	The full load efficiency of the impeller
<i>Units</i>	Ratio
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> Not applicable
<i>Standard Design: Existing Buildings</i>	

Motor Efficiency	
<i>Applicability</i>	All pumps in proposed design that use the detailed modeling method

<i>Definition</i>	The full load efficiency of the pump motor
<i>Units</i>	Ratio
<i>Input Restrictions</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, As designed</u>
<i>Standard Design</i>	The motor efficiency is taken from Table 29, using the next larger motor size than the calculated standard design brake horsepower
<i>Standard Design: Existing Buildings</i>	

Pump Minimum Speed	
<i>Applicability</i>	All two-speed or variable-speed pumps
<i>Definition</i>	The minimum pump speed for a two-speed for variable-speed pump. For two-speed pumps this is typically 0.67 or 0.5. Note: The pump minimum speed is not necessarily the same as the minimum flow ratio, since the system head may change.
<i>Units</i>	Ratio
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, 0.10</u>
<i>Standard Design: Existing Buildings</i>	

Pump Minimum Flow Ratio	
<i>Applicability</i>	Primary chilled water pumps
<i>Definition</i>	The minimum fraction of design flow when the pump is operating at its minimum speed. The pump minimum speed is not necessarily the same as the minimum flow ratio, since the system head may change.
<i>Units</i>	Ratio
<i>Input Restrictions</i>	As designed For compliance software that cannot model intermittent pump operation, the minimum flow ratio is set to 0 and the minimum power per the part-load performance curve is fixed at 0.103.
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others, For compliance software that cannot model intermittent pump operation, the minimum flow ratio is set to 0 and the minimum power per the part-</u>

	load performance curve is fixed at 0.103.
<i>Standard Design: Existing Buildings</i>	

Pump Design Flow (GPM)	
<i>Applicability</i>	All pumps
<i>Definition</i>	The flow rate of the pump at design conditions; derived from the load, and the design supply and return temperatures
<i>Units</i>	gpm or gpm/ton for condenser and primary chilled water pumps
<i>Input Restrictions</i>	Not a user input
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> The temperature change on the evaporator side of the chillers is 20°F (64°F less 44°F) and this equates to a flow of 1.2 gpm/ton. The temperature change on the condenser side of the chillers is 12°F, which equates to a flow of 2.0 gpm/cooling ton. A VSD is required for heating pumps when the service area is greater than or equal to 120,000 ft ² . For hot water pumps servicing boilers, the flow rate in gpm shall correspond to a loop temperature drop of 40°F.
<i>Standard Design: Existing Buildings</i>	

Pump Control Type	
<i>Applicability</i>	All pumps
<i>Definition</i>	The type of control for the pump Choices are: <ul style="list-style-type: none"> • Fixed speed, fixed flow • Fixed speed, variable flow (the default, with flow control via a valve) • Two-speed • Variable speed, variable flow
<i>Units</i>	List, see above
<i>Input Restrictions</i>	As designed; default is “fixed speed, variable flow”, which models the action of a constant speed pump riding the curve against two-way control valves
<i>Standard Design</i>	<u>For healthcare facilities, same as the Proposed Design. For all others,</u> The chilled water pumps shall be modeled as variable-speed, variable flow, and the condenser water pumps shall be modeled as fixed speed. The hot water pumps shall be modeled as fixed-speed, variable flow, riding the curve.

<i>Standard Design:</i> <i>Existing Buildings</i>	
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Pump Operation	
<i>Applicability</i>	All pumps
<i>Definition</i>	The type of pump operation can be either on-demand, standby, or scheduled. On-demand operation means the pumps are only pumping when their associated equipment is cycling. Chiller and condenser pumps are on when the chiller is on and the heating hot water pump operates when its associated boiler is cycling. Standby operation allows hot or chilled water to circulate through the primary loop of a primary/secondary loop system or through a reduced portion of a primary-only system, assuming the system has appropriate three-way valves. Scheduled operation means that the pumps and their associated equipment are turned completely off according to occupancy schedules, time of year, or outside conditions. Under scheduled operation, when the systems are on they are assumed to be in on-demand mode.
<i>Units</i>	List (see above)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	The <u>standard design</u> baseline system pumps are assumed to operate in on-demand mode. The chilled water and condenser pumps are tied to the chiller operation, cycling on and off with the chiller, and the heating hot water pumps are tied to the boiler operation.
<i>Standard Design:</i> <i>Existing Buildings</i>	

Pump Part-Load Curve							
<i>Applicability</i>	All pumps						
<i>Definition</i>	<p>A part-load power curve for the pump:</p> $CIRC - PUMP - FPLR = a + b(PLR) + c(PLR)^2 + d(PLR)^3$ $P_{pump} = P_{design}(CIRC - PUMP - FPLR)$ <p>Where:</p> <table border="1" style="width: 100%; border-collapse: collapse;"> <tr> <td style="text-align: center;">PLR</td> <td>Part-load ratio (the ratio of operating flow rate in gpm to design flow rate in gpm)</td> </tr> <tr> <td style="text-align: center;">P_{pump}</td> <td>Pump power draw at part-load conditions (W)</td> </tr> <tr> <td style="text-align: center;">P_{design}</td> <td>Pump power draw at design conditions (W)</td> </tr> </table>	PLR	Part-load ratio (the ratio of operating flow rate in gpm to design flow rate in gpm)	P_{pump}	Pump power draw at part-load conditions (W)	P_{design}	Pump power draw at design conditions (W)
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P_{design}	Pump power draw at design conditions (W)						

	<table border="1"> <thead> <tr> <th>Coefficient</th> <th>Default (No Reset)</th> <th>DP Reset</th> </tr> </thead> <tbody> <tr> <td>A</td> <td>0</td> <td>0</td> </tr> <tr> <td>B</td> <td>0.5726</td> <td>0.0205</td> </tr> <tr> <td>C</td> <td>-0.301</td> <td>0.4101</td> </tr> <tr> <td>D</td> <td>0.7347</td> <td>0.5753</td> </tr> </tbody> </table>	Coefficient	Default (No Reset)	DP Reset	A	0	0	B	0.5726	0.0205	C	-0.301	0.4101	D	0.7347	0.5753
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A	0	0														
B	0.5726	0.0205														
C	-0.301	0.4101														
D	0.7347	0.5753														
<i>Units</i>	Data structure															
<i>Input Restrictions</i>	Default is “Default (No Reset)”. The Differential Pressure (DP) reset curve may only be selected if the DDC control type building descriptor indicates that the building has DDC controls.															
<i>Standard Design</i>	For healthcare facilities, same as the Proposed Design. For all others, DP Reset curve for chilled water pumps; heating hot water pump power is assumed to be constant even though the pump is riding the curve.															
<i>Standard Design: Existing Buildings</i>																

5.8.6 Variable Refrigerant Flow (VRF) Systems

Variable refrigerant flow systems consist of an outdoor unit and one or more zonal systems as indoor units. The required system level inputs are shown below. Refer to the HVAC zone level systems section for zonal (indoor) units connected to a VRF system. Equipment performance curves are prescribed and defined in Appendix 5.4B for VRF systems.

<u>VRF System Name</u>	
<i><u>Applicability</u></i>	<u>VRF</u>
<i><u>Definition</u></i>	<u>A unique name designating the VRF System</u>
<i><u>Units</u></i>	<u>None</u>
<i><u>Input Restrictions</u></i>	<u>None</u>
<i><u>Standard Design</u></i>	<u>Not applicable</u>

<u>Heat Recovery</u>	
<i><u>Applicability</u></i>	<u>VRF</u>
<i><u>Definition</u></i>	<u>Identification if heat recovery (refrigerant loop) is present.</u>
<i><u>Units</u></i>	<u>Boolean</u>

<u>Input Restrictions</u>	<u>None (default : No)</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Control Priority</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>A control parameter used to determine when outdoor unit is in heating or cooling</u>
<u>Units</u>	<u>List: MasterThermostat Priority or Load Priority</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>ControlZone</u>	
<u>Applicability</u>	<u>Master Thermostat Control Zone</u>
<u>Definition</u>	<u>The name of the control zone that controls the outdoor unit, when the Control Priority is Master Thermostat Priority</u>
<u>Units</u>	<u>None</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Minimum Part-Load Ratio</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>The minimum part-load ratio for the heat pump. Below this ratio the unit will cycle to meet the load.</u>
<u>Units</u>	<u>Unitless</u>
<u>Input Restrictions</u>	<u>0 to 1</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Rated EER</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>Full load cooling efficiency (Btu/h of net cooling output divided by the electrical energy consumption in Watts) per AHRI rating conditions</u>
<u>Units</u>	<u>Btu/h-W</u>
<u>Input Restrictions</u>	<u>None</u>

<u>Standard Design</u>	<u>Not applicable</u>
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<u>Rated COP</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>Full load heating efficiency (net heating output divided by the electrical energy consumption, both in the same units) per AHRI rating conditions</u>
<u>Units</u>	<u>None</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Rated Indoor Type</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>A flag to determine when the VRF system was rated with ducted or unducted indoor units</u>
<u>Units</u>	<u>List: ducted, unducted</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Equivalent Pipe Length</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>The equivalent pipe length between the farthest terminal unit and the condensing unit, including liquid refrigerant line length, fitting losses and other losses.</u>
<u>Units</u>	<u>ft</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Max Vertical Height</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>The vertical height difference between the highest or lowest terminal unit and outdoor unit</u>
<u>Units</u>	<u>ft</u>

<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Defrost Heat Source</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>The defrost heat source type</u>
<u>Units</u>	<u>List – electric or gas</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Defrost Control Strategy</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>The control method for enabling defrost</u>
<u>Units</u>	<u>List – TimedCycle or OnDemand</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Max Defrost Temp</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>The maximum outdoor dry-bulb temperature at which defrost will occur</u>
<u>Units</u>	<u>Deg F</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

<u>Crankcase Heater Capacity</u>	
<u>Applicability</u>	<u>VRF</u>
<u>Definition</u>	<u>The capacity of the resistive defrost heating element</u>
<u>Units</u>	<u>W</u>
<u>Input Restrictions</u>	<u>None</u>
<u>Standard Design</u>	<u>Not applicable</u>

Compressor Quantity	
<i>Applicability</i>	<u>VRF</u>
<i>Definition</i>	<u>The number of compressors in the condenser, which determines crankcase heater operation</u>
<i>Units</i>	<u>Deg F</u>
<i>Input Restrictions</i>	<u>None</u>
<i>Standard Design</i>	<u>Not applicable</u>

Crankcase Heater Cutoff	
<i>Applicability</i>	<u>VRF</u>
<i>Definition</i>	<u>The outdoor dry-bulb temperature below which the crankcase heater can operate</u>
<i>Units</i>	<u>None</u>
<i>Input Restrictions</i>	<u>None</u>
<i>Standard Design</i>	<u>Not applicable</u>

5.8.65.8.7 Plant Management

Plant management is a method of sequencing equipment. Separate plant management schemes may be entered for chilled water systems, hot water systems, etc. The following building descriptors are specified for each load range, e.g., when the cooling load is below 300 tons, between 300 tons and 800 tons, and greater than 800 tons.

Equipment Type Managed	
<i>Applicability</i>	All plant systems
<i>Definition</i>	The type of equipment under a plant management control scheme Choices include: <ul style="list-style-type: none"> • Chilled water cooling • Hot water space heating • Condenser water heat rejection • Service water heating • Electrical generation
<i>Units</i>	None
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Same as the proposed design

<i>Standard Design: Existing Buildings</i>	
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Equipment Schedule	
<i>Applicability</i>	All plant equipment
<i>Definition</i>	A schedule that identifies when the equipment is in service
<i>Units</i>	Data structure
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Operation staging when multiple equipment is used
<i>Standard Design: Existing Buildings</i>	

Equipment Operation	
<i>Applicability</i>	All plant equipment
<i>Definition</i>	<p>Equipment operation can be either on-demand or always-on.</p> <p>On-demand operation means the equipment cycles on when it is scheduled to be in service and when it is needed to meet building loads. Otherwise it is off.</p> <p>Always-on means that equipment runs continuously when it scheduled to be in service. For the purpose of the compliance model, always-on is used for equipment such as chillers that are base-loaded, and on-demand equipment is scheduled to be on only when the base-loaded equipment (one or more) cannot meet the load.</p>
<i>Units</i>	None
<i>Input Restrictions</i>	As designed; default is on-demand
<i>Standard Design</i>	Assume on-demand operation
<i>Standard Design: Existing Buildings</i>	

Equipment Staging Sequence	
<i>Applicability</i>	All plant equipment
<i>Definition</i>	The staging sequence for plant equipment (chillers and boilers) indicates how multiple pieces of equipment will be staged on and off when a single piece of equipment is unable to meet the load. In both the proposed <u>and standard -design-and-baseline design</u> , the compliance software uses the

	optimal sequence to determine plant staging based on part-load performance. This descriptor is used to identify sequencing when the plant contains unequal equipment, where the order in which the plant equipment is enabled affects plant energy use.
<i>Units</i>	Structure – an array, where each element includes a) the load range, in minimum tons and maximum tons; and b) a list of equipment that is enabled to operate. The equipment will run in the priority matching the order of the equipment listed.
<i>Input Restrictions</i>	As designed; user may specify load ranges for staging each plant equipment.
<i>Standard Design</i>	Not applicable The standard design chiller and boiler plant each consist of one or more equal chillers or boilers, so the loading order is not applicable.
<i>Standard Design: Existing Buildings</i>	

5.8.75.8.8 Thermal Energy Storage

The compliance model inputs below document the requirements to model a chilled water thermal energy storage system with compliance software. Some systems (ice storage, eutectic salts) cannot be modeled with compliance software.

Thermal Energy Storage Systems Name	
<i>Applicability</i>	All thermal energy storage systems
<i>Definition</i>	A unique descriptor for thermal energy storage systems
<i>Units</i>	Text, unique
<i>Input Restrictions</i>	Where applicable, this should match the tags that are used on the plans such that a plan reviewer can make a connection.
<i>Standard Design</i>	Not applicable

Thermal Energy Storage Systems Type	
<i>Applicability</i>	All thermal energy storage systems
<i>Definition</i>	The type of thermal energy storage system being used. Chilled water storage system is the only currently supported option.
<i>Units</i>	List chilled water
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable

Discharge Priority	
<i>Applicability</i>	All thermal energy storage systems
<i>Definition</i>	A descriptor determines whether the storage system or a chiller will operate first to meet cooling loads during the discharge period. Storage priority will normally provide larger demand charge savings but requires a larger storage system. Chiller priority allows use of a significantly smaller storage system but demand reduction will be smaller.
<i>Units</i>	List storage or chiller
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable

Operation Mode Schedule	
<i>Applicability</i>	All thermal energy storage systems
<i>Definition</i>	<p>A schedule which controls operating mode of the thermal energy storage system.</p> <p>A thermal energy storage system can be discharging (supplying chilled water to meet cooling loads), charging (receiving chilled water to be stored for later use), or off. The operation mode schedule specifies one of these modes for each of the 8,760 hours in a year.</p>
<i>Units</i>	Data structure - thermal energy storage mode schedule, specifies charging, discharging, or off on an hourly basis
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable

Rated Capacity	
<i>Applicability</i>	All thermal energy storage systems
<i>Definition</i>	<p>The design cooling capacity of the thermal energy storage system.</p> <p>The rated cooling capacity of the thermal energy storage system is determined by design flow rate of the thermal energy storage system and the temperature difference between the fluid system supply and return water temperature during discharging.</p>
<i>Units</i>	Btu/hr
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable

Tank Location	
<i>Applicability</i>	All thermal energy storage systems
<i>Definition</i>	The location of the heat pump water heater for determining losses and heat energy interaction with the surroundings
<i>Units</i>	List zone, outdoors, or underground
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable

Tank Shape	
<i>Applicability</i>	All thermal energy storage systems
<i>Definition</i>	The shape of the energy storage system tank used to calculate surface area of the tank for heat gain/loss calculations
<i>Units</i>	List: Vertical cylinder, Horizontal cylinder, or rectangular
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable

Tank Volume	
<i>Applicability</i>	All thermal energy storage systems
<i>Definition</i>	The volume of water held in the thermal energy storage system tank. The tank volume and the rated capacity will determine how long the storage system can meet the load.
<i>Units</i>	Gallons
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable

Tank Height	
<i>Applicability</i>	All thermal energy storage systems
<i>Definition</i>	For vertical cylinder or rectangular tank, the height will be the maximum internal height of water held in the upright storage tank. For horizontal cylinder tank, the height of the storage tank will be the inner diameter of the storage tank.
<i>Units</i>	Feet

<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable

Tank Length to Width Ratio	
<i>Applicability</i>	All thermal energy storage systems
<i>Definition</i>	The length to width ratio of a rectangular storage tank in plan view. It is required only if tank shape is rectangular. If the tank is square, the length to width ratio is one. For a rectangular tank, the ratio will be greater than one since the length of the tank is always greater than the width of the tank. This is used to determine the surface area of the tank.
<i>Units</i>	Unitless ratio
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable

Tank R-Value	
<i>Applicability</i>	All thermal energy storage systems
<i>Definition</i>	The insulation applied to the tank used in calculating the tank U-factor
<i>Units</i>	R-value (h-ft ² -F/Btu)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable

5.9 Miscellaneous Energy Uses

Miscellaneous energy uses are defined as those that may be treated separately since they have little or no interaction with the conditioned thermal zones or the HVAC systems that serve them.

5.9.1.1 System Loads and Configuration

Water Heating System Name	
<i>Applicability</i>	All water heating systems
<i>Definition</i>	A unique descriptor for each water heating system. A system consists of one or more water heaters, a distribution system, an estimate of hot water use, and a schedule for that use. Nonresidential

	buildings will typically have multiple systems, perhaps a separate electric water heater for each office break room, etc. Other building types such as hotels and hospitals may have a single system serving the entire building.
<i>Units</i>	Text, unique
<i>Input Restrictions</i>	Where applicable, this should match the tags that are used on the plans such that a plan reviewer can make a connection.
<i>Standard Design</i>	The naming convention for the <u>standard design</u> baseline building system shall be similar to the proposed design.
<i>Standard Design: Existing Buildings</i>	

Water Heating Peak Use	
<i>Applicability</i>	All water heating systems, required
<i>Definition</i>	<p>An indication of the peak hot water usage (e.g., service to sinks, showers, kitchen appliances, etc.). When specified per occupant, this value is multiplied by design occupancy density values and modified by service water heating schedules to obtain hourly load values which are used in the simulation.</p> <p>Peak consumption is commonly specified as gallons per hour (gph) per occupant, dwelling unit, hotel room, patient room, or floor area. If consumption is specified in gph, then additional inputs would be needed such as supply temperature, cold water inlet temperature, etc.</p> <p>Software that specifies peak use as a thermal load in Btu/h can apply ACM rules for the mains (cold water inlet) temperature and supply temperature to convert the prescribed peak use from gph/person to Btu/h-person. The thermal load does not include conversion efficiencies of water heating equipment.</p>
<i>Units</i>	gph/person
<i>Input Restrictions</i>	<p>For nonresidential spaces, prescribed values from Appendix 5.4A if a service hot water heating system is installed; otherwise, all values are 0.</p> <p>For high-rise residential spaces and residential living spaces of hotels and motels (guestrooms), the rules in the <i>Residential ACM Reference Manual</i> are followed.</p>
<i>Standard Design</i>	<p>Prescribed values from Appendix 5.4A if a service hot water heating system is installed; otherwise, all values are 0.</p> <p>For high-rise residential spaces and residential living spaces of hotels and motels (guestrooms), the rules in the <i>Residential ACM Reference Manual</i> are followed.</p>

<i>Standard Design: Existing Buildings</i>	
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Water Heating Schedule	
<i>Applicability</i>	All water heating systems
<i>Definition</i>	A fractional schedule reflecting the time pattern of water heating use. This input modifies the water heating peak use described above.
<i>Units</i>	Data structure - schedule, fractional
<i>Input Restrictions</i>	The schedules from Appendix 5.4A shall be used. For high-rise residential spaces and residential living spaces of hotels and motels (guestrooms), the rules in the <i>Residential ACM Reference Manual</i> are followed.
<i>Standard Design</i>	The schedules from Appendix 5.4A shall be used. For high-rise residential spaces and residential living spaces of hotels and motels (guestrooms), the rules in the <i>Residential ACM Reference Manual</i> are followed.
<i>Standard Design: Existing Buildings</i>	

Water Heating System Configuration	
<i>Applicability</i>	All water heating systems
<i>Definition</i>	The configuration and layout of the water heating system including the number of water heaters; the size, location, length and insulation of distribution pipes; recirculation systems and pumps; and any other details about the system that would affect the energy model.
<i>Units</i>	Data structure
<i>Input Restrictions</i>	None
<i>Standard Design</i>	<p><u>For healthcare facility spaces, the same as Proposed. For high-rise residential buildings, the rules in the Residential ACM Reference Manual shall be followed.</u></p> <p><u>For all other spaces, the standard design-baseline building will shall have one gas storage water heater if any of the spaces have a Space Water Heating Fuel Type of Gas (from Appendix 5.4A), and the standard design building will have on electric water heater if the any of the spaces have a Space Water Heating Fuel Type of Electric. The baseline building shall have one gas storage water heater, except for high-rise residential buildings which shall follow the rules in the Residential ACM Reference Manual.</u></p>

<i>Standard Design: Existing Buildings</i>	
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Water Mains Temperature Schedule	
<i>Applicability</i>	All water heating systems
<i>Definition</i>	A monthly temperature schedule indicating the water mains temperature. This temperature and the setpoint temperature are used to convert the load into a water flow rate.
<i>Units</i>	Data structure - schedule, °F
<i>Input Restrictions</i>	For nonresidential spaces, the schedules from Appendix 5.4A shall be used. The water mains temperature schedule shall be fixed for a given climate zone. For high-rise residential spaces and residential living spaces of hotels and motels (guestrooms), the rules in the <i>Residential ACM Reference Manual</i> are followed.
<i>Standard Design</i>	For nonresidential spaces, the schedules from Appendix 5.4A shall be used. The water mains temperature schedule shall be fixed for a given climate zone. For high-rise residential spaces and residential living spaces of hotels and motels (guestrooms), the rules in the <i>Residential ACM Reference Manual</i> are followed.
<i>Standard Design: Existing Buildings</i>	

5.9.1.2 Water Heaters

This section describes the building descriptors for water heaters. Typically, a building will have multiple water heating systems and each system can have multiple water heaters, so these building descriptors may need to be specified more than once.

Water Heater Name	
<i>Applicability</i>	All water heaters
<i>Definition</i>	A unique descriptor for each water heater in the system. Some systems will have multiple pieces of equipment. For instance, a series of water heaters plumbed in parallel or a boiler with a separate storage tank.
<i>Units</i>	Text, unique
<i>Input Restrictions</i>	Where applicable, this should match the tags that are used on the plans such that a plan reviewer can make a connection.

<i>Standard Design</i>	The naming convention for the <u>standard design</u> baseline building system shall be similar to the proposed design.
<i>Standard Design: Existing Buildings</i>	

Water Heater Type and Size	
<i>Applicability</i>	All water heaters
<i>Definition</i>	This building descriptor includes information needed to determine the criteria from baseline standards. The choices and the associated rated capacity (heat input rate) are listed in the <i>2015 Appliance Efficiency Regulations</i> , except that oil-fired water heaters and boilers are not supported.
<i>Units</i>	List conventional, heat pump split, or heat pump packaged
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<p>For healthcare facility spaces, the same as Proposed. For all other spaces, For all other spaces, the <u>standard design</u> baseline building will <u>shall have one gas storage water heater if any of the spaces have a space water heating fuel type of gas (from Appendix 5.4A), and the standard design building will have on electric water heater if the any of the spaces have a space water heating fuel type of electric</u></p> <p>Gas storage water heater for nonresidential buildings and common, nonresidential spaces of high-rise residential buildings and hotel-motel guestrooms.</p> <p>For high-rise residential spaces and residential living spaces of hotels and motels (guestrooms), the rules in the <i>Residential ACM Reference Manual</i> are followed.</p>
<i>Standard Design: Existing Buildings</i>	Same as proposed if water heater is existing

Rated Capacity	
<i>Applicability</i>	All water heaters
<i>Definition</i>	<u>For gas and electric resistance water heaters, the heating capacity of a water heater (input rate) at the rated conditions specified in DOE 10 CFR Part 430 or ANSI Z21.10. For heat pump water heaters, the rated heating capacity supplied to the water (output rate).</u>
<i>Units</i>	Thousands of British thermal units per hour (MBH)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	<u>The capacity of the standard design water heaters will be based on the</u>

	<p><u>larger of the total design hot water consumption rate (gallons/hr) of all the spaces in the building or 75 gallons per hour. The consumption rate is converted to Btu/hr (-x (design supply temp – 55) x 8.2877 pounds/gallon x 1 Btu/pound-F-). That value is multiplied by 0.60 to find the heat that must be supplied to the water.</u></p> <p><u>All of the water heaters in the proposed design are similarly converted to a total Btu/hr heat output, summed across the water heaters, and multiplied by 0.60.</u></p> <p><u>The standard designbaseline then uses the smaller of these values and divides by thermal efficiency to find energy input.</u></p> <p><u>The standard design capacity equals 60 percent of the total adjusted proposed design capacity (combined rated capacity, converted to gph, and storage capacity in gallons).</u></p> <p><u>The adjusted proposed capacity is equal to the smaller of the actual proposed design capacity and 150% of the required capacity from a sizing run. In the event that the standard design capacity for a gas storage tank is less than 20 gallons, both the storage volume and total capacity shall be increased in proportion to maintain a minimum standard design storage volume of 20 gallons.</u></p> <p><u>If the standard design has both a gas water heater and an electric water heater, the total capacity will be pro-rated between the two based on the total hot water consumption rate of the spaces with water heating fuel type of electric or gas.</u></p> <p><u>If the standard design has both a gas water heater and electric water heater, the capacity of both water heaters shall be set at 50% of the total required capacity for the building.</u></p>
<p><i>Standard Design: Existing Buildings</i></p>	

<p>Storage Volume</p>	
<p><i>Applicability</i></p>	<p>Gas-fired<u>Storage</u> water heaters</p>
<p><i>Definition</i></p>	<p>The storage-volume of a gas-fired<u>storage</u> water heater used in the standby loss calculations and standard designbaseline calculations of energy factor</p>
<p><i>Units</i></p>	<p>Gallons</p>
<p><i>Input Restrictions</i></p>	<p>As designed</p>
<p><i>Standard Design</i></p>	<p><u>The volume of the standard design water heaters will be based on the larger of the total design hot water consumption rate (gallons/hr) of all the spaces in the building or 75 gallons per hour. That value is multiplied by 1 hour and 0.40 to determine the storage volume.</u></p> <p><u>All of the water heaters in the proposed design are similarly converted to a total Btu/hr heat output, summed across the water heaters. This value</u></p>

	<p><u>is multiplied by 0.40, and converted to gallons $(\frac{V}{(design\ supply\ temp - 55)} / 8.2877\ pounds/gallon / 1\ Btu/pound-F\ x\ 1\ hr-)$.</u></p> <p><u>The standard design baseline then uses the smaller of these values.</u></p> <p><u>If the standard design has both a gas water heater and an electric water heater, the total volume will be pro-rated between the two based on the total hot water consumption rate of the spaces with water heating fuel type of electric or gas.</u></p> <p><u>The compliance software shall size the storage tank to 40 percent of the proposed total rated capacity and storage volume for the peak hour, in gallons, but shall be set at a minimum of 20 gallons.</u></p> <p>For high-rise residential spaces and residential living spaces of hotels and motels (guestrooms), the rules in the <i>Residential ACM Reference Manual</i> are followed.</p>
<p><i>Standard Design:</i> <i>Existing Buildings</i></p>	

Input Power	
<i>Applicability</i>	Heat pump water heaters
<i>Definition</i>	<p>The total design electrical input to a heat pump water heater at design conditions.</p> <p>This power includes the input to the compressor, controls, evaporator fan, and pump (if present).</p>
<i>Units</i>	Kilowatts (kW)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable
<i>Standard Design:</i> <i>Existing Buildings</i>	

Storage Tank Location	
<i>Applicability</i>	Heat pump water heaters
<i>Definition</i>	<p>The location storage volume of a gas-fired of a heat pump water heater. This is used in the standby loss calculations and baseline calculations of energy factor.</p>
<i>Units</i>	<p>List:</p> <ul style="list-style-type: none"> • Conditioned • Unconditioned

<i>Input Restrictions</i>	List see above
<i>Standard Design</i>	
<i>Standard Design: Existing Buildings</i>	

Energy Factor																										
<i>Applicability</i>	Equipment covered by the National Appliance Energy Conservation Act (NAECA), which includes small storage and instantaneous water heaters																									
<i>Definition</i>	The energy factor (EF) is the ratio of the energy delivered by the water heater divided by the energy used, in the same units. EF is calculated according to the DOE 10 CFR Part 430 test procedure, which specifies a 24-hour pattern of draws, a storage temperature, inlet water temperature, and other test conditions. These conditions result in the energy delivered for the test period. Energy inputs are measured for the same test period and the EF ratio is calculated.																									
<i>Units</i>	Unitless ratio																									
<i>Input Restrictions</i>	<p>Building descriptors for the proposed design should be consistent with equipment specified on the construction documents or observed in the candidate building.</p> <p>For water heaters manufactured after June 1, 2017 that contain a Uniform Energy Factor, the EF shall not be input by the user, but shall be calculated by:</p> $F = \frac{(NUb) - (N^2PUa)}{d(U - N) + c(N^2P - NPU) - Ub + NPUa}$ <p>Where:</p> <table border="1" style="margin-left: 20px;"> <tr> <td><i>F</i></td> <td>Energy Factor</td> </tr> <tr> <td><i>N</i></td> <td>Recovery Efficiency</td> </tr> <tr> <td><i>P</i></td> <td>Power Input (W)</td> </tr> <tr> <td><i>U</i></td> <td>UEF</td> </tr> <tr> <td><i>N</i></td> <td>Recovery efficiency defined as 0.80 for storage water heater UEF to EF conversion</td> </tr> </table> <table border="1" style="margin-left: 20px;"> <thead> <tr> <th>Draw Pattern</th> <th>a</th> <th>b</th> <th>c</th> <th>d</th> </tr> </thead> <tbody> <tr> <td>Very Small</td> <td>0.250266</td> <td>57.5</td> <td>0.039864</td> <td>67.5</td> </tr> <tr> <td>Low</td> <td>0.065860</td> <td>57.5</td> <td>0.039864</td> <td>67.5</td> </tr> </tbody> </table>	<i>F</i>	Energy Factor	<i>N</i>	Recovery Efficiency	<i>P</i>	Power Input (W)	<i>U</i>	UEF	<i>N</i>	Recovery efficiency defined as 0.80 for storage water heater UEF to EF conversion	Draw Pattern	a	b	c	d	Very Small	0.250266	57.5	0.039864	67.5	Low	0.065860	57.5	0.039864	67.5
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Draw Pattern	a	b	c	d																						
Very Small	0.250266	57.5	0.039864	67.5																						
Low	0.065860	57.5	0.039864	67.5																						

	Medium	0.045503	57.5	0.039864	67.5
	High	0.029794	57.5	0.039864	67.5
<i>Standard Design</i>	<p>For nonresidential buildings and nonresidential spaces, the <u>eEnergy Factor</u> for the <u>standard design</u> baseline building system shall be determined from the <i>Appliance Efficiency Regulations</i>.</p> <p>For high-rise residential spaces and residential living spaces of hotels and motels (guestrooms), the rules in the <i>Residential ACM Reference Manual</i> are followed.</p>				
<i>Standard Design: Existing Buildings</i>					

Uniform Energy Factor	
<i>Applicability</i>	Equipment covered by NAECA that is rated after June 1, 2017 with a Uniform Energy Factor (UEF) that includes small storage and instantaneous water heaters
<i>Definition</i>	The UEF defines an efficiency level for a specific targeted use pattern
<i>Units</i>	Unitless ratio
<i>Input Restrictions</i>	Must meet mandatory minimum requirements defined by federal or state appliance efficiency standards
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

First Hour Rating	
<i>Applicability</i>	Water heating storage tanks with a UEF rating
<i>Definition</i>	The first hour rating is a measure of the overall capacity of the water heater that incorporates both the heat input rate and the tank storage capacity and is used to determine the draw pattern.
<i>Units</i>	gal/hr
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable

<i>Standard Design: Existing Buildings</i>	
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Draw Pattern	
<i>Applicability</i>	Storage water heating tanks with a UEF rating
<i>Definition</i>	The draw pattern is determined as: very small, low, medium, or high from the user entry of first hour rating (FHR). Very small: 0-18 gal/hr Low: 18-51 gal/hr Medium: 51-75 gal/hr High: > 75 gal/hr
<i>Units</i>	List: <ul style="list-style-type: none"> • Very small • Low • Medium • High
<i>Input Restrictions</i>	Not user editable. Draw pattern is determined from FHR user input.
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

Thermal Efficiency	
<i>Applicability</i>	Oil and gas-fired water heaters not covered by NAECA
<i>Definition</i>	The full load efficiency of a water heater at rated conditions expressed as a dimensionless ratio of output over input. It is also referred to as recovery efficiency.
<i>Units</i>	Unitless ratio
<i>Input Restrictions</i>	Building descriptors for the proposed design should be consistent with equipment specified on the construction documents or observed in the candidate building.
<i>Standard Design</i>	For nonresidential buildings and nonresidential spaces, the thermal efficiency is determined from Table F-2 in the <i>2015 Appliance Efficiency Regulations</i> . For high-rise residential spaces and residential living spaces of hotels

	and motels (guestrooms), the rules in the <i>Residential ACM Reference Manual</i> are followed.
<i>Standard Design: Existing Buildings</i>	

Tank Standby Loss					
<i>Applicability</i>	Water heaters not covered by NAECA				
<i>Definition</i>	The tank standby loss for storage tanks, which includes the effect of recovery efficiency				
<i>Units</i>	Btu/h for the entire tank				
<i>Input Restrictions</i>	Standby loss is calculated as: $STBY = 577.5 \times S \times VOL$ Where: <table border="1" style="margin-left: 20px;"> <tr> <td style="text-align: center;"><i>S</i></td> <td>The standby loss fraction listed in the Energy Commission’s Appliance Database of Certified Water Heaters</td> </tr> <tr> <td style="text-align: center;"><i>VO</i></td> <td>The actual storage capacity of the water heater as listed in the Energy Commission’s Appliance Database of Certified Water Heaters (gallons)</td> </tr> </table>	<i>S</i>	The standby loss fraction listed in the Energy Commission’s Appliance Database of Certified Water Heaters	<i>VO</i>	The actual storage capacity of the water heater as listed in the Energy Commission’s Appliance Database of Certified Water Heaters (gallons)
<i>S</i>	The standby loss fraction listed in the Energy Commission’s Appliance Database of Certified Water Heaters				
<i>VO</i>	The actual storage capacity of the water heater as listed in the Energy Commission’s Appliance Database of Certified Water Heaters (gallons)				
<i>Standard Design</i>	Table F-2 of the <i>2015 Appliance Efficiency Standards</i>				
<i>Standard Design: Existing Buildings</i>					

Tank Off-Cycle Loss Coefficient	
<i>Applicability</i>	Water heaters
<i>Definition</i>	The tank standby loss coefficient (UA) for the water heater. For small water heaters covered by NAECA, the loss coefficient is a derived parameter, a function of the EF and recovery efficiency.
<i>Units</i>	Btu/h - °F
<i>Input Restrictions</i>	For NAECA covered water heaters, the loss coefficient is calculated by: $UA = \frac{\frac{1}{EF} - \frac{1}{RE}}{67.5 \left(\frac{24}{41094} - \frac{1}{RE(P_{on})} \right)}$ Where:

	<i>EF</i>	The energy factor of the rated water heater (unitless)
	<i>RE</i>	The recovery efficiency of the rated water heater. If this data is not available the default shall be 0.78 for gas water heaters and 0.93 for electric water heaters.
	<i>P_{on}</i>	The input power to the water heater, in Btu/h
<i>Standard Design</i>	For nonresidential spaces, 10 Btu/h-F. For high-rise residential spaces and residential living spaces of hotels and motels (guestrooms), the rules in the <i>Residential ACM Reference Manual</i> are followed	
<i>Standard Design: Existing Buildings</i>		

Off Cycle Parasitic Losses	
<i>Applicability</i>	Water heaters
<i>Definition</i>	The rate of parasitic losses, such as a pilot light or controls, when the water heater is not heating
<i>Units</i>	Watts
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	0
<i>Standard Design: Existing Buildings</i>	

Off Cycle Fuel Type	
<i>Applicability</i>	Water heaters
<i>Definition</i>	The type of fuel that serves energy using parasitic equipment, such as a pilot light or controls, when the water heater is not heating
<i>Units</i>	List electricity, gas, oil, or propane
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

On-Cycle Parasitic Losses

<i>Applicability</i>	Water heaters
<i>Definition</i>	The rate of parasitic losses, such as a pilot light or controls, when the water heater is not heating. It may be different than off-cycle losses if the flue energy is considered.
<i>Units</i>	Watts
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

On-Cycle Fuel Type	
<i>Applicability</i>	Water heaters
<i>Definition</i>	The type of fuel that serves energy using parasitic equipment, such as a pilot light or controls, when the water heater is not heating
<i>Units</i>	List electricity, gas, oil, or propane
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

Water Heater Ambient Location	
<i>Applicability</i>	Water heaters
<i>Definition</i>	The location of the water heater for determining losses and energy interaction with the surroundings
<i>Units</i>	List schedule, zone, outdoors
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

Water Heater Compressor Location	
<i>Applicability</i>	Heat pump water heaters
<i>Definition</i>	The location of the heat pump compressor for determining losses and

	energy interaction with the surroundings. The air temperature at the compressor location also controls the compressor's crankcase heater operation.
<i>Units</i>	List zone, outdoors
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

Tank Standby Loss Fraction	
<i>Applicability</i>	Storage tank water heaters
<i>Definition</i>	The tank standby loss fraction for storage tanks
<i>Units</i>	Unitless
<i>Input Restrictions</i>	Prescribed to the value listed in the Appliance Database of Certified Water Heaters
<i>Standard Design</i>	Not applicable The part-load curve procedure in Title 24 can be an alternate method of specifying the effects of standby and parasitic losses on performance. The primary method is to specify a loss coefficient for the storage tank.
<i>Standard Design: Existing Buildings</i>	

Fuel Water Heater Part-Load Efficiency Curve	
<i>Applicability</i>	Water heating equipment for which a loss coefficient is not specified (alternate method)
<i>Definition</i>	A set of factors that adjust the full-load thermal efficiency for part load conditions; set as a curve
<i>Units</i>	Percent (%)

<p><i>Input Restrictions</i></p>	<p>The following prescribed curve shall be used based on user inputs. The curve shall take the form of a quadratic equation as follows:</p> $Fuel_{partload} = Fuel_{design} \times FHeatPLC$ $FHeatPLC = \left(a + b \left(\frac{Q_{partload}}{Q_{rated}} \right) \right)$ <p>For Title 24, the coefficients shall be determined by:</p> $a = \frac{STBY}{INPUT}$ $b = \frac{(INPUT \times RE) - STBY}{SRL}$ $PLR_n = \frac{SRL \times F_{whpl(n)}}{INPUT \times RE}$ <p>Recovery efficiency substituted with thermal efficiency when applicable.</p> <p>For boilers, instantaneous gas or other storage type water heaters, not in the scope of covered consumer products as defined in the Title 10 or the Code of Federal Regulations, Part 430:</p> $STBY = 577.5 \times S \times VOL$ <p>Required inputs and standard and proposed design assumptions depend on the type of water heater and whether or not it is a DOE covered consumer product.</p> <p>Where:</p> <table border="1" data-bbox="467 1360 1419 1875"> <tr> <td><i>FHeatPLC</i></td> <td>The fuel heating part load efficiency curve</td> </tr> <tr> <td><i>Fuel_{partload}</i></td> <td>The fuel consumption at part-load conditions (Btu/h)</td> </tr> <tr> <td><i>Fuel_{design}</i></td> <td>The fuel consumption at design conditions (Btu/h)</td> </tr> <tr> <td><i>Q_{partload}</i></td> <td>The water heater capacity at part-load conditions (Btu/h)</td> </tr> <tr> <td><i>Q_{rated}</i></td> <td>The water heater capacity at design conditions (Btu/h)</td> </tr> <tr> <td><i>PLR_n</i></td> <td>Part-load ratio for the nth hour and shall always be less than 1</td> </tr> <tr> <td><i>INPUT</i></td> <td>The input capacity of the water heater expressed in Btu/hr</td> </tr> <tr> <td><i>STBY</i></td> <td>Hourly standby loss expressed in Btu/hr. For large</td> </tr> </table>	<i>FHeatPLC</i>	The fuel heating part load efficiency curve	<i>Fuel_{partload}</i>	The fuel consumption at part-load conditions (Btu/h)	<i>Fuel_{design}</i>	The fuel consumption at design conditions (Btu/h)	<i>Q_{partload}</i>	The water heater capacity at part-load conditions (Btu/h)	<i>Q_{rated}</i>	The water heater capacity at design conditions (Btu/h)	<i>PLR_n</i>	Part-load ratio for the n th hour and shall always be less than 1	<i>INPUT</i>	The input capacity of the water heater expressed in Btu/hr	<i>STBY</i>	Hourly standby loss expressed in Btu/hr. For large
<i>FHeatPLC</i>	The fuel heating part load efficiency curve																
<i>Fuel_{partload}</i>	The fuel consumption at part-load conditions (Btu/h)																
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<i>PLR_n</i>	Part-load ratio for the n th hour and shall always be less than 1																
<i>INPUT</i>	The input capacity of the water heater expressed in Btu/hr																
<i>STBY</i>	Hourly standby loss expressed in Btu/hr. For large																

		storage gas water heaters, STBY is listed in the Energy Commission’s appliance database. The value includes pilot energy and standby losses. For all systems, refer to equation N2-62.
	<i>SRL</i>	The standard recovery load, taken from Appendix 5.4A, in Btu/hr, adjusted for the number of occupants according to the occupancy schedules.
	<i>S</i>	The standby loss fraction listed in the Energy Commission’s Appliance Database of Certified Water Heaters
	<i>VOL</i>	The actual storage capacity of the water heater as listed in the Appliance Database of Certified Water Heaters
<i>Standard Design</i>	Not applicable	
<i>Standard Design: Existing Buildings</i>		

5.9.1.3 Recirculation Systems

This section describes the building descriptors for hot water recirculation systems. The standard design~~baseline building~~ has a recirculation system when the proposed design does. This is one aspect of the water heating system configuration (see above).

Recirculating systems shall follow the rules set forth in Appendix E of the *Residential ACM Reference Manual*.

5.9.1.4 Water Heating Auxiliaries

External Storage Tank Insulation	
<i>Applicability</i>	All water heating systems that have an external storage tank
<i>Definition</i>	Some water heating systems have a storage tank that is separate from the water heater(s) that provides additional storage capacity. This building descriptor addresses the heat loss related to the external tank, which is an additional load that must be satisfied by the water heater(s).
<i>Units</i>	R-value (h-ft ² -F/Btu)
<i>Input Restrictions</i>	As specified in manufacturer data and documented on the construction documents
<i>Standard Design</i>	Heat loss associated with the storage tank in the <u>standard design</u> baseline building shall meet the requirements for an unfired storage tank in the baseline standards which is an insulation R-value of 12.5. The surface area and location of the storage tank shall be the same as the proposed design.

<i>Standard Design: Existing Buildings</i>	
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External Storage Tank Area	
<i>Applicability</i>	All water heating systems that have an external storage tank
<i>Definition</i>	Some water heating systems have a storage tank that is separate from the water heater(s) that provides additional storage capacity. This documents the entire exterior surface area of the tank.
<i>Units</i>	ft ²
<i>Input Restrictions</i>	As specified in manufacturer specifications
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

External Storage Tank Location	
<i>Applicability</i>	All water heating systems that have an external storage tank
<i>Definition</i>	Location of the storage tank, used to determine the heat loss rate and energy exchange with the surroundings
<i>Units</i>	List schedule, zone, outdoors
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Not applicable
<i>Standard Design: Existing Buildings</i>	

Heat Recovery	
<i>Applicability</i>	Water heating systems that are coupled to heat recovery equipment
<i>Definition</i>	Building equipment such as air conditioners, chillers, and gas-fired generators that produce thermal energy that may be recovered and used to heat water. The heat-producing characteristics are generally defined for the equipment that is producing the heat, not the equipment that is receiving the heat (water heaters in this case). The building descriptors will vary depending on the equipment. The models for heat-producing equipment need to produce output on an hourly basis so that the schedule of heat production and heating needs can be aligned and evaluated in the water heating model.

<i>Units</i>	Data structure - depends on the equipment producing the heat
<i>Input Restrictions</i>	No restrictions, other than agreement with the construction documents <u>As designed</u>
<i>Standard Design</i>	<u>Not applicable for nonresidential buildings.</u>
<i>Standard Design: Existing Buildings</i>	

Solar Thermal	
<i>Applicability</i>	Water heating systems with a solar thermal system
<i>Definition</i>	<p>A solar thermal water heating system consists of one or more collectors. Water is passed through these collectors and is heated under the right conditions. There are two general types of solar water heaters: integrated collector storage (ICS) systems and active systems. Active systems include pumps to circulate the water, storage tanks, piping, and controls. ICS systems generally have no pumps and piping is minimal.</p> <p>Solar systems may be tested and rated as a complete system or the collectors may be separately tested and rated. Solar Rating & Certification Corporation (SRCC) OG-300 is the test procedure for whole systems and SRCC OG-100 is the test procedure for collectors. The building descriptors used to define the solar thermal system may vary with each software application and with the details of system design.</p> <p>The solar fraction shall be estimated by the f-chart procedure for solar water heating systems.</p>
<i>Units</i>	Unitless fraction
<i>Input Restrictions</i>	<u>For high-rise residential buildings, the solar fraction provided by the solar DHW system shall be between 0 and 1. For all other buildings, the value is 0 (solar thermal may not be modeled for compliance.)</u>
<i>Standard Design</i>	<u>The standard design baseline building has no solar auxiliary system.</u>
<i>Standard Design: Existing Buildings</i>	

Combined Space Heating and Water Heating	
<i>Applicability</i>	Projects that use a boiler to provide both space heat and water heating
<i>Definition</i>	A system that provides both space heating and water heating from the same equipment, generally the space heating boiler. Such systems are restricted by the baseline standards but may be modeled in the candidate building. The restrictions are due to the misalignment of the space heating load and the water heating load. The first is highly intermittent and weather dependent, and the latter is more constant and not generally

	weather-related.
<i>Units</i>	Data structure
<i>Input Restrictions</i>	The proposed design may have a combined space and water heating system.
<i>Standard Design</i>	The <u>standard design</u> baseline building shall be modeled with separate space heating and water heating systems.
<i>Standard Design: Existing Buildings</i>	

5.9.2 Exterior Lighting

Outdoor lighting requirements are specified in Section 140.7 of the standards. Outdoor lighting shall not be modeled in the proposed design or standard design, and no tradeoffs are available with other building end uses or systems. Outdoor lighting shall meet all prescriptive requirements in the standards.

~~5.9.3 Swimming Pools~~

~~Swimming pools must meet applicable mandatory requirements and are not required to be modeled for Title 24 compliance or reach.~~

~~5.9.45.9.3~~ Other Electricity Use

This set of building descriptors should be used to include any miscellaneous electricity use that would add to the electric load of the building and would be on the building meter. These energy uses are assumed to be outside the building envelope and do not contribute heat gain to any thermal zone.

Miscellaneous Electric Power	
<i>Applicability</i>	All buildings with miscellaneous electric equipment located on the building site
<i>Definition</i>	The power for miscellaneous equipment
<i>Units</i>	Watts (W)
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	

Miscellaneous Electric Schedule

<i>Applicability</i>	All buildings with miscellaneous electric equipment located on the building site
<i>Definition</i>	The schedule of operation for miscellaneous electric equipment that is used to convert electric power to energy use
<i>Units</i>	Data structure - schedule, fractional
<i>Input Restrictions</i>	The schedule specified for the building should match the operation patterns of the system.
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	

5.9.55.9.4 Other Gas Use

This set of building descriptors should be used to include any miscellaneous gas use that would add to the load of the building and would be on the building meter. These energy uses are assumed to be outside the building envelope and do not contribute heat gain to any thermal zone.

Other Gas Power	
<i>Applicability</i>	All buildings that have commercial gas equipment
<i>Definition</i>	Gas power is the peak power which is modified by the schedule (see below).
<i>Units</i>	Btu/h-ft ²
<i>Input Restrictions</i>	As designed
<i>Standard Design</i>	Same as the proposed design
<i>Standard Design: Existing Buildings</i>	

Other Gas Schedule	
<i>Applicability</i>	All buildings that have commercial gas equipment
<i>Definition</i>	The schedule of operation for commercial gas equipment that is used to convert gas power to energy use
<i>Units</i>	Data structure - schedule, fractional
<i>Input Restrictions</i>	Continuous operation is prescribed
<i>Standard Design</i>	Same as the proposed design

<i>Standard Design:</i> <i>Existing Buildings</i>	
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5.10 Common Data Structures

This section describes common data structures. The data structures presented here define objects and example parameters needed to define them. The parameters described are the most common for energy simulation engines. However, other parameters or data constructs are acceptable. The fields used by the simulation program must be mapped to the fields used by the building descriptor.

5.10.1 Schedule

This data structure provides information on how equipment, people, lights, or other items are operated on an hourly basis. The ultimate construct of a schedule is an hourly time series for the simulation period, typically 8,760 hours (365 days, 24 hours per day). Software has often built up the hourly schedule from 24-hour schedules for different day types such as weekdays, Saturdays, Sundays, holidays, etc.

There are several types of schedules:

- **Temperature** schedules specify a temperature to be maintained in a space, a temperature to be delivered from an air handler, or the leaving temperature from a chiller or other equipment.
- **Fraction** schedules specify the fraction of lights that are on, the fraction of people that are in the space, the fraction of maximum infiltration, or other factors.
- **On/off** schedules specify when equipment is operating or when infiltration is occurring.
- **Time period** schedules define periods of time for equipment sequencing, utility tariffs, etc. A time period schedule typically breaks the year in to two or more seasons. For each season, day types are identified such as weekday, Saturday, Sunday, and holidays. Each day type in each season is then divided into time periods.

5.10.2 Holidays

A series of dates defining holidays for the simulation period. Dates identified are operated for the schedule specified for holidays.

5.10.3 Surface Geometry

This data structure represents the location, size, and position of a surface. Surfaces include roofs, walls, floors, and partitions. Surfaces are typically planar and can be represented in various manners, including:

- Rectangular surfaces, which may be represented by a height and width along with the X, Y, and Z of surface origin, and the tilt and azimuth.
- By a series of vertices (X, Y, and Z coordinates defining the perimeter of a surface). More complex polygons may be represented in this manner.

5.10.4 Opening Geometry

This data structure represents the location and size of an opening within a surface. The most common method of specifying the geometry of an opening is to identify the parent surface, the height and width of the opening, and the horizontal and vertical offset (X and Y coordinates relative to the origin of the parent surface). An opening can also include a recess into the parent surface, which provides shading. However, other geometric constructs are acceptable.

5.10.5 Opening Shade

This data structure describes the dimensions and position of external shading devices such as overhangs, side fins, or louvers that shade the opening. Overhangs are specified in terms of the projection distance, height above the opening, and extension distance on each side of the opening.

5.10.6 Construction Assembly

This data structure describes the layers that make up the construction of a wall, roof, floor, or partition. Typically, a construction consists of a sequence of materials, described from the outside surface to the inside surface.

5.10.7 Fenestration Construction

This data structure describes the frame, glass, and other features of a window or skylight. Information may be defined in multiple ways but the criterion are published as a combination of U-factor, solar heat gain coefficient (SHGC), and visible light transmission (VT). Some simulation programs use more detailed methods of describing the performance of fenestration that take into account the angle of incidence of sun striking the fenestration and other factors, such as the properties of each pane and the fill. The compliance software only uses whole window performance properties (U-factor, SHGC, VT).

5.10.8 Material

This data structure describes a material that is used to build up a construction assembly. Typical material properties include specific heat, density, conductivity, and thickness. Materials can also be described in terms of their thermal resistance. The latter approach is sometimes used to approximate construction layers that are not homogeneous, such as framing members in combination with cavity insulation.

5.10.9 Slab Construction

This data structure describes the composition of a slab-on-grade. The compliance model has building descriptors for the perimeter length and the F-factor, which represents the heat loss per lineal foot.

5.11 Exterior Surface Properties

This data structure describes the characteristics of exterior surfaces. Exterior surface properties may include emissivity, reflectivity, and roughness. The first two govern radiation exchange from the surface, while the latter governs the magnitude of the exterior air film resistance.

5.11.1 Occupant Heat Rate

This data structure represents the rate of heat and moisture generated by building occupants. This is typically specified in terms of a sensible heat rate and a latent heat rate. Both are specified in Btu/h.

5.11.2 Furniture and Contents

This data structure represents the thermal mass effect of furniture and other building contents. This is expressed in terms of lb/ft² for the space in question.

5.11.3 Reference Position in a Space

This data structure locates a reference point in a space, typically for the purposes of daylighting control. The typical construct for the reference point is a set of coordinates (X, Y, and Z) relative to the space coordinate system.

5.12 Two-Dimensional Curve

This data structure explains one parameter in terms of another. An example is a curve that modifies the efficiency of an air conditioner relative to the fraction of time that the equipment operates within the period of an hour. The relationship can be expressed in terms of the X and Y coordinates of points on the curve, or it can be expressed as an equation.

5.12.1 Three-Dimensional Curve

This data structure explains one parameter in terms of two others. An example is a curve that modifies the efficiency of an air conditioner relative to the outside air dry-bulb temperature and the wet-bulb temperature of air returning to the coil. The relationship is a three-dimensional surface and can be expressed in terms of the X and Y coordinates of points on the curve, or it can be expressed as an equation.

5.12.2 Temperature Reset Schedule

This data structure describes the relationship between one temperature and another. For example, the independent variable might be outside air temperature and the dependent variable is supply air temperature. In this case, a common schedule would be to set the supply air temperature at 55°F when the outside air temperature is 80°F or warmer and at 62°F when the outside air temperature is 58°F or cooler with the supply air temperature scaling between 55°F and 62°F when the outside air temperature is between 80°F and 58°F.